

REACTING FLUIDS LABORATORY

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EFFECTS UPON CONVECTIVE AND RADIATIVE
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FINITE-RATE CHEMISTRY EFFECTS UPON CONVECTIVE AND RADIATIVE HEATING OF AN ATMOSPHERIC ENTRY VEHICLE

by
Guillermo Perez
under the direction of
Richard C. Farmer
&
Ralph W. Pike

Prepared Under
Grant NGR 19-001-059

by
Department of Chemical Engineering
Louisiana State University
Baton Rouge, Louisiana 70803

for

Langley Research Center
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

LOUISIANA STATE UNIVERSITY



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PREFACE

This report gives the results for a finite-rate, stagnation-line analysis of the radiative heating of a phenolic-nylon ablator. The analysis includes flowfield coupling with the ablator surface, binary diffusion, and a coupled line and continuum radiation model. This report serves as a user's manual and operating instructions for the computer programs listed in this document. This analysis has been incorporated into an around-the-body analysis by these same principal investigators. Copies of the decks which are used to accomplish this later analysis have been supplied to Dr. James N. Moss, grant monitor, of the NASA, Langley Research Center.

This report also served as Guillermo Perez's dissertation requirement in obtaining a Doctor of Philosophy degree in Chemical Engineering.

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NOMENCLATURE

English

| | |
|----------|--|
| B_V | Planckian radiation intensity ($m/t^2 \times$ no. of particles) |
| C_i | Mass fraction (mass of i /unit mass of fluid) $C_i = \frac{\rho_i}{\rho}, \quad \sum_i C_i = 1$ |
| C_p | Specific heat at constant pressure ($L^2/t^2 \times T$) |
| C_{pi} | Species specific heat at constant pressure $(L^2/t^2 \times T \times \text{mole of } i)$ |
| D | Diffusion coefficient (L^2/t) |
| D_i^T | Thermal diffusion coefficient ($m/L \times t$) |
| D_{ij} | Binary diffusion coefficient (L^2/t) |
| E | Radiative flux divergence ($m/L \times t^2$) |
| f | Velocity function defined by Eg. 4.14 |
| f_j | Forward reaction rate constant for the j th reaction |
| F | Gibbs free energy (mL^2/t^2) |
| F_i | Species Gibbs free energy ($mL^2/t^2 \times \text{mole of } i$) |
| H | Total enthalpy, $H = h + \frac{u^2}{2}$ (L^2/t^2) |
| h | Static enthalpy, $h = Q + P/\rho$ (L^2/t^2), also Planck's constant |
| h_i | Species static enthalpy ($L^2/t^2 \times \text{mole of } i$) |
| I_V | Spectral radiation intensity ($m/t^2 \times$ no. of particles) |

| | |
|-----------|--|
| J_{iy} | Mass diffusion component for the i th species in a direction normal to the body surface ($m/L^2 \times t$) |
| k | Coefficient of thermal conductivity ($mL/t^3 \times T$) |
| K_j | Equilibrium constant for the j th reaction |
| M_i | Molecular weight of species i (mass of i /mole of i) |
| m | Number of chemical reactions occurring in the flow-field |
| m_i | Mass of i (m) |
| N | Number density (particles/ L^3) |
| n | Number of chemical species present in the flow-field |
| n_i | Species molal density (moles of i / L^3) |
| P | Static pressure ($m/L \times t^2$) or (F/L^2) |
| Q | Internal energy per unit mass, including chemical energy (L^2/t^2), also total energy flux to a surface (m/t^3) |
| $q_{R,y}$ | Radiative heat flux component in a direction normal to the body surface (m/t^3) or ($E/L^2 \times t$) |
| Q_c | Convective energy flux to a surface (m/t^3) |
| Q_D | Diffusive energy flux to a surface (m/t^3) |
| Q_R | Radiative energy flux to a surface (m/t^3) |
| R | Body nose radius (L) |
| Re | Reynolds number $\rho_{\infty, 0} U_{\infty} R / \mu_{s, 0}$ |

| | |
|-------------|---|
| Re_δ | Reynolds number $\rho_{s,0} U_\infty R/\mu_{s,0}$ |
| IR | Universal gas constant ($\text{mL}^3/\text{t}^2 \times T \times \text{no. of moles}$) |
| r | Cylindrical body radius defined in Fig. 2.1 (L) |
| r_j | Backward reaction rate constant for the jth reaction |
| S | Entropy ($\text{L}^2/\text{t}^2 \times T$) |
| S_i | Species entropy ($\text{L}^2/\text{t}^2 \times T \times \text{mole of } i$) |
| T | Thermodynamic temperature (T) |
| t | Time (t) |
| U_∞ | Freestream velocity (L/t) |
| u | Velocity component in a direction parallel to the body surface (L/t) |
| v | Velocity component in a direction normal to the body surface (L/t) also frequency (1/t) |
| x | Body oriented coordinate defined in Fig. 2.1 |
| y | Body oriented coordinate defined in Fig. 2.1 |
| y_i | Mole fraction of species i, $\sum_i y_i = 1$ |

Greek

| | |
|------------|--|
| α_e | Volumetric absorption coefficient, effective ($\text{L}^2 \times \text{no. of particles}/\text{L}^3$) |
| δ | Shock detachment distance (L) |

| | |
|------------------|--|
| $\tilde{\delta}$ | Transformed shock detachment distance |
| ϵ | Difference between the body and shock angle $\epsilon = \theta - \phi$ radians |
| η | Dorodnitsyn variable |
| θ | Body angle (radians) |
| κ | Local body curvature (1/L) |
| $\tilde{\kappa}$ | $1 + y$ |
| Λ | Diffusional or radiative flux divergence (see Eq. 2.12) (m/L x t ²) |
| λ | $(\tilde{\mu}^{-2/3} \mu)$ (m/L x t) |
| μ | Ordinary viscosity (m/L x t) |
| $\tilde{\mu}$ | Bulk viscosity (m/L x t) |
| ν | Frequency (1/t) |
| ξ | Nondimensional x-coordinate |
| ρ | Density (m/L ³) |
| ρ_i | Partial density of species i, $\rho_i = n_i M_i$ (m of i/L ³) |
| $\tilde{\rho}$ | Density ratio across shock |
| ϕ | Shock angle (radians) |
| ω_i | Generation of species i (m/L ³ x t) |

Subscripts

| | |
|-------------|---|
| D | Diffusion |
| e | Edge conditions |
| i | Species i |
| n | Normal component |
| r | Radiation |
| t | Tangential component or total quantity |
| v | Spectral |
| w | Wall quantities |
| x | Component in a direction parallel to the body surface |
| y | Component in a direction normal to the body surface |
| ∞ | Freestream conditions |
| s, δ | Quantities immediately behind the shock |

Superscripts

| | |
|---|--|
| A | 0 or 1 denoting two-dimensional or axisymmetric respectively (an exponent) |
| T | Thermal |
| * | Denotes dimensional variables |
| o | Standard state quantity |
| - | Evaluated on char side of ablator interface |
| + | Evaluated on flow-field side of ablator interface |

** Symbols not listed are defined where used.

\$ Abbreviations mean: m, mass; L, length; t, time; T, temperature; F, force; E, energy

ABSTRACT

A mathematical model of the aerothermochemical environment along the stagnation line of a planetary return spacecraft using an ablative thermal protection system was developed and solved for conditions typical of atmospheric entry from planetary missions. The model, implemented as a Fortran IV computer program, was designed to predict viscous, reactive and radiative coupled shock layer structure and the resulting body heating rates. The analysis includes flow-field coupling with the ablator surface, binary diffusion, coupled line and continuum radiative and equilibrium or finite-rate chemistry effects. The gas model used includes thermodynamic, transport, kinetic and radiative properties of air and ablation product species, including 19 chemical species and 16 chemical reactions. Specifically, the impact of non-equilibrium chemistry effects upon stagnation line shock layer structure and body heating rates was investigated.

The mathematical model forms a set of twenty-six algebraic, differential and integrodifferential non-linear equations subject to two-point boundary conditions. The numerical solution was carried out by decoupling, linearizing and finite differencing the equations in a iterative, globally-implicit manner.

The model was used to determine the finite-rate chemistry and chemical-equilibrium stagnation line heating rate of a 9 foot entry vehicle protected by a phenolic-nylon ablator and moving at 50,000 feet/second when the free-stream air density is 8.85×10^{-8} slugs/ft³ and the mass injection rate equals .05. The results obtained predict significantly different shock layer properties for the finite-rate

case and for the chemical equilibrium case. The ablation products entering the shock layer through the body wall react much slower than under equilibrium conditions and the air components entering the shock layer through the shock are not de-ionized, as predicted by the equilibrium analysis, but remain "frozen" throughout most of the shock region. The total heating rate to the body was found to be significantly lower under non-equilibrium than under equilibrium chemistry conditions. This difference was attributed to lower concentrations of optically active species, such as N and O, over most of the shock layer in the non-equilibrium chemistry case.

On the basis of this research it was concluded that a finite-rate stagnation-line shock layer solution which contains a reasonable kinetics model to describe atmospheric entries protected by phenolic-nylon ablators was successfully developed. It was also concluded that, for the flight conditions considered, finite-rate chemistry effects are significant since both shock layer structure and body heating rates are markedly different from those predicted by chemical equilibrium analyses. It was recommended that additional studies be carried out on: 1.) improving the computational speed of the solution; 2.) finite-rate chemistry effects under flight conditions different from those considered in this work; 3.) the effect of assuming more realistic shock and wall boundary conditions; and 4.) determining solutions for coupled non-equilibrium ablator response and shock layers. The computational speed of the solution should be improved so that SLAC is developed from a research program to an effective engineering tool. Studies of finite-rate chemistry effects under flight conditions different from those considered here should be carried out to determine the range of

conditions over which solutions may be obtained using the implemented model, and the range of conditions over which non-equilibrium chemistry effects are significant and how they affect the body heating rates. The boundary conditions studies should investigate the possible non-equilibrium composition of air, including detailed studies of the pertinent kinetics at the shock, (in the present work the chemical equilibrium composition was used), and the use of boundary conditions of the third kind at the wall. Coupled solutions for both non-equilibrium ablator behavior and shock layers should be determined to obtain a complete solution to the quasi-steady entry problem.

CHAPTER 1

INTRODUCTION

THE NATURE OF THE REENTRY PROBLEM

The launching of a suborbital, orbital or superorbital flight requires that some means of propulsion be utilized to propel the spacecraft against the force of gravity. In practice, a chemically-fueled rocket engine is used. The increase in the speed of the spacecraft results in an increase in its kinetic energy while its motion through the gravitational force field results in an increase in its potential energy. This kinetic and potential energy would be converted into the same amount of kinetic energy if the spacecraft were allowed to fall freely towards the Earth and there was no atmosphere. This would result in fantastic landing speeds and the destruction of the spacecraft upon its collision with the surface of the Earth. Obviously, additional propulsive power would have to be used in a direction opposite to the force of gravity in order to decelerate the spacecraft until reasonable landing speeds are obtained. This need for additional propulsive power results in a decrease of the payload since some of the weight of the spacecraft has to be used for the propulsion hardware. N.A.S.A.'s Moon landings are accomplished precisely in this manner.

Deceleration of entry vehicles can be accomplished by: 1) use of propulsive power against the pull of gravity, and 2) use of aerodynamic drag. Landing of spacecrafts on celestial bodies lacking an atmosphere

precludes the use of the second technique. Aerodynamic braking requires transfer of the body's kinetic and potential energy to the atmosphere and results in the heating of the air in the vicinity of the body. Part of the energy gained by the air is then transferred to the body by convection and radiation creating a need for thermal protection of the spacecraft. Both the propulsive and aerodynamic drag braking techniques require a reduction of the payload since in the former the propulsion hardware and in the latter the heat shield must be a part of the spacecraft. The optimum technique to be used varies with mission characteristics and should be chosen according to a number of factors, such as the level of heat shielding technology and payload. Less payload penalty is suffered by using aerodynamic braking than propulsive braking for braking from hyperbolic approach velocities to orbital velocities at Mars (Ref. 1.1).

DESCRIPTION OF THE FLOW-FIELD IN THE VICINITY OF THE BODY

Of primary importance, in establishing entry heating levels, is the magnitude of the speeds that will be encountered. The range of Earth entry speeds associated with several mission objectives is given in Figure 1.1 (Ref. 1.2). The use of the gravitational field of Venus to alter the interplanetary trajectories between Mars and Earth (Venus swingby) is shown to result in significantly reduced Earth entry speeds when compared to direct trajectories. Of particular interest are entry speeds of about 50,000 fps (about 15 km/sec) since the capability of entering at these speeds will permit a wide variety of trips to Mars, Venus and the asteroids.

A blunt body of the type used by NASA in its manned spacecraft program (Figure 1.2) would show, under conditions prevalent during atmospheric entry at speeds of approximately 50,000 fps, a standing bow

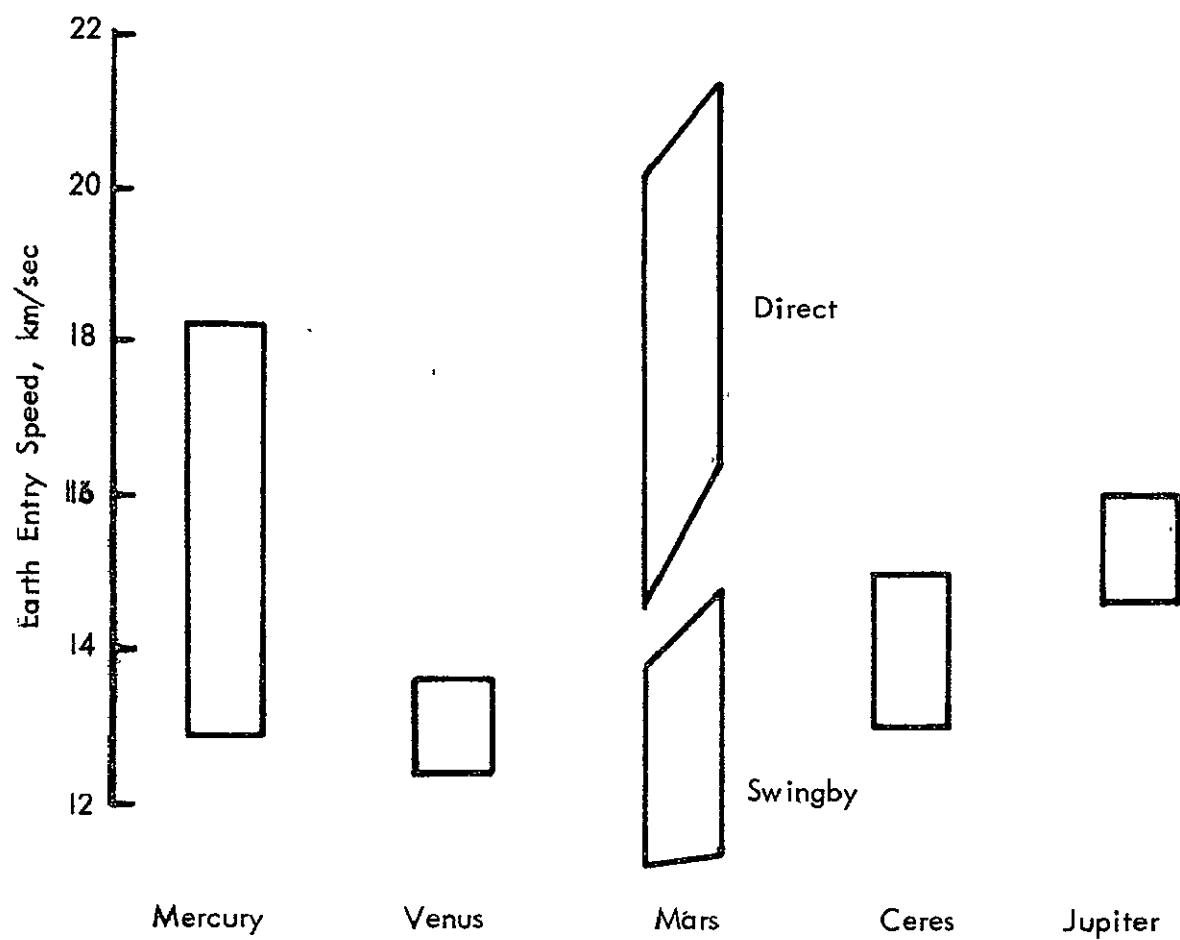


Figure 1.1 Earth Entry Speeds for Several Mission Objectives
(From Ref. 1.2)

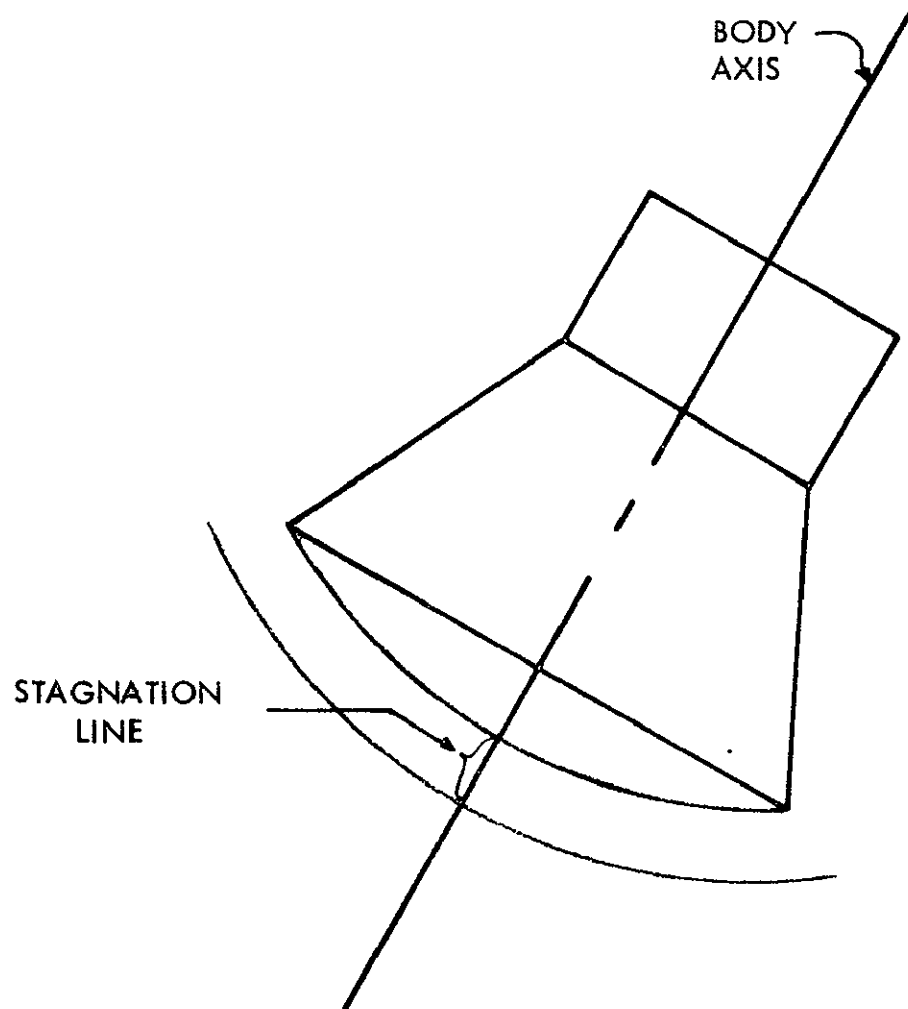


Figure 1.2 Flow-Field in the Vicinity of a Blunt Body

shock due to the deceleration of the air in the vicinity of the body. The blunt body of Figure 1.2 is axisymmetric and is shown flying at a zero angle of attack (its axis of symmetry coincides with the translational vector), this means that the shock and the flow field between the shock and the body are axisymmetric. The region of the flow field between the shock and the surface of the body may be divided into the stagnation line, so-called because the velocity is zero somewhere along this line, and the around-the-body region. When the body wall is impermeable, the stagnation point is at the wall where both the normal and tangential components of velocity are zero.

It is important to note, that even though viscous effects are in part responsible for the generation of the high temperatures present, they are not solely responsible since even for inviscid fluids there would be a deceleration of the fluid at the stagnation line. Thus the difference between viscous and inviscid flow fields would not be basic in nature.

THERMAL PROTECTION SYSTEMS

The task of protecting the vehicle from the heat generated during atmospheric entry may be accomplished in a number of ways: use of heat sinks, transpiration cooling, radiation cooling, ablative cooling, etc. The use of a heat sink as a thermal protection system is very limited by the lack of suitable materials with high heat capacity and low thermal conductivity. Transpiration cooling involves the injection of a cool fluid into the shock layer so that it absorbs heat and insulates the vehicle from the high temperatures. Radiation cooling involves the use of a highly reflective structure in order to block the transfer of radiative heat into the body. Ablative heat shields operate with materials that undergo endothermic physical and chemical changes.

Of all the methods of thermal protection, ablative cooling has been the most successful. This type of thermal protection system may be either a non-charring or a charring ablator. The former type of ablator is one that changes from a solid to a gaseous state and enters the shock layer counter to the heat flow; typical of these materials is Teflon. Charring ablators, on the other hand, decompose to a porous char and relatively low molecular weight gases. The latter type ablator has enjoyed widespread use in the space program.

The principal thermal protection mechanisms in a charring ablator are shown in Figure 1.3. Blockage occurs when the pyrolysis gases are injected into the shock layer and prevent most of the convective heat flux from reaching the body surface. The ablation products in the shock layer absorb part of the radiative flux from the shock and also react endothermically. Char re-radiation stops part of the radiative flux to the body while, if the temperature is high enough, sublimation of the char absorbs part of the incoming energy. The further "cracking" of pyrolysis gases and the process of heating those gases from the temperature at the decomposition zone to the shock layer temperature and their subsequent injection into the shock layer constitute additional protection mechanisms. The decomposition process in the pyrolysis zone which initially produces the pyrolysis gas also absorbs heat. Finally, any heat not absorbed elsewhere may be stored in the virgin material in which case the process acts as a simple heat sink.

THE NEED FOR A BETTER UNDERSTANDING OF THE AERODYNAMIC BRAKING PROCESS

Results from a study of the possible effects of uncertainties in analyses upon heat shield weight are given in Fig. 1.4 (Ref. 1.2). The factors considered were uncertainties in: air absorption coefficients,

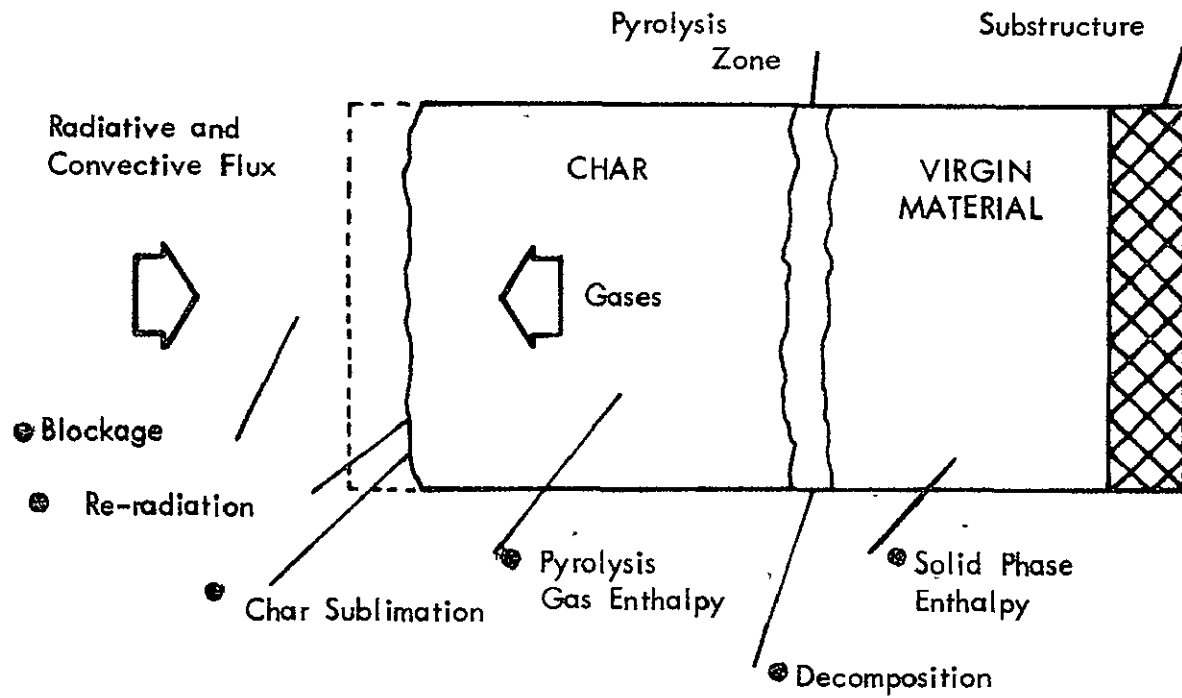


Figure 1.3 Thermal Protection Mechanisms (From Ref. 1.2)

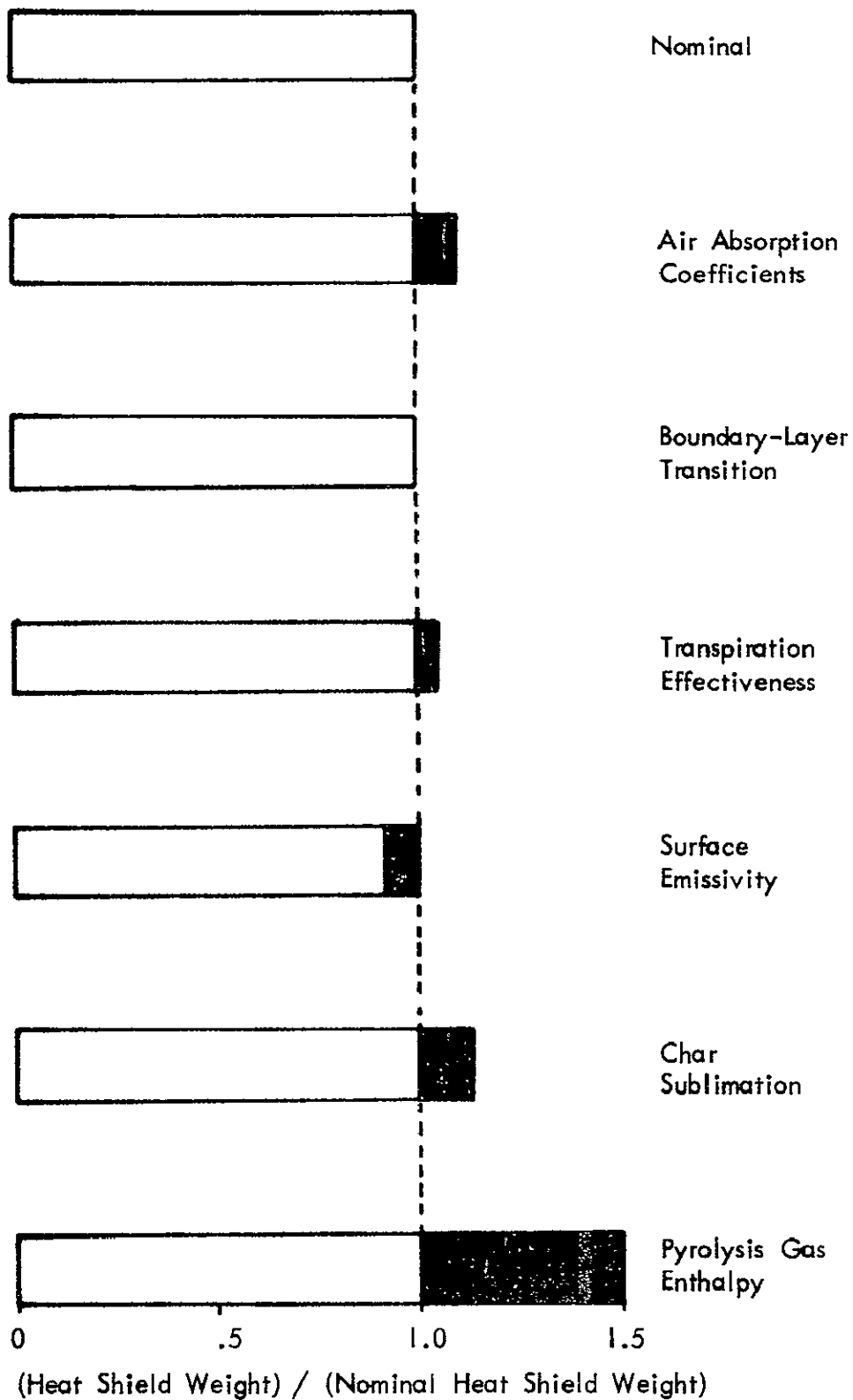


Figure 1.4 Results of Uncertainty Analysis for an Entry Speed of 15 km/sec
(From Ref. 1.2)

boundary layer transition, transpiration effectiveness, surface emissivity, char sublimation, and pyrolysis gas enthalpy. The entry speed considered is 50,000 fps and the weights are normalized to those for the nominal conditions as shown at the top. Results for each perturbation are stated. It should be noted that the only perturbation causing a decrease in heat shield weight is the surface emissivity. Since the effect of some of the uncertainties can be additive it is evident that a weight increase of 50 percent or more is possible. For the range of entry speeds of interest in this study the heat shield is estimated to represent between 10 and 20 percent of the entry-vehicle weight. Due to the uncertainties mentioned above this value can represent as high as 15 to 30 percent of the entry vehicle weight with a corresponding reduction in the payload returnable to Earth. Such an effect is clearly not negligible.

One obvious way of reducing these uncertainties is by experimental verification of the thermal environment about the spacecraft. Such a verification requires duplication of a number of parameters like: flight velocity, free-stream density, enthalpy and flow energy, etc. Unfortunately, the presently existing ground testing facilities are not capable of simultaneous duplication of all the important parameters (Refs. 1.2, 1.4). The alternatives to ground testing are: testing under actual flight conditions or developing accurate mathematical models to describe the aerodynamic braking process. Because of the very high costs involved in flight testing, the importance of improving existing analytic models is evident.

The pyrolysis gas enthalpy uncertainty, being the largest has since being studied (Ref. 1.4). For entry speeds greater than 50,000

fps, the effect of the radiation heat transfer uncertainty should become increasingly important since at these higher speeds the ratio of heat transfer by radiation to heat transfer by convection increases. A recent study by Engel (Ref. 1.5) resulted in development of a model that includes line and continuum radiation and equilibrium chemistry with no diffusion. Each (Ref. 1.6) used the same radiation model used by Engel and obtained solutions for binary and multicomponent diffusion of gas mixtures in chemical equilibrium

Radiative properties are dependent, among other things, upon chemical composition. For this reason it is important to use the correct chemical model when computing heating rates. The chemical equilibrium assumption greatly simplifies the problem since it eliminates the need for consideration of chemical kinetics. When conditions in the flow-field are such that the actual chemical composition is close to the chemical equilibrium composition, then the use of an equilibrium model is valid. This generally occurs when the pressure and temperature are high. On the other hand for low pressures and temperatures the difference between the actual and equilibrium compositions may be large and a finite-rate chemistry model must be used.

STATEMENT OF OBJECTIVES

This study was undertaken in an effort to reduce uncertainties in reentry heating resulting from non-equilibrium chemistry effects. The state-of-the art upon which improvements will be made is given in Refs. 1.5 and 1.6. Specifically, the current research will try to fulfill the following objectives:

1. Develop and solve a mathematical model of the flow-field along the stagnation-line of the shock layer. The model shall

include the effects of line and continuum radiation, thermodynamic and transport properties of air and ablation products, binary diffusion, and finite-rate chemistry.

2. Determine whether finite-rate chemistry effects significantly change the heating rate from the one obtained by assuming the flow is in chemical equilibrium.

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CHAPTER 2

DEVELOPMENT OF THE FLOW-FIELD EQUATIONS

The mathematical model which describes the flow-field at the stagnation line is developed in this chapter. The conservation equations for the flow-field between the body and the shock with the appropriate wall and shock boundary conditions are developed. From these equations, the stagnation line conservation equations and boundary conditions are derived.

CONSERVATION EQUATIONS OF THE SHOCK LAYER

The mathematical model of the flowfield from the body surface to the shock is obtained from the laws of conservation of mass, momentum and energy and also from the formulation of a gas model. The gas model used will be discussed in the latter part of this chapter as well as in Chapter 3. The complete body oriented conservation equations for a multicomponent, viscous, radiating, chemically reacting fluid at steady state were developed by Engel (Ref. 2.1). The body oriented coordinate system and shock layer geometrical relations pertinent to these conservation equations are given in Figure 2.1.

However, these equations are far too complex to allow solution with presently available techniques. Fortunately, it has been found that, in general, solution of the complete conservation equations is not necessary since some of the terms in the equations do not significantly contribute to the solution and may therefore be neglected. Of

course, which terms, if any, are negligible varies with flow conditions and must be determined in each particular case. It will be shown below that, for the flight conditions of interest, the shock layer is laminar, thin is a continuum and these facts can be used to obtain the order of magnitude of the terms in the conservation equation.

According to the results of Hayes and Probstein (Ref. 2.2) the gas behind a bow shock of a hypervelocity vehicle is a continuum when the free-stream Reynolds number (Re) satisfies the relation

$$Re = \frac{\rho_{\infty}^* U_{\infty}^* R^*}{\mu_{\delta,0}^*} > 100 \quad (2.1)$$

Further, for free-stream Reynolds numbers greater than 100, the standoff distance divided by body radius (δ^*/R^*) has been shown to be approximately equal to the ratio of the pre-shock density (ρ_{∞}^*) to the post-shock density (ρ_{δ}^*) (Ref. 2.2).

$$\frac{\delta^*}{R^*} \approx \frac{\rho_{\infty}}{\rho_{\delta}^*} = \bar{\rho} \quad (2.2)$$

Finally, for hypersonic Mach numbers the density ratio across the shock ($\bar{\rho}$) is of the order of one tenth and less for dissociating gases.

$$\bar{\rho} \leq .10 \quad (2.3)$$

An estimate of the dimensionless standoff distance can be obtained from Eqs. 2.1 and 2.2.

$$\frac{\delta^*}{R^*} \leq .10 \quad (2.4)$$

In this chapter a superscript * will denote dimensional variables unless it is explicitly stated otherwise.

Since the dimensionless distance in a direction parallel to the body (X^*/R^*) is of order 1 and greater while the dimensionless distance in a direction normal to the body is of order $\frac{1}{10}$ and less, this means that the shock layer is thin.

In order to determine the order of magnitude of the terms in the conservation equations following the methods of Ref. 2.3, the equations are nondimensionalized by using constants characteristic of the flow field, for example

$$\begin{aligned}\xi &= \frac{X^*}{R^*} \\ y &= \frac{y^*}{R^*} \\ u &= \frac{u^*}{U_{\infty}^*} \\ v &= \frac{v^*}{U_{\infty}^*}\end{aligned}\tag{2.5}$$

The order of magnitude of the dependent and independent variables is determined by using the largest possible magnitude of the dimensional variable, for example,

$$\begin{aligned}\xi \sim \frac{R^*}{R^*} &= 1 \\ y \sim \frac{\delta^*}{R^*} &= \bar{\rho} \\ u \sim \frac{U_{\infty}^*}{U_{\infty}^*} &= 1\end{aligned}\tag{2.6}$$

Using the above procedure, the orders of magnitude of the terms in the conservation equations were determined (Ref. 2.1) and are given in Table 2.1. All terms of order $\bar{\rho}^2$ and higher have been dropped from all equations except the y-momentum equation. The dimensionless variables appearing in Eqs. 2.7 - 2.12 are defined in Table 2.2. It must be noted that Eqs. 2.8, 2.9 and 2.10 contain terms of unknown order of magnitude.

TABLE 2.1
LISTING OF CONSERVATION EQUATION WITH
ORDER ASSESSMENT RESULTS

Global continuity:

$$\begin{array}{cc} O[1] & O[1] \\ \frac{\partial}{\partial \xi} (r^A \rho u) + \frac{\partial}{\partial y} (\kappa r^A \rho v) = 0 \end{array} \quad (2.7)$$

Species continuity:

$$\begin{array}{ccc} O[1] & O[1] & O[1] \\ \frac{\partial}{\partial \xi} (r^A \rho C_i u) + \frac{\partial}{\partial y} (\kappa r^A \rho C_i v) = - \frac{\partial}{\partial y} (\kappa r^A J_{1y}) + \kappa r^A \omega_i \end{array} \quad (2.8)$$

ξ - momentum:

$$\begin{array}{ccc} O[1] & O[1] & O[\bar{\rho}] \\ \rho r^A u \frac{\partial u}{\partial \xi} + \rho \kappa r^A v \frac{\partial u}{\partial y} + \rho \kappa r^A uv + r^A \frac{\partial P}{\partial \xi} \\ O\left[\frac{1}{\rho^2}\right] & O\left[\frac{1}{\rho}\right] & O\left[\frac{1}{\rho}\right] \\ - \frac{1}{R_e} \left\{ \frac{\partial}{\partial y} \left(\kappa r^A \mu \frac{\partial u}{\partial y} \right) - \frac{\partial}{\partial y} (\kappa r^A \mu u) + \kappa r^A \mu \frac{\partial u}{\partial y} \right\} = 0 \end{array} \quad (2.9)$$

TABLE 2.1 (Cont.)

y - momentum: ($O[\bar{\rho}]$ and larger terms)

$$\begin{aligned}
 & \begin{array}{ccc} O[\bar{\rho}] & O[\bar{\rho}] & O[1] \end{array} \\
 & \rho r^A_u \frac{\partial v}{\partial \xi} + \rho r^{A\sim}_{\kappa v} \frac{\partial v}{\partial y} - \rho \tilde{\kappa} r^A_u{}^2 + \tilde{\kappa} r^A \frac{\partial P}{\partial y} \\
 & \begin{array}{ccc} O\left[\frac{1}{\rho}\right] & & O\left[\frac{1}{\rho}\right] \end{array} \\
 & - \frac{1}{R_e} \left\{ \frac{\partial}{\partial \xi} \left(r^A_{\mu} \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial y} \left(\lambda \frac{\partial}{\partial \xi} (r^A_u) \right) \right. \\
 & \begin{array}{ccc} O\left[\frac{1}{\rho}\right] & & O\left[\frac{1}{\rho}\right] \end{array} \\
 & \left. + \frac{\partial}{\partial y} \left(\lambda \frac{\partial}{\partial y} (\tilde{\kappa} r^A_v) \right) + \frac{\partial}{\partial y} \left(2 \tilde{\kappa} r^A_{\mu} \frac{\partial v}{\partial y} \right) \right\} = 0
 \end{aligned} \tag{2.10}$$

y - momentum: ($O[\bar{\rho}^2]$ and larger terms)

$$\begin{aligned}
 & \begin{array}{ccc} O[\bar{\rho}] & O[\bar{\rho}] & O[1] \end{array} \\
 & \cancel{\rho r^A_u \frac{\partial v}{\partial \xi}}^0 + \rho r^{A\sim}_{\kappa v} \frac{\partial v}{\partial y} - \cancel{\rho \tilde{\kappa} r^A_u{}^2}^0 + \tilde{\kappa} r^A \frac{\partial P}{\partial y} \\
 & \begin{array}{ccc} O\left[\frac{1}{\rho}\right] & & O\left[\frac{1}{\rho}\right] \end{array} \\
 & - \frac{1}{R_e} \left\{ \frac{\partial}{\partial \xi} \left(r^A_{\mu} \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial y} \left(\lambda \frac{\partial}{\partial \xi} (r^A_u) \right) \right. \\
 & \begin{array}{ccc} O\left[\frac{1}{\rho}\right] & O\left[\frac{1}{\rho}\right] & O[1] \end{array} \\
 & \left. + \frac{\partial}{\partial y} \left(\lambda \frac{\partial}{\partial y} (\tilde{\kappa} r^A_v) \right) + \frac{\partial}{\partial y} \left(2 \tilde{\kappa} r^A_{\mu} \frac{\partial v}{\partial y} \right) - \frac{\partial}{\partial \xi} (r^A_{\mu\kappa} u) \right. \\
 & \begin{array}{ccc} O[1] & O[1] & O[1] \end{array} \\
 & \left. - \frac{\lambda_{\kappa}}{\tilde{\kappa}} \frac{\partial}{\partial \xi} (r^A_u) - \frac{\lambda_{\mu}}{\tilde{\kappa}} \frac{\partial}{\partial y} (\tilde{\kappa} r^A_v) - 2\mu \frac{\kappa r^A}{\tilde{\kappa}} \frac{\partial u}{\partial \xi} \right\} = 0
 \end{aligned} \tag{2.11}$$

TABLE 2.1 (Cont.)

Energy:

$$\begin{aligned}
& \begin{matrix} 0[1] & 0[1] & 0[1] & 0[1] \end{matrix} \\
& r^A_{\rho u} \frac{\partial H}{\partial \xi} + \tilde{\kappa} r^A_{\rho v} \frac{\partial H}{\partial y} = -2 \left(\Lambda_{D,y} + \Lambda_{R,y} \right) \\
& \begin{matrix} 0 \left[\frac{1}{-2} \right] & 0 \left[\frac{1}{-\rho} \right] \end{matrix} \qquad (2.12) \\
& + \frac{2}{R_e} \left\{ \frac{\partial}{\partial y} \left(\tilde{\kappa} r^A_{\mu u} \frac{\partial u}{\partial y} \right) - \frac{\partial}{\partial y} \left(\kappa r^A_{\mu u}{}^2 \right) \right\}
\end{aligned}$$

TABLE 2.2

DEFINITION OF DIMENSIONLESS VARIABLES

$$\xi = \frac{x^*}{R^*} \quad y = \frac{y^*}{R^*} \quad u = \frac{u^*}{U_\infty^*} \quad v = \frac{v^*}{U_\infty^*}$$

$$\rho = \frac{\rho^*}{\rho_{\infty,0}^*} \quad \mu = \frac{\mu^*}{\mu_{s,0}^*} \quad \lambda = \frac{\lambda^*}{\mu_{s,0}^*} \quad \delta = \frac{\delta^*}{R^*}$$

$$r = \frac{r^*}{R^*} \quad \kappa = \kappa^* R^* \quad P = \frac{P^*}{\rho_\infty^* (U_\infty^*)^2} \quad H = \frac{H^*}{H_s^*}$$

$$h = \frac{h_s^*}{H_s^*} \quad h = \frac{h^*}{h_s^*} \quad \text{where} \quad H_s^* = \frac{1}{2} U_\infty^{*2}$$

$$\tilde{\kappa} = 1 + \kappa y \quad \omega_i = \frac{R^* \omega_1^*}{\rho_\infty^* U_\infty^*} \quad J_i = \frac{J_i^*}{\rho_\infty^* U_\infty^*} \quad \Lambda_{R,x} = \frac{\Lambda_{R,x}^*}{\rho_\infty^* (U_\infty^*)^3}$$

$$\Lambda_{R,y} = \frac{\Lambda_{R,y}^*}{\rho_\infty^* (U_\infty^*)^3} \quad \Lambda_{D,x} = \frac{\Lambda_{D,x}^*}{\rho_\infty^* (U_\infty^*)^3} \quad \Lambda_{D,y} = \frac{\Lambda_{D,y}^*}{\rho_\infty^* (U_\infty^*)^3}$$

where

$$\Lambda_{R,x}^* = \frac{\partial}{\partial x^*} (r^* A_{q_{R,x}}^*) \quad \Lambda_{R,y}^* = \frac{\partial}{\partial y^*} (r^* A_{\kappa q_{R,y}}^*)$$

$$\Lambda_{D,x}^* = \frac{\partial}{\partial x^*} (r^* A_{q_{D,x}}^*) \quad \Lambda_{D,y}^* = \frac{\partial}{\partial y^*} (r^* A_{\kappa q_{D,y}}^*)$$

This means that these terms can not be dropped from the equations since they may or may not be significant.

Two additional assumptions were made in order to simplify the equations of the flow-field. The first of these consists of assuming that Stokes' Postulate (Ref. 2.3) is valid throughout the flow-field. This yields

$$\lambda^* = -\frac{2}{3} \mu^* \quad (2.13)$$

and simplifies Eqs. 2.10 and 2.11. The radiative energy term in Eq. 2.12 was simplified by assuming that the shock layer geometry is approximated locally by an infinite plane slab in which radiative transport properties vary only across the slab. This assumption yields

$$\begin{aligned} \Lambda_{Ry} &= \frac{\partial (\tilde{\kappa} r^A q_{R,y})}{\partial y} \\ &= -2\pi \tilde{\kappa} r^A \int_0^\infty \alpha_v (2B_v - I_v) dv \end{aligned} \quad (2.14)$$

As a result of the bulk viscosity assumption and the planar radiative transfer model Eqs. 2.7-2.12 may be written in a more usable form. The resulting equations are known as the thin shock layer equations and are given in Table 2.3. These equations are also referred to as the second order boundary layer equations with curvature effects.

If all the terms of order $\bar{\rho}$ or higher are dropped from the conservation equations they are then known as the first order shock layer equations or the first order boundary layer equations (Table 2.4).

The first order boundary layer equations (Table 2.4) were chosen to describe the flowfield for the following reasons: (1) the Reynolds numbers encountered during Earth atmospheric reentry from typical Mars

TABLE 2.3
SECOND ORDER SHOCK LAYER EQUATIONS

Global continuity:

$$\frac{\partial}{\partial x} (r^A \rho u) + \frac{\partial}{\partial y} (\tilde{\kappa} r^A \rho v) = 0 \quad (2.15)$$

Species continuity:

$$\frac{\partial}{\partial x} (r^A \rho C_i u) + \frac{\partial}{\partial y} (\tilde{\kappa} r^A \rho C_i v) = - \frac{\partial}{\partial y} (\tilde{\kappa} r^A J_{i,y}) + \tilde{\kappa} r^A \omega_i \quad (2.16)$$

x - Momentum:

$$\begin{aligned} \rho r^A u \frac{\partial u}{\partial x} + \rho \tilde{\kappa} r^A v \frac{\partial u}{\partial y} + \rho \tilde{\kappa} r^A uv = - r^A \frac{\partial P}{\partial x} \\ + \frac{\partial}{\partial y} (\tilde{\kappa} r^A \mu \frac{\partial u}{\partial y}) - \kappa u \frac{\partial r^A}{\partial y} \end{aligned} \quad (2.17)$$

y - Momentum: ($O[\bar{\rho}]$ and larger terms)

$$\begin{aligned} \rho r^A u \frac{\partial v}{\partial x} + \rho \tilde{\kappa} r^A v \frac{\partial v}{\partial y} - \rho \tilde{\kappa} r^A u^2 = - \tilde{\kappa} r^A \frac{\partial P}{\partial y} \\ + \frac{\partial}{\partial x} (r^A \mu \frac{\partial u}{\partial y}) - \frac{2}{3} \frac{\partial}{\partial y} (\mu \frac{\partial r^A u}{\partial x}) + \frac{4}{3} \frac{\partial}{\partial y} (\tilde{\kappa} r^A \mu \frac{\partial v}{\partial y}) \\ - \frac{2}{3} \frac{\partial}{\partial y} (\tilde{\kappa} r^A \mu v + \tilde{\kappa} \mu v \frac{\partial r^A}{\partial y}) \end{aligned} \quad (2.18)$$

TABLE 2.3 (Cont.)

y - Momentum: $(0 [\rho^{-2}] \text{ and larger terms})$

$$\begin{aligned}
 & \rho r^A u \frac{\partial v}{\partial x} + \rho r^A \tilde{\kappa} v \frac{\partial v}{\partial y} - \rho \kappa r^A u^2 = - \tilde{\kappa} r^A \frac{\partial P}{\partial y} \\
 & + \frac{\partial}{\partial x} (r^A \mu \frac{\partial u}{\partial y}) - \frac{2}{3} \frac{\partial}{\partial y} (\mu \frac{\partial r^A u}{\partial x}) + \frac{4}{3} \frac{\partial}{\partial y} (\tilde{\kappa} r^A \mu \frac{\partial v}{\partial y}) \\
 & - \frac{2}{3} \frac{\partial}{\partial y} (\kappa r^A \mu v) - \frac{\partial}{\partial x} (r^A \kappa \mu u) + \frac{2}{3} \mu \frac{\kappa}{\tilde{\kappa}} \frac{\partial}{\partial x} (r^A u) \\
 & + \frac{2}{3} \mu \frac{\kappa}{\tilde{\kappa}} \frac{\partial}{\partial y} (\tilde{\kappa} r^A v) - 2 \mu \frac{\kappa}{\tilde{\kappa}} r^A \frac{\partial u}{\partial x} - \frac{2}{3} \frac{\partial}{\partial y} (\tilde{\kappa} \mu v \frac{\partial r^A}{\partial y})
 \end{aligned} \tag{2.19}$$

Energy:

$$\begin{aligned}
 & r^A \rho u \frac{\partial H}{\partial x} + \tilde{\kappa} r^A \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left[\tilde{\kappa} r^A \left\{ -k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} \right. \right. \\
 & \left. \left. - \frac{P}{N^2} \sum_i \sum_{j \neq i} \frac{N_i}{m_i} \frac{D_i^T}{D_{ij}} \left(\frac{J_{i,y}}{\rho_j} - \frac{J_{i,y}}{\rho_i} \right) \right\} \right] - \tilde{\kappa} r^A \frac{\partial q_{R,y}}{\partial y} \\
 & \frac{\partial}{\partial y} (\tilde{\kappa} r^A \mu u \frac{\partial u}{\partial y}) - \frac{\partial}{\partial y} (\kappa r^A \mu u^2)
 \end{aligned} \tag{2.20}$$

TABLE 2.4
FIRST ORDER SHOCK LAYER EQUATIONS

(Order determined at $R_e = \rho_\infty U_\infty R / \mu_{\delta,0} = 100$)

Global continuity:

$$\frac{\partial}{\partial x} (\rho r^A u) + \frac{\partial}{\partial y} (\rho \kappa r^A v) = 0 \quad (2.21)$$

Species continuity:

$$\frac{\partial}{\partial x} (r^A \rho C_i u) + \frac{\partial}{\partial y} (\kappa r^A \rho C_i v) = - \frac{\partial}{\partial y} (\kappa r^A J_{i,y}) + \kappa r^A \omega_i \quad (2.22)$$

x - Momentum:

$$\rho r^A u \frac{\partial u}{\partial x} + \rho \kappa r^A v \frac{\partial u}{\partial y} = - r^A \frac{\partial P}{\partial x} + \frac{\partial}{\partial y} \left[\kappa r^A \mu \frac{\partial u}{\partial y} \right] \quad (2.23)$$

y - Momentum:

$$\rho \kappa u^2 = \kappa \frac{\partial P}{\partial y} \quad (2.24)$$

Energy:

$$\begin{aligned} r^A \rho u \frac{\partial H}{\partial x} + \kappa r^A \rho v \frac{\partial H}{\partial y} &= \frac{\partial}{\partial y} \left(\kappa r^A k \frac{\partial T}{\partial y} \right) - \frac{\partial}{\partial y} \left(\kappa r^A \left\{ \sum_i h_i J_{i,y} \right. \right. \\ &\quad \left. \left. - \frac{P}{N^2} \sum_i \sum_{j \neq i} \frac{N_i}{m_i} \frac{D_i^T}{D_{i,j}} \left(\frac{J_{i,y}}{\rho_j} - \frac{J_{i,y}}{\rho_i} \right) \right\} \right) - \kappa r^A \frac{\partial q_{R,y}}{\partial y} \\ &+ \frac{\partial}{\partial y} \left(\kappa r^A \mu u \frac{\partial u}{\partial y} \right) \end{aligned} \quad (2.25)$$

are of the order of 10^{4*} making second order terms even less significant than was shown by the order of magnitude analysis; (2) there are inherent approximations in the theoretical models and experimental data used to predict thermodynamic, transport, radiation and chemical kinetics properties; therefore, there is no need for more detail in the equations solved than in the properties used in the equations.

Further simplification of the conservation equations was obtained by noticing that for the flight conditions and body radii of interest in this work the shock layer is very thin, therefore the relation $\tilde{\kappa} = 1 + \kappa y$ may be approximated by $\tilde{\kappa} = 1$. In addition, neglecting thermal diffusion yields the so-called boundary layer equations (Table 2.5).

BOUNDARY CONDITIONS

As was mentioned above, the boundary layer equations are parabolic and thus require the specification of boundary conditions on the stagnation line ($x = 0$), wall ($y = 0$) and shock ($y = \delta$). The stagnation line boundary conditions are obtained by taking the limit of Eqs. 2.26 - 2.30 as $x \rightarrow 0$. The resulting equations form the basis for most of the work performed in this dissertation and they will be developed in the next section. Wall and shock boundary conditions will be discussed below.

Wall Boundary Conditions

The development of wall boundary conditions may be performed in two different ways. The first, and most frequently used technique

For typical entry conditions of $U_{\infty}^ = 50,000$ fps, $\rho_{\infty}^* = 10^{-5}$ lbm/ft³ (corresponding to an altitude of 214,000 ft (Ref. 2.4)) and $R^* = 10$ ft, a value of $T_{\delta}^* = 15,000$ °K is obtained, therefore $\mu_{\delta,0}^* = 10^{-4}$ lbm/ft sec (Ref. 2.5) and $Re = \frac{\rho_{\infty}^* R^* U_{\infty}^*}{\mu_{\delta,0}^*} \approx 5 \times 10^4$.

TABLE 2.5
BOUNDARY LAYER EQUATIONS

Global continuity:

$$\frac{\partial}{\partial x} (r_w^A \rho u) + r_w^A \frac{\partial}{\partial y} (\rho v) = 0 \quad (2.26)$$

Species continuity:

$$\frac{1}{r_w^A} \frac{\partial}{\partial x} (r_w^A \rho C_i u) + \frac{\partial}{\partial y} (\rho C_i v) = - \frac{\partial}{\partial y} (J_{i,y}) + \omega_i \quad (2.27)$$

x - Momentum:

$$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = - \frac{\partial P}{\partial x} + \frac{\partial}{\partial y} (\mu \frac{\partial u}{\partial y}) \quad (2.28)$$

y - Momentum:

$$0 = \frac{\partial P}{\partial y} \quad (2.29)$$

Energy:

$$\begin{aligned} \rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} (k \frac{\partial T}{\partial y}) - \frac{\partial}{\partial y} \left\{ \sum_i h_i J_{i,y} \right. \\ \left. - \frac{\partial q_{R,y}}{\partial y} \right. \\ \left. + \frac{\partial}{\partial y} (\mu u \frac{\partial u}{\partial y}) \right\} \end{aligned} \quad (2.30)$$

consists of writing mass, momentum, and energy balances across the ablator-shock layer interface (Fig. 2.2). A second technique involves the use of the conservation equations of the flow-field which are integrated from y^+ to y^- and then the resulting equations are evaluated as $\Delta y \rightarrow 0$. Both of these methods should yield identical results since the conservation equations used in the second technique are mass, momentum, and energy balances at any point in the flow-field. The first method has the advantage that the physical significance of each term in the resulting equations is more readily evident. The second technique, on the other hand will assure that all the necessary terms have been considered.

In this work the second technique discussed above was applied to the boundary layer equations (Eqs. 2.26 - 2.30) in order to obtain the surface balances needed. For example, the global continuity equation is multiplied by dy and integrated from y^+ to y^-

$$\int_{y^-}^{y^+} \frac{1}{r_w} \frac{\partial(r_w \rho u)}{\partial x} dy + \int_{y^-}^{y^+} \frac{\partial(\rho v)}{\partial y} dy = 0$$

to yield

$$\left[\frac{1}{r_w} \frac{\partial(r_w \rho u)}{\partial x} \right]_{av} \Delta y + (\rho v)^+ - (\rho v)^- = 0$$

where $\Delta y = y^+ - y^-$ and use has been made of the mean value theorem to integrate the first term. When the limit as $\Delta y \rightarrow 0$ of the equation above is taken the result is

$$(\rho v)^- = (\rho v)^+ \quad (2.31)$$

This same procedure was used with the other equations and the resulting relations are given in Table 2.6.

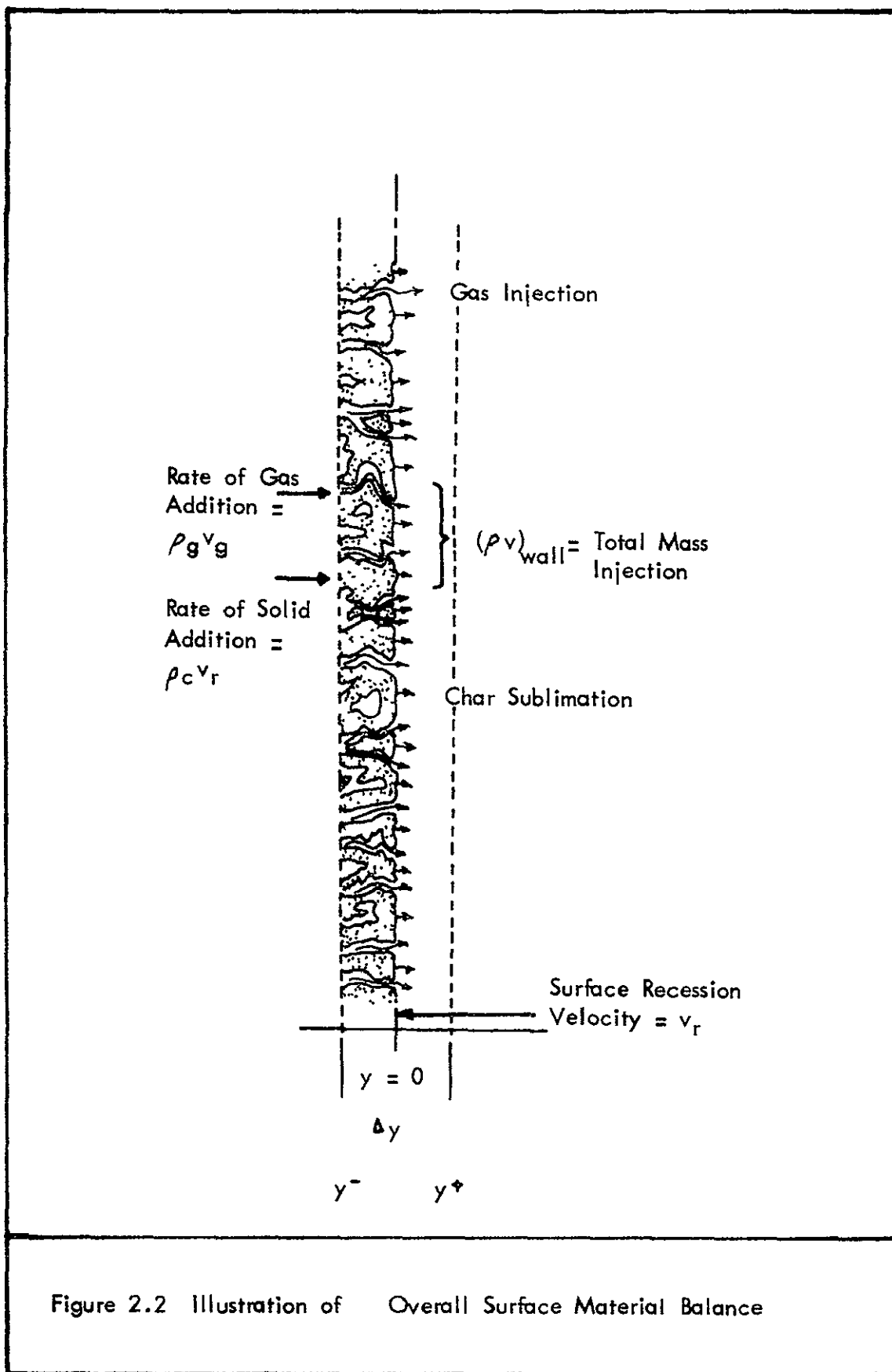


TABLE 2.6
SURFACE BALANCE EQUATIONS

Global Mass Balance:

$$(\rho v)^- = (\rho v)^+ \quad (2.31)$$

Species Mass Balance:

$$(\rho v c_i - J_i)^- = (\rho v c_i - J_i)^+ \quad (2.32)$$

X-momentum:

$$(\rho v u - \mu \frac{\partial u}{\partial y})^- = (\rho v u - \mu \frac{\partial u}{\partial y})^+ \quad (2.33)$$

y-momentum:

$$p^- = p^+ \quad (2.34)$$

Energy:

$$\begin{aligned} & (\rho v H - k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} + q_{R,y} - \mu u \frac{\partial u}{\partial y})^- \\ & = (\rho v H - k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} + q_{R,y} - \mu u \frac{\partial u}{\partial y})^+ \end{aligned} \quad (2.35)$$

In these surface balance equations all the dependent variables on the char side of the interface (those with a superscript $-$) must be interpreted as being dependent upon both the solid and gaseous phases. For example $\rho^- = \rho_g + \rho_c$ and $(\rho v)^- = \rho_g v_g + \rho_c v_c$. In addition it must be noticed that if the flow in the char is one dimensional ($u^- = 0$), as it is usually assumed to be, the x - momentum equation is identically zero while the terms with arrows through them in the energy balance also become zero. This same situation exists on the stagnation line since u is zero for all values of y.

Shock Boundary Conditions

The bow shock is governed by three basic conservation equations, corresponding to the three physical principles of conservation of mass, momentum, and energy. These equations, commonly known as the Rankine-Hugoniot equations are given in rectangular coordinates by (Ref. 2.6):

Continuity

$$\rho_{\infty}^* V_{\infty,n}^* = \rho_{\delta}^* V_{\delta,n}^* \quad (2.36)$$

Momentum

$$\text{(normal)} \quad \rho_{\infty}^* + \rho_{\infty}^* V_{\infty,n}^{*2} = \rho_{\delta}^* + \rho_{\delta}^* V_{\delta,n}^{*2} \quad (2.37)$$

$$\text{(tangential)} \quad V_{\infty,t}^* = V_{\delta,t}^* \quad (2.38)$$

Energy

$$h_{\infty}^* + \frac{1}{2} V_{\infty,n}^{*2} + \frac{1}{2} V_{\infty,t}^{*2} = h_{\delta}^* + \frac{1}{2} V_{\delta,n}^{*2} + \frac{1}{2} V_{\delta,t}^{*2} \quad (2.39)$$

Using Figure 2.3 the above equations can be written in body oriented coordinates. The development of these equations in curvilinear coordinates was performed by Engel (Ref. 2.1) and the resulting equations are given below in dimensionless form

$$v_{\delta} = \sin \phi \sin \epsilon - \bar{\rho} \cos \phi \cos \epsilon \quad (2.40)$$

$$u_{\delta} = \sin \phi \cos \epsilon - \bar{\rho} \cos \phi \sin \epsilon \quad (2.41)$$

$$P_{\delta} = (1 - \bar{\rho}) \cos^2 \phi + \cancel{P_{\infty}^0} \quad (2.42)$$

$$h_{\delta} = (1 - \bar{\rho}^2) \cos \phi + \cancel{h_{\infty}^0} \quad (2.43)$$

Again, the dimensionless variables used above were defined as given in Table 2.2. As shown in Eqs. 2.42 and 2.43 the terms P_{∞} and h_{∞} may be neglected since they are of order $\bar{\rho}^{-2}$.

For a given shock shape and flight conditions, if the post-shock fluid is assumed to be pure air in chemical equilibrium, i.e., no diffusion of ablation products to the shock, then the velocity of the fluid at the shock and also its thermodynamic state (pressure, enthalpy, composition, etc.) can be found from Eqs. 2.40 - 2.43 and a technique for obtaining the equilibrium composition of a fluid. In practice a chemical equilibrium program is usually available and a value of $\bar{\rho}$ is guessed, from Eqs. 2.42 and 2.43 P_{δ} and h_{δ} are computed and the equilibrium program is used to obtain the composition and a new value of $\bar{\rho}$ is obtained. This procedure is repeated until the guessed and computed values of $\bar{\rho}$ agree.

Therefore, boundary conditions of the first kind, i.e., the dependent variables are specified, may be used at the shock ($y = \delta$). These boundary conditions are given below

$$\begin{aligned} u &= u_{\delta} \\ v &= v_{\delta} \\ P &= P_{\delta} \\ h &= h_{\delta} \\ Ci &= Ci_{\delta}(P, h) \text{ (Assuming chemical equilibrium)} \\ I_v(T_{w\delta}) &= 0 \text{ (No precursor radiation)} \end{aligned} \quad (2.44)$$

The additional boundary condition shown above results from assuming that no radiation from the free-stream air crosses the shock.

STAGNATION LINE BOUNDARY LAYER EQUATIONS*

The stagnation line boundary layer equations are obtained by formally taking the limit as $x \rightarrow 0$ of the boundary layer equations (Eqs. 2.26 - 2.30). The resulting equations were obtained by Engel (Ref. 2.1) and are given here in dimensional form for axisymmetric flow in Table 2.7. Some comments concerning the derivation of the equations in Table 2.7 are pertinent: 1) at the stagnation line the component of velocity tangential to the body (u) is zero because of symmetry; 2) the stagnation line x-momentum equation is obtained by first differentiating the x-momentum equation (Eq. 2.28) with respect to x and then taking the limit as $x \rightarrow 0$ of the resulting equation. This procedure is used because taking the limit of the x-momentum equation yields $(\partial P / \partial x)_{x=0} = 0$, a relationship which does not contain any useful information.

The wall and shock boundary conditions needed in order to be able to integrate Eqs. 2.45 - 2.49 were obtained by taking the limit of Eqs. 2.31 - 2.35 and 2.40 - 2.43 as $x \rightarrow 0$. Before taking the limit of the surface balance equations the flow in the char was assumed to be one-dimensional ($u^- = u^+ = 0$). As was discussed earlier, this condition implies that the terms crossed by an arrow in the energy balance equation are zero while the x-momentum balance is identically zero. In lieu of the x-momentum balance, the fact that u is zero for all x locations at the wall ($(\partial u / \partial x)^+ = 0$) may be used with the stagnation line global continuity equation (Eq. 2.45) to yield

$$\frac{\partial (\rho'v)^+}{\partial y} = 0$$

*In this section all variables are dimensional.

TABLE 2.7

STAGNATION LINE BOUNDARY LAYER EQUATIONS

Global continuity:

$$2 \frac{\partial u}{\partial x} = - \frac{1}{\rho} \frac{\partial}{\partial y} (\rho v) \quad (2.45)$$

Species continuity:

$$\rho v \frac{\partial C_i}{\partial y} = - \frac{\partial}{\partial y} (J_{i,y}) + \omega_i \quad (2.46)$$

x - Momentum:

$$\begin{aligned} \frac{\partial}{\partial y} \left[\mu \frac{\partial}{\partial y} \left(\frac{1}{\rho} \frac{\partial}{\partial y} (\rho v) \right) \right] - \rho v \frac{\partial}{\partial y} \left(\frac{1}{\rho} \frac{\partial}{\partial y} (\rho v) \right) \\ + \frac{\rho}{2} \left(\frac{1}{\rho} \frac{\partial}{\partial y} (\rho v) \right)^2 + 2 \left(\frac{\partial^2 P}{\partial x^2} \right)_{x=0} = 0 \end{aligned} \quad (2.47)$$

y - Momentum:

$$\frac{\partial P}{\partial y} = 0 \quad (2.48)$$

Energy:

$$\begin{aligned} \rho v \frac{\partial H}{\partial y} = - \frac{\partial}{\partial y} \left[-k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} \right. \\ \left. - \frac{\partial q_{R,y}}{\partial y} \right] \end{aligned} \quad (2.49)$$

The stagnation line wall boundary conditions are given in Table 2.8. At the shock $\phi = 0$ and the shock boundary conditions simplify to those given in Table 2.8. The reader should notice that while all the shock boundary conditions are of the first kind, the wall boundary conditions are of the first, second and third kind.

MATHEMATICAL DESCRIPTION OF THE PROBLEM

Before any attempt is made to describe the mathematics of the problem, it is convenient to introduce into the equations the binary diffusion assumption

$$J_{i,y} = -\rho D \frac{\partial c_i}{\partial y} \quad (i = 1, \dots, n) \quad (2.51)$$

and use the relation

$$H = h + v^2/2 \quad (2.52)$$

to write the energy equation (Eq. 2.49) in terms of static enthalpy instead of total enthalpy. When this is done the resulting conservation equations are:

Global continuity:

$$2 \left(\frac{\partial u}{\partial x} \right)_{x=0} = -\frac{1}{\rho} \frac{d(\rho v)}{dy} \quad (2.45)$$

Species continuity:

$$\frac{d(\rho D \frac{dc_i}{dy})}{dy} - \rho v \frac{dc_i}{dy} + \omega_1 = 0 \quad (i = 1, \dots, n) \quad (2.53)$$

x - momentum:

$$\frac{d}{dy} \left[\mu \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \right] - \rho v \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \quad (2.47)$$

$$+ \frac{\rho}{2} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right)^2 + \left(\frac{\partial^2 p}{\partial x^2} \right)_{x=0} = 0$$

TABLE 2.8

WALL AND SHOCK BOUNDARY CONDITIONS FOR STAGNATION LINE PROBLEM

Wall ($y = 0$):

$$\rho v = (\rho v)^- \quad (2.50a)$$

$$\rho v C_i - J_i = (\rho v C_i - J_i)^- \quad (2.50b)$$

$$\frac{\partial(\rho v)}{\partial y} = 0 \quad (2.50c)$$

$$P = P^- \quad (2.50d)$$

$$(\rho v H - k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} + q_{R,y}) =$$

$$(\rho v H - k \frac{\partial T}{\partial y} + \sum_i h_i J_{i,y} + q_{R,y})^- \quad (2.50e)$$

Shock ($y = \delta$):

$$v = v_\delta = - \frac{\rho_\infty}{\rho_{\delta,o}} U_\infty \cos \varepsilon \quad (2.50f)$$

$$u = u_\delta = \frac{\rho_\infty U_{\delta,o}}{\rho_{\delta,o}} \sin \varepsilon \quad (2.50g)$$

$$P = P_\delta = (1 - \frac{\rho_\infty}{\rho_{\delta,o}}) \rho_\infty U_\infty \quad (2.50h)$$

$$h = h_\delta = (1 - (\frac{\rho_\infty}{\rho_{\delta,o}})^2) \frac{U_\infty}{2} \quad (2.50i)$$

$$C_i = C_i(\rho_{\delta,o}, h_\delta) \quad (2.50j)$$

y - momentum:

$$\frac{dP}{dy} = 0 \quad (2.48)$$

Energy:

$$\rho v \frac{dh}{dy} + \rho v^2 \frac{dv}{dy} = \frac{d}{dy} \left(k \frac{dT}{dy} - \rho D \sum_i^n h_i \frac{dC_i}{dy} \right) - \frac{dq_{r,y}}{dy} \quad (2.54)$$

The mathematical model of the stagnation line is given by the above set of $4 + n$ (for a system with n chemical species) ordinary integro-differential equations and a set of two-point boundary conditions given in Table 2.8. The integro-differential equations are coupled, nonlinear, of first, second and third order. They are integral since the flux divergence term $(dq_{r,y}/dy)$ in the energy equation is an integral given by Eq. 2.14. These equations contain the $7 + n$ unknowns $\rho, v, (\partial u/\partial x)_{x=0}, P, h, T$, and C_i ($i = 1, \dots, n$) where both $(\partial u/\partial x)_{x=0}$ and $(\partial^2 P/\partial x^2)_{x=0}$ are at most functions of y . This discussion assumes that thermodynamic, transport, radiative, and kinetics properties (D, ω_i, μ, k , and h_i) are known in terms of the dependent variables listed above. These properties will be discussed in Chapter 3. It is evident that since the $4 + n$ equations contain $7 + n$ unknowns three more independent relations are needed. These are provided by the thermal equation of state

$$P = \left(\sum_i^n \frac{C_i}{M_i} \right) \rho RT \quad (2.55)$$

the caloric equation of state;

$$h = \sum_i^n C_i h_i \quad (2.56)$$

and the Rankine-Hugoniot relation for the shock pressure as a function of the shock angle measured from the stagnation line

$$P_\delta = (1 - \bar{\rho}) \cos^2 \phi \rho_\infty U_\infty^2 \quad (2.42)$$

From the y-momentum equation and the fact that $(\frac{\partial P}{\partial x})_{x=0} = 0$ (as discussed in the previous section) it is evident that

$$(\frac{\partial^2 P}{\partial x^2})_{x=0} = (\frac{\partial^2 P_\delta}{\partial x^2})_{x=0} = \text{constant} \quad (2.57)$$

Substitution of Eq. 2.42 in Eq. 2.57 gives

$$(\frac{\partial^2 P}{\partial x^2})_{x=0} = -2 (1 - \bar{\rho}) (\frac{d\phi}{dx})_{x=0}^2 \rho_\infty U_\infty^2 \quad (2.58)$$

This development makes it possible to rewrite the x-momentum equation as x-momentum

$$\begin{aligned} \frac{d}{dy} \left[\mu \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \right] - \rho v \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \\ + \frac{\rho}{2} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right)^2 - 4 (1 - \bar{\rho}) \left(\frac{d\phi}{dx} \right)_{x=0}^2 \rho_\infty U_\infty^2 \end{aligned} \quad (2.59)$$

Therefore, for a known shock geometry (known $\phi(x)$) the conservation equations contain $6 + n$ unknowns instead of $7 + n$ as before. A further reduction in the number of equations that need to be solved may be accomplished by eliminating h from the energy equation by substituting the caloric equation of state as follows. From Eq. 2.56

$$\begin{aligned} \frac{dh}{dy} &= \sum_i^n h_i \frac{dC_i}{dy} + \sum_i^n C_i \frac{dh_i}{dy} \\ &= \sum_i^n h_i \frac{dC_i}{dy} + \left(\sum_i^n C_i \frac{dh_i}{dT} \right) \frac{dT}{dy} \end{aligned} \quad (2.60)$$

$$\begin{aligned}
&= \sum_i^n h_i \frac{dC_i}{dy} + \left(\sum_i^n C_i C_{Pi} \right) \frac{dT}{dy} \\
&= \sum_i^n h_i \frac{dC_i}{dy} + C_P \frac{dT}{dy}
\end{aligned}$$

Substituting Eq. 2.60 in Eq. 2.54 gives

Energy:

$$\begin{aligned}
\rho v C_P \frac{dT}{dy} &= - \rho v^2 \frac{dv}{dy} - \rho v \left(\sum_i^n h_i \frac{dC_i}{dy} \right) \\
&+ \frac{d[k \frac{dT}{dy} + \rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right)]}{dy} - \frac{dq_{R,y}}{dy}
\end{aligned} \tag{2.61}$$

In this manner the unknowns have been reduced to ρ , v , $(\partial u / \partial x)_x = 0$,

T , and C_i ($i = 1, \dots, n$).

At this point it is pertinent to ask, how many of Eqs. 2.45, 2.53, 2.48, 2.55, 2.59, and 2.61 must be solved simultaneously? It is evident that the y -momentum equation (Eq. 2.48), containing only one dependent variable, is not coupled to the other equations. In fact, it can be integrated anatically to yield

$$P(y) = P_\delta = \text{constant} \tag{2.62}$$

In addition, the global continuity equation (Eq. 2.45) contains only two dependent variables and can be left out of the set that must be solved initially. This has the effect of eliminating $(\partial u / \partial x)_x = 0$ from the unknowns in the equations that must be solved. This variable may be readily obtained from Eq. 2.45 once the ρ and v profiles are known. The answer to the question that was asked at the beginning of this paragraph is that Eqs. 2.53, 2.55, and 2.59, and 2.61 in the unknowns ρ , v , T , and C_i ($i = 1, \dots, n$) must be solved simultaneously.

Let us now consider the boundary conditions that must be used with these equations. The species continuity equation (Eq. 2.53) being second order requires two boundary conditions for C_i . These may be provided by Eqs. 2.50 b and 2.50 j

$$\rho v C_i + \frac{dC_i}{dy} = (\rho v C_i + \rho D \frac{dC_i}{dy})^- \text{ at } y = 0 \quad (2.63)$$

$$C_i = C_{i,\delta} \quad \text{at } y = \delta \quad (2.50j)$$

Notice that the binary diffusion assumption (Eq. 2.51) was substituted in Eq. 2.50b to obtain Eq. 2.63. The x-momentum equation (Eq. 2.59) is third order and requires three boundary conditions for ρv . These are given by Eqs. 2.50a, 2.50c,

$$\rho v = (\rho v)_w \text{ at } y = 0 \quad (2.50a)$$

$$\frac{d(\rho v)}{dy} = 0 \text{ at } y = 0 \quad (2.50c)$$

and

$$\rho v = \rho_\delta v_\delta \text{ at } y = \delta \quad (2.64)$$

The energy equation (Eq. 2.61) being second order in T requires two boundary conditions in addition to the ones already discussed in connection with radiation. The first of these boundary conditions is provided by the wall energy balance (Eq. 2.50e), this relation is written in terms of the variables previously discussed by substituting Eqs. 2.51, 2.52 and 2.56

$$\begin{aligned} \rho v \left(\sum_i^n C_i h_i + \frac{v^2}{2} \right) - k \frac{dT}{dy} - \rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right) + q_{R,y} = \\ \left[\rho v \left(\sum_i^n C_i h_i + \frac{v^2}{2} \right) - k \frac{dT}{dy} - \rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right) + q_{R,y} \right]^- \end{aligned} \quad (2.65)$$

The second temperature boundary condition needed is given by

$$T = T_{\delta} \quad \text{at } y = \delta \quad (2.65)$$

This completes the boundary conditions needed. Of these the most complex ones are those given by Eqs. 2.63 and 2.65. The boundary condition given by Eq. 2.63 may be simplified by noticing that for fairly high mass injection rates $((\rho v)_{\text{wall}}/\rho_{\infty} U_{\infty} > 0.05)$ mass transfer by diffusion is small compared to mass transfer by convection near the wall. This means that the diffusion terms in Eq. 2.63 may be neglected to yield

$$C_i = C_i^- \quad \text{at } y = 0 \quad (i = 1, \dots, n) \quad (2.67)$$

where use has been made of Eq. 2.50a to cancel the ρv terms from the equation above. The wall energy balance (Eq. 2.65) may also be simplified by neglecting the kinetic energy of the fluid $(\frac{v^2}{2})$ since it is small when compared to the thermal energy. When this is done Eq. 2.65 becomes

$$\begin{aligned} \rho v \left(\sum_i^n C_i h_i \right) - k \frac{dT}{dy} - \rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right) + q_{R,y} = \\ \left[\rho v \left(\sum_i^n C_i h_i \right) - k \frac{dT}{dy} - \rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right) + q_{R,y} \right]^- \end{aligned} \quad (2.68)$$

Since all the other boundary conditions are of the first and second kind and uncoupled, while Eq. 2.68 is of the third kind and coupled to the species continuity equation, it was expedient to assume that the char is at the sublimation temperature of carbon:

$$T = T_w \quad \text{at } y = 0 \quad (2.69)$$

Until now we have assumed that δ (the shock stand-off distance) is known, a necessary condition to be able to integrate the equations of the flow-field. In practice, δ is unknown and therefore must be guessed to be able to compute it correctly. This boundary condition is obtained by

evaluating the global continuity equation at the shock

$$\frac{d(\rho v)}{dy} = -2 \rho_{s,0} \left(\frac{\partial u}{\partial x} \right)_{x=0} \text{ at } y = \delta \quad (2.70)$$

Table 2.9 gives a summary of the stagnation line model including the conservation equations, the equation of state and the boundary conditions.

REFERENCES

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- 2.5 Hansen, C.F., "Approximations for the Thermodynamic and Transport Properties of High-Temperature Air," NASA Technical Report R-50, 1959
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TABLE 2.9

SUMMARY OF THE STAGNATION LINE MODEL

Governing Equations:

Species continuity:

$$\frac{d(\rho D \frac{dC_i}{dy})}{dy} - \rho v \frac{dC_i}{dy} + \omega_i = 0 \quad (2.53)$$

X-momentum:

$$\frac{d}{dy} \left[\mu \frac{d(\rho v)}{dy} \right] - \rho v \frac{d(\rho v)}{dy} \quad (2.59)$$

$$+ \frac{\rho}{2} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right)^2 - 4(1-\bar{p}) \left(\frac{\partial u}{\partial x} \right)_{x=0}^2 = \rho_\infty U_\infty^2 = 0$$

Energy:

$$\begin{aligned} \frac{d}{dy} \left(k \frac{dT}{dy} \right) - \rho v C_p \frac{dT}{dy} + \frac{d}{dy} \left[\rho D \left(\sum_i^n h_i \frac{dC_i}{dy} \right) \right] \\ - \rho v^2 \frac{dv}{dy} - \rho v \left(\sum_i^n h_i \frac{dC_i}{dy} \right) - \frac{dq_{R,y}}{dy} = 0 \end{aligned} \quad (2.61)$$

Thermal equation of state:

$$\rho = \frac{1}{R} T \left(\sum_i^n \frac{C_i}{M_i} \right) = P_\delta \quad (2.55)$$

TABLE 2.9 (CONTINUED)

Boundary Conditions:

Wall ($y = 0$)

$$1. \quad \rho v = (\rho v)_w \quad (2.50a)$$

$$2. \quad \frac{d(\rho v)}{dy} = 0 \quad (2.50c)$$

$$3. \quad C_1 = C_{1,w} \quad (2.67)$$

$$4. \quad T = T_w \quad (2.68)$$

Shock ($y = \delta$)

$$1. \quad \rho v = (\rho v)_\delta \quad (2.64)$$

$$2. \quad \frac{d(\rho v)}{dy} = -2\rho_{\delta,o} \left(\frac{\partial u \delta}{\partial x} \right)_{x=0} \quad (2.70)$$

$$3. \quad C_1 = C_{1,\delta} \quad (2.50j)$$

$$4. \quad T = T_\delta \quad (2.69)$$

CHAPTER 3

THERMODYNAMIC, TRANSPORT, KINETIC, AND RADIATIVE PROPERTIES

The thermodynamic, transport, kinetic, and radiative properties of the mixture of air and ablation species which are necessary to solve the conservation equations that were presented in the previous chapter are evaluated with the methods described in this chapter.

THERMODYNAMIC PROPERTIES

Thermodynamic Properties of Pure Species at Standard State

The specific heat at constant pressure, enthalpy, entropy, and Gibbs free energy of the pure species at one atmosphere pressure (standard state) were obtained by Esch, et. al. (Ref. 3.1). This was accomplished by curve fitting with the method of least squares, temperature polynomials to data found in the literature. The polynomials used to fit the thermodynamic properties are given in Table 3.1. For each species, two different fits were made, one for low temperatures ($1000^{\circ}\text{K} \leq T \leq 6000^{\circ}\text{K}$) and the other for high temperatures ($6000^{\circ}\text{K} \leq T \leq 15,000^{\circ}\text{K}$). For each species, the coefficients for the expressions in Table 3.1 are given in Table 3.2 for the two temperature ranges. Each term in Eqs. 3.1 - 3.4 is dimensionless, temperature is in $^{\circ}\text{K}$ and $R = 1.987 \text{ cal/gmole} - ^{\circ}\text{K}$, therefore the units of the thermodynamic properties are as given below

$$C_{Pi} = [\text{ cal/gmole of } i - ^{\circ}\text{K}]$$

TABLE 3.1 POLYNOMINAL EQUATIONS FOR
STANDARD THERMODYNAMIC PROPERTIES

Specific Heat

$$\frac{C_{p,i}^{\circ}}{R} = A_{1,i} + A_{2,i}T + A_{3,i}T^2 + A_{4,i}T^3 + A_{5,i}T^4 \quad (3.1)$$

Enthalpy

$$\frac{h_i^{\circ}}{RT} = A_{1,i} + \frac{A_{2,i}}{2}T + \frac{A_{3,i}}{3}T^2 + \frac{A_{4,i}}{4}T^3 + \frac{A_{5,i}}{5}T^4 + \frac{A_{6,i}}{T} \quad (3.2)$$

Entropy

$$\frac{S_i^{\circ}}{R} = A_{1,i} \ln T + A_{2,i}T + \frac{A_{3,i}}{2}T^2 + \frac{A_{4,i}}{3}T^3 + \frac{A_{5,i}}{4}T^4 + A_{7,i} \quad (3.3)$$

Free Energy

$$\frac{F_i^{\circ}}{RT} = A_{1,i}(1 - \ln T) - \frac{A_{2,i}}{2}T - \frac{A_{3,i}}{6}T^2 - \frac{A_{4,i}}{12}T^3 - \frac{A_{5,i}}{20}T^4 + \frac{A_{6,i}}{T} - A_{7,i} \quad (3.4)$$

TABLE 3.2

POLYNOMIAL COEFFICIENTS FOR THERMODYNAMIC PROPERTY CORRELATIONS

| Species | A1,i | | A2,i | | A3,i | | A4,i | | A5,i | | A6,i | | A7,i | | T* |
|---------|---------|----|-------------|--|-------------|--|-------------|--|-------------|--|----------|----|----------|----|----|
| C+ | 0.2609E | 01 | -0.1393E-03 | | 0.5959E-07 | | -0.1037E-10 | | 0.6345E-15 | | 0.2168E | 06 | 0.3709E | 01 | L |
| | 0.2528E | 01 | 0.4869E-05 | | -0.7026E-08 | | 0.1134E-11 | | -0.3476E-16 | | 0.2168E | 06 | 0.4139E | 01 | H |
| H | 0.2500E | 01 | -0.8243E-06 | | 0.6421E-09 | | -0.1720E-12 | | 0.1457E-16 | | 0.2547E | 05 | -0.4612E | 00 | L |
| | 0.3934E | 01 | -0.1776E-02 | | 0.6013E-06 | | -0.7819E-10 | | 0.3482E-14 | | 0.2547E | 05 | -0.8598E | 01 | H |
| N+ | 0.2727E | 01 | -0.2820E-03 | | 0.1105E-06 | | -0.1551E-10 | | 0.7847E-15 | | 0.2254E | 06 | 0.3645E | 01 | L |
| | 0.2499E | 01 | -0.3725E-05 | | 0.1147E-07 | | -0.1102E-11 | | 0.3078E-16 | | 0.2254E | 06 | 0.4950E | 01 | H |
| O+ | 0.2491E | 01 | 0.2762E-04 | | -0.1881E-07 | | 0.3807E-11 | | -0.1028E-15 | | 0.1879E | 06 | 0.4424E | 01 | L |
| | 0.2944E | 01 | -0.4108E-03 | | 0.9156E-07 | | -0.5848E-11 | | 0.1190E-15 | | 0.1879E | 06 | 0.1750E | 01 | H |
| E- | 0.2500E | 01 | 0.3440E-06 | | -0.1954E-09 | | 0.3937E-13 | | -0.2573E-17 | | -0.7450E | 03 | -0.1173E | 02 | L |
| | 0.2508E | 01 | -0.6332E-05 | | 0.1364E-08 | | -0.1094E-12 | | 0.2934E-17 | | -0.7450E | 03 | -0.1208E | 02 | H |
| C | 0.2612E | 01 | -0.2030E-03 | | 0.1095E-06 | | -0.1695E-10 | | 0.8590E-15 | | 0.8542E | 05 | 0.4144E | 01 | L |
| | 0.2141E | 01 | 0.3219E-03 | | -0.5498E-07 | | 0.3604E-11 | | -0.5564E-16 | | 0.8542E | 05 | 0.6874E | 01 | H |
| CN | 0.3411E | 01 | 0.4897E-03 | | 0.1005E-06 | | -0.3473E-10 | | 0.2361E-14 | | 0.4745E | 05 | 0.4746E | 01 | L |
| | 0.3473E | 01 | 0.7337E-03 | | -0.9088E-07 | | 0.4847E-11 | | -0.1018E-15 | | 0.5420E | 05 | 0.4152E | 01 | H |
| CO | 0.3254E | 01 | 0.9698E-03 | | -0.2647E-06 | | 0.3037E-10 | | -0.1177E-14 | | -0.1434E | 05 | 0.4875E | 01 | L |
| | 0.3366E | 01 | 0.8027E-03 | | -0.1968E-06 | | 0.1940E-10 | | -0.5549E-15 | | -0.1434E | 05 | 0.4263E | 01 | H |
| C2 | 0.4443E | 01 | -0.2885E-03 | | 0.3036E-06 | | -0.6244E-10 | | 0.3915E-14 | | 0.9787E | 05 | -0.1090E | 01 | L |
| | 0.4026E | 01 | 0.4857E-03 | | -0.7026E-07 | | 0.4666E-11 | | -0.1142E-15 | | 0.9787E | 05 | 0.1090E | 01 | H |
| C2H | 0.3485E | 01 | 0.3563E-02 | | -0.1237E-05 | | 0.1866E-09 | | -0.1013E-13 | | 0.5809E | 05 | 0.4784E | 01 | L |
| | 0.5307E | 01 | 0.8966E-03 | | -0.1378E-06 | | 0.9251E-11 | | -0.2278E-15 | | 0.5809E | 05 | -0.5288E | 01 | H |

* Temperature Range L = 1000-6000°K, H = 6000-15000°K

TABLE 3.2 (Continued)

| Species | A _{1,i} | | A _{2,i} | | A _{3,i} | | A _{4,i} | | A _{5,i} | | A _{6,i} | | A _{7,i} | | T* |
|-------------------------------|------------------|----|------------------|-------------|------------------|-------------|------------------|----|------------------|----|------------------|--|------------------|--|----|
| C ₂ H ₂ | 0.3891E | 01 | 0.5717E-02 | -0.1957E-05 | 0.2931E-09 | -0.1585E-13 | 0.2590E | 05 | 0.6520E | 00 | L | | | | |
| | 0.6789E | 01 | 0.1503E-02 | -0.2295E-06 | 0.1534E-10 | -0.3763E-15 | 0.2590E | 05 | -0.1539E | 02 | H | | | | |
| C ₃ | 0.4002E | 01 | 0.3541E-02 | -0.1318E-05 | 0.2064E-09 | -0.1144E-13 | 0.9423E | 05 | 0.2020E | 01 | L | | | | |
| | 0.2213E | 02 | -0.1759E-01 | 0.5565E-05 | -0.6758E-09 | 0.2825E-13 | 0.9423E | 05 | -0.1021E | 03 | H | | | | |
| C ₃ H | 0.3965E | 01 | 0.6200E-02 | -0.2265E-05 | 0.3717E-09 | -0.2262E-13 | 0.6283E | 05 | 0.3467E | 01 | L | | | | |
| | 0.3965E | 01 | 0.6200E-02 | -0.2265E-05 | 0.3717E-09 | -0.2262E-13 | 0.6283E | 05 | 0.3467E | 01 | H | | | | |
| C ₄ H | 0.5874E | 01 | 0.7403E-02 | -0.2729E-05 | 0.4437E-09 | -0.2637E-13 | 0.7605E | 05 | -0.4010E | 01 | L | | | | |
| | 0.5874E | 01 | 0.7403E-02 | -0.2729E-05 | 0.4437E-09 | -0.2637E-13 | 0.7605E | 05 | -0.4010E | 01 | H | | | | |
| HCN | 0.3654E | 01 | 0.3444E-02 | -0.1258E-05 | 0.2169E-09 | -0.1430E-13 | 0.1442E | 05 | 0.2373E | 01 | L | | | | |
| | 0.3654E | 01 | 0.3444E-02 | -0.1258E-05 | 0.2169E-09 | -0.1430E-13 | 0.1442E | 05 | 0.2373E | 01 | H | | | | |
| H ₂ | 0.3358E | 01 | 0.2794E-03 | 0.9372E-07 | -0.2948E-10 | 0.2141E-14 | -0.1018E | 04 | -0.3548E | 01 | L | | | | |
| | 0.3363E | 01 | 0.4656E-03 | -0.5127E-07 | 0.2802E-11 | -0.4905E-16 | -0.1018E | 04 | -0.3716E | 01 | H | | | | |
| N | 0.2474E | 01 | 0.9097E-04 | -0.7814E-07 | 0.2218E-10 | -0.1489E-14 | 0.5609E | 05 | 0.4300E | 01 | L | | | | |
| | 0.2746E | 01 | -0.3909E-03 | 0.1338E-06 | -0.1191E-10 | 0.3369E-15 | 0.5609E | 05 | 0.2872E | 01 | H | | | | |
| O | 0.2670E | 01 | -0.1970E-03 | 0.7193E-07 | -0.8901E-11 | 0.4002E-15 | 0.2915E | 05 | 0.4504E | 01 | L | | | | |
| | 0.2548E | 01 | -0.5952E-04 | 0.2701E-07 | -0.2798E-11 | 0.9380E-16 | 0.2915E | 05 | 0.5049E | 01 | H | | | | |
| N ₂ | 0.3221E | 01 | 0.9878E-03 | -0.2907E-06 | 0.3938E-10 | -0.2000E-14 | -0.1043E | 04 | 0.4326E | 01 | L | | | | |
| | 0.3727E | 01 | 0.4684E-03 | -0.1140E-06 | 0.1154E-10 | -0.3293E-15 | -0.1043E | 04 | 0.1294E | 01 | H | | | | |
| O ₂ | 0.3316E | 01 | 0.1151E-02 | -0.3726E-06 | 0.6186E-10 | -0.3666E-14 | -0.1044E | 04 | 0.5393E | 01 | L | | | | |
| | 0.3721E | 01 | 0.4254E-03 | -0.2835E-07 | 0.6050E-12 | -0.5186E-17 | -0.1044E | 04 | 0.3254E | 01 | H | | | | |

*Temperature Range L = 1000-6000°K, H = 6000-15,000°K

$$h_i^\circ = [\text{cal/gmole of } i]$$

$$S_i^\circ = [\text{cal/gmole of } i - ^\circ\text{K}]$$

$$F_i^\circ = [\text{cal/gmole of } i]$$

Thermodynamic Properties of Pure Species at Arbitrary State

Expressions for the thermodynamic properties of pure species at arbitrary pressures and temperatures are predicted in the following manner. The species are assumed to be ideal gases so that (Ref. 3.2)

$$h_i = h_i(T) = h_i^\circ \quad (3.5)$$

and since $C_{Pi} = (\partial h_i / \partial T)_P$ then

$$C_{Pi} = C_{Pi}(T) = C_{Pi}^\circ \quad (3.6)$$

In other words, both the enthalpy and the specific heat at constant pressure are independent of pressure. On the other hand let us examine entropy, from the first law of thermodynamics

$$dh_i = TdS_i + \frac{1}{X_i} dP \quad (3.7)$$

where X_i is the molar concentration of species i (moles of i /volume).

Rewriting Eq. 3.7 gives

$$dS_i = \frac{1}{T} dh_i - \frac{1}{X_i T} dP \quad (3.8)$$

From the thermal equation of state for an ideal gas

$$P = X_i RT \quad (3.9)$$

and Eq. 3.8 can be rewritten as

$$dS_i = \frac{1}{T} dh_i - R d(\ln P) \quad (3.10)$$

and integrated to yield

$$S_i(T,P) = \int \frac{1}{T} dh_i - R \ln P + A_{8,i} \quad (3.11)$$

where $A_{8,i}$ is an integration constant. In order to evaluate $A_{8,i}$ we make use of the fact that $S_i(T,1) = S_i^\circ$ and obtain (by evaluating Eq. 3.11 at $P = 1$ atmosphere).

$$A_{8,i} = S_i^\circ - \int \frac{1}{T} dh_i \quad (3.12)$$

Substitution of Eq. 3.12 in Eq. 3.11 yields

$$S_i = S_i(T,P) = S_i^\circ - R \ln P \quad (3.13)$$

The Gibbs free energy is defined as

$$F_i = h_i - TS_i \quad (3.14)$$

substitution of Eqs. 3.5 and 3.13 in the above equation results in

$$F_i = h_i^\circ - TS_i^\circ - RT \ln P \quad (3.15)$$

or

$$F_i = F_i(T,P) = F_i^\circ + RT \ln P \quad (3.16)$$

since

$$F_i^\circ = h_i^\circ - TS_i^\circ$$

Thermodynamic Properties of Mixtures

For each species in a mixture consisting of a total of n gaseous species the thermodynamic properties are given by

$$h_i = h_i(T) = h_i^\circ \quad (3.17)$$

$$C_{Pi} = C_{Pi}(T) = C_{Pi}^\circ \quad (3.18)$$

$$S_i = S_i(T,P) = S_i^\circ - R \ln P_i \quad (3.19)$$

$$F_i = F_i(T,P) = F_i^\circ + RT \ln P_i \quad (3.20)$$

where P_i is the partial pressure of species i (atmosphere). Mixture properties are then given by

$$h = \sum_i^n y_i h_i \text{ [cal./gmole of mixture]} \quad (3.21)$$

$$C_p = \sum_i^n y_i C_{pi} \text{ [cal./gmole of mixture - } ^\circ\text{K]} \quad (3.22)$$

$$S = \sum_i^n y_i S_i \text{ [cal./gmole of mixture - } ^\circ\text{K]} \quad (3.23)$$

$$F = \sum_i^n y_i F_i \text{ [cal./gmole of mixture]} \quad (3.24)$$

where y_i is the i^{th} species mole fraction (gmole of i /gmole of mixture) and the units of each property are given by the terms in brackets.

TRANSPORT PROPERTIES

Presently available transport properties data for air and ablation products are extremely limited because of the difficulties involved with experimental measurements at the extremely high temperatures of interest. Thus, it is necessary to rely heavily on properties obtained from theoretical predictions like the Chapman-Enskog kinetic theory (Ref. 3.3). At the lower temperatures, where ionization has not yet begun to occur, the classical first order Chapman-Enskog kinetic theory has been found to be reasonably accurate. However for higher temperatures the use of more rigorous kinetic models becomes necessary. The transport properties used in the present work were obtained by Esch (Ref. 3.4); in order to obtain simple closed form expressions for species viscosity and thermal conductivity the theoretical results reported in the literature were curve fit with polynomial expressions in temperature.

Viscosity

The species viscosity data obtained from theoretical predictions

were curve fit to a second degree polynomial in temperature

$$\mu_i = a_i + b_i T + c_i T^2 \quad (3.25)$$

In the above relation temperature is in °K, μ_i is in lbm of 1/ft-sec and the coefficients a_i , b_i , and c_i are given in Table 3.3. For a mixture consisting of n gaseous species viscosity was predicted by the Buddenberg-Wilke model (Ref. 3.5)

$$\mu = \frac{\sum_i^n y_i \mu_i}{\sum_i^n y_i \phi_{ij}} \quad (3.26)$$

where

$$\phi_{ij} = \frac{1}{\sqrt{8}} \left(1 + \frac{M_i}{M_j} \right)^{-\frac{1}{2}} \left[1 + \left(\frac{\mu_i}{\mu_j} \right)^{\frac{1}{2}} \left(\frac{M_j}{M_i} \right)^{\frac{1}{4}} \right]^2 \quad (3.27)$$

and M_i is the i^{th} species molecular weight.

Thermal Conductivity

The species thermal conductivities were obtained from linear fits of theoretical results:

$$K_i = d_i + e_i T \quad (3.28)$$

again T is in °K, k_i is in °K, k_i is in BTU/ft - sec - °R and the coefficients d_i and e_i are given in Table 3.4. Mixture thermal conductivity is compared by

$$k = \frac{\sum_i^n y_i k_i}{\sum_i^n y_i \phi_{ij}} \quad (3.29)$$

where as before ϕ_{ij} is given by Eq. 3.27.

Binary Diffusion

The coefficient of binary diffusion was assumed to be a function of temperature and pressure as follows (Ref. 3.4):

$$D = \frac{8.128 \times 10^{-8} T^{1.659}}{P} \quad (3.30)$$

TABLE 3.3
CONSTANTS FOR VISCOSITY CORRELATION

$$\mu_i = a_i + b_i T + c_i T^2 \quad \frac{\text{lbm}}{\text{ft-sec}}$$

| Species | $a \times 10^5$ | $b \times 10^7$ | $c \times 10^{12}$ | Temperature Range ($^{\circ}\text{K}$) |
|-------------------------------|-----------------|-----------------|--------------------|--|
| O ₂ | 1.693 | 0.1496 | -0.2276 | 2,000-10,000 |
| N ₂ | 0.970 | 0.1613 | -0.1916 | 2,000-10,000 |
| O | 1.519 | 0.1875 | -0.2228 | 2,000-10,000 |
| N | 0.253 | 0.2206 | -0.3737 | 2,000-10,000 |
| O ⁺ | 0.0 | 0.0500 | -0.1000 | 8,000-15,000 |
| N ⁺ | 0.0 | 0.0500 | -0.1000 | 8,000-15,000 |
| e ⁻ | 0.0 | 0.0500 | -0.1000 | 8,000-15,000 |
| C | 1.997 | 0.1772 | -0.3378 | 5,000-10,000 |
| H | 0.294 | 0.0889 | -0.0811 | 4,000-10,000 |
| H ₂ | -0.079 | 0.0791 | -0.0886 | 4,000-10,000 |
| CO | 2.404 | 0.1363 | -0.2184 | 4,000- 9,000 |
| C ₃ | 2.019 | 0.1179 | -0.1655 | 1,000- 5,000 |
| CN | 2.404 | 0.1363 | -0.2184 | 4,000- 9,000 |
| C ₂ H | 2.404 | 0.1363 | -0.2184 | 4,000- 9,000 |
| C ₂ H ₂ | 1.396 | 0.0842 | -0.6939 | 1,000- 5,000 |
| C ₃ H | 2.019 | 0.1179 | -0.1655 | 1,000- 5,000 |
| C ₄ H | 2.019 | 0.1179 | -0.1655 | 1,000- 5,000 |
| HCN | 1.378 | 0.0965 | -0.0948 | 1,000- 5,000 |
| C ₂ | 1.931 | 0.1393 | -0.2575 | 4,000- 9,000 |
| C ⁺ | 0.0 | 0.0500 | -0.1000 | 8,000-15,000 |

TABLE 3.4
CONSTANTS FOR THERMAL CONDUCTIVITY CORRELATION

$$k_i = a_i + b_i T \text{ (BTU/ft-sec-}^\circ\text{R)}$$

| Species | $a \times 10^5$ | $b \times 10^8$ | Temperature Range ($^\circ\text{K}$) |
|-------------------------------|-----------------|-----------------|---|
| O ₂ | 1.019 | 0.4901 | 2,000-10,000 |
| N ₂ | 0.654 | 0.6457 | 2,000-10,000 |
| O | 1.250 | 0.7092 | 2,000-10,000 |
| N | 1.281 | 0.8593 | 2,000-10,000 |
| O ⁺ | 26.0 | 0.0 | 8,000-15,000 |
| N ⁺ | 26.0 | 0.0 | 8,000-15,000 |
| e ⁻ | 26.0 | 0.0 | 8,000-15,000 |
| C | 2.506 | 0.7479 | 5,000-10,000 |
| H | 2.496 | 5.129 | 4,000-10,000 |
| H ₂ | 3.211 | 5.344 | 4,000-10,000 |
| CO | 0.859 | 0.6233 | 1,000- 5,000 |
| C ₃ | 0.630 | 0.5804 | 1,000- 5,000 |
| CN | 0.859 | 0.6233 | 2,000-10,000 |
| C ₂ H | 1.126 | 0.7439 | 1,000- 5,000 |
| C ₂ H ₂ | 1.126 | 0.7439 | 1,000- 5,000 |
| C ₃ H | 0.630 | 0.5804 | 1,000- 5,000 |
| C ₄ H | 0.630 | 0.5804 | 1,000- 5,000 |
| HCN | 0.486 | 0.8714 | 1,000- 5,000 |
| C ₂ | 0.859 | 0.6233 | 1,000- 5,000 |
| C ⁺ | 26.0 | 0.0 | 8,000-15,000 |

where T is in $^{\circ}\text{K}$, P is in atmospheres, and D is in ft^2/sec .

Chemical Kinetics

Formulation of a kinetics model that describes the chemical reactions in the flow-field requires the investigation of innumerable possible reactions that can take place among the air and ablation product species that are present in the shock layer. Because of the complexity associated with kinetics models involving a large number of reactions only those reactions that are important may be considered. In other words, we seek to develop the simplest model capable of accurately describing the chemical kinetics of the flow-field.

Suppose that a number m of chemical reactions of the form

$$\sum_{i=1}^n v_{ij}' S_i \rightleftharpoons \sum_{i=1}^n v_{ij}'' S_i \quad (j = 1, \dots, m) \quad (3.31)$$

where n is the total number of species in the system, and v_{ij}' and v_{ij}'' are the stoichiometric coefficients of the reactants and products in the j^{th} reaction, respectively, have been chosen as a kinetics model, then the rate of generation of species i due to finite-rate chemical reactions (ω_i) is given by the Law of Mass Action (Ref. 3.6) as

$$\begin{aligned} \omega_i = & \sum_j^m (v_{ij}' - v_{ij}'') f_j M_j \prod_{k=1}^n \left(\frac{\rho C_k}{M_k} \right)^{v_{kj}'} \\ & - \sum_j^m (v_{ij}' - v_{ij}'') r_j M_i \prod_{k=1}^n \left(\frac{\rho C_k}{M_k} \right)^{v_{kj}''} \quad (i = 1, \dots, n) \end{aligned} \quad (3.32)$$

where ρ is the density, C_i is the i^{th} species mass fraction, M_i is the molecular weight of species i , and f_j and r_j are the forward and backward reaction rate constants for the j^{th} reaction, respectively. The forward and backward reaction rate constants are generally taken as functions of temperature of the form (Ref. 3.7)

$$f_j = a_{fj} T^{b_{fj}} \exp(-e_{fj}/T) \quad (j = 1, \dots, m) \quad (3.33)$$

$$r_j = a_{rj} T^{b_{rj}} \exp(-e_{rj}/T) \quad (j = 1, \dots, m) \quad (3.34)$$

where a_{fj} , T^{bfj} and a_{rj} , T^{brj} are the frequency factors, and e_{fj} and e_{rj} are the activation energies for the j^{th} forward and backward reactions, respectively.

In practice, the forward reaction rate constant is normally measured experimentally while the backward reaction rate constant is obtained as follows: At equilibrium

$$(v_{ij}'' - v_{ij}') f_j M_i \prod_{k=1}^n \left(\frac{\rho C_k}{M_k}\right)^{v_{kj}'} = (v_{ij}'' - v_{ij}') r_j M_i \prod_{k=1}^n \left(\frac{\rho C_k}{M_k}\right)^{v_{kj}''} \quad (3.35)$$

or

$$\frac{f_j}{r_j} = \prod_{k=1}^n \left(\frac{\rho C_k}{M_k}\right)^{(v_{ki}'' - v_{kj}') } \quad (j = 1, \dots, m) \quad (3.36)$$

The equilibrium constant for the j^{th} reaction (K_j) is given by (Ref. 3.8)

$$K_j = \frac{\prod_{k=1}^n (v_{kj}'' - v_{kj}')}{(RT)^k} \prod_{k=1}^n \left(\frac{\rho C_k}{M_k}\right)^{(v_{kj}'' - v_{kj}') } \quad (j = 1, \dots, m) \quad (3.37)$$

where R is the ideal gas constant. Substitution of Eq. 3.37 in Eq.

3.36 and solving for r_j yields

$$r_j = (RT)^{\sum_{k=1}^n (v_{kj}'' - v_{kj}') } f_j K_j^{-1} \quad (j = 1, \dots, m) \quad (3.38)$$

Therefore, knowing the forward reaction rate constant and the equilibrium constant the backward rate constant can be obtained from Eq. 3.38.

There are two methods commonly used to obtain the equilibrium constant of a reaction. The first method consists of computing the chemical equilibrium composition of the system for a range of different temperatures and using this data with Eq. 3.37 to compute the equilibrium constant for different values of temperature. This set of values of equilibrium constant versus temperature can be curve fit with

analytical expression for K_j

$$K_j = \gamma_j \exp (\beta_j/T) \quad (j = 1, \dots, m) \quad (3.39)$$

where γ_j and β_j are constants. The second method consists of using the expression for the equilibrium constant obtained from thermodynamics (Ref. 3.9)

$$K_j = \exp (-\Delta F_j^\circ / RT) \quad (j = 1, \dots, m) \quad (3.40)$$

where ΔF_j° is the standard free energy for the j th reaction and is given by

$$\Delta F_j^\circ = \sum_{k=1}^n (v_{kj}'' - v_{kj}') F^\circ \quad (j = 1, \dots, m) \quad (3.41)$$

$$\begin{aligned} \frac{\Delta F_j^\circ}{RT} = & B_{1,j} (1 - \ln T) - \frac{B_{2,j}}{2} T - \frac{B_{3,j}}{6} T^2 - \frac{B_{4,j}}{12} T^3 \\ & - \frac{B_{5,j}}{20} T^4 + \frac{B_{6,j}}{T} - B_{7,j} \end{aligned}$$

where

$$B_{\ell,j} = \sum_{k=1}^n (v_{kj}'' - v_{kj}') A_{\ell,k} \quad (\ell = 1, \dots, 7)$$

Therefore the equilibrium constant may be obtained from the species standard free energy expressions given in Tables 3.1 and 3.2. The first method has the advantage that when it is used the backward reaction rate constant has the same form as Eq. 3.34; this can be seen by eliminating f_j and K_j from Eq. 3.38 by substituting Eqs. 3.33 and 3.39 to yield

$$r_j = (RT) \frac{\sum_{k=1}^n (v_{kj}'' - v_{kj}')}{\gamma_j} \frac{a_{fj}^{b_{fj}}}{T^{b_{fj}}} \exp [-(e_{fj} + \beta_j)/T] \quad (j = 1, \dots, m) \quad (3.42)$$

Comparing Eqs. 3.42 and 3.34 it is evident that

$$\begin{aligned} a_{rj} &= \frac{(RT)^k}{\gamma_j} \sum_{k=1}^n (v_{kj}'' - v_{kj}') a_{fj} \\ b_{rj} &= b_{fj} + \sum_{k=1}^n (v_{kj}'' - v_{kj}') \\ e_{rj} &= e_{fj} + \beta_j \end{aligned} \quad (3.43)$$

The second method, on the other hand, does not yield an expression of

the form of Eq. 3.34 since ΔF_j° is a polynomial expression (see Eq. 3.4). The resulting expression is therefore cumbersome to use. On the other hand the second method has the advantage that K_j is readily obtained from the species standard free energy and does not require a routine for computing chemical equilibrium as does the first method.

Up until this point, this discussion has been limited to the case where the kinetics model was already determined; however, as will be shown below, establishing an appropriate kinetics model is not a simple task. Information has been collected over a period of years on the chemistry of the reactions of ablation products and air species, and this information is stored in the form of a computer implemented data management file which presently contains several thousand reactions. It was necessary to examine this extensive set of reactions to arrive at a listing of most probable reactions (Ref. 3.10). These probable reactions were selected involving 28 species. Currently, it is not possible to consider a solution of the conservation equations with this many species and reactions. Furthermore, rate data were not available for many of them. Therefore, this preliminary list was then reexamined for key reactions to represent the chemical system. The selection was based on the species anticipated to have the largest compositions and presumably, therefore, dominate the energy level of the system. These key reactions are given in Table 3.5. In Table 3.6 the previously omitted ablation product and combustion reactions are given. Shown in Table 3.7 are the additional air and hydrogen combustion reactions. The values of the coefficients a_{fj} , b_{fj} , and e_{fj} of the forward reaction rate constants (see Eq. 3.33) for the reactions in Table 3.5 are given in Table 3.8. The forward rate constants listed

TABLE 3.5
SELECTED IMPORTANT CHEMICAL REACTIONS IN
THE SHOCK LAYER OF AN ABLATING BODY

| | | | | | | | | | |
|-----|-------------------------------|---|---|---|------------------|---|----------------|---|---|
| 1. | CO | + | N | = | CN | + | O | | |
| 2. | C ₂ H | + | H | = | C ₂ | + | H ₂ | | |
| 3. | N ₂ | + | M | = | 2N | + | M | | |
| 4. | H ₂ | + | M | = | 2H | + | M | | |
| 5. | O | + | M | = | O 2 | + | E - | + | M |
| 6. | N | + | M | = | N 2 | + | E - | + | M |
| 7. | CN | + | M | = | C | + | N | + | M |
| 8. | CHN | + | M | = | CN | + | H | + | M |
| 9. | C ₂ H | + | M | = | C ₂ | + | H | + | M |
| 10. | C ₂ H ₂ | + | M | = | C ₂ H | + | H | + | M |
| 11. | CO | + | M | = | C | + | O | + | M |
| 12. | C ₃ | + | M | = | C ₂ | + | C | + | M |
| 13. | C ₂ | + | M | = | 2C | + | M | | |
| 14. | C | + | M | = | C 2 | + | E - | + | M |
| 15. | C ₃ H | + | M | = | C ₂ H | + | C | + | M |
| 16. | C ₄ H | + | M | = | C ₃ H | + | C | + | M |

TABLE 3.6 ADDITIONAL ABLATION PRODUCT AND
COMBUSTION REACTIONS

| <u>CH₄, CH₃, CH₂, CH Reactions</u> | <u>C₂H₂, C₂H, Reactions</u> |
|---|---|
| 1. CH ₄ = CH ₃ + H | 1. C ₂ H ₂ + H = C ₂ H + H ₂ |
| 2. CH ₄ = CH ₂ + H ₂ | 2. C ₂ H ₂ + O = CH ₂ + CO |
| 3. CH ₃ = CH ₂ + H | 3. C ₂ H ₂ + OH = C ₂ H + H ₂ O |
| 4. CH ₂ = CH + H | 4. C ₂ H + H = C ₂ + H ₂ |
| 5. CH ₂ + H = CH + H ₂ | 5. C ₂ H + O = CH + CO |
| 6. CH = C + H | |
| 7. C + H ₂ = CH + H | <u>Other Reactions</u> |
| 8. CH ₂ + O = CO + H ₂ | 1. C ₂ + H = C + CH |
| | |
| <u>CN Reactions</u> | |
| 1. 2CN = C ₂ + N ₂ | |
| 2. CN + O = N + CO | |

TABLE 3.7 ADDITIONAL AIR AND HYDROGEN COMBUSTION REACTIONS

Air Reactions

1. $O_2 + M = 2O + M$
2. $N_2 + M = N^+ + e^- + M$
3. $NO + M = N + O + M$
4. $NO + O = O_2 + N$
5. $N_2 + O = NO + N$
6. $N + O = NO^+ + e^-$

Hydrogen Combustion Reactions

1. $H + O_2 = OH + O$
2. $O + H_2 = OH + H$
3. $OH + H_2 = H_2O + H$
4. $2OH = H_2O + O$
5. $H + OH + M = H_2O + M$
6. $H + O + M = OH + M$

TABLE 3.8 COEFFICIENTS OF THE
FORWARD REACTION RATE CONSTANTS

| j^{th} Reaction | a_{fj} | b_{fj} | e_{fj} | Comments |
|--------------------------|----------|----------|----------|------------|
| 1 | 8.0 E 18 | .5 | 71,000 | ϵ |
| 2 | 4.5 E 11 | .5 | 35,000 | Ref. 3.11 |
| 3 | 1.0 E 21 | -1.5 | 224,900 | 3.12 |
| 4 | 3.6 E 18 | -.82 | 103,000 | 3.13 |
| 5 | 2.8 E 12 | .5 | 313,000 | 3.13 |
| 6 | 2.9 E 12 | .5 | 333,000 | 3.13 |
| 7 | 2.2 E 20 | -1.0 | 131,800 | 3.14 |
| 8 | 8.4 E 18 | .5 | 120,000 | ϵ |
| 9 | 9.5 E 18 | .5 | 140,000 | ϵ |
| 10 | 9.5 E 18 | .5 | 117,000 | ϵ |
| 11 | 8.5 E 19 | -1.0 | 257,900 | 3.14 |
| 12 | 1.0 E 19 | .5 | 190,000 | ϵ |
| 13 | 9.3 E 18 | .5 | 155,000 | ϵ |
| 14 | 9.4 E 18 | .5 | 265,000 | ϵ |
| 15 | 9.5 E 18 | .5 | 165,000 | ϵ |
| 16 | 9.5 E 18 | .5 | 145,000 | ϵ |

ϵ Activation energy computed using the method described by Semenov
(Ref. 3.15), and the frequency factor computed using collision
theory (Ref. 3.16).

were the best experimental values or were computed theoretically. With these constants, $R = 1.987 \text{ cal/gmole} - ^\circ\text{K}$ and temperature in $^\circ\text{K}$, the units of f_j are $\text{cm}^3/\text{gm} - \text{mole} - \text{sec}$.

Originally the method proposed to obtain the equilibrium constants for the reactions in Table 3.5 was by a curve fit of equilibrium data. The chemical equilibrium composition for different values of temperature (from $1,000^\circ\text{K}$, to $15,000^\circ\text{K}$) were computed with a free energy minimization chemical equilibrium program. This information was then used with Eq. 3.37 to obtain the value of the equilibrium constant for different temperatures. The results of this operation are shown in Fig. 3.1 for the reaction $\text{C}_3\text{H} \rightleftharpoons \text{C}_2\text{H} + \text{C}$ where $\ln K$ is plotted versus $1/T$ (dotted line). These actual values of K were then least squares fit with an expression of the form of Eq. 3.39. The resulting analytical expression for K is also plotted in Fig. 3.1 (solid line). It can be seen that using the analytical expression for K instead of the actual values introduces significant error, for example at $T = 10,000^\circ\text{K}$ the percentage error in K is 1000. This difficulty encountered in this reaction was found to be typical in some of the other reactions as well. For this reason it was found more desirable to use the second method discussed above for obtaining K_j . This was accomplished by obtaining ΔF_j° from Eq. 3.41 and then using Eq. 3.40 to obtain K_j . Table 3.9 contains the values of the coefficients $B_{1,j}$, $B_{2,j}$, ..., $B_{7,j}$ of Eq. 3.41 for the reactions in Table 3.5. The equilibrium constant K_j obtained, if plotted in Fig. 3.1, is undistinguishable from the curve of the actual values of K .

Radiative Properties

The radiation flux divergence ($dq_{R,y}/dy$) appearing in Eq. 2.61

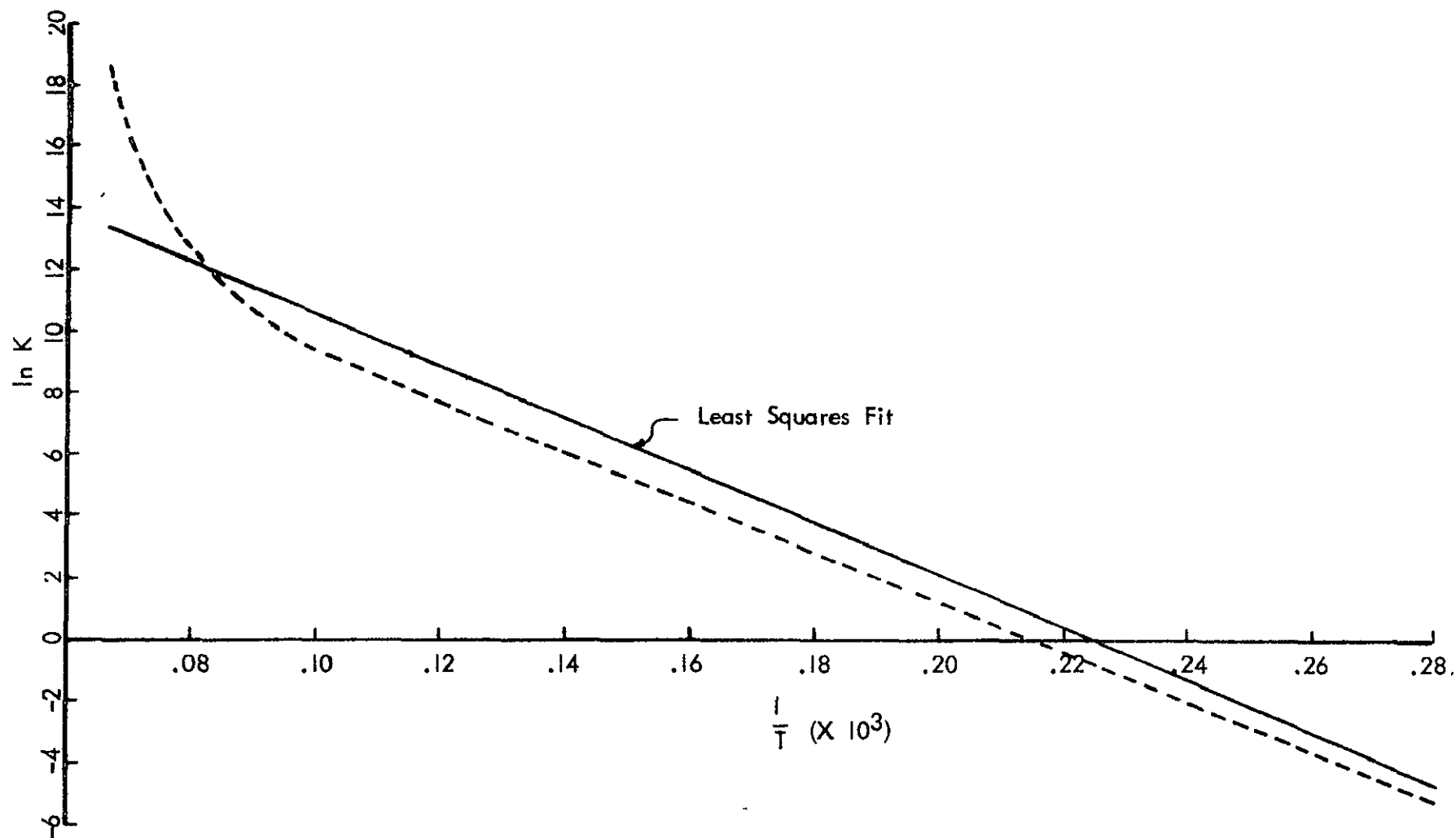


Figure 3.1 Equilibrium Constant and Its Linear Least Squares Fit For The Reaction $\text{C}_3\text{H} \rightleftharpoons \text{C}_2\text{H} + \text{C}$

TABLE 3.9

COEFFICIENTS OF THE EQUILIBRIUM CONSTANT CORRELATION

| REACTION(*) | B _{1,j} | B _{2,j} | B _{3,j} | B _{4,j} | B _{5,j} | B _{6,j} | B _{7,j} | T(**) |
|-------------|----------------------------------|----------------------------------|-----------------------|-----------------------|------------------|------------------|------------------|-------|
| 1 | 0.3530E+00-0.7681E-03 | 0.5153E-06-0.9618E-10 | 0.5427E-14 | 0.3485E+05 | 0.7500E-01 | | | L |
| | -0.9100E-01 | 0.2624E-03-0.8700E-09-0.5441E-11 | 0.2100E-15 | 0.4160E+05 | 0.2066E+01 | | | H |
| 2 | 0.1816E+01-0.3571E-02 | 0.1634E-05-0.2783E-09 | 0.1617E-13 | 0.1329E+05-0.8961E+01 | | | | L |
| | -0.1852E+01 | 0.1831E-02-0.5850E-06 | 0.7641E-10-0.3417E-14 | 0.1329E+05 | 0.1126E+02 | | | H |
| 3 | 0.1727E+01-0.8059E-03 | 0.1344E-06 | 0.4980E-11-0.9780E-15 | 0.1132E+06 | 0.4274E+01 | | | L |
| | 0.1765E+01-0.1250E-02 | 0.3816E-06-0.3536E-10 | 0.1003E-14 | 0.1132E+06 | 0.4450E+01 | | | H |
| 4 | 0.1642E+01-0.2810E-03-0.9244E-07 | 0.2914E-10-0.2112E-14 | 0.5196E+05 | 0.2626E+01 | | | | L |
| | 0.4505E+01-0.4018E-02 | 0.1254E-05-0.1592E-09 | 0.7013E-14 | 0.5196E+05-0.1348E+02 | | | | H |
| 5 | 0.2321E+01 | 0.2250E-03-0.9094E-07 | 0.1275E-10-0.5056E-15 | 0.1580E+06-0.1181E+02 | | | | L |
| | 0.2904E+01-0.3576E-03 | 0.6591E-07-0.3159E-11 | 0.2813E-16 | 0.1580E+06-0.1538E+02 | | | | H |
| 6 | 0.2753E+01-0.3726E-03 | 0.1884E-06-0.3765E-10 | 0.2271E-14 | 0.1686E+06-0.1238E+02 | | | | L |
| | 0.2261E+01 | 0.3808E-03-0.1210E-06 | 0.1070E-10-0.3032E-15 | 0.1686E+06-0.1000E+02 | | | | H |
| 7 | 0.1675E+01-0.6017E-03-0.6914E-07 | 0.3996E-10-0.2991E-14 | 0.9406E+05 | 0.3698E+01 | | | | L |
| | 0.1414E+01-0.8027E-03 | 0.1697E-06-0.1315E-10 | 0.3831E-15 | 0.8731E+05 | 0.5594E+01 | | | H |
| 8 | 0.2257E+01-0.2955E-02 | 0.1359E-05-0.2518E-09 | 0.1668E-13 | 0.5850E+05 | 0.1912E+01 | | | L |
| | 0.3753E+01-0.4486E-02 | 0.1768E-05-0.2902E-09 | 0.1768E-13 | 0.6525E+05-0.6819E+01 | | | | H |

* SEE TABLE 3.5 FOR REACTIONS

** TEMPERATURE RANGE L= 1,000-6,000 DEG. K, H= 6,000-15,000 DEG. K

TABLE 3.9

COEFFICIENTS OF THE EQUILIBRIUM CONSTANT CORRELATION (Continued)

| REACTION(*) | B _{1,j} | B _{2,j} | B _{3,j} | B _{4,j} | B _{5,j} | B _{6,j} | B _{7,j} | T(**) |
|-------------|----------------------------------|-----------------------|-----------------------|-----------------------|------------------|------------------|------------------|-------|
| 9 | 0.3458E+01-0.3852E-02 | 0.1541E-05-0.2492E-09 | 0.1406E-13 | 0.6525E+05-0.6335E+01 | | | | L |
| | 0.2653E+01-0.2187E-02 | 0.6688E-06-0.8277E-10 | 0.3596E-14 | 0.6525E+05-0.2220E+01 | | | | H |
| 10 | 0.2094E+01-0.2155E-02 | 0.7206E-06-0.1067E-09 | 0.5735E-14 | 0.5766E+05 | 0.3671E+01 | | | L |
| | 0.2452E+01-0.2382E-02 | 0.6930E-06-0.8428E-10 | 0.3630E-14 | 0.5766E+05 | 0.1504E+01 | | | H |
| 11 | 0.2028E+01-0.1370E-02 | 0.4461E-06-0.5622E-10 | 0.2436E-14 | 0.1289E+06 | 0.3773E+01 | | | L |
| | 0.1323E+01-0.5403E-03 | 0.1688E-06-0.1859E-10 | 0.5931E-15 | 0.1289E+06 | 0.7660E+01 | | | H |
| 12 | 0.3053E+01-0.4032E-02 | 0.1731E-05-0.2858E-09 | 0.1621E-13 | 0.8906E+05 | 0.1034E+01 | | | L |
| | -0.1596E+02 | 0.1840E-01-0.5690E-05 | 0.6841E-09-0.2842E-13 | 0.8906E+05 | 0.1101E+03 | | | H |
| 13 | 0.7810E+00-0.1175E-03-0.8460E-07 | 0.2854E-10-0.2197E-14 | 0.7297E+05 | 0.9378E+01 | | | | L |
| | 0.2560E+00 | 0.1581E-03-0.3970E-07 | 0.2542E-11 | 0.2920E-17 | 0.7297E+05 | 0.1266E+02 | | H |
| 14 | 0.2497E+01 | 0.6404E-04-0.5011E-07 | 0.6619E-11-0.2271E-15 | 0.1306E+06-0.1216E+02 | | | | L |
| | 0.2895E+01-0.3234E-03 | 0.4932E-07-0.2579E-11 | 0.2381E-16 | 0.1306E+06-0.1481E+02 | | | | H |
| 15 | 0.2132E+01-0.2840E-02 | 0.1138E-05-0.2021E-09 | 0.1335E-13 | 0.8068E+05 | 0.5461E+01 | | | L |
| | 0.3483E+01-0.4981E-02 | 0.2072E-05-0.3588E-09 | 0.2234E-13 | 0.8068E+05-0.1881E+01 | | | | H |
| 16 | 0.7030E+00-0.1406E-02 | 0.5735E-06-0.8895E-10 | 0.4609E-14 | 0.7220E+05 | 0.1162E+02 | | | L |
| | 0.2320E+00-0.8811E-03 | 0.4090E-06-0.6840E-10 | 0.3694E-14 | 0.7220E+05 | 0.1435E+02 | | | H |

* SEE TABLE 3.5 FOR REACTIONS

** TEMPERATURE RANGE L= 1,000-6,000 DEG. K. H= 6,000-15,000 DEG. K

is defined as follows (see Eq. 2.14)

$$\frac{dq_{R,y}}{dy} = - 2 \pi \int \alpha_v (2B_v - I_v) dv \quad (3.44)$$

where

$q_{R,y}$ = radiative flux in a direction normal to the body

α_v = volumetric absorption coefficient,

B_v = Plankian radiation intensity,

I_v = spectral radiation intensity,

v = frequency

Radiative mechanisms in high temperature gases may be categorized into those which produce radiation of a given frequency and those which produce radiation over a wide spectrum. The first of these groups of mechanisms is known as a line radiation mechanism (results from electronic transition between the bound energy levels in atoms or molecules), while the second group are commonly referred to as continuum radiation mechanisms (transfer between ionic states for atomic and molecular species, and transitions between two free energy levels in which free electrons are present in both the initial and final states).

One consequence of the existence of line and continuum radiative mechanisms is that the volumetric absorption coefficient (α_v) for a high temperature gas varies discontinuously with wavelength. This fact requires that the integration of Eq. 3.44 be carried out on a piecewise basis over the frequency domain. The frequency range is divided into regions (bands) within which the discontinuous variations are averaged. Continuum radiation bands are used to represent regions of continuous radiation while line radiation bands are used to model the effect of the various discontinuous contributions. As in the numerical

integration of continuous functions, the use of more bands (smaller intervals) leads to a more accurate representation of the radiative process.

The radiative model used in the present work was developed by Engel (Ref. 3.17) from a model originally developed by Wilson (Ref. 3.18). The computer program developed by Engel computes both the radiative flux ($q_{R,y}$), and the radiative flux divergence ($dq_{R,y}/dy$) for a mixture of air and ablation products. The model uses nine line frequency bands and twelve continuum bands. It considers the following species and radiation mechanisms:

| | | | | |
|---|-----------|----------------|------------------|-----------|
| H | | O ₂ | CO | |
| C | Line and | N ₂ | C ₃ | Continuum |
| O | Continuum | C ₂ | C ₂ H | |
| N | | H ₂ | | |

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CHAPTER 4

THE SOLUTION METHOD

The mathematical model of stagnation-line flow contains a series of assumptions designed to simplify the mathematics of the problem without significantly reducing the accuracy of the solution. However, the resulting equations are still non-linear, integral, and coupled and have variable coefficients, therefore an analytical solution cannot be obtained. This means that integration of the equations must be accomplished numerically. Before an attempt is made to formulate a numerical solution scheme, it is convenient to nondimensionalize and transform the equations of the stagnation line model (Table 2.9). After the model has been transformed, the behavior of specific terms, equations, and the entire system of equations will be investigated so that a solution method may be established. The solution method will then be implemented with a computer program.

NONDIMENSIONALIZATION AND TRANSFORMATION

Using the dimensionless variables given in Table 4.1, the stagnation line model can be written in the dimensionless form given in Table 4.2. Notice that the only difference between the dimensionless variables given in Table 4.1 and those used for the order of magnitude assesment in Chapter 2 (Table 2.2) is that in the former $\rho_{\delta,0}^*$ is used in place of ρ_{∞}^* . This has the effect of introducing into the momentum equation (Eq. 4.2) a Reynolds Number based upon post-shock density as opposed to the free-stream

TABLE 4.1

DIMENSIONLESS VARIABLES

$$u = \frac{u^*}{U_\infty^*}$$

$$v = \frac{v^*}{U_\infty^*}$$

$$P = \frac{P^*}{\rho_{\delta,o}^* U_\infty^{*2}}$$

$$\rho = \frac{\rho^*}{\rho_{\delta,o}^*}$$

$$x = \frac{x^*}{R^*}$$

$$y = \frac{y^*}{R^*}$$

$$T = \frac{T^*}{T_{\delta,o}^*}$$

$$h = \frac{h^*}{\frac{1}{2} U_\infty^{*2}}$$

$$h_1 = \frac{h_1^*}{\frac{1}{2} U_\infty^{*2}}$$

$$C_p = \frac{T_{\delta,o}^* C_p^*}{\frac{1}{2} U_\infty^{*2}}$$

$$D = \frac{D^*}{R^* U_\infty^*}$$

$$\mu = \frac{\mu^*}{\mu_{\delta,o}^*}$$

$$k = \frac{T_{\delta,o}^* k^*}{R^* P_{\delta,o}^* U_\infty^{*3}}$$

$$\omega_i = \frac{R^* \omega_i^*}{P_{\delta,o}^* U_\infty^*}$$

$$J_1 = \frac{J_1^*}{\rho_{\delta,o}^* U_\infty^*}$$

$$q_{R,y} = \frac{q_{R,y}^*}{\rho_{\delta,o}^* U_\infty^{*3}}$$

$$\frac{R}{M_i} = \frac{T_{\delta,o}^*}{U_\infty^{*2}} \frac{R^*}{M_i^*}$$

$$Re_\delta = \frac{\rho_{\delta,o}^* U_\infty^* R^*}{\mu_{\delta,o}^*}$$

$$\bar{\rho} = \frac{\rho_\infty^*}{\rho_{\delta,o}^*}$$

$$\delta = \frac{\delta^*}{R^*}$$

TABLE 4.2

DIMENSIONLESS STAGNATION LINE MODEL

EQUATIONS:

Species Continuity:

$$\frac{d}{dy} (\rho D \frac{dC_i}{dy}) - \rho v \frac{dC_i}{dy} + \omega_i = 0 \quad (i = 1, \dots, n) \quad (4.1)$$

X-momentum:

$$\begin{aligned} \frac{1}{Re_s} \frac{d}{dy} \left[\mu \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \right] - \rho v \frac{d}{dy} \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right) \\ + \frac{1}{2} \rho \left(\frac{1}{\rho} \frac{d(\rho v)}{dy} \right)^2 - 4 \bar{\rho} (1 - \bar{\rho}) \left(\frac{d\phi}{dx} \right)_{x=0}^2 = 0 \end{aligned} \quad (4.2)$$

Energy:

$$\begin{aligned} \frac{2k}{C_p} \frac{d^2 h}{dy^2} + \left[-\rho v + \frac{d}{dy} \left(\frac{2k}{C_p} \right) \right] \frac{dh}{dy} = \frac{2dq_{R,y}}{dy} \\ + 2\rho v^2 \frac{dv}{dy} + \frac{d}{dy} \left(\frac{2k}{C_p} - \rho D \right) \left(\sum_i^n h_i \frac{dC_i}{dy} \right) \end{aligned} \quad (4.3)$$

Caloric Equation of state:

$$h = \sum_i^n C_i h_i(T) \quad (4.4)$$

TABLE 4.2 (CONTINUED)

Thermal Equation of State:

$$\rho \left[R T + \left(\sum_{i=1}^n \frac{C_i}{M_i} \right) \right] = P_\delta \quad (4.5)$$

Boundary Conditions:

Wall ($y = 0$)

$$1. \quad \rho v = (\rho v)_w$$

$$2. \quad \frac{d(\rho v)}{dy} = 0$$

$$3. \quad C_i = C_{i,w} \quad (i = 1, \dots, n)$$

$$4. \quad h = h_w$$

Shock ($y = \delta$)

$$1. \quad \rho v = (\rho v)_\delta = v_\delta$$

$$2. \quad \frac{d(\rho v)}{dy} = -2 \left(\frac{\partial u}{\partial x} \right)_{x=\delta} \quad (4.6)$$

$$3. \quad C_i = C_{i,\delta} \quad (i = 1, \dots, n)$$

$$4. \quad h = h_\delta$$

Reynolds number previously used.

The mathematical model, as originally developed in Chapter 2 (Table 2.9), included the energy equation in temperature form (Eq. 2.61). During the present investigation the early attempts at solving the model concentrated on this formulation. However, after much computational experimentation led consistently to failure, it was decided to try a different formulation of the energy equation. The energy equation in terms of dimensionless temperature

$$\begin{aligned} \frac{d(k \frac{dT}{dy})}{dy} - \frac{1}{2} \rho v C_p \frac{dT}{dy} + \frac{1}{2} \frac{d[\rho D (\sum_{i=1}^n h_i \frac{dC_i}{dy})]}{dy} \\ - \rho v^2 \frac{dv}{dy} - \frac{1}{2} \rho v (\sum_{i=1}^n h_i \frac{dC_i}{dy}) - \frac{dq_{R,y}}{dy} = 0 \end{aligned} \quad (4.3a)$$

was transformed to an enthalpy form by nondimensionalizing Eq. 2.60 to yield

$$\frac{dT}{dy} = \frac{1}{C_p} \frac{dh}{dy} - \frac{1}{C_p} \sum_{i=1}^n h_i \frac{dC_i}{dy} \quad (4.3b)$$

and substituting Eq. 4.3b in Eq. 4.3a to yield Eq. 4.3 (see Table 4.2).

The model with the temperature form of the energy equation consisted of $3 + n$ equations in the unknowns ρ , v , T and C_i ($i = 1, \dots, n$). The modified model includes the enthalpy as an additional unknown (temperature continues to be an unknown since the thermodynamic, transport, radiative and chemical kinetics properties are explicit function of T), and therefore requires an additional equation. This equation is the caloric equation of state (Eq. 4.4). The boundary conditions on the energy equation are now $h = h_w$ at $y = 0$ and $h = h_\delta$ at $y = \delta$.

The equations given in Table 4.2 can now be transformed using the Dorodnitsyn transformation defined as:

$$\eta = \frac{\int_0^y \frac{\rho}{\delta} dy}{\int_0^y \rho dy} = \frac{\int_0^y \frac{\rho}{\tilde{\delta}} dy}{\tilde{\delta}} \quad (4.7)$$

From the above equation

$$\frac{d}{dy} = \frac{\rho}{\delta} \frac{d}{dn} \quad (4.8)$$

and this relation yields the equations given in Table 4.3. Notice that the Dorodnitsyn transformation converts the independent variable y to the new independent variable η . Before proceeding to discuss the numerical solution to Eqs. 4.9 - 4.11 and Eqs. 4.4 and 4.5 it is convenient to rewrite the equations in a form more suitable for numerical solution.

Species continuity:

Expanding the second order term in Eq. 4.9 and rearranging gives

$$\rho^2 D \frac{d^2 C_i}{d\eta^2} + \left[\frac{d(\rho^2 D)}{d\eta} - \tilde{\delta} \rho v \right] \frac{dC_i}{d\eta} + \frac{\tilde{\delta}^2}{\rho} \omega_i = 0 \quad (i = 1, \dots, n) \quad (4.13)$$

X-momentum:

Let us define a new dependent variable as follows

$$f = \frac{\rho v}{\left(\frac{d(\rho v)}{d\eta} \right)_{\eta=1}} \quad (4.14)$$

therefore

$$\frac{df}{d\eta} = \frac{\frac{d(\rho v)}{d\eta}}{\left(\frac{d(\rho v)}{d\eta} \right)_{\eta=1}} \quad (4.15)$$

When the global continuity equation is nondimensionalized and transformed into η - space the result is

$$2 \left(\frac{\partial u}{\partial x} \right)_{x=0} = - \frac{1}{\tilde{\delta}} \frac{d(\rho v)}{d\eta} \quad (4.16)$$

TABLE 4.3

TRANSFORMED STAGNATION LINE MODEL

EQUATIONS:

Species Continuity:

$$\frac{d}{d\eta} (\rho^2 D \frac{dC_i}{d\eta}) - \tilde{\delta} \rho v \frac{dC_i}{d\eta} + \frac{\tilde{\delta}^2}{\rho} \omega_i = 0 \quad (i = 1, \dots, n) \quad (4.9)$$

X-momentum:

$$\begin{aligned} \frac{d}{d\eta} [\rho \mu \frac{d^2(\rho v)}{d\eta^2}] - \tilde{\delta} Re_{\delta} \rho v \frac{d^2(\rho v)}{d\eta^2} \\ + \frac{\tilde{\delta}}{2} Re_{\delta} (\frac{d(\rho v)}{d\eta})^2 - 4\tilde{\delta}^3 Re_{\delta} \frac{\rho}{\rho} (1-\rho) (\frac{d\phi}{dx})^2_x = 0 \end{aligned} \quad (4.10)$$

Energy:

$$\frac{2}{\tilde{\delta}} \frac{\rho k}{C_p} \frac{d^2 h}{d\eta^2} + [-\rho v + \frac{1}{\tilde{\delta}} \frac{d}{d\eta} (\frac{2\rho k}{C_p})] \frac{dh}{d\eta} = \quad (4.11)$$

$$2 \frac{\tilde{\delta}}{\rho} \frac{dq_{R,y}}{dy} + 2\rho v^2 \frac{dv}{d\eta} + \frac{1}{\tilde{\delta}} \frac{d}{d\eta} (\frac{2\rho k}{C_p} - \rho^2 D) (\sum_1^n h_i \frac{dC_i}{d\eta})$$

Caloric Equation of State:

$$h = \sum_i C_i h_i \quad (4.4)$$

TABLE 4.3 (CONTINUED)

Thermal Equation of State:

$$\rho R T \left(\sum_{i=1}^n \frac{C_i}{M_i} \right) = P_\delta \quad (4.5)$$

Boundary Conditions:

Wall ($\eta = 0$)Shock ($\eta = 1$)

1. $\rho v = (\rho v)_w$

1. $\rho v = (\rho v)_\delta = v_\delta$

(4.12)

2. $\frac{d(\rho v)}{d\eta} = 0$

2. $\frac{d(\rho v)}{d\eta} = -2\tilde{\delta} \left(\frac{\partial u_\delta}{\partial x} \right)_{x=0}$

3. $C_i = C_{i,w} \quad (i = 1, \dots, n)$

3. $C_i = C_{i,\delta} \quad (i = 1, \dots, n)$

4. $h = h_w$

4. $h = h_\delta$

Substitution of Eq. 4.16 in Eqs. 4.14 and 4.15 gives

$$f = - \frac{\rho v}{2\tilde{\delta}} \frac{\partial u_{\delta,0}}{\partial x} \quad (4.17)$$

$$\frac{df}{d\eta} = \frac{\left(\frac{\partial u}{\partial x}\right)_{x=0}}{\left(\frac{\partial u_{\delta}}{\partial x}\right)_{x=0}} \quad (4.18)$$

Making use of L'Hospital's rule,

$$\lim_{x \rightarrow 0} \frac{u}{u_{\delta}} = \frac{\left(\frac{\partial u}{\partial x}\right)_{x=0}}{\left(\frac{\partial u_{\delta}}{\partial x}\right)_{x=0}} \quad (4.19)$$

and

$$\frac{df}{d\eta} \approx \lim_{x \rightarrow 0} \frac{u}{u_{\delta}} \quad (4.20)$$

follows from Eqs. 4.18 and 4.19.

Substitution of Eq. 4.17 in Eq. 4.10 gives

$$\begin{aligned} \frac{d}{d\eta} \left(\mu \frac{d^2 f}{d\eta^2} \right) + R_{e_{\delta}}^{-2} \tilde{\delta}^2 \left(\frac{\partial u_{\delta,0}}{\partial x} \right) \left[2f \frac{d^2 f}{d\eta^2} - \left(\frac{df}{d\eta} \right)^2 \right. \\ \left. + 2 \frac{\rho}{\rho} (1 - \rho) \frac{\left(\frac{d\phi}{dx} \right)^2_{x=0}}{\frac{\partial u_{\delta,0}}{\partial x}} \right] = 0 \end{aligned} \quad (4.21)$$

The boundary conditions for this equation are

Wall ($\eta = 0$):

$$1. f = - \frac{(\rho v) w}{2\tilde{\delta}} \frac{\partial u_{\delta,0}}{\partial x}$$

$$2. \frac{df}{d\eta} = 0$$

Shock ($\eta = 1$):

$$1. f = - \frac{v_{\delta}}{2\tilde{\delta}} \frac{\partial u_{\delta,0}}{\partial x}$$

$$2. \frac{df}{d\eta} = 1$$

The momentum equation, being third order, can be expanded into two coupled equations; one first order and the other second order by defining

$$Z = \frac{1}{\tilde{\delta}} \frac{df}{d\eta} \quad (4.22)$$

and substituting into Eq. 4.21

$$\begin{aligned} \rho u \frac{d^2 Z}{d\eta^2} + [2R_{e\delta} \tilde{\delta} \frac{\partial u_{\delta,o}}{\partial x} f + \frac{d(\rho u)}{d\eta}] \frac{dZ}{d\eta} \\ - R_{e\delta} \tilde{\delta}^3 \frac{\partial u_{\delta,o}}{\partial x} Z^2 + 2R_{e\delta} \tilde{\delta} \frac{\bar{\rho}}{\rho} (1 - \bar{\rho}) \frac{(\frac{d\phi}{dx})^2}{\frac{\partial u_{\delta,o}}{\partial x}} = 0 \end{aligned} \quad (4.23)$$

The resulting boundary conditions for Eqs. 4.22 and 4.23 are

Wall ($\eta = 0$):

Shock ($\eta = 1$):

$$1. \quad f = f_w = - \frac{(\rho v)w}{2\tilde{\delta} \frac{\partial u_{\delta,o}}{\partial x}}$$

$$1. \quad f = f_\delta = - \frac{v_\delta}{2\tilde{\delta} \frac{\partial u_{\delta,o}}{\partial x}}$$

$$2. \quad Z = 0$$

$$2. \quad Z = 1/\tilde{\delta}$$

As was discussed in Chapter 2, four boundary conditions need to be satisfied by the x - momentum equation, three because the equation is third order and the other since the shock stand-off distance (δ) is unknown.

In terms of Eqs. 4.22 and 4.23, this means that these equations must satisfy all four of Eqs. 4.24. This is done in practice by applying the two boundary conditions on Z and f shown above to Eqs. 4.23 and 4.22, respectively. Integration of Eq. 4.22 gives

$$f = \tilde{\delta} \int_0^\eta Z d\eta + f_w \quad (4.25)$$

Evaluating this relation at $\eta = 1$ and solving for $\tilde{\delta}$ yields

$$\tilde{\delta} = \frac{f_\delta - f_w}{\int_0^1 Z d\eta} \quad (4.26)$$

It is easily seen that substitution of Eqs. 4.25 and 4.26 in Eq. 4.23 yields a second order, non-linear, integro-differential equation with Z as the only dependent variable (assuming that $\rho(\eta)$ and $\mu(\eta)$ are known). This equation can be solved for Z and then Eq. 4.25 is used to complete the solution to the problem. An alternative method of solution used in the present work is iteration: guess a value of $\tilde{\delta}$ and solve Eqs. 4.23 and 4.25 for f and Z , use the computed Z in Eq. 4.26 to obtain a new value of $\tilde{\delta}$ and repeat the above procedure using the new value of $\tilde{\delta}$ as guess until the guessed and computed values of $\tilde{\delta}$ are equal.

Energy:

Expansion of the second term on the right-hand side of Eq. 4.11 gives

$$\begin{aligned} \frac{1}{\tilde{\delta}} \frac{d}{d\eta} \left[\left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i h_i \frac{dC_i}{d\eta} \right) \right] &= \frac{1}{\tilde{\delta}} \frac{d}{d\eta} \left[\left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i h_i \frac{dC_i}{d\eta} \right) \right] \\ &+ \frac{1}{\tilde{\delta}} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i \frac{dh_i}{d\eta} \frac{dC_i}{d\eta} \right) + \frac{1}{\tilde{\delta}} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i h_i \frac{d^2 C_i}{d\eta^2} \right) \end{aligned} \quad (4.27)$$

or

$$\frac{2\rho k}{\tilde{\delta} C_p} \frac{d^2 h}{d\eta^2} + \left[-\rho v + \frac{1}{\tilde{\delta}} \frac{d}{d\eta} \left(\frac{2\rho k}{C_p} \right) \right] \frac{dh}{d\eta} = \quad (4.28)$$

$$\begin{aligned} \frac{2\tilde{\delta}}{\rho} \frac{dq_{R,y}}{dy} + 2\rho v^2 \frac{dv}{d\eta} + \frac{1}{\tilde{\delta}} \frac{d}{d\eta} \left[\left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i h_i \frac{dC_i}{d\eta} \right) \right] \\ + \frac{1}{\tilde{\delta}} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i \frac{dh_i}{d\eta} \frac{dC_i}{d\eta} \right) + \frac{1}{\tilde{\delta}} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i h_i \frac{d^2 C_i}{d\eta^2} \right) \end{aligned}$$

A summary of the dimensionless transformed equations is given in Table 4.4.

METHODS OF COMPUTING REACTING FLOWS

Historically the first attempts at solving the finite-rate chemistry

TABLE 4.4

SUMMARY OF DIMENSIONLESS TRANSFORMED EQUATIONS

EQUATIONS:

Species Continuity:

$$\rho^2_D \frac{d^2 C_i}{d\eta^2} + \left[\frac{d(\rho^2_D)}{d\eta} - \tilde{\delta} \rho' v \right] \frac{dC_i}{d\eta} + \frac{\tilde{\delta}^2}{\rho} \omega_i = 0 \quad (i = 1, \dots, n) \quad (4.13)$$

X-Momentum:

$$f = - \frac{\rho v}{2\tilde{\delta} \frac{\partial u_{\delta,0}}{\partial x}} \quad (4.17)$$

$$\begin{aligned} \rho \mu \frac{d^2 z}{d\eta^2} + [2R_{e_\delta} \tilde{\delta}^2 \left(\frac{\partial u_{\delta,0}}{\partial x} \right) f + \frac{d(\rho \mu)}{d\eta}] \frac{dz}{d\eta} \\ - R_{e_\delta} \tilde{\delta}^3 \left(\frac{\partial u_{\delta,0}}{\partial x} \right) z^2 + 2R_{e_\delta} \tilde{\delta} \frac{\bar{\rho}}{\rho} (1 - \bar{\rho}) \frac{\left(\frac{d\phi}{dx} \right)^2}{\frac{\partial u_{\delta,0}}{\partial x}} = 0 \end{aligned} \quad (4.23)$$

$$f = \tilde{\delta} \int_0^\eta z \, d\eta + f_w \quad (4.25)$$

$$\tilde{\delta} = \frac{f_\delta - f_w}{\int_0^1 z \, d\eta} \quad (4.26)$$




TABLE 4.4 (CONTINUED)

Energy:

$$\begin{aligned}
& \frac{2}{\delta} \frac{\rho k}{C_p} \frac{d^2 h}{d\eta^2} + \left[-\rho v + \frac{1}{\delta} \frac{d}{d\eta} \left(\frac{2\rho k}{C_p} \right) \right] \frac{dh}{d\eta} = \\
& \frac{2\delta}{\bar{\rho}} \frac{dq_{R,y}}{dy} + 2\rho v^2 \frac{dv}{d\eta} + \frac{1}{\delta} \frac{d}{d\eta} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i^n h_i \frac{dC_i}{d\eta} \right) \\
& + \frac{1}{\delta} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i^n \frac{dh_i}{d\eta} \frac{dC_i}{d\eta} \right) + \frac{1}{\delta} \left(\frac{2\rho k}{C_p} - \rho^2 D \right) \left(\sum_i^n h_i \frac{d^2 C_i}{d\eta^2} \right) \quad (4.28)
\end{aligned}$$

Caloric Equation of State:

$$h = \sum_i^n C_i h_i \quad (4.4)$$

Thermal Equation of State:

$$p = R T \left(\sum_i^n \frac{C_i}{M_i} \right) = P_\delta \quad (4.5)$$

problem were concentrated upon one-dimensional flows with no diffusion. The equations describing this physical situation are of the following form:

$$\frac{dy_i}{dt} = f_i(y_1, y_2, \dots, y_n) \quad (i = 1, 2, \dots, n) \quad (4.29)$$

$$y_i(0) = Y_{i,0}$$

where the concentrations of species in an n-species system (y_1, \dots, y_n) are the dependent variables and time (t) is the independent variable. These equations are non-linear, coupled through the right-hand-side term (f_1, \dots, f_n), and are subject to initial boundary conditions of the first kind. In attempting to numerically solve the above set of equations it was found that the classical integration techniques (Euler, Runge-Kutta, etc.) could be used to compute the solution for cases where the reaction rates were relatively slow, but for fast reaction rates (those occurring when the flow is near chemical equilibrium) the time required to integrate the equations became exorbitant. When the latter condition existed the equations were referred to as "stiff" equations by Curtiss and Hirschfelder (Ref. 4,1) because this type of problem arises also in overcontrolled servomechanical systems.

Numerical techniques for solving differential equations are generally judged on the basis of stability, accuracy, and speed of computation. Stability means that the numerical solution must follow the general form of the true solution. Accuracy measures how closely the numerical solution approximates the true solution. It should be clear that an algorithm may be stable but inaccurate. The computational speed depends upon the allowable stepsize and on the computational effort required to perform each step.

The solution of Eqs. 4.29 is normally carried out by locally linearizing the equations and solving at each step the resulting set of linear equations. The linearization is carried out by expanding each f_i in a truncated Taylor series about a local point t_m to yield

$$\frac{dy_i}{dt} = f_{i,m} + \sum_{j=1}^n \left(\frac{\partial f_i}{\partial y_j} \right)_m (y_j - y_{j,m}) \quad (i = 1, \dots, n) \quad (4.30)$$

or

$$\frac{dy_i}{dt} = \sum_{j=1}^n A_{ij} y_j + B_i \quad (i = 1, \dots, n) \quad (4.31)$$

where

$$A_{ij} = \left(\frac{\partial f_i}{\partial y_j} \right)_m \quad \text{and} \quad B_i = F_{i,m} - \sum_{j=1}^n \left(\frac{\partial f_i}{\partial y_j} \right)_m y_{j,m}$$

Equation 4.31 may be written as

$$\frac{d\bar{y}}{dt} = \bar{A} \bar{y} + \bar{B} \quad (4.32)$$

using matrix notation where $\bar{y} = [y_1, y_2, \dots, y_n]^T$,

$$\bar{B} = [B_1, B_2, \dots, B_n]^T \quad \text{and}$$

$$\bar{A} = \begin{bmatrix} \frac{\partial f_1}{\partial y_1} & \frac{\partial f_1}{\partial y_2} & \dots & \frac{\partial f_1}{\partial y_n} \\ \frac{\partial f_2}{\partial y_1} & \frac{\partial f_2}{\partial y_2} & \dots & \frac{\partial f_2}{\partial y_n} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{\partial f_n}{\partial y_1} & \frac{\partial f_n}{\partial y_2} & \dots & \frac{\partial f_n}{\partial y_n} \end{bmatrix}$$

Techniques for solving Eq. 4.32 are generally classified as explicit, implicit and locally exact. Explicit techniques are those for which the values of each of the dependent variables at any step may be expressed explicitly in terms of the dependent variables at preceding steps. All

predictor-corrector methods, as well as all variants of the Runge-Kutta method are explicit. Implicit techniques are those for which the values of each of the variables at any step depends on itself and on the other dependent variables at that step as well as on the dependent variables at preceeding steps. An implicit technique when applied to Eq. 4.32 always results in a set of linear algebraic simultaneous equations which must be solved for the vector \bar{y} at each step. Locally exact techniques solve Eq. 4.32 exactly to yield

$$y_i = \sum_{j=1}^n C_{ij} y_{1j} \exp(\lambda_j h) + y_{i,p} \quad (i = 1, \dots, n) \quad (4.33)$$

where the C_{ij} are integration constants, the λ_j are the eigenvalues of \bar{A} , h is the stepsize and $y_{i,p}$ are particular solutions to Eq. 4.32. This solution assumes that all the eigenvalues of \bar{A} are distinct.

All explicit techniques have finite stability boundaries, that is, they are stable for some stepsizes and unstable for others. Under conditions of stiffness, explicit techniques must use a very small stepsize in order to remain stable. On the other hand, they can be very accurate and require a modest degree of computational effort to compute a step. Some implicit techniques are unconditionally stable, in other words, the allowable stepsize is not restricted by stability requirements. However, their accuracy is generally limited and, because a set of linear algebraic equations must be solved, they require a fair amount of computational effort to perform a step. Locally exact methods are unconditionally stable and very accurate but, because the eigenvalues of \bar{A} must be computed for each step, require a large degree of computational effort to compute a step.

The numerical solution of stiff equations has been studied extensively and a number of techniques were proposed to cope with this problem

(Refs. 4.1 - 4.15).

Curtiss and Hirschfelder (Ref. 4.1) proposed an implicit technique for solving a single stiff equation. Emanuel (Ref. 4.2) studied the application of predictor-corrector and Runge-Kutta techniques to the chemical equations in near equilibrium flows. The locally exact method was applied by Moretti (Ref. 4.3) to the problem of combustion of hydrogen in air at constant pressure. He made a comparison of the computing time required using the locally exact method and the fourth order Runge-Kutta method.

Treanor (Ref. 4.5) developed an explicit method for solving non-linear stiff equations. It assumes that the basic equations can be approximated by a linearized form, but not the form given in Eqs. 4.30. Instead they are linearized as follows:

$$\begin{aligned} \frac{dy_i}{dt} = & - (P_i)_m y_i + (A_i)_m \\ & + (B_i)_m h + (C_i)_m h^2 \end{aligned} \quad (i=1, \dots, n) \quad (4.34)$$

where the P_i , A_i , B_i and C_i are constants in a given step but are allowed to vary from step to step.

Lomax and Bailey (Ref. 4.6) and Bailey (Ref. 4.12) studied the solution to the problem of air flow behind a normal shock by applying different numerical techniques. The techniques utilized were the explicit modified Euler method, the implicit modified Euler method and Treanor's method.

Magnus and Schechter (Ref. 4.15) developed a unified theory of numerical techniques based on rational approximations to the exponential terms in Eqs. 4.33. They showed that a number of explicit and implicit techniques are really rational approximations of different order. The

implicit methods studied, which include the method of collocation and the subdomain method, were shown to be unconditionally stable and the subdomain method was applied to the problems of hydrogen-air combustion and dissociating air.

From the work that has been carried out it has become evident that there is no "best" method for solving Eqs. 4.29. The optimum method to be used depends upon the degree of "stiffness" of the equations. If the flow is essentially frozen, classical explicit techniques such as fourth order Runge-Kutta are better than any implicit or locally exact method because they would be accurate and faster. If the flow is near chemical equilibrium, an unconditionally stable implicit technique is optimum because the stepsize would be restricted only by the truncation error. In general, the locally exact technique is less convenient than the explicit or implicit methods, but it yields more information about the system of equations.

This discussion has assumed that the problem represented by Eqs. 4.29 is "well behaved", this means that the solution is inherently stable, and our discussion of stability was centered around what is called induced instability, that is, instability induced by the numerical technique used. However, it can also happen that the problem is "ill posed" or inherently unstable (Ref. 4.16). This can happen, for example, when the general solution to Eqs. 4.29 contains exponentially growing terms which must be suppressed to fulfill the boundary conditions imposed on the problem. For example, the general solution to the equation

$$\frac{dy}{dt} = y - t \quad (4.35)$$

is given by

$$y = C e^t + t + 1 \quad (4.36)$$

where C is an integration constant. If the initial condition is given as $y(0) = 1$, which makes $C = 0$ so that the solution is a simple linear term growing only slowly in comparison with the exponential term, any numerical technique for solving Eq. 4.35 will introduce the exponential term into the computed solution resulting in inherent instability.

This type of condition is common in problems described by differential equations of the type

$$D_i(x) \frac{d^2 y_i}{dx^2} + E_i(x) \frac{dy_i}{dx} = F_i(Y_1, Y_2, \dots, Y_n) \quad (i = 1, \dots, n) \quad (4.37)$$

where the D_i and E_i are functions of the independent variable x ; the F_i are nonlinear functions of y_1, y_2, \dots, y_n , and Eqs. 4.37 are subject to the following boundary conditions

$$y_i(0) = y_{i,0} \quad (i = 1, \dots, n) \quad (4.38)$$

$$y_i(1) = y_{i,1} \quad (i = 1, \dots, n) \quad (4.39)$$

One physical situation described by Eqs. 4.37 - 4.39 is that arising from similar boundary layer flows including mass diffusion and finite-rate chemical reactions.

When the equations are inherently unstable the methods discussed above for solving Eqs. 4.29 are not effective. Such methods are usually referred to as "marching" techniques because the solution starts at one of the boundaries and, for each step taken, the solution marches towards the other boundary. When marching techniques are applied to a two-point boundary value problem, such as that represented by Eqs. 4.37 - 4.39, it is necessary to transform the problem into an initial value problem.

This is accomplished by guessing the value of the $\frac{dy_i}{dx}$ at $x = 0$ and using the marching technique to compute the y_i at $x = 1$, if the computed y_i at $x = 1$ differs from the desired values, it is necessary to iterate until the boundary conditions at $x = 1$ are met.

Because of the problems associated with the use of marching techniques, boundary layer flow problems including mass diffusion and finite-rate chemical reactions have been approached with what are known as "globally implicit" techniques. These are techniques which work directly with the two-point boundary value problem and produce in one step the solution over the domain of interest.

Fay and Kay (Ref. 4.16) used a globally implicit method to solve the equations describing a laminar, dissociating, nitrogen boundary layer at the stagnation point of an axisymmetric body. The boundary layer equations were first linearized about an initial trial solution (or previous iteration) in such a way that the equations are uncoupled (except implicitly through the trial solution). The equations were then cast into an implicit finite-difference form and solved in sequence, the trial solutions being updated after each solution had been obtained. This sequence of solutions was continued until the original differential equations were satisfied to the desired degree of accuracy.

Blottner (Ref. 4.17) studied techniques for solving the viscous shock layer flow at the stagnation point of a blunt body for air with finite-rate chemical reactions. He found that a globally implicit technique, such as the one used by Fay and Kay, produced converged solutions in a reasonable amount of time. Adams, et. al., (Ref. 4.18) applied the technique used by Fay and Kay to compute the inviscid and viscous flow fields around spherically blunted cone geometries, including injection of inert argon or chemically reacting carbon dioxide with chemical reactions

taking place at a finite -rate.

Liu (Ref. 4.19) studied the problem of hydrogen injection into air at an axisymmetric point including mass diffusion and non-equilibrium chemistry. He studied the applicability of various techniques, such as an implicit marching technique developed by Lomax (Ref. 4.10) and the globally implicit method used by Fay and Kay, and found that some of the methods could not be applied to the reacting flow problem and those that could be applied did not work for near chemical equilibrium flows.

To be successful, any proposed solution method to the stagnation-line, finite-rate equations must be in accord with these previous studies.

DECOUPLING THE EQUATIONS

Although it is desirable to solve the equations in Table 4.4 in a coupled manner, practical considerations make it necessary to decouple them. It must be noticed that any attempt to solve the equations coupled would require the simultaneous solution of $7 + n$ (in our problem $n = 19$) non-linear, ordinary integro-differential and linear integral and algebraic equations.

An illustration of how coupled equations can be uncoupled is given by considering the problem of solving

$$\frac{dL_1}{dt} = a_{11} L_1 + a_{12} L_2 \quad (4.40)$$

$$\frac{dL_2}{dt} = a_{21} L_1 + a_{22} L_2 \quad (4.41)$$

where L_1 and L_2 are the dependent variables and the a_{ij} 's are constants. If the function L_2 is guessed, ($L_2(o)$), then this value can be used in Eq. 4.40 to give

$$\frac{dL_1^{(1)}}{dt} - a_{11} L_1^{(1)} = a_{12} L_2^{(o)} \quad (4.42)$$

where $L_2^{(0)}$ is a known function of t . This equation can be solved for $L_1^{(1)}$ and this function can be used as a guess for L_1 in Eq. 4.41 to give

$$\frac{dL_2}{dt} - a_{22} L_2^{(1)} = a_{21} L_1^{(1)} \quad (4.43)$$

The function $L_2^{(1)}$ obtained from solving the equation above can then be compared to $L_2^{(0)}$; if they are equal the problem is solved, if not $L_2^{(1)}$ is used as the new $L_2^{(0)}$ and the procedure described above is repeated. It is easy to see that the price that must be paid for reducing the original problem to that posed by Eqs. 4.42 and 4.43 is, in general, a greater amount of computation needed.

The scheme used to decouple the equations in Table 4.4 is given in Figure 4.1 while Table 4.5 contains the uncoupled equations. First of all, values of $\tilde{\delta}^{(0)}$, $\rho^{(0)}$, $(\rho\mu)^{(0)}$, and $C_i^{(0)}$ ($i = 1, \dots, n$) are guessed. With these functions it is possible to solve Eqs. 4.44 and 4.45 for $f^{(1)}$ and $z^{(1)}$ (this section of the flowchart is that enclosed by the dotted lines), this is accomplished by guessing $f^{(0)}$ and solving Eq. 4.44 for $z^{(1)}$, then Eq. 4.45 is solved for $f^{(1)}$. If $f^{(1)} = f^{(0)}$ we proceed to solve for $v^{(1)}$ and $\delta^{(1)}$, if not, $f^{(0)}$ and $f^{(1)}$ are used to compute the next guess $(f^{(0)}) = (1 - \lambda_1) f^0 + \lambda_1 f^{(1)}$, where $0 < \lambda_1 \leq 1$ and the process is repeated. Once $f^{(1)}$ and $z^{(1)}$ are known, Eqs. 4.46 and 4.47 are solved for $v^{(1)}$ and $\tilde{\delta}^{(1)}$, respectively. If $\tilde{\delta}^{(1)} = \tilde{\delta}^{(0)}$ the iterative process

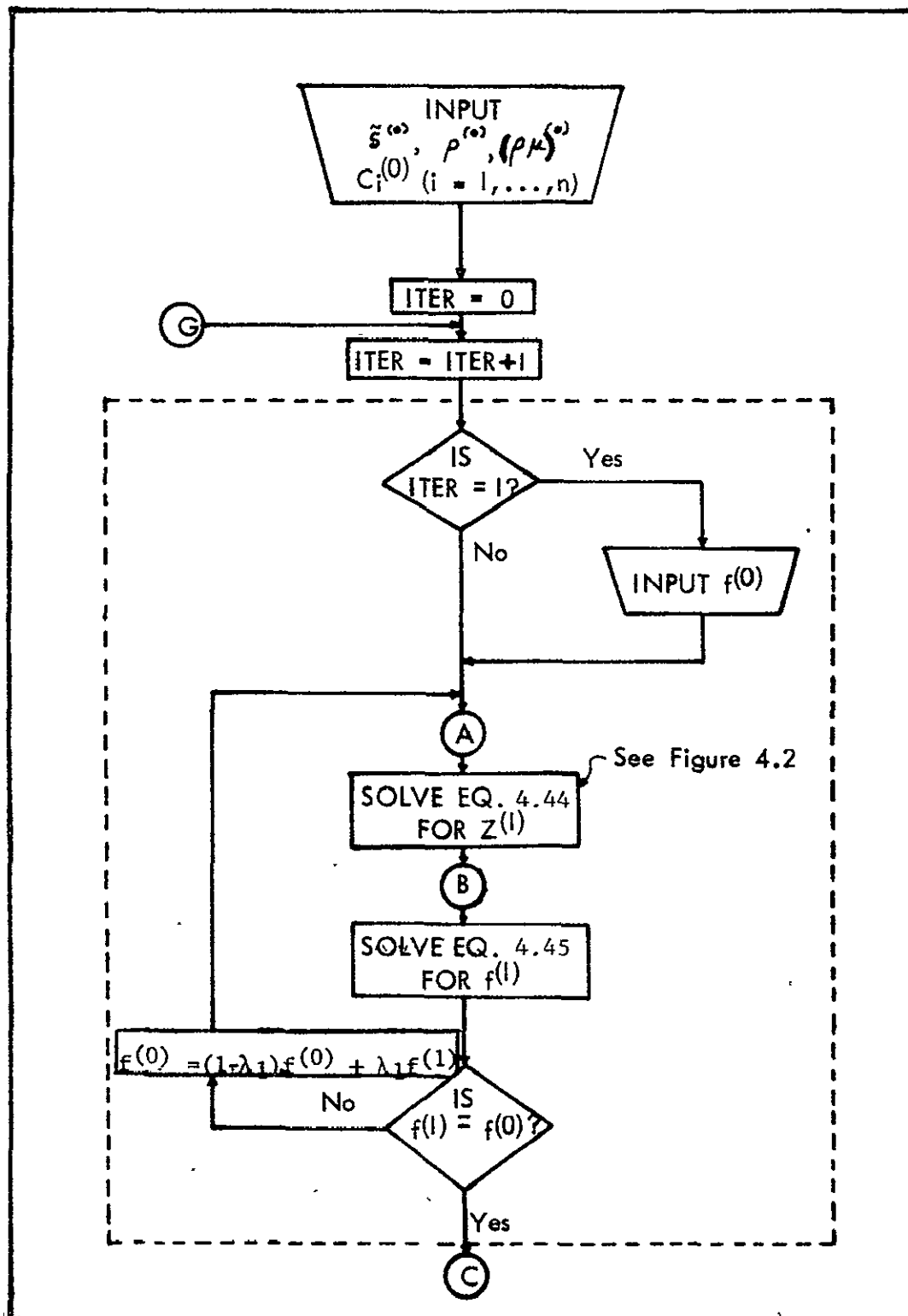


Figure 4.1 Problem Flowchart

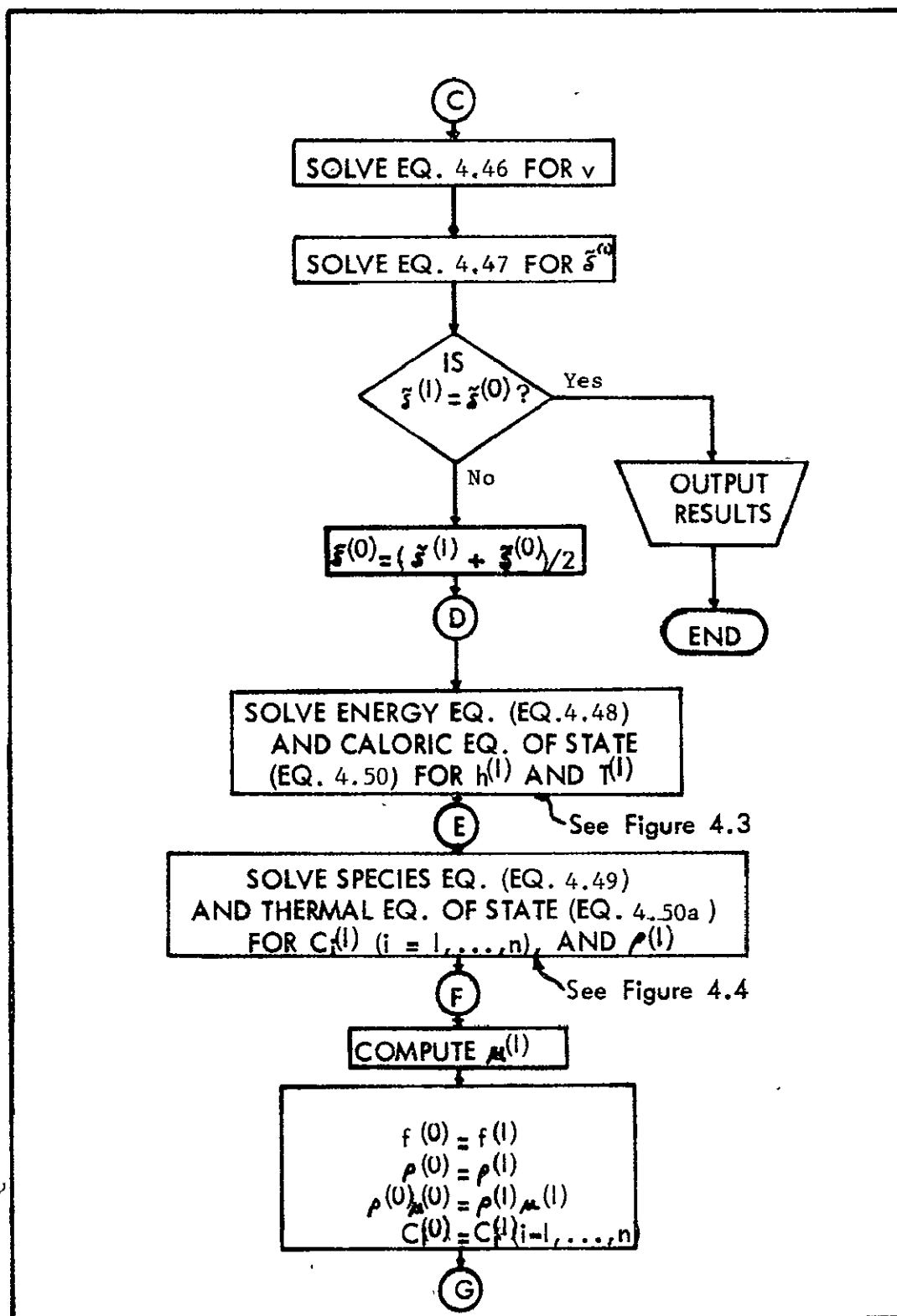


Figure 4.1 Problem Flowchart (continued)

TABLE 4.5
DECOUPLED EQUATIONS

χ - Momentum

$$\rho^{(0)} \mu^{(0)} \frac{d^2 z^{(1)}}{d\eta^2} + \left[2 \operatorname{Re}_{\delta} \tilde{\delta}^{(0)^2} \left(\frac{\partial \mu_{\delta,0}}{\partial \chi} \right) f^{(0)} + \frac{d(\rho^{(0)} \mu^{(0)})}{d\eta} \right] \frac{dz^{(1)}}{d\eta} \quad (4.44)$$

$$- \operatorname{Re}_{\delta} \tilde{\delta}^{(0)^3} \left(\frac{\partial \mu_{\delta,0}}{\partial \chi} \right) z^{(1)^2} + 2 \operatorname{Re}_{\delta} \tilde{\delta}^{(0)} \frac{\bar{\rho}}{\rho^{(0)}} (1 - \bar{\rho}) \frac{\left(\frac{d\phi}{d\chi} \right)^2}{\frac{\partial \mu_{\delta,0}}{\partial \chi}} \chi = 0$$

$$f^{(1)} = \tilde{\delta}^{(0)} \int_0^{\eta} z^{(1)} d\eta + f\omega \quad (4.45)$$

$$v^{(1)} = - 2 \frac{\tilde{\delta}^{(0)}}{\rho^{(0)}} \left(\frac{\partial \mu_{\delta,0}}{\partial \chi} \right) f^{(1)} \quad (4.46)$$

$$\tilde{\delta}^{(1)} = \frac{f_{\delta} - f_{\omega}}{\int_0^{\eta} z^{(1)} d\eta} \quad (4.47)$$

Energy:

$$\frac{2}{\tilde{\delta}^{(1)}} \left(\frac{\rho k}{c\rho} \right)^{(0)} \frac{d^2 h^{(1)}}{d\eta^2} + \left[-\rho^{(0)} v^{(1)} + \frac{1}{\tilde{\delta}^{(1)}} \frac{d}{d\eta} \left(\frac{2\rho k}{c\rho} \right)^{(0)} \right] \frac{dh^{(1)}}{d\eta} =$$

$$\frac{2\tilde{\delta}^{(1)}}{\rho^{(0)}} \frac{dq^{(0)}}{dy} R_{,y} + 2\rho^{(0)} (v^{(1)})^2 \frac{dv^{(1)}}{d\eta} + \frac{1}{\tilde{\delta}^{(1)}} \frac{d}{d\eta} \left[\frac{2\rho k}{c\rho} - \rho^2 D \right]^{(0)} \quad (4.48)$$

$$\left(\sum_i h_i \frac{d C_i}{d\eta} \right)^{(0)} \Bigg]$$

TABLE 4.5 (Continued)

$$+ \frac{1}{\tilde{\delta}^{(1)}} \left(\frac{2\rho k}{c_p} - \rho^2 D \right)^{(0)} \left(\sum_i^n \frac{dh_i}{dn} \frac{dC_i}{dn} \right)^{(0)} + \frac{1}{\tilde{\delta}^{(1)}} \left[\left(\frac{2\rho k}{c_p} - \rho^2 D \right)^{(0)} \right. \\ \left. \left(\sum_i^n h_i \frac{d^2 C_i}{dn^2} \right)^{(0)} \right]$$

Species Continuity:

$$\rho^{(0)^2} D^{(1)} \frac{d^2 C_i^{(1)}}{dn^2} + \left[\frac{d(\rho^{(0)^2} D^{(1)})}{dn} - \tilde{\delta}^{(1)} \rho^{(0)} v^{(1)} \right] \frac{d C_i^{(1)}}{dn} \\ + \frac{\tilde{\delta}^{(1)^2}}{\rho^{(0)}} \omega_i^{(1)} = 0 \quad (4.49)$$

Caloric Equation of State:

$$T^{(1)} = \text{fn} \{h^{(1)}, C_1^{(0)}, \dots, C_n^{(0)}\} \quad (4.50)$$

Thermal Equation of State:

$$\rho^{(1)} = \frac{P_\delta}{\left(\sum_i^n \frac{C_i^{(1)}}{M_i} \right) IRT^{(1)}} \quad (4.50a)$$

is ended and the results output, if not the values of $\rho^{(0)}$, $C_i^{(0)}$ ($i = 1, \dots, n$), $v^{(1)}$ and $\tilde{\delta}^{(1)}$ are used in the energy equation (Eq. 4.48) to solve for $h^{(1)}$, and the caloric equation of state (Eq. 4.50) is solved for $T^{(1)}$. Then the species equation (Eq. 4.49) and the thermal equation of state (Eq. 4.50a) are solved for $C_i^{(1)}$ ($i = 1, \dots, n$) and $\rho^{(1)}$. Once the temperature and concentration profiles are known, the new viscosity ($\mu^{(1)}$) is computed, and $\tilde{\delta}^{(1)}$, $\rho^{(1)}$, $(\rho\mu)^{(1)}$ and $C_i^{(1)}$ ($i=1, \dots, n$) can be used as guesses for the next iteration. The flowcharts for the operations noted in Figure 4.1 are given in Figures 4.2 - 4.4 to be developed below.

LINEARIZATION OF THE EQUATIONS

In the previous section the equations were decoupled (Table 4.5); this has the effect of making each equation independent from the rest except through the iterative process. However, the equations are still nonlinear with variable coefficients. In considering the kinds of numerical techniques that may be used to solve these equations, the fact that the boundary conditions are given on two different points in the flow-field suggests that "globally implicit" finite-difference approximations would be preferable to so-called "shooting" or "marching" techniques since the latter requires specifications of all boundary conditions on one point in the flow-field. As used in this work globally implicit means that each differential equation is substituted by finite-difference approximations over the entire flow-field, and this results in a set of simultaneous algebraic equations that when solved yield the value of the dependent variable at different points in the flow field. If this finite-difference technique is applied directly to the nonlinear equations the problem is reduced to solving a set of non-linear

algebraic equations; however, due to the present lack of efficient techniques for solving sets of non-linear algebraic equations it is more convenient to linearize the differential equations before they are written in finite-difference form. When this is done the problem is reduced to obtaining the solution to a set of linear algebraic equations, a much easier problem.

Consider a non-linear ordinary differential equation of the form

$$\frac{dL}{dt} \{t\} = g \{L, t\} \quad (4.51)$$

where $g \{L, t\}$ is the non-linear term. It is evident that there are various ways of linearizing this equation; the simplest one being to use an approximation to the solution $L^{(0)}$ to write

$$\frac{dL^{(1)}}{dt} = g \{L^{(0)}, t\} \quad (4.25)$$

This equation is now linear since $L^{(0)}$ is a known function of t , it may be solved for $L^{(1)}$ and if $L^{(1)} = L^{(0)}$ the answer has been found, if not $L^{(1)}$ is used as the new guess and the process is repeated. Another commonly used linearizing technique is to expand the non-linear term in Eq. 4.51 about the function $L^{(0)}$ in a Taylor series truncated after the second term to yield

$$g \{L^{(1)}, t\} = g \{L^{(0)}, t\} + \frac{\partial g \{L^{(0)}, t\}}{\partial L} (L^{(1)} - L^{(0)}) \quad (4.53)$$

and by substitution of this relation in Eq. 4.51

$$\frac{dL^{(1)}}{dt} - \frac{\partial g\{L^{(0)}, t\}}{\partial t} L^{(1)} = g\{L^{(1)}, t\} - \frac{\partial g\{L^{(0)}, t\}}{\partial t} L^{(0)} \quad (4.54)$$

This technique is known as quasilinearization (Refs. 4.20 and 4.21), and has been used extensively to solve problems in fluid and orbital mechanics. It can be demonstrated that if the sequence $\{L^{(K)}\}$ converges, that is if $\lim_{K \rightarrow \infty} \{L^{(K)}\} = L$, it does it monotonically (for example if $L^{(0)} < L$ for all t monotonicity of convergence means $L^{(0)} \leq L^{(1)} \leq \dots \leq L$, and quadratically

$$\left| L^{(K+1)} - L^{(K)} \right| = N \left| L^{(K)} - L^{(K-1)} \right| \quad (4.55)$$

where N is a constant. Although these two properties make the technique extremely powerful, in practice the computation of $\partial g / \partial t$ may be very costly. Therefore, in some instances the simpler technique given by Eq. 4.52 may be preferable.

In the present application use will be made of both the simple linearization and quasilinearization techniques.

X - momentum:

The non-linear term in Eq. 4.44 is the $Z^{(1)^2}$ term, and quasilinearization yields

$$\begin{aligned} Z^{(1)^2} &= Z^{(0)^2} + 2Z^{(0)} (Z^{(1)} - Z^{(0)}) \\ &= 2Z^{(0)} Z^{(1)} - Z^{(0)^2} \end{aligned} \quad (4.56)$$

when this relation is substituted in Eq. 4.44 the result is

$$\frac{d^2 Z^{(1)}}{d\eta^2} + 2 \left[\frac{\text{Re}_{\delta} \tilde{\delta}^{(0)^2}}{\rho^{(0)} \mu^{(0)}} \frac{\partial u_{\delta 0}}{\partial x} f(Q) + \frac{d}{d\eta} \left(\frac{1}{\rho} \right)^{(0)} \mu^{(0)} \right] \frac{dZ^{(1)}}{d\eta}$$

$$-2 \frac{\text{Re}_\delta \delta^{(0)3}}{\rho^{(0)} \mu^{(0)}} \frac{\partial \mu_{\delta,0}}{\partial \chi} z^{(0)} z^{(1)} = \quad (4.57)$$

$$-2 \frac{\text{Re}_\delta \tilde{\delta}^{(0)}}{\rho^{(0)} \mu^{(0)}} \left[\frac{\bar{\rho} (1 - \bar{\rho})}{\rho^{(0)}} \frac{\left(\frac{d\phi}{dx} \right)^2_{x=0}}{\frac{\partial \mu_{\delta,0}}{\partial \chi}} + \frac{\delta^{(0)2}}{2} \frac{\partial \xi_{\delta,0}}{\partial \chi} z^{(0)2} \right]$$

This equation is now linear and may be solved as shown in Figure 4.2.

Energy:

The energy equation (Eq. 4.48) is linear in $h^{(1)}$, and therefore linearization is not needed. Inversion of the caloric equation of state is accomplished as follows:

From Eq. 4.4

$$h^{(1)} = \sum_i^n c_i^{(0)} h_i \{T\} \quad (4.4)$$

therefore the problem consists of finding a root of the equation

$$G\{T\} = h^{(1)} - \sum_i^n c_i^{(0)} h_i \{T\}$$

Since the $h_i \{T\}$ are nonlinear functions of T , the Newton-Raphson technique was used:

$$G\{T^{(K+1)}\} = G\{T^{(K)}\} + \frac{dG}{dT}^{(K)} (T^{(K+1)} - T^{(K)})$$

$$T^{(K+1)} - T^{(K)} = - \frac{G\{T^{(K)}\}}{\frac{dG}{dT}^{(K)}}$$

$$\Delta T^{(K)} = - \frac{h^{(1)} - \sum_i^n c_i^{(0)} h_i (T^{(K)})}{-\sum_i^n c_i^{(0)} c_{\rho i} (T^{(K)})} = \frac{h^{(1)} - h^{(K)}}{c_\rho^{(K)}}$$

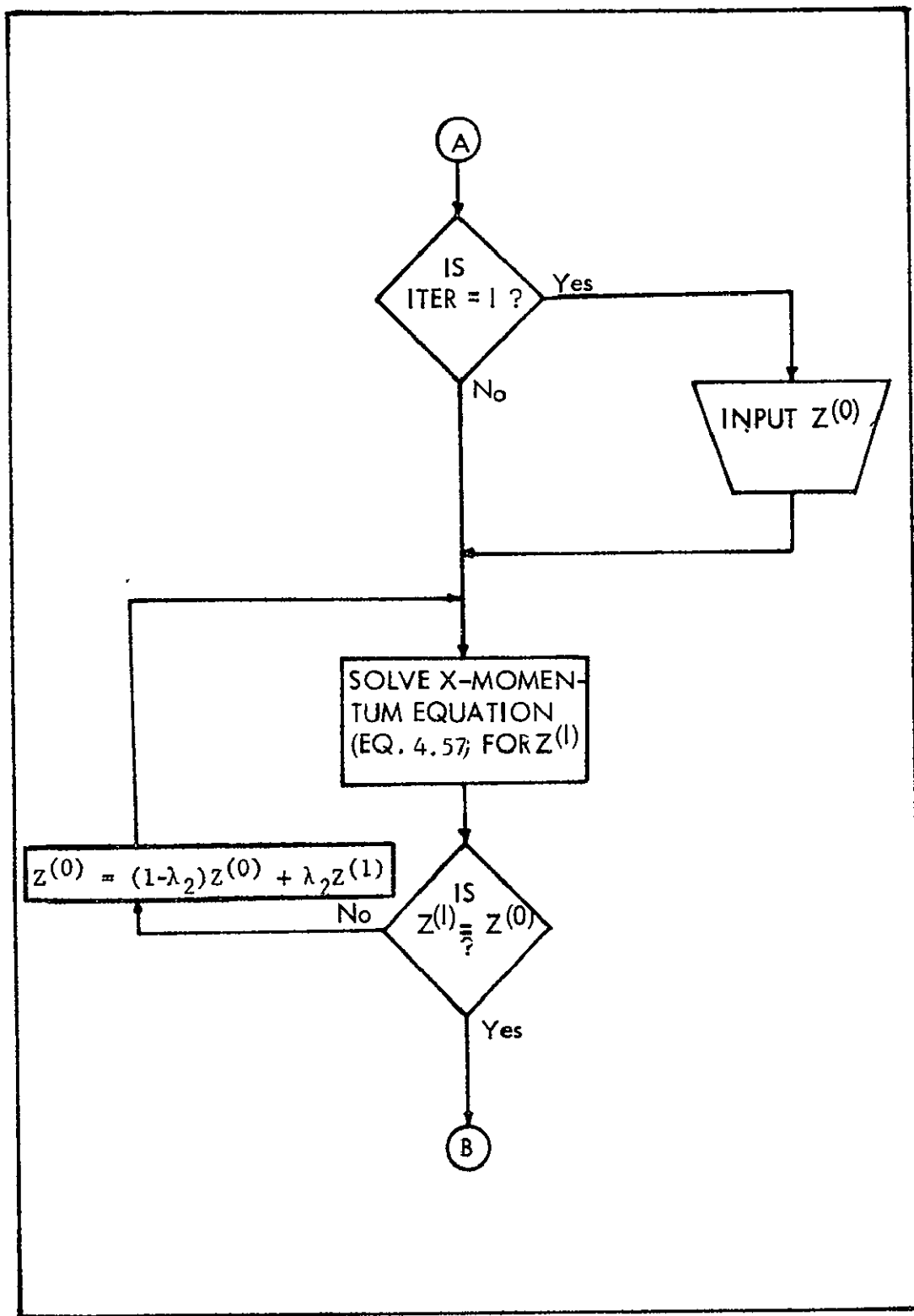


Figure 4.2 Solution of X-momentum Equation

Using underrelaxation to obtain convergence

$$\begin{aligned} T^{(K+1)} &= T^{(K)} + \lambda_3 \Delta T^{(K)} \\ &= T^{(K)} + \lambda_3 \frac{h^{(1)} - h^{(K)}}{c_p^{(K)}} \quad 0 < \lambda_3 \leq 1 \end{aligned} \quad (4.58)$$

Once $T^{(K+1)}$ has converged it becomes the new $T^{(1)}$.

The flowchart for the solution of the equation of energy and the caloric equation of state for $h^{(1)}$ and $T^{(1)}$ is given in Fig. 4.3.

Species Continuity:

The reaction rate term in Eq. 4.49 is non-linear, therefore, quasilinearization gives

$$\omega_i^{(1)} = \omega_i^{(0)} + \frac{\partial \omega_i^{(0)}}{\partial C_i} (C_i^{(1)} - C_i^{(0)}) \quad (i = 1, \dots, n) \quad (4.59)$$

and the species equation becomes

$$\begin{aligned} \rho^{(0)^2} D^{(1)} \frac{d^2 C_i^{(1)}}{d \eta^2} + \left[\frac{d (\rho^{(0)^2} D^{(1)})}{d \eta} - \tilde{\delta}^{(1)} \rho^{(0)} v^{(1)} \right] \frac{d C_i^{(1)}}{d \eta} \\ + \frac{\tilde{\delta}^{(1)^2}}{\rho^{(0)}} \frac{\partial \omega_i^{(0)}}{\partial C_i} C_i^{(1)} = \frac{\tilde{\delta}^{(1)^2}}{\rho^{(0)}} \left(\frac{\partial \omega_i^{(0)}}{\partial C_i} C_i^{(0)} - W_i^{(0)} \right) \end{aligned} \quad (i = 1, \dots, n) \quad (4.60)$$

The scheme for solving the equation above and the thermal equation of state is given in Figure 4.4.

NUMERICAL SOLUTION OF THE EQUATIONS

As written in the previous section, the conservation equations (Eqs. 4.57, 4.58, and 4.60) are of the form

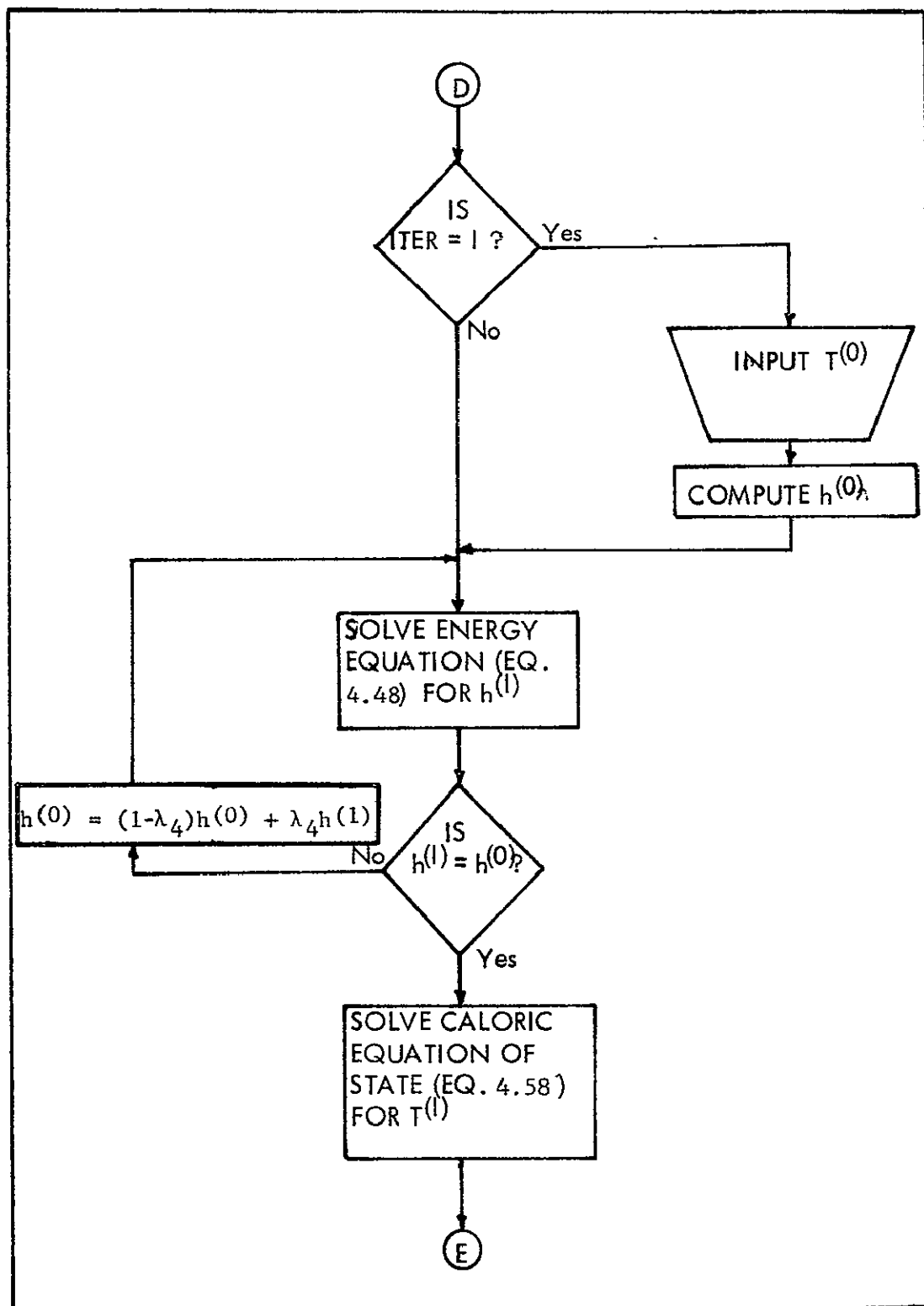


Figure 4.3 Solution of Energy Equation

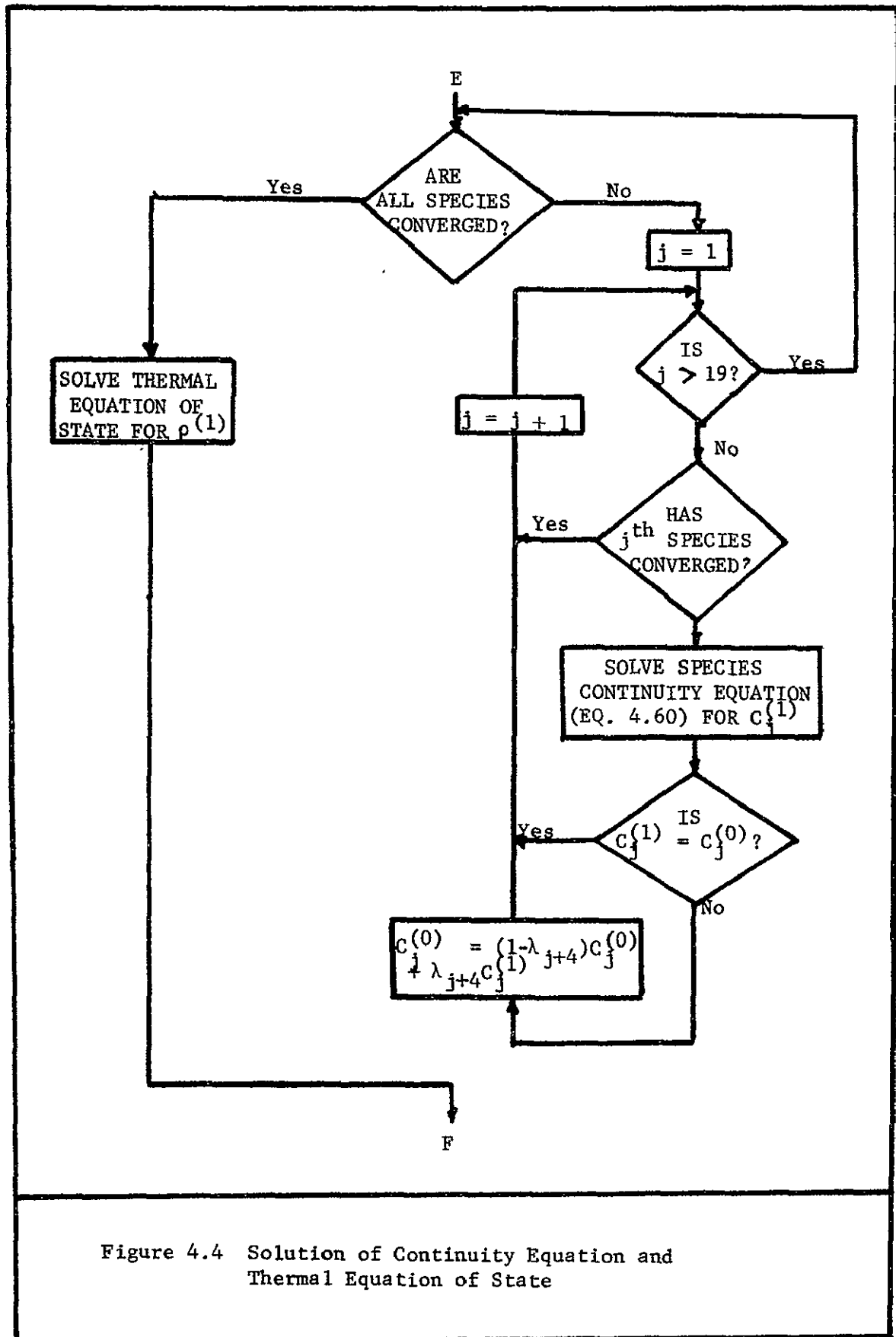


Figure 4.4 Solution of Continuity Equation and Thermal Equation of State

$$\alpha_1 \frac{d^2 W}{dn^2} + \alpha_2 \frac{dW}{dn} + \alpha_3 W = \alpha_4 \quad (4.61)$$

where α_1 , α_2 , α_3 , and α_4 are functions of η . The numerical solution of Eq. 4.61 was accomplished by substituting finite-difference approximations for the derivatives in the equation. These approximations were obtained as follows: Knowing the value of W and its derivatives at any point η_j in the flow-field it is possible to write the Taylor series expression for the value of W at point η_{j-1}

$$W_{j-1} = W_j + \frac{dW_j}{dn} (\eta_{j-1} - \eta_j) + \frac{d^2 W_j}{dn^2} \left(\frac{\eta_{j-1} - \eta_j}{2} \right)^2 + 0 (\eta_{j-1} - \eta_j)^3 \quad (4.62)$$

where $W_j = W(\eta_j)$. Knowing W_j and its derivatives it is also possible to write an expression for W_{j+1}

$$W_{j+1} = W_j + \frac{dW_j}{dn} (\eta_{j+1} - \eta_j) + 0 (\eta_{j+1} - \eta_j)^2 \quad (4.63)$$

Neglecting higher order terms and solving Eqs. 4.62 and 4.63 first for dW_j/dn and then for $d^2 W_j/dn^2$ yields

$$\begin{aligned} \frac{dW_j}{dn} \approx & \frac{\Delta \eta_{j-1}}{\Delta \eta_j (\Delta \eta_j + \Delta \eta_{j-1})} W_{j+1} + \frac{(\Delta \eta_j - \Delta \eta_{j-1})}{\Delta \eta_j \Delta \eta_{j-1}} W_j \\ & + \frac{(-\Delta \eta_j)}{\Delta \eta_{j-1} (\Delta \eta_j + \Delta \eta_{j-1})} W_{j-1} \end{aligned} \quad (4.64)$$

$$\begin{aligned}
\frac{d^2 W_j}{d\eta^2} \approx & \frac{2}{\Delta\eta_j (\Delta\eta_j + \Delta\eta_{j-1})} W_{j+1} + \frac{(-2)}{\Delta\eta_j \Delta\eta_{j-1}} W_j \\
& + \frac{2}{\Delta\eta_{j-1} (\Delta\eta_j + \Delta\eta_{j-1})} W_{j-1}
\end{aligned}
\tag{4.65}$$

where $\Delta\eta_j = \eta_{j+1} - \eta_j$.

When Eq. 4.61 is evaluated at η_j and Eqs. 4.64 and 4.65 are introduced the result is

$$\begin{aligned}
& \left[\frac{2\alpha_{1,j} - \Delta\eta_j \alpha_{2,j}}{\Delta\eta_{j-1} (\Delta\eta_j + \Delta\eta_{j-1})} \right] W_{j-1} + \left[\frac{-2\alpha_{1,j} + (\Delta\eta_j - \Delta\eta_{j-1}) \alpha_{2,j}}{\Delta\eta_j \Delta\eta_{j-1}} \right. \\
& \quad \left. + \alpha_{3,j} \right] W_j \\
& + \left[\frac{2\alpha_{1,j} + \Delta\eta_{j-1} \alpha_{2,j}}{\Delta\eta_j (\Delta\eta_j + \Delta\eta_{j-1})} \right] W_{j+1} = \alpha_{4,j}
\end{aligned}
\tag{4.66}$$

Equation 4.66 is of the form

$$A_j W_{j-1} + B_j W_j + C_j W_{j+1} = D_j \tag{4.67}$$

Letting $\eta_0 = 0$ and $\eta_m = 1$ this means that W_0 and W_m are known since they are boundary conditions, when Eq. 4.67 is written for $1 \leq j \leq (m-1)$ the resulting set of equations is given in matrix notation by

$$\begin{bmatrix}
 B_1 & C_1 & & & \\
 A_2 & B_2 & C_2 & & \\
 A_3 & B_3 & C_3 & & \\
 \cdot & \cdot & \cdot & \cdot & \\
 \cdot & \cdot & \cdot & \cdot & \\
 \cdot & \cdot & \cdot & \cdot & \\
 & A_{m-2} & B_{m-2} & C_{m-2} & \\
 & & A_{m-1} & B_{m-1} & \\
 & & & &
 \end{bmatrix}
 \begin{bmatrix}
 W_1 \\
 W_2 \\
 W_3 \\
 \cdot \\
 \cdot \\
 \cdot \\
 W_{m-2} \\
 W_{m-1}
 \end{bmatrix}
 =
 \begin{bmatrix}
 D_1 - A_1 W_0 \\
 D_2 \\
 D_3 \\
 \cdot \\
 \cdot \\
 \cdot \\
 D_{m-2} \\
 D_{m-1} - C_{m-1} W_m
 \end{bmatrix}
 \quad (4.68)$$

The Thomas algorithm for tridiagonal matrices (Ref. 4.22) is a very efficient scheme for solving sets of linear algebraic equations involving tridiagonal matrices and has been used in this work to solve Eq. 4.68.

The solution of Equations (4.68) was accomplished by writing a computer program to perform the necessary calculations. This program (SLAC- Stagnation Line Analysis with Chemistry) includes the thermodynamic, transport, kinetic, and radiative properties discussed in Chapter 3, as well as, a subroutine to calculate local chemical equilibrium. The equilibrium calculation uses a free energy minimization technique developed by Del Valle and Pike (Ref. 4.23). SLAC is discussed in Appendix A and listed in Appendix B.

This program is a tool which models the stagnation region of an ablator-protected entry vehicle. Adequate boundary-conditions to describe the bow-shock and ablating surface, as described herein, must be specified to utilize this program.

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CHAPTER 5

RESULTS OF STAGNATION REGION HEATING ANALYSIS

The SLAC program was used to compute the finite-rate and equilibrium-chemistry stagnation-line heating rate of a 9 foot entry vehicle moving at 50,000 feet/second when the free-stream air density is 8.85×10^{-8} slugs/ft³ and the mass injection rate $(\rho_w v_w / \rho_\infty U_\infty)$ equals .05. The vehicle is protected by a phenolic-nylon ablator with the following elemental composition (elemental mass fraction) : 73.03% carbon, 7.29% hydrogen, 4.96% nitrogen and 14.72% oxygen. The char surface was assumed to be at the sublimation temperature of carbon at .1 atmosphere pressure ($T_w = 3,450^\circ\text{K}$). The shock was assumed to be concentric ($\frac{d\epsilon}{dx} = 0$, $\frac{d\phi}{dx} = 1$).

These flight conditions and the rate of mass injection were used because they correspond to the conditions obtained by Engel (Ref. 5.1) when the ablator and the equilibrium shock layer solutions are coupled. In other words, according to his results, a vehicle moving under the flight conditions listed above and protected by phenolic-nylon ablators at a 5% rate. Furthermore, Esch (Ref. 5.2) also obtained chemical equilibrium results for the same flight conditions and injection rate. This means that the equilibrium results obtained during the present work may be compared to those obtained by the two investigators mentioned above. The comparison is more valid than in most cases since the same routine was used by all the investigators to solve the Rankine-Hugoniot equations. This means that the shock boundary conditions used were identical. In addition, the radiation model used was the same.

A complete solution to this entry-heating problem would admit non-equilibrium chemistry effects in the ablator response analysis. Such considerations could give different ablator species and (outside) wall temperatures. Even if the ablator effluent into the shock layer were in equilibrium, an entire new series of calculations similar to the ones done by Engel (Ref. 5.1) should be performed to determine the non-equilibrium coupled solution. Ablator non-equilibrium is not in the scope of this research, and non-equilibrium coupling must await the development of an adequate non-equilibrium shock layer analysis. This later development is precisely the subject of this research.

The wall and shock boundary conditions used in this study are given in Table 5.1. Table 5.2 presents a summary of the flight conditions used and the shock layer properties obtained. Since changes in flight conditions strongly affect the behavior of both chemical kinetics and shock layer calculations, the model and the solution scheme developed in this work should be studied over a wide range of flight conditions to determine its adequacy. In the present work only one set of flight conditions was considered, but it is believed that since the conditions are typical of those encountered during atmospheric entry, the model and its solution should work for a range of conditions similar to the ones considered.

COMPUTATION OF BODY HEATING RATE

Since computation of body heating rates for equilibrium and finite-rate chemistry was one of the stated objectives of this investigation, it is proper to discuss how this computation is performed.

As was discussed in Chapter 2, the stagnation line energy boundary condition at the body surface is given in dimensional form by

TABLE 5.1 WALL AND SHOCK BOUNDARY CONDITIONS

| Wall ($y = 0$) | Shock ($y = \delta$) |
|--|---|
| 1. $\rho \dot{v} = 2.23 \times 10^{-4} \frac{\text{slug}}{\text{ft}^2\text{-sec}}$ | 1. $\rho v = -4.45 \times 10^{-4} \frac{\text{slug}}{\text{ft}^2\text{-sec}}$ |
| 2. $\frac{d(\rho v)}{d y} = 0$ | 2. $\frac{d(\rho v)}{d y} = 8.5 \times 10^{-2} \frac{\text{slug}}{\text{ft}^3\text{-sec}}$ |
| 3. $T = 3,450^\circ\text{K}$ | 3. $T = 13,000^\circ\text{K}$ |
| 4. $C_i =$ Chemical equilibrium composition of ablation products at $3,450^\circ\text{K}$ and 0.1 atmosphere | 4. $C_i =$ Chemical equilibrium composition of air at $13,000^\circ\text{K}$ and 0.1 atmosphere |

TABLE 5.2 SUMMARY OF FLIGHT CONDITIONS AND
SHOCK LAYER PROPERTIES

$$\rho_{\infty} = 8.85 \times 10^{-8} \text{ slug/ft}^3$$

$$U_{\infty} = 5 \times 10^4 \text{ ft/sec}$$

$$\rho_w = 8.44 \times 10^{-6} \text{ slug/ft}^3$$

$$v_w = 2.64 \times 10^1 \text{ ft/sec}$$

$$T_w = 3,450^\circ\text{K}$$

$$\rho_{\delta} = 1.7 \times 10^{-6} \text{ slug/ft}^3$$

$$v_{\delta} = -2.62 \times 10^3 \text{ ft/sec}$$

$$T_{\delta} = 13,000^\circ\text{K}$$

$$\rho_{\delta} = 0.1 \text{ atm}$$

$$\bar{p} = 0.0536$$

$$R = 9 \text{ ft}$$

$$\text{Injection rate} = 5\%$$

$$\begin{aligned}
 (\rho v H - k \frac{dT}{dy} + \sum_1^n h_i J_{i,y} + q_{r,y})^+ &= \\
 (\rho v H - k \frac{dT}{dy} + \sum_1^n h_i J_{i,y} + q_{r,y})^- &= Q
 \end{aligned}
 \tag{2.50e}$$

The heating rate Q is the net amount of energy heating the body. It must be recalled that the + and - superscripts refer to the shock layer and char sides of the interface, respectively. Q is computed by:

$$Q = (\rho v H - k \frac{dT}{dy} + \sum_1^n h_i J_{i,y} + q_{r,y})^+ \tag{5.1}$$

Preliminary calculations showed the first term in this equation to be negligible when compared to the other terms in the equation, for this reason it was not included in the computation of Q . The heating rate to the body was expressed as

$$Q = Q_c + Q_d + Q_R \tag{5.2}$$

where

$$Q_c = -k \frac{dT}{dy} = \text{Heat transfer by convection} \tag{5.3}$$

$$Q_d = -\sum_1^n h_i J_{i,y} = \text{Heat transfer by diffusion} \tag{5.4}$$

$$Q_R = q_{R,y} = \text{Heat transfer by radiation} \tag{5.5}$$

FINITE-RATE CHEMISTRY RESULTS

The SLAC program was used to compute non-equilibrium temperature, enthalpy, density, velocity and mass fractions profiles, and the body heating rate for the flight conditions previously discussed. Two sets of runs were performed, one using the chemical kinetics model described in Chapter 3, and the other using a modified kinetics model developed

by Balhoff (Ref. 5.3). The modified model assumes the same reactions used in the model described in Chapter 3, but the coefficients of the forward reaction rate constants were modified as shown in Table 5.3. Comparison of these coefficients to those in Table 3.8 shows that while the reactions used are the same, Balhoff's chemistry model uses significantly smaller reaction rate constants in 9 of the 16 reactions considered. Balhoff's model is in effect a modification of the original model used since it was found out that the frequency factors computed using collision theory were in error.

The results from both runs are presented here (Table 5.4) to give an indication of how sensitive the heating rate is to the value of the rate constants used. It must be pointed out that the solution using the rates in Table 5.3 were obtained in about 5% of the computer time, it took to obtain the solution with the original chemistry model (18 hours vs. 45 minutes in an IBM 360/65). This was probably due to the fact that the rates used in the original chemistry model were much faster than those in the modified model, and this made convergence of the species solution more difficult.

CHEMICAL EQUILIBRIUM RESULTS

Results obtained from the SLAC program using an equilibrium chemistry model for the flight conditions previously discussed provide profiles of temperature, enthalpy, density, velocity and mass fractions from the body surface ($y/\delta = 0$) to the shock ($y/\delta = 1$), and the body heating rate. It was maintained above that one of the reasons for choosing the flight conditions studied was the fact that both Engel (Ref. 5.1) and Esch (Ref. 5.2) had studied the same condition and, therefore, the validity of the equilibrium results obtained from SLAC

TABLE 5.3 MODIFIED COEFFICIENTS OF THE
FORWARD REACTION RATE CONSTANTS

| j th Reaction | a_{tj} | b_{fj} | e_{fj} | Comments |
|--------------------------|-----------|----------|----------|----------|
| 1 | 1.09 E 10 | 0.5 | 71,000 | * |
| 2 | 4.50 E 11 | 0.5 | 35,000 | |
| 3 | 1.10 E 21 | -1.5 | 224,900 | |
| 4 | 3.60 E 18 | -0.82 | 103,000 | |
| 5 | 2.80 E 12 | 0.5 | 313,000 | |
| 6 | 2.90 E 12 | 0.5 | 333,000 | |
| 7 | 2.20 E 20 | -1.0 | 131,800 | |
| 8 | 1.09 E 10 | 0.5 | 120,000 | * |
| 9 | 1.22 E 10 | 0.5 | 140,000 | * |
| 10 | 1.23 E 10 | 0.5 | 117,000 | * |
| 11 | 8.50 E 19 | -1.0 | 257,900 | |
| 12 | 1.28 E 10 | 0.5 | 190,000 | * |
| 13 | 1.20 E 10 | 0.5 | 155,000 | * |
| 14 | 1.21 E 10 | 0.5 | 165,000 | * |
| 15 | 1.09 E 10 | 0.5 | 115,000 | * |
| 16 | 1.09 E 10 | 0.5 | 145,000 | * |

*This rate constant is different from that used in the model
described in Chapter 3.

TABLE 5.4 NON-EQUILIBRIUM HEATING RATES AS PREDICTED BY
TWO SETS OF RATE CONSTANTS

| Rate Constants | # Q_c | # Q_d | # Q_R | # Q |
|----------------|------------|------------|------------|----------|
| Table 5.3 | 30 | 1 | 138 | 169 |
| Table 3.8 | 4 | 1 | 106 | 111 |

watts/cm²

could be corroborated.

Before proceeding to discuss and compare the results obtained to those obtained by Engel (Ref. 5.1) and Esch (Ref. 5.2), it is relevant to discuss the differences in the models used. Firstly, Engel used transport properties of air throughout the flow field (including the region from the stagnation point to the body) while Esch and the present work used properties of air and ablation products throughout the flow field. Secondly, the species wall boundary conditions are given by (See Eq. 2.50b).

$$(\rho v)_w C_{i,w} - J_{i,w} = (\rho v)_w C_i^- \quad (i = 1, \dots, n) \quad (5.6)$$

where the term J_i^- , which corresponds to mass diffusion inside the char has been neglected, and the C_i^- are the mass fractions obtained from the ablator response analysis. Esch assumed that mass diffusion could be described by using a binary diffusion coefficient ($J_i = -\rho D \frac{dC_i}{dy}$),

and used boundary conditions of the third kind:

$$(\rho v)_w C_{i,w} + \left(D \frac{dC_i}{dy} \right)_w = (\rho v)_w C_i^- \quad (i = 1, \dots, n) \quad (5.7)$$

Engel assumed there was no diffusion throughout the flow field

($J_i(y) = J_{i,w} = 0$), and used boundary conditions of the first kind:

$$C_{i,w} = C_i^- \quad (i = 1, \dots, n) \quad (5.8)$$

In the present investigation mass diffusion was allowed throughout the shock layer but, in order to simplify the solution to the species equations, the boundary conditions used at the wall were of the first kind:

$$C_{i,w} = C_i^- \quad (i = 1, \dots, n) \quad (5.9)$$

Lastly, both Engel and Esch solved the energy equation in temperature form while in the present work an enthalpy form of the energy equation was used. As was discussed in Chapter 4, many attempts were made at solving the energy equation in temperature term, however, it was not possible to obtain a solution, and therefore, an enthalpy form of the equation was used.

The chemical equilibrium heating rates obtained by Engel, Esch and the present investigation are compared in Table 5.5. It must be noticed that Engel and Esch did not include the heat transfer to the body by diffusion.

The significant difference in convective heating rates, with the present investigation yielding an essentially negligible heating rate by convection, results because the present investigation predicts smaller temperature gradients near the wall than those predicted by Engel and Esch. In other words, the present investigation predicts a higher convective heating blockage effect resulting from injection of ablation products into the flow field.

COMPARISON OF PREDICTED NON-EQUILIBRIUM AND EQUILIBRIUM HEATING RATES

From the results shown in Tables 5.4 and 5.5 it is evident that the predicted equilibrium heating rate is approximately three times as large as the non-equilibrium heating rate, and for both cases most of the body heating is the result of radiation.

The lower radiative heating rate predicted by the non-equilibrium computation may be explained by examining the changes in chemical composition which occur in the shock layer. When the chemical compositions predicted by the equilibrium and the non-equilibrium calculations are

TABLE 5.5 EQUILIBRIUM HEATING RATES OBTAINED BY
DIFFERENT INVESTIGATORS

| INVESTIGATOR | $Q_c^{\#}$ | $Q_d^{\#}$ | $Q_R^{\#}$ | $Q^{\#}$ |
|--------------------------|------------|------------|------------|----------|
| Engel (Ref. 5.1) | 50 | ♣ | 350 | 400 |
| Esch (Ref. 5.2) | 120 | ♣ | 397 | 517 |
| Present Investigation | 3 | 1 | 388 | 392 |

♣ Did not consider
heat transfer by diffusion

$\#$ watts/cm²

compared, it becomes evident that they are markedly different and that finite-rate chemistry effects are present throughout the flow-field (The complete set of dimensionless temperature, enthalpy, density, velocity and mass fraction profiles for the two non-equilibrium and the equilibrium runs are presented in Appendix C. Those profiles needed in the following discussion will be reproduced in this chapter.)

An examination of the chemical composition of equilibrium ablation products shows that as they leave the char surface and are subjected to rapidly increasing temperatures, the decomposition process begun in the char continues as the "large" molecules, C_3H , C_2H , C_4H , HCN , C_2H_2 , and H_2 are broken down (for example see Fig. 5.1) into simpler species such as C_3 , C_2 , CO , CN , N_2 , C and H (see Fig. 5.2). As the ablation products flow towards the stagnation point, this process continues as C_3 , C_2 , CO , CN , and N_2 are converted to C , C^+ , N , and O (see Figs. 5.2 and 5.3).

Near the stagnation point, most of the carbon is ionized to C^+ and, due to diffusion of ablation products past the stagnation point, the concentration of C^+ persists for some distance past the stagnation point before decreasing to zero (see Figs. 5.3 and 5.4).

In the region from the char surface to the stagnation point the non-equilibrium results predict that the ablation products will decompose at a much slower rate, when compared to the rate predicted by the equilibrium results. The "large" molecules (C_3H , C_2H , C_4H , HCN , C_2H_2 , and H_2) and others (C_3 , CO , CN , and N_2) begin to break down into C , C^+ , C_2 , N , N^+ , O^+ , and H (see Figs. 5.1, - 5.3). But the ablation products reach the stagnation point before this process is completed and the results are much lower concentrations of C , C^+ , N , O , and H in

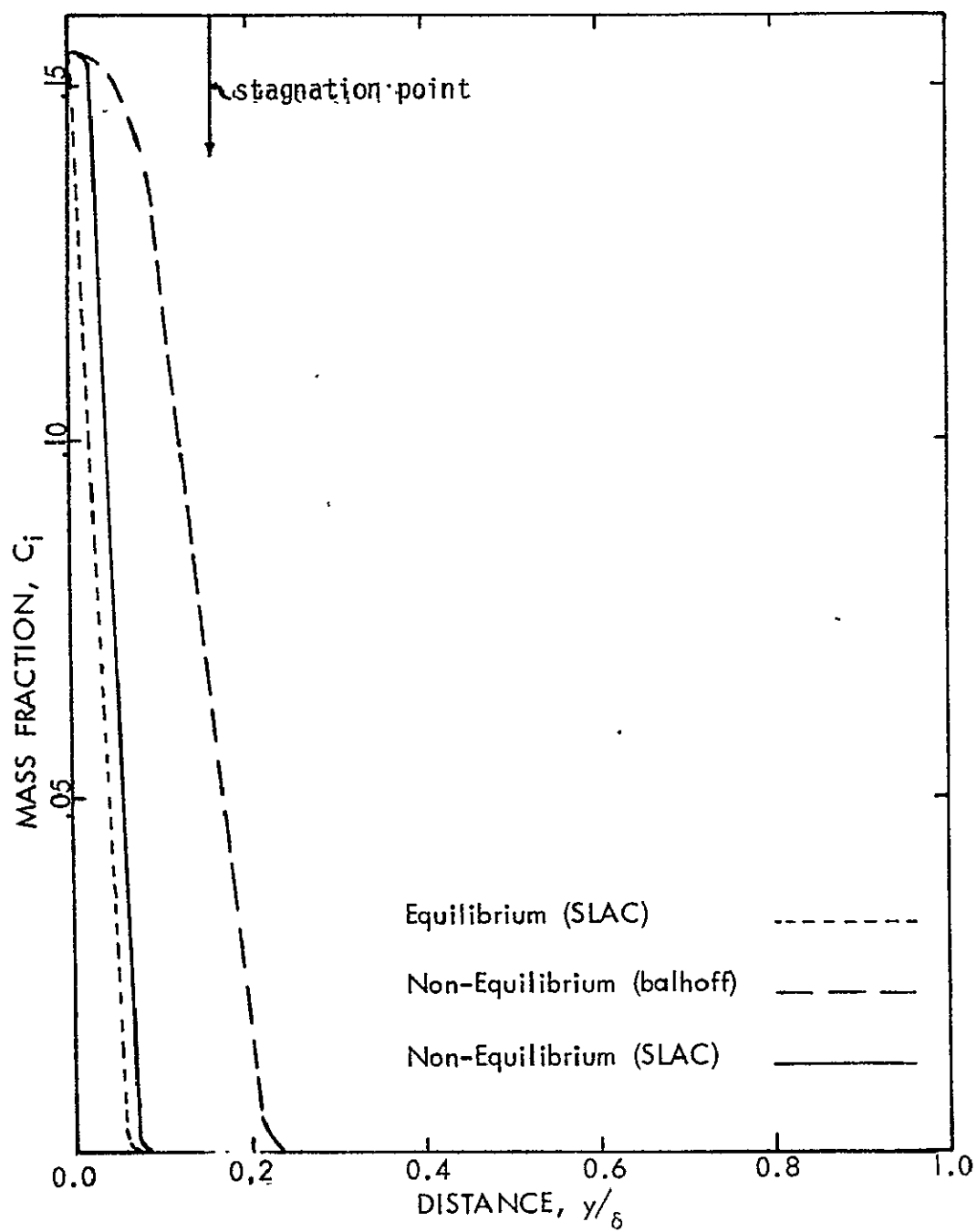


Figure 5.1 Mass Fraction Profiles for C_3H

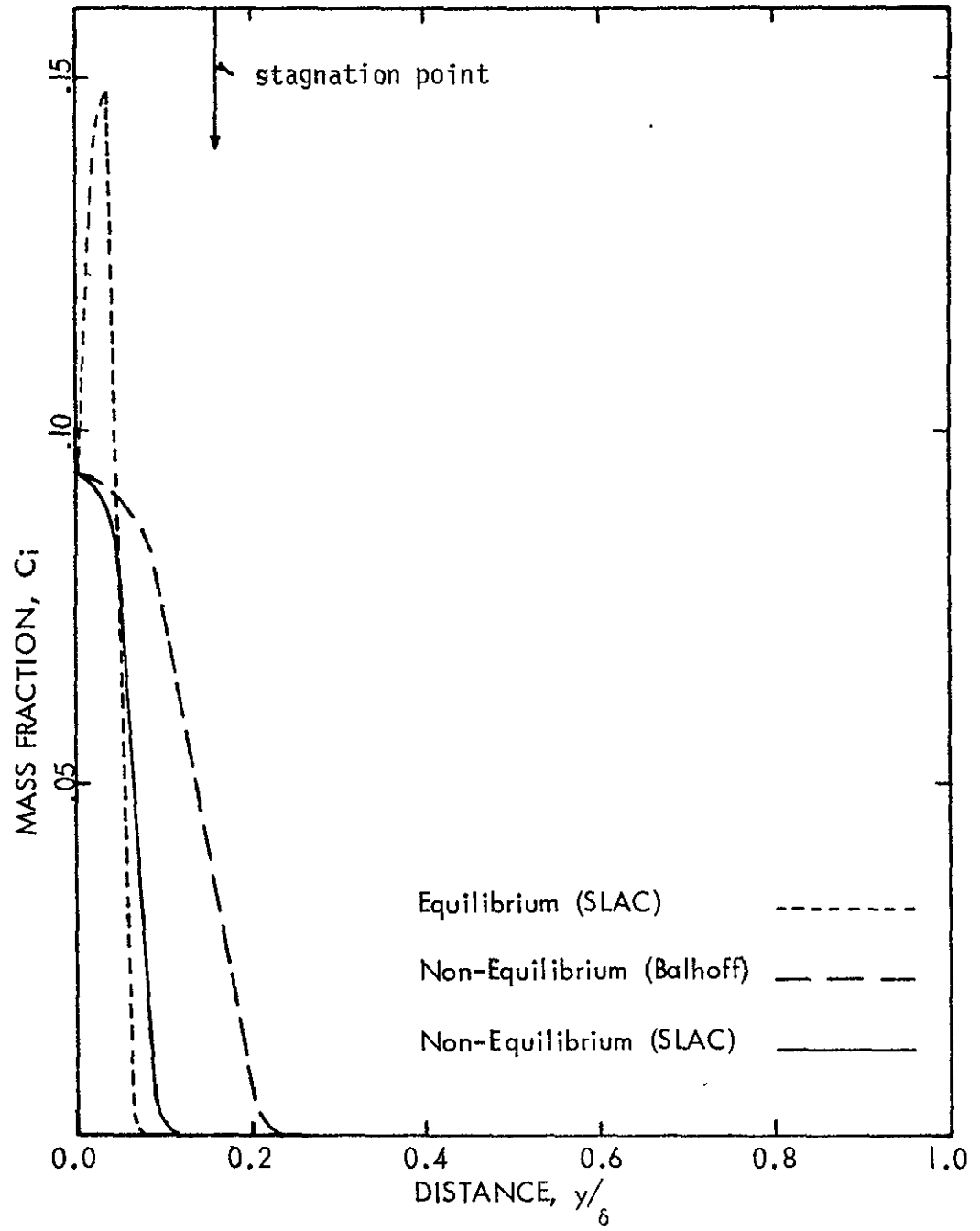


Figure 5.2 Mass Fraction Profiles for C_3

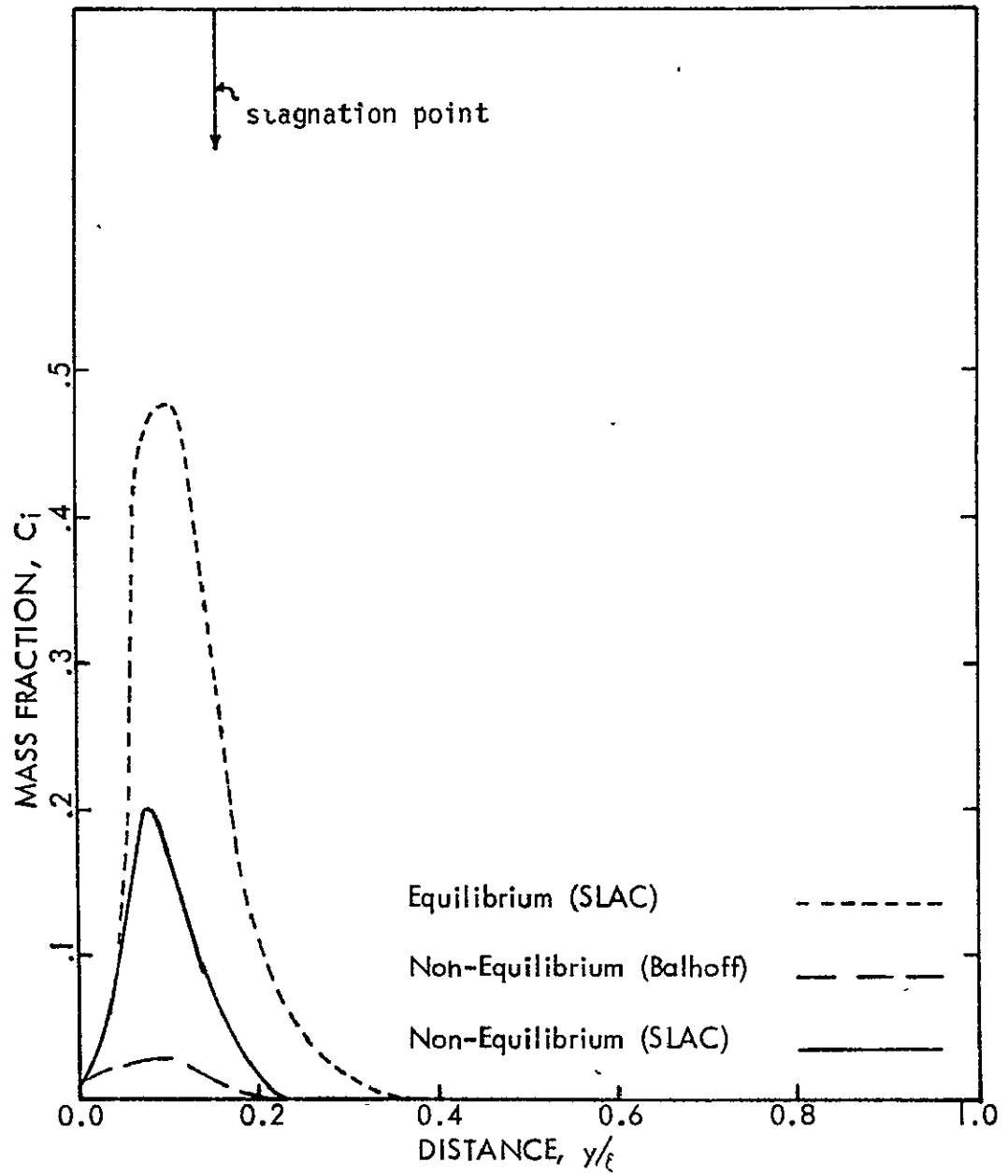


Figure 5.3 Mass Fraction Profiles for C

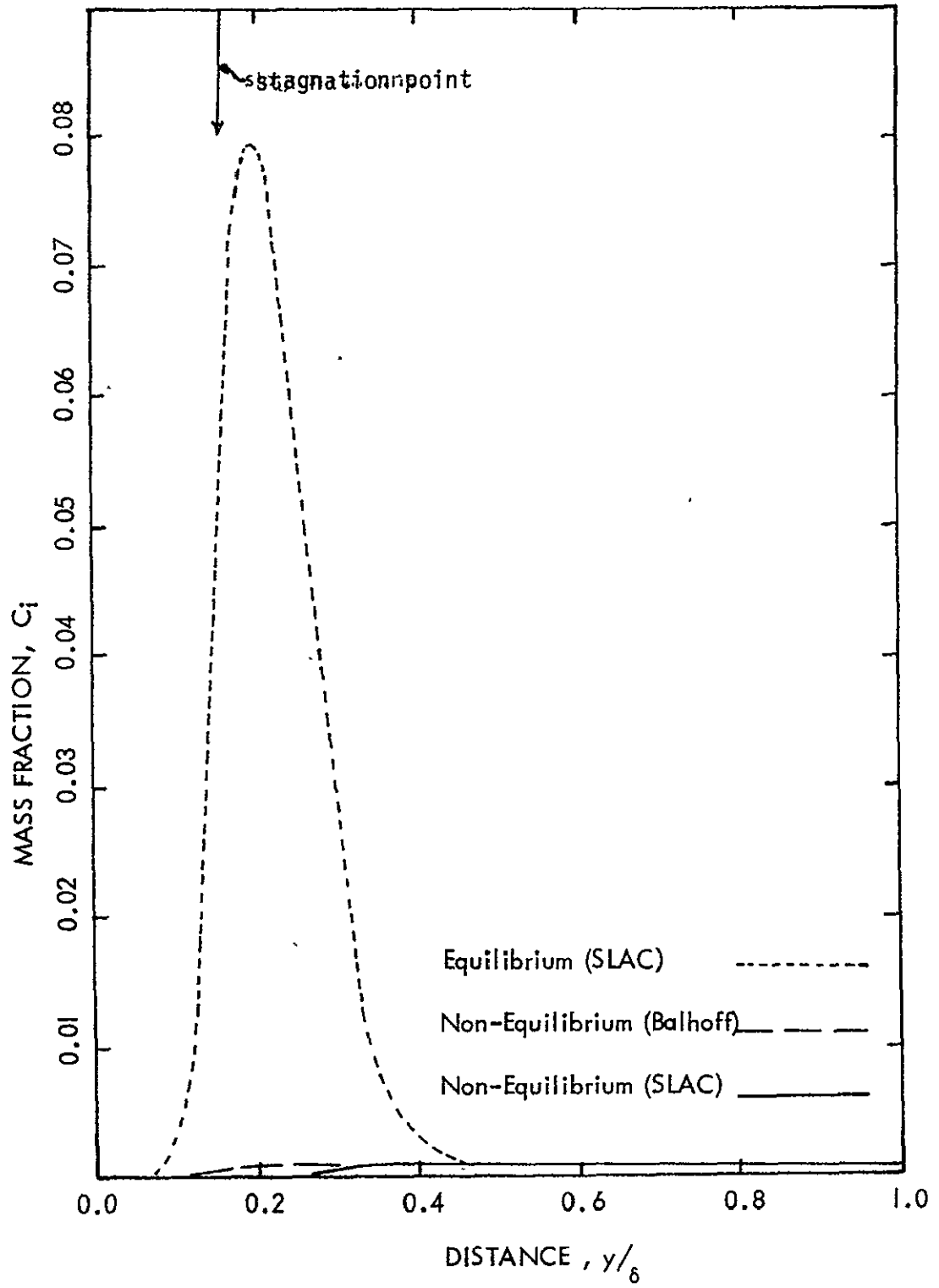


Figure 5.4 Mass Fraction Profiles for C^+

this region (see Fig. 5.4).

In the shock layer region of the flow-field, equilibrium results predict that as the air components are cooled down a process of rapid de-ionization occurs with most of the N^+ and O^+ being converted to N and O (see Figs. 5.5 and 5.6). On the other hand, non-equilibrium results predict that the air components will flow from the shock to the stagnation point without undergoing any changes until very close to the stagnation point where their concentrations begin to decrease and fall rapidly past the stagnation point. As a result of these "frozen" concentrations, the region between the shock and the stagnation point has much lower concentrations of N and O (and much higher concentrations of N^+ and O^+) than those predicted by the equilibrium analysis (see Figs. 5.5 and 5.6).

Therefore, the non-equilibrium analysis predicts much lower concentrations of C, CN, C^+ , N_2 , N, O, and H and much higher concentrations of C_2 , CO, C_3H , C_2H , C_4H , HCN, C_2H_2 , N^+ , O^+ , H_2 , and e^- than the equilibrium analysis. Since C, N, O, and H line and continuum mechanisms are the major contributors to radiative energy transport in the shock layer (Ref. 5.4), under non-equilibrium chemistry conditions the resulting radiative heat transfer to the body is significantly lower than the one predicted by chemical equilibrium analyses.

DIFFICULTIES ASSOCIATED WITH THE SOLUTION

This research demonstrates that the speed of obtaining a solution, if one can be obtained, depends on a large number of factors including: the formulation of the model, how realistic the values used for properties are, the numerical techniques used, and how such techniques are implemented.

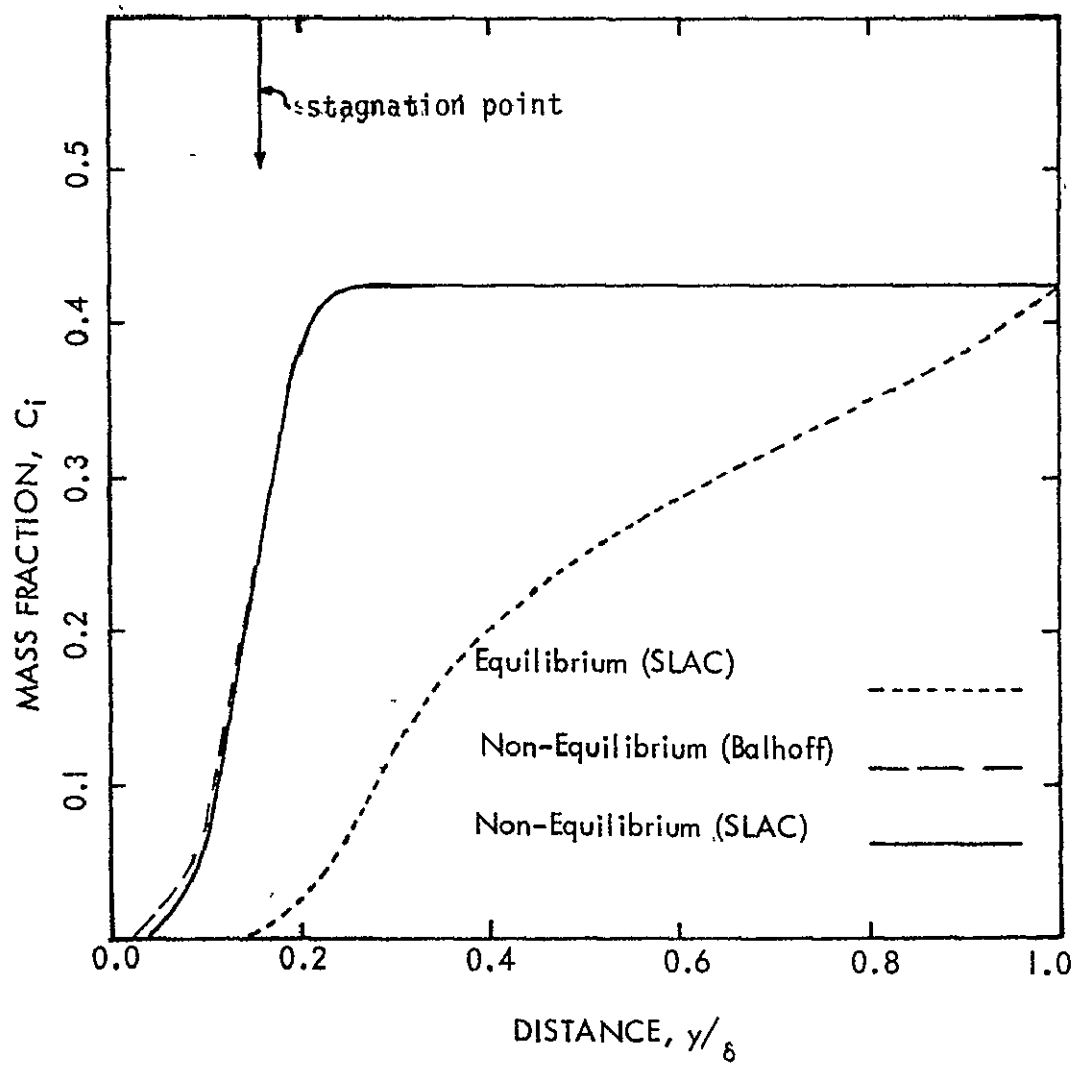


Figure 5.5 Mass Fraction Profiles for N^+

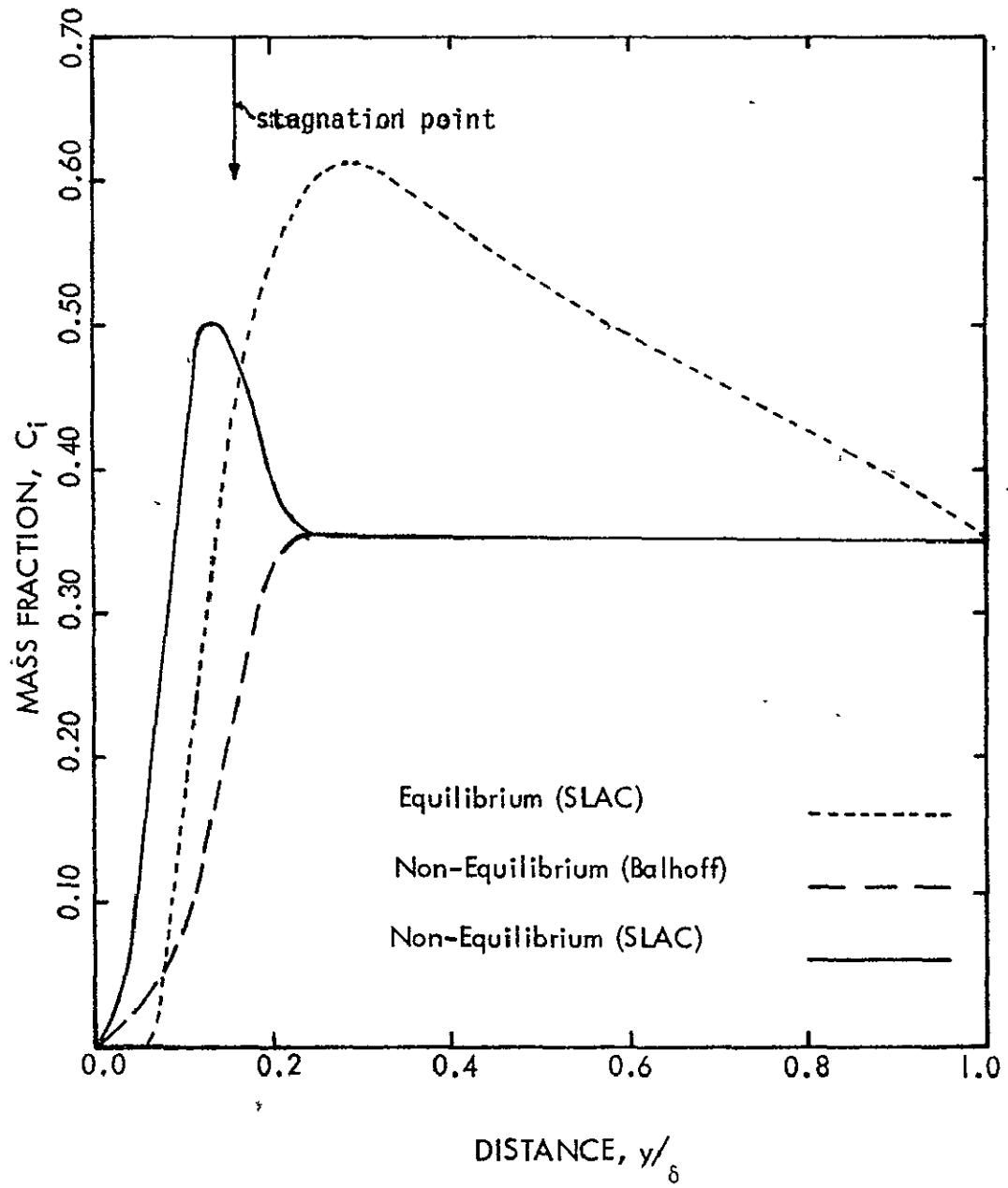


Figure 5.6 Mass Fraction Profiles for N

The attempts at using the energy equation in terms of temperature failed because the temperature profile is very sensitive to the values of the other dependent variables and the energy equation in this form is apparently more strongly coupled to the species equations. The enthalpy formulation, on the other hand, permits uncoupling of the equations and makes it possible to obtain a converged solution.

However, although a solution was determined using the energy equation in terms of enthalpy, obtaining such a solution was extremely difficult. Much of this difficulty arose because of the use of unrealistically high reaction rates. The non-equilibrium solution using the original chemical kinetics model took about 18 hours of time in an IBM 360/65 computer. The solution using the modified kinetics model took about 45 minutes in the same computer. This means that the unrealistic rates increased the difficulty in obtaining a solution.

As was discussed in Chapter 4, a number of numerical techniques have been demonstrated to be inappropriate for solving chemical kinetics problems. However, even those techniques which may be valid will not work unless they are implemented properly. For example, the scheme used to uncouple and iterate on the equations will, in many cases, make the difference between success and failure. Also, the weighting factors used to obtain new guesses from the old and new solutions (the λ 's in Figs. 4.1 - 4.4) determine if the solution will converge. Unfortunately, there is no straightforward procedure for determining how to implement a technique properly; a great deal of experimentation is required before success is attained.

REFERENCES

- 5.1 Engel, Carl D., Ablation and Radiation Coupled Viscous Hypersonic Shock Layers, PhD. Dissertation, Louisiana State University, Baton Rouge, Louisiana (1971)
- 5.2 Esch, Donald D., Stagnation Region Heating of a Phenolic-Nylon Ablator During Return from Planetary Missions, Ph.D. Dissertation Louisiana State University, Baton Rouge, Louisiana (1971).
- 5.3 Balhoff, John F., Research Associate, Chemical Engineering Department, Louisiana State University, Private Communication.
- 5.4 Engel, op, cit., Vol. I, p. 126

CHAPTER 6

CONCLUSIONS AND RECOMMENDATIONS

CONCLUSIONS

From the results obtained in the present investigation the following conclusions can be derived:

A finite-rate stagnation-line shock layer solution which contains a reasonable kinetics model to describe atmospheric entries protected by phenolic-nylon ablators was developed.

The model was used to determine the non-equilibrium shock layer structure and body heating rates under flight conditions typical of re-turn from planetary missions. Although only one set of conditions was considered, it is believed that the method of solution is adequate for a range of conditions similar to the ones considered.

2. Finite-rate chemistry effects are significant for the flight conditions considered.

The results obtained show that as the ablation products enter the shock layer they react much slower than under equilibrium conditions. The air components entering through the shock are not de-ionized, as the equilibrium analysis predicts, but remain "frozen" throughout most of the shock region. The validity of these results are, of course, dependent upon the accuracy of the chemical kinetics model used. The degree of confidence in the results is high because: the kinetics model used is

the most complete that has been used in this type of application, the reaction rates used were the best rates available, and for both sets of reaction rates (the original and the modified) the same finite-rate chemistry effects were present.

3. For the flight conditions studied the total heating rate to the body is significantly lower under non-equilibrium than under equilibrium chemistry conditons.

The predicted non-equilibrium heating rate is approximately one third as large as the predicted equilibrium heating rate. This results from the fact that the non-equilibrium radiative heating rate is approximately one third as large as the equilibrium radiative heating, and in both instances, most of the total heating results from radiation. The significantly lower radiative heating rate is a direct result of finite rate chemistry effects which result in much lower concentrations of C, N, O, and H, the species responsible for most of the line and continuum radiative transfer.

RECOMMENDATIONS

Considering the conclusions presented above, it is recommended that:

1. The mathematical model for the stagnation-line analysis of an entry vehicle presented herein should be studied further to determine if its computational speed can be increased.

The present version of SLAC should be developed from the research tool it is today until it becomes an effective engineering tool. The principal obstacle that needs to be overcome is the amount of computer time required to obtain a solution. The computational speed is a function of: the model formulation, how realistic the properties used are,

the schemes for uncoupling and iterating on the equations, and the numerical techniques used. Much improvement in computational speed was obtained by considering different variations on all these factors. It is believed that the computational speed of the present solution could be increased by at least a factor of 2 by further consideration of uncoupling and iterating schemes.

2. Studies of finite-rate chemistry effects be carried out under flight conditions different from those considered in the present work.

The objectives of the proposed studies should be to determine: a) the range of flight conditions over which solutions may be obtained using the implemented model; and b) the range of conditions over which finite-rate chemistry effects are significant and how they affect the heating rates to the body.

3. Research be carried out on the effect of assuming more realistic shock and wall boundary conditions.

This investigation should consider using non-equilibrium compositions of air at the shock as boundary conditions and using boundary conditions of the third kind at the wall. The use of non-equilibrium compositions of air will result in increased concentrations of N and O and less N^+ and O^+ when compared to the equilibrium boundary conditions. This might result in increased radiative heating rates since N and O are optically more active than N^+ and O^+ . By perturbing the shock boundary conditions used in SLAC an estimate of the importance of this effect could be developed. In the present investigation the mass and energy boundary conditions used at the wall were of the first kind, however the mass and energy surface balances yield conditions of the third kind.

A study should be carried out to determine if using the more complete conditions has any significant effect on the heating rate.

4. Coupled solutions for both non-equilibrium ablation products and shock layers should be determined.

A complete solution to the quasi-steady entry problem requires that the ablator response be coupled to the existing flight conditions through the shock layer. This can be accomplished by performing multiple calculations of ablation and shock layer behaviour and matching conditions at the interface between the ablator surface and the flow field. At present, ablator-shock layer response under chemical equilibrium conditions has been studied, however, additional studies should be carried out coupling the finite-rate solutions determined from SLAC to those obtained from a non-equilibrium ablator-response model.

APPENDIX A

USER'S MANUAL FOR SLAC

INTRODUCTION

This appendix will serve as a user's manual for the SLAC (Stagnation Line Analysis with Chemistry) program. The program implements a model designed to predict the stagnation line viscous, reactive, and radiative coupled shock layer structure, and the resulting heating rates produced by a blunt body during super orbital entry into planetary atmospheres. The problem was formulated in Chapter 1, and the equations describing the flow-field (the stagnation line boundary layer equations (See Table 2.7), and the wall and shock boundary conditions (See Table 2.8), were derived in Chapter 2. The thermodynamic, transport, radiative, and chemical kinetic properties are described in Chapter 3. The program can also be used to compute flows in chemical equilibrium, and for this purpose it utilizes a free energy minimization routine developed by Del Valle and Pike (Ref. A.1). The numerical procedures used to solve the model are described in Chapter 4.

The SLAC program results from extensions performed on a program (VISRAD) primarily developed by Engel (Ref. A.2), and in a program (SLAB) developed by Esch (Ref. A.3) from VISRAD. These programs result from the efforts of many individuals over a considerable period of time. SLAC was designed to be used for thermal environment prediction studies. It

represents a significant tool for studying a variety of atmospheric entry heating problems. The program, written in FORTRAN IV, is capable of performing the following types of analyses:

- Stagnation line solutions
- Coupled diffusive, convective and radiative flux calculations
- Emission, and line and continuum radiation calculations
- Binary diffusion calculations
- Finite-rate or equilibrium chemistry calculations

PROGRAM PROCEDURES

The SLAC computer program was developed following a philosophy of minimizing user's effort and maximizing program flexibility and adaptability. Accordingly, the basic program logic as shown in Figure A.1 is quite simple. However these basic subprograms are supported by 22 subroutines and 7 function subprograms. Each of these modules performs computational functions which are of a basic nature (e.g., computation of thermodynamic properties), and allows for modification, substitution, addition or deletion of existing modules with minimum effort.

In order to minimize input requirements, three techniques were used. The first consists of internal initialization of values for temperature, density, viscosity, and stand-off distance which are necessary to start the solution procedure. If better guesses are available they may be input as discussed in the next section. The second technique involves internal specification in BLOCK DATA of problem-defining parameters such as the elemental composition at the surface and thermodynamic curve-fit constants which are changed quite infrequently. The third technique consists of internally selecting program options if an option variable is left blank on an input card. In this procedure the most commonly used options are performed when a blank is input.

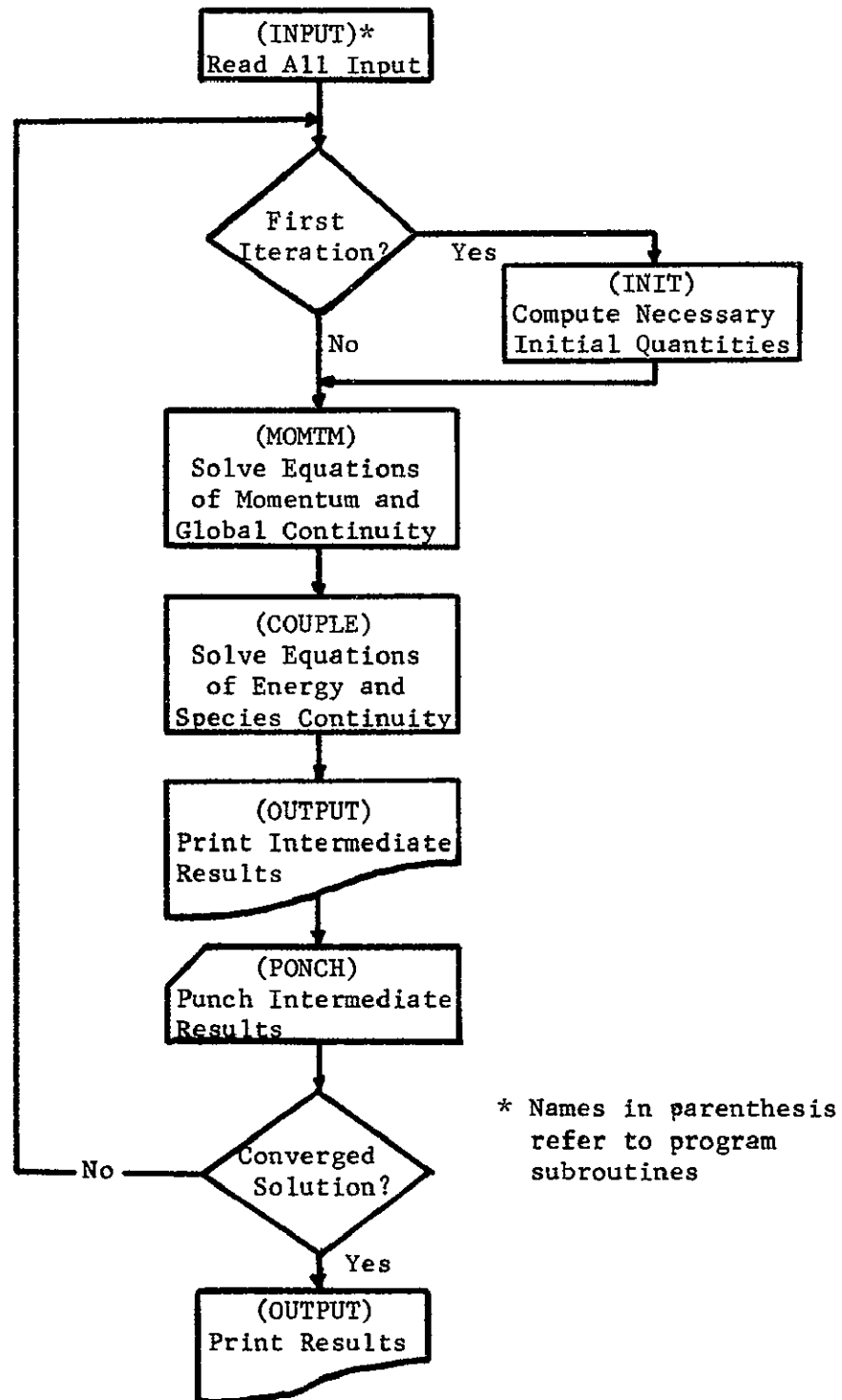


FIGURE A.1 PROGRAM FLOWCHART

The start up of the SLAC program can be achieved in a number of ways. As stated previously, internal guesses are available to begin the iteration procedure. Two types of temperature profiles are available. One for no mass injection and the other for mass injection. These profiles are usually quite satisfactory as initial guesses if an emission radiation coupled problem is to be run. However, if a line and continuum radiation coupled problem which includes mass injection, and finite-rate chemistry is to be run the internal guess may not be accurate enough. Consequently, it might be necessary to input a better temperature profile guess.

INPUT GUIDE

All inputs to the SLAC computer program are read from cards supplied by the user; no tapes are required. The basic inputs consist of parameters defining flight conditions (free-stream velocity and density), blunt body radius, wall temperature and mass injection rate. Additional input parameters are required to determine which program options are desired, and to provide the necessary guesses of dependent variables. After each overall iteration, the program outputs a deck of cards containing the values of the dependent variables and other pertinent parameters in such a manner that a given case run may be interrupted and continued at a later time by using the produced deck of punched cards to restart the run. Multiple case runs, and hence entire trajectories, can be processed by placing the input data for each new case behind the data for the previous one.

Table A.1 presents the card input formats for SLAC and Table A.2 provides a corresponding definition of variables.

In single case runs Card 10 must be followed by a card with the characters END punched in columns 1, 2 and 3 to indicate the run has ended. In multiple case runs the END card should be placed after Card

TABLE A.1 CARD INPUT FOR SLAC

| <u>Card</u> | <u>Variables</u> | <u>Format</u> |
|-------------|---|------------------------|
| 1 | TITLE, IEM | 18A4, I8 |
| 2 | KEEP, NETA, IRAD, ITYPE, MAXM, MAXE MAXD, LT, IPHI, FPRCT, TPRCT, IDEBUG | 9I5, 2E12.0, 2X, I1 |
| 3 | UNIF, RINF, R, TWK, HTOTAL, RVW | 6E12.0 |
| 4 | DELTA, DTIL, RZB, RE, PDTIL | 5E12.0 |
| 5 | T(I) | 6E12.0 |
| 6A | RHØ(I) | 6E12.0 |
| 6B | RM(I) | 6E12.0 |
| 7 | DEPS | E12.0 |
| 8 | ETA(I) | 6E12.0 |
| 9 | NDEBUG, TOL, IAB | I5, 5X E10.4, I5 |
| 10 | CWALL(J) | 5E15.8 |

TABLE A.2 VARIABLE DEFINITIONS FOR SLAC

| <u>Card</u> | <u>Variable</u> | <u>Description</u> |
|-------------|-----------------|---|
| 1 | TITLE | Title for identification of the problem |
| | IEM | Number of overall iterations performed. IEM = 0 for case run initiation, IEM>0 for restarts |
| 2 | KEEP | Indicator to determine if the temperature profile from the previous case is to be kept as a guess for the current case. KEEP = 0 Temperature not kept KEEP = 1 Temperature kept |
| | NETA | The number of points to be used in the shock layer profile. If NETA = 0, a set of 51 equally spaced points will be used. If NETA>0 card 8 must be read. |
| | IRAD | A variable used to specify the type of solution. IRAD = 1 Convective solution only = 2 Uncoupled radiation = 3 Coupled radiation solution |
| | ITYPE | A variable used to specify the type of radiation model to be used. ITYPE = 0 Line and continuum radiation model = 1 Emission radiation model |
| | MAXM | Maximum number of iterations allowed in the internal momentum loop. If MAXM = 0, it is internally set = 15. |
| | MAXE | Maximum number of iterations allowed in the energy-species continuity equation and in the overall momentum-energy-species loop. If MAXE = 0, it is internally set = 15. |
| | MAXD | Maximum number of iterations allowed in the external momentum loop. If MAXD = 0, it is internally set = 15. |
| | | |
| | | |
| | | |

| <u>Card</u> | <u>Variable</u> | <u>Description</u> |
|-------------|-----------------|--|
| | LT | Indicator to determine if a temperature guess and if ρ and ρv - guesses are to be read in. LT = 0 Cards 5 and 6 are not read. = 1 Card 5 but not card 6 is read. = 2 Cards 5 and 6 are read. |
| | IPHI | Indicator to determine if the shock curvature is to be input. IPHT = 0 $d\epsilon/d\epsilon = 0$ is internally set. = 1 Card 7 is required for input. |
| | FPRCT | Convergence tolerance for each point the f' profile. If FPRCT = 0.0 it is internally set = .005. |
| | TPRCT | Convergence tolerance for each point in the T profile. If TPRCT = 0.0, it is internally set = .005. |
| | IDEBUG | A switch to allow intermediate printout to be obtained at each iteration IDEBUG = 0 No print. = 1 Print is given |
| 3 | UNIF | The free-stream flight velocity (U_∞) in feet/sec. |
| | RINF | The free-stream density (ρ_∞) in slugs/ft ³ . |
| | R | Principal body radius in feet. |
| | TWK | Wall Temperature in degrees Kelvin. |
| | HTOTAL | Total free-stream enthalpy in ft ² /sec. ² . If HTOTAL = 0.0, it is set to $U_\infty^2/2$. (Free-stream static enthalpy is assumed negligible). |
| | RVW | Mass injection rate $(\rho v)_w/(\rho U)_\infty$. |
| 4 | DELTA | An initial guess for the shock standoff distance δ/R . If DELTA = 0.0, a guess is supplied by program. |

| <u>Card</u> | <u>Variable</u> | <u>Description</u> |
|-------------|----------------------|--|
| | DTIL | A guess for the transformed standoff distance δ/R . The program will also supply this value if DTIL = 0.0 |
| | RZB | The density ratio across the shock $\bar{\rho} = \rho_{\infty}/\rho_{\delta}$. If RZB is input as 0.0, the code will determine a value. |
| | RE | The Reynolds number for the problem, $Re_s = U_{\infty} R \rho_{\delta} / \mu_{\delta}$. This quantity is determined by the program if RE is input as 0.0. |
| | PDTIL | Convergence tolerance placed on δ for total solution convergence. If PDTIL = 0.0, it is internally set = .001. |
| 5 | T(I), I = 1, NETA | An initial guess for the dimensionless shock layer temperature profile (T/T_{δ}). This card is required only if LT > 0. |
| 6A | RH0(I), I=1, NETA | An initial guess for the dimensionless shock layer density profile (ρ/ρ_{δ}). This card is required only if LT = 2. |
| 6B | RM(I), I=1, NETA | An initial guess for the dimensionless shock layer ρv profile ($\rho v/\rho_{\delta} v_{\delta}$). This card is required only if LT = 2. |
| 7 | DEPS | The stagnation line shock curvature ($d\epsilon/dx$). If IPHI = 0, then $d\epsilon/dx = -.0$ is internally set. If IPHI = 1, Card 7 is read and $d\epsilon/dx$ is supplied by the user. |
| 8 | ETA(I), I=1, NETA | The grid shock layer points at which the solution profiles are to be computer. If NETA = 0, $\Delta\eta$ is set to 0.02 and ETA (I) is computed by the program. (ETA(1)=0.0 \rightarrow wall, ETA (NETA)=1.0 \rightarrow shock). |
| 9 | NDEBUG | Debug option to output thermodynamic curve-fit equations and intermediate results from NDEBUG = 0 No output. = 1 Output given. |

| <u>Card</u> | <u>Variable</u> | <u>Description</u> |
|-------------|----------------------|---|
| | TOL | Convergence criteria for CHEMEQ. If TOL is input as 0.0, the code will set it to 0.001. |
| | IAB | A variable used to specify the type of chemistry model to be used. IAB = 0 Finite-rate chemistry IAB = 1 Equilibrium chemistry |
| 10 | CWALL(J), J=1,NSP | Wall mass fractions. NSP = 20 <div style="display: flex; flex-wrap: wrap;"> <div style="width: 25%;">J = 1→O₂</div> <div style="width: 25%;">6→N⁺</div> <div style="width: 25%;">11→CO</div> <div style="width: 25%;">16→C₃H</div> <div style="width: 25%;">2→N₂</div> <div style="width: 25%;">7→E⁻</div> <div style="width: 25%;">12→C₃</div> <div style="width: 25%;">17→C₄H</div> <div style="width: 25%;">3→O</div> <div style="width: 25%;">8→C</div> <div style="width: 25%;">13→CN</div> <div style="width: 25%;">18→HCN</div> <div style="width: 25%;">4→N</div> <div style="width: 25%;">9→H</div> <div style="width: 25%;">14→C₂H</div> <div style="width: 25%;">19→C₂</div> <div style="width: 25%;">5→O⁺</div> <div style="width: 25%;">10→H₂</div> <div style="width: 25%;">15→C₂H₂</div> <div style="width: 25%;">20→C⁺</div> </div> |

10 of the last case to be run.

Some caution should be exercised when preparing an input for this program. The program considers 5 elements, including electrons, and twenty species listed under CWALL (J). The set of species was selected for an air atmosphere and an phenolic-nylon ablator. If another ablator is selected for study and this set of species is appropriate, no alteration of the program is required. All that is required is a card input of wall mass fractions of the ablator selected on Card 10. If extensive study of a different ablator using this program is anticipated, the user may find it convenient to change the wall composition stated in BLOCK DATA under CWALL rather than reading in the data for each run.

If required, a change to another set of species can be made with comparatively little difficulty. Thermodynamic and transport properties may be altered by changing the curve - fit constants in BLOCK DATA. The thermodynamic curve - fit equations were listed in Table 3.1 and the correspondence between the coefficients in the table and those in the program is

| For | | | | | |
|-----------------|---|----------------|------------|--|-----|
| 1000 < T ≤ 6000 | | | 6000 < T°K | | |
| AI | = | A ₁ | = | | AI1 |
| BI | = | A ₂ | = | | BII |
| CI | = | A ₃ | = | | CII |
| DI | = | A ₄ | = | | DII |
| EI | = | A ₅ | = | | EII |
| FI | = | A ₆ | = | | FII |
| GI | = | A ₇ | = | | GII |

where the coefficients are dimensioned to include a value for each species.

The species ordering is given in the SP array with corresponding ordering in SMW (i.e. species molecular weight) array in BLOCK DATA. The transport properties curve-fit equations are listed in Tables 3.3 (viscosity) and 3.4 (thermal conductivity) and the correspondence between the coefficients in those tables and those in the program is:

Viscosity:

| Table 3.3 | | SLAC |
|-----------|---|------|
| a | = | V1 |
| b | = | V2 |
| c | = | V3 |

Thermal Conductivity

| Table 3.4 | | SLAC |
|-----------|---|------|
| a | = | K1 |
| b | = | K2 |

If new species and reactions are to be included in the finite-rate calculations, then subroutine FG2 must be modified accordingly. Finally, a study should be made to determine if radiatively important species are to be included.

OUTPUT DESCRIPTION

This section presents a description of the program output and definitions of the output symbols. The program produces both printed output and punched card output. The printed output provides a human-readable record of input parameters, intermediate results and the final solution. The punched card output provides a machine-readable medium for restarting runs.

Printed Output

The first page of output is a print of the input data. This is provided for a check of the input and an identification of the problem.

All quantities on this page are defined in the Input Guide section. Most of the second page also contains problem specification data which is self explanatory. Following the guessed nondimensional stand-off distance (DELTA) and transformed stand-off distance (DELTA) and transformed stand-off distance (DTIL), a listing of the dimensional stand-off distance computed at each iteration is given if the line-continuum radiation model is used.

Species number densities for those species used in the radiation calculation during the final iteration are printed on the third page. The fourth page provides an output of radiative fluxes computed during the final iteration. The continuum contribution and line contribution to the spectral flux is printed for three ETA points (ETA = 0.0 = wall, ETA = stagnation point, ETA = 1.0 = shock) as a function of frequency intervals and frequency centers respectively. The columns of fluxes in watts/cm² denoted by Q PLUS and Q MINUS designates fluxes toward the surface and away from the surface respectively.

From the fifth until the thirtieth page, results of the overall iteration, either final or intermediate, are presented. The fifth page begins with one of the following messages: (1) "INTERMEDIATE PRINT AT ITERATION NO. a MCONV = b ECONV = c DCONV = d" where a is the overall iteration number, and b, c, and d are either T (true) or F (false) depending upon if the momentum - global continuity equation, energy-species continuity equations, and the shock standoff distance have converged, respectively. If this message is printed at least one of b, c, or d must be F. (2) "SOLUTION CONVERGED IN e ITERATIONS", where e is the number of overall iterations it took to converge. Following either one of these two messages is a printout of the shock stand-off distance parameters DELTA, DTIL; the convective (QC), radiative (QR), diffusive

(QD) and total heating rate with the respective units printed. To the left of the heating rate data the density ratio across the shock (RB) and the mass injection rate (RVW) are stated. Following the heating rate data is a print of some of the solution profiles as a function of the shock layer coordinates ETA and (Y/D). The solution profiles printed are:

F' = velocity function

$RV = \rho v / \rho_{\infty} U_{\infty}$ (nondimensional mass flux per unit area)

$T/TD = T/T_{\delta}$ (nondimensional temperature)

$E = 1E$ (radiative flux divergence)

$V = V/U_{\infty}$ (nondimensional normal velocity)

$V(FT/SEC)$ = dimensional normal velocity

G = nondimensional total enthalpy

$H(STATIC)$ = nondimensional static enthalpy

These profiles appear in part of page 5 and in page 6.

The shock layer thermodynamic and transport properties as a function of ETA and Y/D are printed in the following pages. The profiles printed are:

$P(ATM)$ = pressure in atmospheres

$T(DEG. KEL.)$ = temperature in degrees Kelvin

$RHO (SLUGS/FT^3)$ = density in slugs/cubic feet

$M (LBM/FT-SEC)$ = viscosity in lbm/(ft-sec)

$RM (LBF^2-SEC^3/FT^6)$ = product of density and viscosity in
 $lbf^2 - sec^3/ft^6$

$K(BTU/FT-SEC-R)$ = thermal conductivity in Btu/(ft-sec-°R)

The next four pages present the profiles of the mass fractions of O_2 , N_2 , O , N , O^+ , N^+ , e^- , C , H , H_2 , CO , C_3 , CN , C_2H , and C_2H_2 as a function of ETA. Page 13 contains the profiles of mixture specific heat at

constant pressure (CP), mass fractions for C_3H , C_4H , HCN and C^+ , and the mixture molecular weight (AMW) as a function of ETA.

The above description is for a standard output. However, if the intermediate print option is used (i.e. IDEBUG > 0) or if the thermodynamic fits and intermediate results from the chemical equilibrium calculation are desired (i.e. NDEBUG = 1), additional information is printed.

If message number (2) is printed, the run has ended and the program proceeds to the next case or stops if there are no additional cases to be run. If message number (1) is printed, the solution has not converged and additional iterations must be carried out.

Punched Card Output

After each overall iteration the program outputs a deck of cards containing all relevant data on flight conditions, body characteristics, and shock layer structure. These data are punched by a subroutine named PUNCH in such a manner that the cards may be used to restart a run at a later time. For example, if the program is allowed to run for 3 hours, during that time the program might perform 4 overall iterations and it would output 4 decks of punched cards, one for each overall iteration. If the solution has not converged after the 3 hours have run out, the fourth deck of punched cards may be used to restart the run at a later time.

REFERENCES

- A.1. Del Valle, E.G., and R.W. Pike, "Computation of the Equilibrium Composition of Reacting, Gas-Solid Mixtures with Material and Energy Balance Constraints," NASA-RFL-10, March 1970, Reacting Fluids Laboratory, Louisiana State University, Baton Rouge, Louisiana.

- A.2 Engel, Carl D., Ablation and Radiation Coupled Viscous Hyper-sonic Shock Layers, Ph.D. Dissertation, Louisiana State University, Baton Rouge, Louisiana (1971).
- A.3 Esch, Donald D., Stagnation Region Heating of a Phenolic-Nylon Ablator During Return from Planetary Missions, Ph.D. Dissertation, Louisiana State University, Baton Rouge, Louisiana (1971).

APPENDIX B
SLAC PROGRAM LISTING

| | | | | | |
|----|--|--|---------------|------|------|
| C | **** | S L A C | **** | MAIN | 10 |
| C | STAGNATION LINE HEATING ANALYSIS FOR A VISCOUS HYPERSONIC | | | MAIN | 20 |
| C | SHOC LAYER WITH FINITE-RATE OR EQUILIBRIUM CHEMISTRY AND | | | MAIN | 30 |
| C | RADIATIVE HEAT TRANSFER. | | | MAIN | 40 |
| C | G. PEREZ, C.D. ENGEL, AND D.D. ESCH | | | MAIN | 50 |
| C | JUNE, 1972 | | | MAIN | 60 |
| | COMMON /FRSTRM/ | U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | | MAIN | 70 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | | | MAIN | 80 |
| | LOGICAL MCONV,ECONV,DCONV | | | MAIN | 90 |
| | COMMON /RFLUX/ | E(60),IRAD,ITYPE | | MAIN | 100 |
| | COMMON/NUMBER/ | NSP,NNS,NE,NC | | MAIN | 110 |
| | COMMON/EQ2/ | AA(20,5),ICODE(20) | | MAIN | 120 |
| | COMMON/EQ3/ | IA(20,5) | | MAIN | 130 |
| C | | | | MAIN | 140 |
| C | | | | MAIN | 150 |
| C | | | | MAIN | 160 |
| C | ** | D R I V E R | P R O G R A M | ** | MAIN |
| C | | | | MAIN | 170 |
| C | | | | MAIN | 180 |
| C | | | | MAIN | 190 |
| 1 | CONTINUE | | | MAIN | 200 |
| C | | | | MAIN | 210 |
| C | ** | READ AND PRINT ALL INPUT DATA | ** | MAIN | 220 |
| C | | | | MAIN | 230 |
| | CALL INPUT | | | MAIN | 240 |
| C | | | | MAIN | 250 |
| C | ----- | FLOAT AA(I,J) MATRIX.... | | MAIN | 260 |
| C | | | | MAIN | 270 |
| | DO 30 I=1,NSP | | | MAIN | 280 |
| | DO 30 J=1,NE | | | MAIN | 290 |
| 30 | AA(I,J)=IA(I,J) | | | MAIN | 300 |
| C | | | | MAIN | 310 |
| C | ** | COMPUTE NECESSARY INITIAL QUANTITIES | ** | MAIN | 320 |
| C | | | | MAIN | 330 |
| | IF(IEM.EQ.0)CALL INIT | | | MAIN | 340 |
| | IF(ITYPE.EQ.0)CALL RADIN | | | MAIN | 350 |

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|---|--|----------|
| C | | MAIN 360 |
| C | **** START OVERALL ITERATION **** | MAIN 370 |
| C | | MAIN 380 |
| | 1000 CONTINUE | MAIN 390 |
| | IEM = IEM+1 | MAIN 400 |
| C | | MAIN 410 |
| C | ** SOLVE MOMENTUM EQUATION ** | MAIN 420 |
| C | | MAIN 430 |
| | CALL MOMTM | MAIN 440 |
| C | | MAIN 450 |
| C | | MAIN 460 |
| C | | MAIN 470 |
| C | ** SOLVE ENERGY AND SPECIES EQUATIONS ** | MAIN 480 |
| C | | MAIN 490 |
| | CALL COUPLE | MAIN 500 |
| C | | MAIN 510 |
| C | ** INTERMEDIATE PRINTOUT ** | MAIN 520 |
| C | | MAIN 530 |
| | CALL OUTPUT (2) | MAIN 540 |
| | CALL PONCH | MAIN 550 |
| C | | MAIN 560 |
| C | | MAIN 570 |
| C | | MAIN 580 |
| C | ** CHECK SIMULTANEOUS MOMENTUM AND ENERGY CONVERGENCE ** | MAIN 590 |
| C | | MAIN 600 |
| | IF(IEM.GT.MAXD) GO TO 3000 | MAIN 610 |
| | IF(.NOT.MCONV) GO TO 1000 | MAIN 620 |
| | IF(.NOT.ECONV) GO TO 1000 | MAIN 630 |
| | IF(.NOT.DCONV) GO TO 1000 | MAIN 640 |
| C | ** PRINT ALL OUTPUT ** | MAIN 650 |
| C | | MAIN 660 |
| | CALL OUTPUT (1) | MAIN 670 |
| C | | MAIN 680 |
| C | ** CONVERGED , GO BACK TO RUN ANOTHER CASE ** | MAIN 690 |
| C | | MAIN 700 |

| | | |
|---|---|----------|
| | GO TO 1 | MAIN 710 |
| C | | MAIN 720 |
| | 3000 CONTINUE | MAIN 730 |
| C | | MAIN 740 |
| C | ** MOMENTUM AND ENERGY DID NOT CONVERGE SIMULTANEOUSLY ** | MAIN 750 |
| C | | MAIN 760 |
| | CALL OUTPUT (3) | MAIN 770 |
| | GO TO 1 | MAIN 780 |
| | END | MAIN 790 |

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|---|--|----------|
| | SUBROUTINE INPUT | INPU 10 |
| C | | INPU 20 |
| C | | INPU 30 |
| C | ** ROUTINE TO READ AND PRINT ALL INPUT DATA ** | INPU 40 |
| C | | INPU 50 |
| C | | INPU 60 |
| | COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDIL | INPU 70 |
| | COMMON/CONV1/HDAMP | INPU 80 |
| | COMMON /DEL/ DELTA,DTIL,DTILS | INPU 90 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | INPU 100 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | INPU 110 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CPNF | INPU 120 |
| | COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),CC(5,60) | INPU 130 |
| | COMMON/PROP2/ MU(60),RM(60), AK(60) | INPU 140 |
| | COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | INPU 150 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | INPU 160 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | INPU 170 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTGTAL | INPU 180 |
| | COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | INPU 190 |
| | COMMON /YL/ETA(60),YEND(60) | INPU 200 |
| | COMMON/TIT/TITLE(18) | INPU 210 |
| C | | INPU 220 |
| C | | INPU 230 |
| | COMMON/EQ1/AI(20), BI(20), CI(20), DI(20), EI(20), FI(20), GI(20), | INPU 240 |
| X | AII(20),BII(20),CII(20),DII(20),EII(20),FII(20),GII(20) | INPU 250 |
| | COMMON/EQ2/AA(20,5),ICODE(20) | INPU 260 |
| | COMMON/EQ3/IA(20,5) | INPU 270 |
| | COMMON/ID/SP(20),EL(5) | INPU 280 |
| | COMMON/WT/SMW(20),AWT(5) | INPU 290 |
| | COMMON/NUMBER/ASP,NNS,NE,NC | INPU 300 |
| | COMMON/SP1/SS,TOL,NDEBUG | INPU 310 |
| C | | INPU 320 |
| | REAL MU,MUDZ | INPU 330 |
| | LOGICAL MCONV,ECONV,DCONV | INPU 340 |
| | DATA END /'END '/ | INPU 350 |

| | | |
|-----|---|----------|
| C | | INPU 360 |
| C | ** INPUT FORMATS ** | INPU 370 |
| C | | INPU 380 |
| 100 | FORMAT (18A4,I8) | INPU 390 |
| 101 | FORMAT (9I5,2E12.0,2X,I1) | INPU 400 |
| 102 | FORMAT (6E12.0) | INPU 410 |
| 107 | FORMAT(I5, 5X,E10.4,I5) | INPU 420 |
| 108 | FORMAT(5E15.8) | INPU 430 |
| C | | INPU 440 |
| C | ** OUTPUT FORMATS ** | INPU 450 |
| C | | INPU 460 |
| 200 | FORMAT (1H1 , 18A4,I8 ////) | INPU 470 |
| 201 | FORMAT (12H0 INPUT DATA ///) | INPU 480 |
| 202 | FORMAT (9H0KEEP = I5 | INPU 490 |
| 1 | / 9H NETA = I5 | INPU 500 |
| 2 | / 9H MAXM = I5 | INPU 510 |
| 3 | / 9H MAXE = I5 | INPU 520 |
| 4 | / 9H MAXD = I5 | INPU 530 |
| 5 | / 9H FPRCT = 1PE15.6 | INPU 540 |
| 6 | / 9H TPRCT = E15.6 | INPU 550 |
| 7 | / 9H LT = I5 | INPU 560 |
| 8 | / 9H IDEBUG = I5 | INPU 570 |
| 9 | / 9H IPhi = I5) | INPU 580 |
| 203 | FORMAT(1X/' ** FINITE-RATE CHEMISTRY **'//) | INPU 590 |
| 204 | FORMAT (9H0UINF = 1PE15.6 | INPU 600 |
| 1 | / 9H RINF = E15.6 | INPU 610 |
| 2 | / 9H R = E15.6 | INPU 620 |
| 3 | / 9H TW = E15.6 | INPU 630 |
| 4 | / 9H HTOTAL = E15.6 | INPU 640 |
| 5 | / 9H RVW = E15.6 | INPU 650 |
| 6 | / 9H PDTIL = E15.6 //) | INPU 660 |
| 205 | FORMAT (9H0NDEBUG = I5 | INPU 670 |
| 1 | / 9H TOL = F5.3//) | INPU 680 |
| C | | INPU 690 |
| 206 | FORMAT (20H0INITIAL T PROFILE / (1H , 12F10.5)) | INPU 700 |

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|-----|--|----------|
| 207 | FORMAT (20H0INITIAL RHO PROFILE/ (1H , 12F10.5)) | INPU 710 |
| 208 | FORMAT (20H0INITIAL RM PROFILE/ (1H , 12F10.5)) | INPU 720 |
| 209 | FORMAT (20H0ETA / (1H , 12F10.5)) | INPU 730 |
| 210 | FORMAT(32H * CONVECTIVE CALCULATION ONLY *) | INPU 740 |
| 211 | FORMAT(36H * UNCOUPLED RADIATION CALCULATION *) | INPU 750 |
| 212 | FORMAT(34H * COUPLED RADIATION CALCULATION *) | INPU 760 |
| 213 | FORMAT(36H * CONTINUUM AND LINE CALCULATION *) | INPU 770 |
| 214 | FORMAT(19H * EMISSION MODEL *) | INPU 780 |
| 215 | FORMAT(1X/' ** EQUILIBRIUM CHEMISTRY **'//) | INPU 790 |
| 216 | FORMAT (16H SPECIES INPUTS | INPU 800 |
| 1 | /16H NO. ELEMENTS = 15 | INPU 810 |
| 2 | /25X,5(15,2X,A4)) | INPU 820 |
| 218 | FORMAT (16H NO. SPECIES = 15) | INPU 830 |
| 220 | FORMAT (25X,5(15,2X,A4)) | INPU 840 |
| 222 | FORMAT (16H NO. SOLIDS = 15) | INPU 850 |
| C | | INPU 860 |
| C | CARD 1 ----- | INPU 870 |
| | READ (5,100) TITLE,IEM | INPU 880 |
| | IF (TITLE (1) .EQ. END) STOP | INPU 890 |
| | IF(IEM)6999,5999,6999 | INPU 900 |
| | 5999 CONTINUE | INPU 910 |
| C | | INPU 920 |
| C | **** START FROM SCRATCH **** | INPU 930 |
| C | | INPU 940 |
| C | | INPU 950 |
| C | ** INPUT OPTION PARAMETERS ** | INPU 960 |
| C | | INPU 970 |
| C | ** IRAD = 1 NO RADIATION CALCULATED | INPU 980 |
| C | IRAD = 2 UNCOUPLED SOLUTION | INPU 990 |
| C | IRAD = 3 COUPLED SOLUTION ** | INPU1000 |
| C | ** ITYPE=0 SPECTRAL MODEL WITH LINES | INPU1010 |
| C | ITYPE=1 EMISSION MODEL | INPU1020 |
| C | | INPU1030 |
| | KETA = NETA | INPU1040 |
| C | CARD 2 ----- | INPU1050 |

| | |
|--|----------|
| READ (5,101)KEEP,NETA,IRAD,ITYPE,MAXM,MAXE,MAXD,LT,IPHI, | INPU1060 |
| 1 FPRCT,TPRCT,IDEBUG | INPU1070 |
| META = NETA | INPU1080 |
| IF(KETA .EQ. 0) KEEP = 0 | INPU1090 |
| IF(KEEP. GT. 0) NETA = KETA | INPU1100 |
| C | INPU1110 |
| HDAMP = 0.6 | INPU1120 |
| TDAMP = 0.06 | INPU1130 |
| DDAMP = 0.5 | INPU1140 |
| IF(MAXM .EQ. 0) MAXM=15 | INPU1150 |
| IF(MAXE.EQ. 0) MAXE=5 | INPU1160 |
| IF(MAXD.EQ. 0) MAXD=15 | INPU1170 |
| IF(FPRCT.EQ. 0.0) FPRCT=.005 | INPU1180 |
| IF(TPRCT .EQ. 0.0) TPRCT=0.005 | INPU1190 |
| IF (NETA .EQ. 0) NETA = 51 | INPU1200 |
| IF (IRAD.EQ.0) IRAD =1 | INPU1210 |
| C | INPU1220 |
| C | INPU1230 |
| C ** FREE-STREAM FLIGHT CONDITIONS ** | INPU1240 |
| C CARD 3 ----- | INPU1250 |
| READ(5,102) U INF,R INF,R,TWK,HTOTAL,RVW | INPU1260 |
| UINF2=UINF**2 | INPU1270 |
| IF(KEEP.GT. 0) TWOLD = T(1) | INPU1280 |
| T(1)=TWK | INPU1290 |
| C | INPU1300 |
| IF(HTOTAL .EQ. 0.0) HTOTAL=UINF2/2.0 | INPU1310 |
| C | INPU1320 |
| C ** INITIAL SHOCK QUANTITY ESTIMATES ** | INPU1330 |
| C CARD 4 ----- | INPU1340 |
| READ(5,102) DELTA,DTIL,RZB,RE,PDTIL | INPU1350 |
| IF(PDTIL.EQ.0.0) PDTIL = .001 | INPU1360 |
| C | INPU1370 |
| C ** INPUT INITIAL TEMPERATURE PROFILE ** | INPU1380 |
| C CARD 5 ----- | INPU1390 |
| IF(LT .EQ. 0) GO TO 2800 | INPU1400 |

| | |
|---|----------|
| READ (5,102) (T (I) , I = 1 , NETA) | INPU1410 |
| 2800 CONTINUE | INPU1420 |
| C ** INPUT RHO AND (RHO)(NU) PROFILES ** | INPU1430 |
| C CARD 6 ----- | INPU1440 |
| IF(LT,LT,2) GO TO 2900 | INPU1450 |
| READ(5,102) (RHO(I),I=1,NETA) | INPU1460 |
| READ(5,102) (RM (I),I=1,NETA) | INPU1470 |
| 2900 CONTINUE | INPU1480 |
| C | INPU1490 |
| C ** SHOCK SHAPE (DEPS/DXI) ** | INPU1500 |
| C | INPU1510 |
| IF (IPHI .NE. 0) GO TO 2550 | INPU1520 |
| DEPS = 0.0 | INPU1530 |
| GO TO 2570 | INPU1540 |
| 2550 CONTINUE | INPU1550 |
| C CARD 7 ----- | INPU1560 |
| READ(5,102) DEPS | INPU1570 |
| 2570 CONTINUE | INPU1580 |
| DPHI = 1. -DEPS | INPU1590 |
| C | INPU1600 |
| C | INPU1610 |
| IF (META .GT. 0) GO TO 1000 | INPU1620 |
| IF(KEEP .GT. 0) GO TO 1500 | INPU1630 |
| C | INPU1640 |
| C ** FIXED GRID SIZE CN ETA ** | INPU1650 |
| C | INPU1660 |
| DETA = 0.02 | INPU1670 |
| ETA (1) = 0.0 | INPU1680 |
| DO 500 I = 2 , 51 | INPU1690 |
| ETA (I) = ETA (I-1) + DETA | INPU1700 |
| 500 CONTINUE | INPU1710 |
| C | INPU1720 |
| GO TO 1500 | INPU1730 |
| C | INPU1740 |
| 1000 CONTINUE | INPU1750 |

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|---|--|----------|
| C | | INPU1760 |
| C | ** INPUT ETA POINTS ** | INPU1770 |
| C | CARD 8 ----- | INPU1780 |
| | READ (5,102) (ETA (1) , I = 1 , NETA) | INPU1790 |
| C | | INPU1800 |
| | 1500 CONTINUE | INPU1810 |
| C | | INPU1820 |
| C | -----READ SPECIES PARAMETER CARDS..... | INPU1830 |
| C | CARD 9 ----- | INPU1840 |
| | READ 107, NDBUG,TOL,IAB | INPU1850 |
| C | | INPU1860 |
| | IF(TOL.LE.0.0) TOL = .001 | INPU1870 |
| C | NDBUG=OPTIONAL OUTPUT VARIABLE | INPU1880 |
| C | NC = NUMBER OF GASEOUS COMPONENTS | INPU1890 |
| C | CARD 10 ----- | INPU1900 |
| | READ 108,(CWall(I),I=1,NSP) | INPU1910 |
| | GO TO 7999 | INPU1920 |
| | 6999 CONTINUE | INPU1930 |
| C | | INPU1940 |
| C | **** RE-START **** | INPU1950 |
| C | | INPU1960 |
| | READ(5,1801) FPRCT,TPRCT,DDAMP,TDAMP,PDTIL,HDAMP | INPU1970 |
| | READ(5,1801) DELTA,DTIL,DTILS | INPU1980 |
| | READ(5,1802) U INF, RINF, UINF2, R , RE, ITM, IEM, NETA | INPU1990 |
| | READ(5,1803) KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | INPU2000 |
| | READ(5,1801)RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | INPU2010 |
| | READ(5,1804) NSP,NNS,NE,NC | INPU2020 |
| | READ(5,1801) (PI(J),RHO(J),T(J),J=1,NETA) | INPU2030 |
| | READ (5,1901)(AMW(J),J=1,NETA) | INPU2040 |
| | READ(5,1801) ((C(I,J),J=1,NETA),I=1,NSP) | INPU2050 |
| | READ(5,1801)(HM(J),J=1,NETA) | INPU2060 |
| | READ(5,1801) (E(J),J=1,NETA) | INPU2070 |
| | READ(5,1804) IRAD,ITYPE | INPU2080 |
| | READ(5,1801)DUD,DPHI,TD,RZB,PD,HD,HTOTAL | INPU2090 |
| | READ (5,1801)(F(J),FC(J),Z(J),V(J),J=1,NETA) | INPU2100 |

| | |
|---|----------|
| READ(5,1801)RVW,PRW,TWOLD | INPU2110 |
| READ(5,1801) (CWALL(I),I=1,NSP) | INPU2120 |
| READ(5,1801) (ECWALL(K),K=1,NE) | INPU2130 |
| READ(5,1801) (ETA(J),YOND(J),J=1,NETA) | INPU2140 |
| READ(5,715)NDEBUG,IAB,TOL | INPU2150 |
| READ (5,1801)((CC(K,J),J=1,NETA),K=1,NE) | INPU2160 |
| 1801 FORMAT(6E13.5) | INPU2170 |
| 1802 FORMAT(5E12.5,3I3) | INPU2180 |
| 1803 FORMAT(5I5,3L3,2I5) | INPU2190 |
| 1804 FORMAT(4I4) | INPU2200 |
| 715 FORMAT(2I5,E15.6) | INPU2210 |
| C | INPU2220 |
| 7999 CONTINUE | INPU2230 |
| C | INPU2240 |
| C **** PRINT DATA **** | INPU2250 |
| C | INPU2260 |
| WRITE (6,200) TITLE,IEM | INPU2270 |
| WRITE (6,201) | INPU2280 |
| WRITE (6,202)KEEP,NETA,MAXM,MAXE,MAXD,FPRCT,TPRCT,LT,IDEBUG | INPU2290 |
| 1 ,IPHI | INPU2300 |
| WRITE(6,204) U INF,R INF,R,TWK,HTOTAL,RVW,PDTIL | INPU2310 |
| IF (IRAD.EQ.1) WRITE (6,210) | INPU2320 |
| IF (IRAD.EQ.2) WRITE (6,211) | INPU2330 |
| IF (IRAD.EQ.3) WRITE (6,212) | INPU2340 |
| IF(IRAD.EQ.1) GO TO 300 | INPU2350 |
| IF(ITYPE.EQ.0) WRITE(6,213) | INPU2360 |
| IF(ITYPE.EQ.1) WRITE(6,214) | INPU2370 |
| 300 CONTINUE | INPU2380 |
| WRITE (6,206) (T (I) , I = 1 , NETA) | INPU2390 |
| IF(IFM.EQ.0)T(1) = TWK | INPU2400 |
| WRITE(6,207)(RHO(I),I=1,NETA) | INPU2410 |
| WRITE(6,208)(RM (I),I=1,NETA) | INPU2420 |
| WRITE(6,209)(ETA(1),I=1,NETA) | INPU2430 |
| WRITE(6,217) DEPS | INPU2440 |
| 217 FORMAT(9H0DEPS/DXI /(1H .12F10.5)) | INPU2450 |

| | |
|---|----------|
| IF(IAB.EQ.0)WRITE(6,203) | INPU2460 |
| IF(IAB.EQ.1)WRITE(6,215) | INPU2470 |
| WRITE(6,205)NDEBUG,TOL | INPU2480 |
| WRITE(6,216) NE,(I,EL(I),I=1,NE) | INPU2490 |
| WRITE(6,218) NSP | INPU2500 |
| JJ = 1 | INPU2510 |
| KK = JJ+4 | INPU2520 |
| 30 WRITE(6,220) (I,SP(I),I=JJ,KK) | INPU2530 |
| IF(KK+5.GT.NSP) GO TO 35 | INPU2540 |
| JJ = JJ+5 | INPU2550 |
| KK = JJ +4 | INPU2560 |
| GO TO 30 | INPU2570 |
| 35 KD = NSP -KK | INPU2580 |
| IF(KD.LE.0) GO TO 45 | INPU2590 |
| KK = KK +KD | INPU2600 |
| JJ = JJ +5 | INPU2610 |
| 40 GO TO 30 | INPU2620 |
| 45 CONTINUE | INPU2630 |
| WRITE(6,222) NNS | INPU2640 |
| PRINT 305 | INPU2650 |
| 305 FORMAT(/' SPECIES'/' NAME',9X,'SMW', | INPU2660 |
| 1'WALL MASS FRACTION'/) | INPU2670 |
| DO10I=1,NSP | INPU2680 |
| 10 PRINT302,SP(I),SMW(I),CWALL(I) | INPU2690 |
| 302 FORMAT(1X,A4,1F13.3,E12.4) | INPU2700 |
| IF(NDEBUG.EQ.0)GOTO9999 | INPU2710 |
| PRINT309 | INPU2720 |
| 309 FORMAT(/,' SPECIES',35X,'THERMO-CONSTANTS A-G',29X,'RANGE') | INPU2730 |
| DO11I=1,NSP | INPU2740 |
| PRINT303,SP(I),AII(I),BII(I),CII(I),DII(I),EII(I),FII(I),GII(I) | INPU2750 |
| 11 PRINT304, AI (I),BI (I),CI (I),DI (I),EI (I),FI (I),GI (I) | INPU2760 |
| 303 FORMAT(/,1X,A4,7E12.4,' LOW RANGE') | INPU2770 |
| 304 FORMAT(5X,7E12.4,' HIGH RANGE') | INPU2780 |
| PRINT 307 | INPU2790 |
| 307 FORMAT(/,25X,' AA(I,J) MATRIX',/) | INPU2800 |

```
      DO 12 J=1,NE
12    PRINT 306,(IA(I,J),I=1,NSP)
306  FORMAT(5X,20I5)
C
9999 CONTINUE
      RETURN
      END
```

```
INPU2810
INPU2820
INPU2830
INPU2840
INPU2850
INPU2860
INPU2870
```

| | | |
|---|--|----------|
| | SUBROUTINE INIT | INIT 10 |
| C | | INIT 20 |
| C | | INIT 30 |
| C | ** ROUTINE TO COMPUTE NECESSARY INITIAL QUANTITIES | INIT 40 |
| C | | INIT 50 |
| C | | INIT 60 |
| | COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDIL | INIT 70 |
| | COMMON /DEL/ DFLTA,DTIL,DTILS | INIT 80 |
| | COMMON/EQ2/AA(20,5),ICODE(20) | INIT 90 |
| | COMMON/EQ3/IA(20,5) | INIT 100 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | INIT 110 |
| | COMMON/GUESS/TG1(60),TG2(60) | INIT 120 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | INIT 130 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | INIT 140 |
| | COMMON/PROP1/PI(60),RHQ(60),T(60),AMW(60),C(20,60),EC(5,60) | INIT 150 |
| | COMMON/NUMBER/NSP,NNS,NE,NC | INIT 160 |
| | COMMON/PROP2/ MU(60),RM(60), AK(60) | INIT 170 |
| | COMMON/PROP3/CPS(20,60),HS(20,60),CP(60),HM(60) | INIT 180 |
| | COMMON/VECTOR/ CA(60),CB(60),CC(60),B(60) | INIT 190 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | INIT 200 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | INIT 210 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | INIT 220 |
| | COMMON/SP2/BR,S(20),CSHOCK(5) | INIT 230 |
| | COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | INIT 240 |
| | COMMON/WT/SMW(20),AWT(5) | INIT 250 |
| | COMMON /YL/ETA(60),YGND(60) | INIT 260 |
| | COMMON/DD/D(60) | INIT 270 |
| | COMMON/IT/AC | INIT 280 |
| | REAL MU,MUDZ | INIT 290 |
| C | | INIT 300 |
| | LOGICAL MCONV,ECONV,DCCNV | INIT 310 |
| C | | INIT 320 |
| | MCONV = .FALSE. | INIT 330 |
| | ECONV = .FALSE. | INIT 340 |
| | DCONV = .FALSE. | INIT 350 |

| | |
|--|----------|
| DO 900 I=1,60 | INIT 360 |
| DO 900 J=1,NSP | INIT 370 |
| 900 C(J,I) = 1.0E-20 | INIT 380 |
| C | INIT 390 |
| C | INIT 400 |
| C ** DETERMINE DENSITY RATIO , REYNOLDS NUMBER | INIT 410 |
| C FROM INPUTS OR RANKINE HUGENIOT EQS. ** | INIT 420 |
| C | INIT 430 |
| C GUESSED VALUES | INIT 440 |
| TD = 12000. + .5E-5*(HTOTAL -6.5E+8) | INIT 450 |
| RZB=.06 | INIT 460 |
| C | INIT 470 |
| T(ETA) = 1.0 | INIT 480 |
| HNF = 2.*778.28*32.172/UINF2 | INIT 490 |
| 998 CONTINUE | INIT 500 |
| PD = (1. -RZB)*RINF *UINF2/2116. | INIT 510 |
| HD = HTOTAL/(778.28*32.172) | INIT 520 |
| CPNF = 1.8*778.28*32.172*TD *2. /UINF2 | INIT 530 |
| AKNF = 1.8*778.28*TD *RZB/(R*RINF*UINF*UINF2) | INIT 540 |
| PI(ETA) = PD | INIT 550 |
| CALL GAS(ETA) | INIT 560 |
| RZB1=RINF/(RDZ*RHO(ETA)) | INIT 570 |
| TEST =ABS((RZB-RZB1)/RZB) | INIT 580 |
| IF(TEST .LT. 0.005) GO TO 999 | INIT 590 |
| RZB=.5*(RZB+RZB1) | INIT 600 |
| GO TO 998 | INIT 610 |
| 999 CONTINUE | INIT 620 |
| RE = RDZ*UINF*R*32.174 / MUDZ | INIT 630 |
| C | INIT 640 |
| C | INIT 650 |
| C ** GUESS AT DELTA TO START ** | INIT 660 |
| C | INIT 670 |
| IF(DELTA .EQ. 0.0) DELTA=0.78*RZB | INIT 680 |
| IF(DTIL .EQ. 0.0) DTIL=1.1*DELTA +1.2*RVW | INIT 690 |
| WRITE(6,200) RZB,RE | INIT 700 |

| | | |
|-----|---|----------|
| 200 | FORMAT(14H0DENSITY RATIO ,5X,12HREYNOLDS NO. /2E15.6) | INIT 710 |
| | WRITE(6,201) DELTA,DTIL | INIT 720 |
| 201 | FORMAT(6H0DELTA,13X,4HDTIL /2E15.6) | INIT 730 |
| C | | INIT 740 |
| 997 | CONTINUE | INIT 750 |
| | DO 995 I=1,NETA | INIT 760 |
| | PI(I) = PD | INIT 770 |
| | E(I) =0.0 | INIT 780 |
| 995 | CONTINUE | INIT 790 |
| C | ** RANKIN-HUGONIOT RELATIONS ** | INIT 800 |
| C | | INIT 810 |
| | VD = -RZB | INIT 820 |
| | TW = T(1) | INIT 830 |
| | T(1) = T(1)/TD | INIT 840 |
| C | | INIT 850 |
| C | ** STAGNATION POINT LIMIT QUANTITIES ** | INIT 860 |
| C | | INIT 870 |
| | DUD = DPHI + RZB*(1.-DPHI) | INIT 880 |
| C | NONDIMENSIONALIZING FACTORS | INIT 890 |
| | AKNF = 1.8*778.28*TD *RZB/(R*RINF*UINF*UINF2) | INIT 900 |
| | CPNF = 1.8*778.28*32.172*TD *2. /UINF2 | INIT 910 |
| C | | INIT 920 |
| C | GUESSED F AND Z PROFILES | INIT 930 |
| C | | INIT 940 |
| | IF(KEEP. GT. 0) GO TO 9 | INIT 950 |
| | N = NETA-2 | INIT 960 |
| | FD = RZB/(2.*DUD*DTIL) | INIT 970 |
| | FW = -RVW*FD | INIT 980 |
| | F(1) = FW | INIT 990 |
| | DO 2 K=2,NETA | INIT1000 |
| | F(K) = (FD-FW)*ETA(K) + FW | INIT1010 |
| 2 | CONTINUE | INIT1020 |
| | DO 3 I=1,N | INIT1030 |
| | Z(I) = ETA(I+1)/DTIL | INIT1040 |
| 3 | CONTINUE | INIT1050 |

| | |
|---|----------|
| C GUESSED T PROFILES | INIT1060 |
| IF(KEEP.GT.0)GOTO9 | INIT1070 |
| IF(LT.GT.0) GO TO 11 | INIT1080 |
| IF(RVW.GT.0.0)GOTO7 | INIT1090 |
| C NO BLOWING T PROFILE | INIT1100 |
| TWG1 = .1033 | INIT1110 |
| DO6K = 2,NETA | INIT1120 |
| TP = TG1(K) +(T(1)- TWG1) | INIT1130 |
| T(K) = TP -(T(1)- TWG1) * ETA(K) | INIT1140 |
| 6 CONTINUE | INIT1150 |
| GO TO 11 | INIT1160 |
| 7 CONTINUE | INIT1170 |
| TWG2 = .3325 | INIT1180 |
| C BLOWING T PROFILE | INIT1190 |
| DO8K = 2,NETA | INIT1200 |
| TP = TG2(K) +(T(1)- TWG2) | INIT1210 |
| T(K) = TP -(T(1)- TWG2) * ETA(K) | INIT1220 |
| 8 CONTINUE | INIT1230 |
| GO TO 11 | INIT1240 |
| 9 CONTINUE | INIT1250 |
| DO 10 K=2,NETA | INIT1260 |
| TP = T(K) +T(1) -TWOLD | INIT1270 |
| T(K) = TP -(T(1)-TWOLD)*ETA(K) | INIT1280 |
| WRITE(6,100) T(K),ETA(K) | INIT1290 |
| 10 CONTINUE | INIT1300 |
| 11 CONTINUE | INIT1310 |
| C | INIT1320 |
| C ** INITIALIZE SHOCK LAYER PARAMETERS FOR VARIABLE STEP SIZE | INIT1330 |
| DO 810 I=NETA,60 | INIT1340 |
| ETA(I)=1.0 | INIT1350 |
| T(I) = 1.0 | INIT1360 |
| E(I) = 0.0 | INIT1370 |
| PI(I) = PD | INIT1380 |
| MU(I)=1.0 | INIT1390 |
| CP(I) = CP(NETA) | INIT1400 |

| | |
|--|----------|
| AK(I) = AK(NETA) | INIT1410 |
| V(I) = VD | INIT1420 |
| F(I) = FD | INIT1430 |
| FC(I)=FD | INIT1440 |
| DO 810 J=1,NSP | INIT1450 |
| C(J,I) = C(J,NETA) | INIT1460 |
| HS(J,I)=1.0 | INIT1470 |
| 810 CONTINUE | INIT1480 |
| 1000 CONTINUE | INIT1490 |
| DO223J=1,60 | INIT1500 |
| DO223K=1,NE | INIT1510 |
| 223 EC(K,J) = 1.E-20 | INIT1520 |
| C | INIT1530 |
| DO221I=1,NSP | INIT1540 |
| C(I,1) = CWALL(I) | INIT1550 |
| 221 CONTINUE | INIT1560 |
| C-----CALCULATE AMW(N) | INIT1570 |
| C | INIT1580 |
| WAMW = 0.0 | INIT1590 |
| DO 25 J=1,NSP | INIT1600 |
| 25. WAMW = WAMW +CWALL(J)/SMW(J) | INIT1610 |
| WAMW = 1./WAMW | INIT1620 |
| 26 AMW(1)= WAMW | INIT1630 |
| C | INIT1640 |
| C | INIT1650 |
| C-----CONVERT WALL AND SHOCK COMPOSITIONS TO AN ELEMENTAL BASIS..... | INIT1660 |
| C | INIT1670 |
| DO331J=1,NE | INIT1680 |
| EC(J,1)=0.0 | INIT1690 |
| EC(J,NETA)=0.0 | INIT1700 |
| DO331I=1,NSP | INIT1710 |
| FAC=AA(I,J)*AWT(J)/SMW(I) | INIT1720 |
| FC(J,1)=EC(J,1) + FAC*C(I,1) | INIT1730 |
| 33 EC(J,NETA)=EC(J,NETA) + FAC*C(I,NETA) | INIT1740 |
| 331 ECWALL(J)=EC(J,1) | INIT1750 |

| | | |
|------|---|----------|
| C | | INIT1760 |
| | DO34N=NETA,60 | INIT1770 |
| | DO34J=1,NE | INIT1780 |
| 34 | EC(J,N)=EC(J,NETA) | INIT1790 |
| C | | INIT1800 |
| C | **** 'COMPUTE MUDZ AND RMDZ' **** | INIT1810 |
| | AC = 8.129E-08*(TD**1.659)/(PI(1)*R*UINF) | INIT1820 |
| | DO1623J=1,NETA | INIT1830 |
| 1623 | D(J) = AC*(T(J)**1.659) | INIT1840 |
| | CALL ELRAT | INIT1850 |
| | CALL CHEMEQ (1,NETA) | INIT1860 |
| | MUDZ = 1.0 | INIT1870 |
| | CALL PROPRT (NSP,NETA,NETA) | INIT1880 |
| | MUDZ = MU(NETA) | INIT1890 |
| | RMDZ = RM(NETA) | INIT1900 |
| | CALL PROPRT (NSP,1,NETA) | INIT1910 |
| | DTILS = .01 | INIT1920 |
| | IF(IDERUG .EQ. 0) RETURN | INIT1930 |
| | WRITE(6,4000) VD,DUD,PD | INIT1940 |
| | WRITE(6,4000) DELTA,DTIL,RZB,RE | INIT1950 |
| 4000 | FORMAT(1H0,6E15.6) | INIT1960 |
| 273 | FORMAT(6E12.0) | INIT1970 |
| 100 | FORMAT(1X,9E14.6) | INIT1980 |
| | RETURN | INIT1990 |
| | END | INIT2000 |

| | |
|--|----------|
| SUBROUTINE MOMTM | MOMT 10 |
| C | MOMT 20 |
| C -----THIS SUBROUTINE SOLVES THE MOMENTUM EQUATION AS A | MOMT 30 |
| C SECOND ORDER EQUATION AND A FIRST ORDER EQUATION ----- | MOMT 40 |
| COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDTIL | MOMT 50 |
| COMMON /DEL/ DELTA,DTIL,DTILS | MOMT 60 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | MOMT 70 |
| COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | MOMT 80 |
| COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | MOMT 90 |
| COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),EC(5,60) | MOMT 100 |
| COMMON/PROP2/ MU(60),RM(60), AK(60) | MOMT 110 |
| COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | MOMT 120 |
| COMMON /RFLUX/ E(60),IRAD,ITYPE | MOMT 130 |
| COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTCTAL | MOMT 140 |
| COMMON/VECTOR/ CA(60),CB(60),CC(60),B(60) | MOMT 150 |
| COMMON /VEL/ F(60),FC(60),Z(60),V(60) | MOMT 160 |
| COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWall(20),ECWall(5) | MOMT 170 |
| COMMON /YL/ETA(60),YOND(60) | MOMT 180 |
| LOGICAL MCONV,ECONV,DCCNV | MOMT 190 |
| C | MOMT 200 |
| C----- INITIALIZED QUANTITIES ----- | MOMT 210 |
| C | MOMT 220 |
| MCONV = .FALSE. | MOMT 230 |
| DTILS = DTIL | MOMT 240 |
| ITM = 1 | MOMT 250 |
| N = NETA -2 | MOMT 260 |
| L = NETA-1 | MOMT 270 |
| AA3 = RZB*(1.-RZB)*DPHI**2/DUD | MOMT 280 |
| DTILS2 = DTILS | MOMT 290 |
| IF(IEM.GT.3) DTIL=.5*(DTIL+DTILS2) | MOMT 300 |
| C | MOMT 310 |
| C | MOMT 320 |
| C-----Z'''+A1*Z'+A2*Z=A3 | MOMT 330 |
| C COMPUTE A1,A2,A3 | MOMT 340 |
| 14 CONTINUE | MOMT 350 |

C----- BOUNDARY CONDITIONS -----

C

RED = RE*DTIL

RED2 = 2.*RED*DTIL*DUD

DTIL2 = DTIL*DTIL

FD = RZB/(2.*DUD*DTIL)

FW = -RVW*FD

F(1) = FW

B(L) = 1./DTIL

ITER = 1

15 CONTINUE

II = 1

DO 20 I=1,N

DET=ETA(I+1)-ETA(I)

DETN=ETA(I+2)-ETA(I+1)

D1 = DETN*(DETN+DET)

D2 = DETN*DET

D3 = DET*(DETN+DET)

RMP = DET*RM(I+2)/D1 + (DETN-DET)*RM(I+1)/D2 - DETN*RM(I)/D3

A1 = (RED2*F(I+1) + RMP)/RM(I+1)

A2 = -RED2*DTIL*Z(I)/RM(I+1)

A3 = -2.*RED*(AA3/(RHO(I+1)*RM(I+1))

1 +DTIL2 *DUD*Z(I)**2/(2.*RM(I+1)))

C

C-----CA*Z(N-1)+CB*Z(N)+CC*Z(N+1)=B

C

COMPUTE CA, CB, CC

CA(II) = (2.-A1*DETN)/D3

CB(I) = A1*(DETN-DET)/D2 - 2./D2 + A2

CC(I) = (2.+A1*DET)/D1

B(I) = A3

II = I

20 CONTINUE

B(N) = B(N) - CC(N)/DTIL

C

CALL TRID (N)

MQMT 360

MQMT 370

MQMT 380

MQMT 390

MQMT 400

MQMT 410

MQMT 420

MQMT 430

MQMT 440

MQMT 450

MQMT 460

MQMT 470

MQMT 480

MQMT 490

MQMT 500

MQMT 510

MQMT 520

MQMT 530

MQMT 540

MQMT 550

MQMT 560

MQMT 570

MQMT 580

MQMT 590

MQMT 600

MQMT 610

MQMT 620

MQMT 630

MQMT 640

MQMT 650

MQMT 660

MQMT 670

MQMT 680

MQMT 690

MQMT 700

| | | |
|--|--|----------|
| C | | MOMT 710 |
| C-----INTEGRATE FIRST ORDER EQUATION----- | | MOMT 720 |
| FC(1)=FW | | MOMT 730 |
| SUM=FW+ (B(1)+FW)*(ETA(2)-ETA(1))*DTIL/2. | | MOMT 740 |
| FC(2)=SUM | | MOMT 750 |
| DO 30 K=3,NETA | | MOMT 760 |
| SUM=SUM+DTIL*(B(K-1)+B(K-2))*(ETA(K) -ETA(K-1))/2. | | MOMT 770 |
| 30 FC(K) = SUM | | MOMT 780 |
| C | | MOMT 790 |
| C-----CHECK FOR CONVERGENCE | | MOMT 800 |
| C | | MOMT 810 |
| DO 40 K=2,NETA | | MOMT 820 |
| PRCT=ABS((FC(K)-F(K))/F(K)) | | MOMT 830 |
| IF (PRCT.GT.FPRCT) GO TO 50 | | MOMT 840 |
| 40 CONTINUE | | MOMT 850 |
| GO TO 90 | | MOMT 860 |
| 50 CONTINUE | | MOMT 870 |
| ITER=ITER+1 | | MOMT 880 |
| DO 60 K=1,NETA | | MOMT 890 |
| 60 F(K)=FC(K) | | MOMT 900 |
| DO 65 I=1,N | | MOMT 910 |
| 65 Z(I)=B(I) | | MOMT 920 |
| IF(ITER.GE.MAXM) GO TO 90 | | MOMT 930 |
| GO TO 15 | | MOMT 940 |
| 90 CONTINUE | | MOMT 950 |
| C | | MOMT 960 |
| C----- COMPUTE NEW DTIL ----- | | MOMT 970 |
| C | | MOMT 980 |
| DTILC = (FD-FW)*DTIL/(F(NETA)-FW) | | MOMT 990 |
| PRCT = ABS((DTIL-DTILC)/DTIL) | | MOMT1000 |
| IF(ITM.GT.MAXM) GO TO 160 | | MOMT1010 |
| ITM = ITM +1 | | MOMT1020 |
| IF(PRCT.LE.PDTIL) GO TO 150 | | MOMT1030 |
| DTIL = DTIL +DDAMP*(DTILC-DTIL) | | MOMT1040 |
| GO TO 14 | | MOMT1050 |

| | | |
|-----|---|----------|
| 150 | CONTINUE | MOMT1060 |
| | DTIL = DTIL+ DDAMP*(DTILC -DTIL) | MOMT1070 |
| | MCONV = .TRUE. | MOMT1080 |
| C | | MOMT1090 |
| C | CHECK MOMENTUM-ENERGY CONVERGENCE | MOMT1100 |
| | PRCT = ABS((DTIL-DTILS)/DTILS) | MOMT1110 |
| | IF(PRCT.LE.PDTIL) DCCNV = .TRUE. | MOMT1120 |
| 160 | CONTINUE | MOMT1130 |
| C | | MOMT1140 |
| | DO 170 K=1,NETA | MOMT1150 |
| 170 | V(K) = -FC(K)*DTIL*2./RHO(K) | MOMT1160 |
| C | DEBUG OUTPUT | MOMT1170 |
| | IF(IDEBUG.EQ. 0) RETURN | MOMT1180 |
| | WRITE(6,102) ITER,ITM | MOMT1190 |
| 102 | FORMAT(10X,2I3/) | MOMT1200 |
| | WRITE(6,100) DTIL,DTILC | MOMT1210 |
| | WRITE(6,101) | MOMT1220 |
| 101 | FORMAT(6X,'ETA',12X,'F',12X,'FC',12X,'RHO',12X,'RM',12X,'VS',12X, | MOMT1230 |
| | 1 'V') | MOMT1240 |
| | DO 120 K=1,NETA | MOMT1250 |
| | VS=-FC(K)*DTIL*UINF*2./RHO(K) | MOMT1260 |
| | WRITE(6,100) ETA(K),F(K),FC(K), RHO(K),RM(K),VS ,V(K) | MOMT1270 |
| 100 | FORMAT(1X,9E14.6) | MOMT1280 |
| 120 | CONTINUE | MOMT1290 |
| | WRITE(6,103) | MOMT1300 |
| 103 | FORMAT(6X,'ETA',13X,'Z',13X,'B',12X,2HF') | MOMT1310 |
| | DO 121 I=1,N | MOMT1320 |
| | U=B(I)*DTIL | MOMT1330 |
| | WRITE(6,100) ETA(I+1),Z(I),B(I),U | MOMT1340 |
| 121 | CONTINUE | MOMT1350 |
| | RETURN | MOMT1360 |
| | END | MOMT1370 |

| | | |
|---|--|----------|
| | SUBROUTINE OUTPUT(N) | OUTP 10 |
| C | | OUTP 20 |
| C | | OUTP 30 |
| C | ** ROUTINE TO PRINT SHOCK LAYER SOLUTION ** | OUTP 40 |
| C | | OUTP 50 |
| C | | OUTP 60 |
| | COMMON/ID/SP(20),EL(5) | OUTP 70 |
| | COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDTIL | OUTP 80 |
| | COMMON /DEL/ DELTA,DTIL,DTILS | OUTP 90 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | OUTP 100 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | OUTP 110 |
| | LOGICAL MCONV,ECONV,DCONV | OUTP 120 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | OUTP 130 |
| | COMMON/NUMBER/NSP,NNS,NE,NC | OUTP 140 |
| | COMMON/PROP1/PI(60),RHC(60), T(60),AMW(60),C (20,60),EC(5,60) | OUTP 150 |
| | COMMON/PROP2/ MU(60),RM(60), AK(60) | OUTP 160 |
| | COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | OUTP 170 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | OUTP 180 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | OUTP 190 |
| | COMMON /SFLUX/QRI(3) | OUTP 200 |
| | COMMON/VECTOR/ CA(60),CB(60),CC(60),B(60) | OUTP 210 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | OUTP 220 |
| | COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | OUTP 230 |
| | COMMON /YL/ETA(60),YOND(60) | OUTP 240 |
| | COMMON /DD/ D(60) | OUTP 250 |
| | COMMON/SP1/SS,TOL,NDEBUG | OUTP 260 |
| | DIMENSION BOUT(60),DQR(30) | OUTP 270 |
| | REAL MU,MUDZ | OUTP 280 |
| | DATA HEAD1/'WALL'/' ,HEAD2/' '/,HEAD3/'SHOC'/' | OUTP 290 |
| C | | OUTP 300 |
| C | ** COMPUTE RADIATION FLUX IF UNCOUPLED PROBLEM ** | OUTP 310 |
| C | | OUTP 320 |
| | IF(ITYPE,NE,0)GOTO20 | OUTP 330 |
| | IF(IRAD .EQ. 2 .AND. N .NE. 2) CALL TRANS(1) | OUTP 340 |
| | IF (IRAD,NE,1) CALL TRANS2 | OUTP 350 |

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| 20 | IF(IRAD.EQ.2.AND.ITYPE.EQ.1) CALL EFLUX | OUTP 360 |
| | WRITE(6,203) | OUTP 370 |
| 203 | FORMAT(1H1) | OUTP 380 |
| C | | OUTP 390 |
| C | ** COMPUTE Y COORDINATE ** | OUTP 400 |
| C | | OUTP 410 |
| | YOND(1) = 0.0 | OUTP 420 |
| | SUM = 0.0 | OUTP 430 |
| | DO 40 K=2,NETA | OUTP 440 |
| | DETA= ETA(K)-ETA(K-1) | OUTP 450 |
| | SUM= SUM +DETA*(1./RHO(K)+1./RHO(K-1))/2. | OUTP 460 |
| | YOND(K) = DTIL*SUM | OUTP 470 |
| 40 | CONTINUE | OUTP 480 |
| | DELTA = YOND(NETA) | OUTP 490 |
| | DO 50 K=1,NETA | OUTP 500 |
| | YOND(K)= YOND(K)/DELTA | OUTP 510 |
| 50 | CONTINUE | OUTP 520 |
| C | | OUTP 530 |
| C | ** COMPUTE CONVECTIVE HEATING RATE ** | OUTP 540 |
| C | | OUTP 550 |
| C | WATTS/CM**2 | OUTP 560 |
| | QC = -AK(1)*RINF*UINF*UINF2* (T(2)-T(1))/ | OUTP 570 |
| | 1 (.88*778.28 *YCND(2)*DELTA*RZB) | OUTP 580 |
| C | BTU/FT**2-SEC | OUTP 590 |
| | QCP=QC*.88 | OUTP 600 |
| C | | OUTP 610 |
| C | ** COMPUTE RADIATIVE FLUX TO SURFACE ** | OUTP 620 |
| C | | OUTP 630 |
| | QR = 0.0 | OUTP 640 |
| | IF(IRAD .EQ. 1) GO TO 445 | OUTP 650 |
| | DO 1100 K=2,NETA | OUTP 660 |
| | QR = QR + QUAD(YOND,E,K) | OUTP 670 |
| 1100 | CONTINUE | OUTP 680 |
| C | WATTS/CM**2 | OUTP 690 |
| | QR =-QR *RINF*UINF2*UINF *DELTA/(685.*RZB) | OUTP 700 |

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| IF(I TYPE.EQ.0)QR=-QRI(1) | OUTP 710 |
| 445 CONTINUE | OUTP 720 |
| C BTU/FT**2-SEC | OUTP 730 |
| QRP=QR*0.88 | OUTP 740 |
| C ** COMPUTE DIFFUSIVE FLOW TO SURFACE ** | OUTP 750 |
| C WATTS/CM**2 | OUTP 760 |
| QD = 0. | OUTP 770 |
| DO1768 I=1,NSP | OUTP 780 |
| 1768 QD = QD + HS(I,1)*(C(I,2) - C(I,1)) | OUTP 790 |
| QD = - RHO(1)*D(1)*QD/(2.*DELTA*YOND(2)) | OUTP 800 |
| QD = RINF*UINF*UINF2*QD/(.88*778.28) | OUTP 810 |
| C BTU/FT**2-SEC | OUTP 820 |
| QDP = .88*QD | OUTP 830 |
| QTOTAL=QC+QR+QD | OUTP 840 |
| QTOTP=QTOTAL*.88 | OUTP 850 |
| C | OUTP 860 |
| C ** DIMENSIONALIZE RHO,MU,P,AND E ** | OUTP 870 |
| C | OUTP 880 |
| DO 450 I = 1 , NETA | OUTP 890 |
| RHO(I)=RHO(I)*RDZ | OUTP 900 |
| MU (I)=MU(I)*MUDZ | OUTP 910 |
| RM (I)=RM(I)*RMDZ | OUTP 920 |
| AK(I) = AK(I)/AKNF | OUTP 930 |
| E(I) = E(I) * RINF * UINF2 * UINF / (20866.0 * R *RZB) | OUTP 940 |
| CP(I) = CP(I)/CPNF | OUTP 950 |
| 450 CONTINUE | OUTP 960 |
| C | OUTP 970 |
| GO TO (1,2,3,4) , N | OUTP 980 |
| C | OUTP 990 |
| 1 WRITE(6,201) IEM | OUTP1000 |
| 201 FORMAT(23H SOLUTION CONVERGED IN ,I3,11H ITERATIONS //) | OUTP1010 |
| GO TO 4 | OUTP1020 |
| C | OUTP1030 |
| 2 WRITE (6,202) IEM,MCONV,ECONV,DCONV | OUTP1040 |
| 202 FORMAT(1H0,37H INTERMEDIATE PRINT AT ITERATION NO. ,I4,10X, | OUTP1050 |

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| 1 | MCONV=' ,L4,5X,'ECONV=' ,L4,5X,'DCONV=' ,L4,/'/) | OUTP1060 |
| | GO TO 4 | OUTP1070 |
| C | | OUTP1080 |
| | 3 CONTINUE | OUTP1090 |
| C | | OUTP1100 |
| 4 | CONTINUE | OUTP1110 |
| C | | OUTP1120 |
| C | ** PRINT SHOCK QUANTITIES AND HEATING RATE ** | OUTP1130 |
| C | | OUTP1140 |
| | WRITE(6,204) DELTA,DTIL | OUTP1150 |
| 204 | FORMAT(1H0, 25X,9H DELTA = ,1PE14.6,10X,7HDTIL = , | OUTP1160 |
| 1 | E15.6) | OUTP1170 |
| C | | OUTP1180 |
| | WRITE (6,210) QC,QCP | OUTP1190 |
| 210 | FORMAT (1H0, 25X,5H QC = E15.6,2X,13H(WATTS/CM**2), | OUTP1200 |
| 1 | 2X,1H=,E15.6,2X,17H(BTU/FT**2 - SEC)) | OUTP1210 |
| | WRITE(6,212) RZB,QR,QRP | OUTP1220 |
| 212 | FORMAT (1H0,6H RB =,F9.4,10X,5H QR = E15.6,2X,13H(WATTS/CM**2), | OUTP1230 |
| 1 | 2X,1H=,E15.6,2X,17H(BTU/FT**2 - SEC)) | OUTP1240 |
| | WRITE(6,213) RVW,QD,QDP | OUTP1250 |
| 213 | FORMAT (1H0,5H RVW =,F9.4,10X,5H QD = E15.6,2X,13H(WATTS/CM**2), | OUTP1260 |
| 1 | 2X,1H=,E15.6,2X,17H(BTU/FT**2 - SEC)) | OUTP1270 |
| C | | OUTP1280 |
| | WRITE(6,215) QTOTAL,QTOTP | OUTP1290 |
| 215 | FORMAT(1H0,16HTOTAL HEATING = E15.6,2X,13H(WATTS/CM**2), | OUTP1300 |
| 1 | 2X,1H=,E15.6,2X,17H(BTU/FT**2 - SEC)) | OUTP1310 |
| C | | OUTP1320 |
| C | ** PRINT Y/D , F AND T PROFILES ** | OUTP1330 |
| C | | OUTP1340 |
| | WRITE(6,205) | OUTP1350 |
| 205 | FORMAT(1H0,7X, 4H ETA, 5X, 4HY/DZ, 8X, 2HF', 8X, 3H RV, 8X , | OUTP1360 |
| 1 | 4HT/TD, 4X ,13H E(WATTS/CM3),4X,2H V, 7X,12H V (FT/SEC) , | OUTP1370 |
| 2 | 5X,2H G,6X,12H H (STATIC) , //) | OUTP1380 |
| | FP = 0.0 | OUTP1390 |
| | NS=NSP | OUTP1400 |

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| C | | OUTP1410 |
| | DO 100 I=1,NETA | OUTP1420 |
| C | COMPUTE ENTHALPIES | OUTP1430 |
| | HSTAT = 0.0 | OUTP1440 |
| | DO 99 J=1,NS | OUTP1450 |
| 99 | HSTAT = HSTAT + HS(J,I)*C(J,I) | OUTP1460 |
| | G = HSTAT + V(I)**2 | OUTP1470 |
| C | | OUTP1480 |
| | HEAD=HEAD2 | OUTP1490 |
| | IF(I .EQ. 1) HEAD=HEAD1 | OUTP1500 |
| | IF (I .EQ. NETA) HEAD=HEAD3 | OUTP1510 |
| | YDZ = YOND(I) | OUTP1520 |
| | IF(I.EQ.NETA) FP =1.0 | OUTP1530 |
| | RV = -FC(I)*DTIL*2. | OUTP1540 |
| | VS = V(I) *UINF | OUTP1550 |
| | WRITE(6,208) HEAD,ETA(I),YDZ,FP,RV,T(I),E(I),V(I),VS,G,HSTAT | OUTP1560 |
| 208 | FORMAT(1H ,A4, F6.3,1P10E12.3) | OUTP1570 |
| | IF(I.LT.NETA-1) FP = Z(I)*DTIL | OUTP1580 |
| | 100 CONTINUE | OUTP1590 |
| C | | OUTP1600 |
| C | ** WRITE OUT SHOCK LAYER GAS PROPERTIES ** | OUTP1610 |
| C | | OUTP1620 |
| | WRITE(6,44) | OUTP1630 |
| 44 | FORMAT(1H1.48X,28H-SHOCK LAYER GAS PROPERTIES-) | OUTP1640 |
| C | | OUTP1650 |
| | WRITE(6,206) | OUTP1660 |
| 206 | FORMAT(1H0.3X,3HETA,8X,4H Y/D,12X,2HP ,12X,2H T,11X,3HRHO,11X,2HMU | OUTP1670 |
| 1 | , 12X, 3HRMU,11X,2H K) | OUTP1680 |
| C | | OUTP1690 |
| | WRITE(6,207) | OUTP1700 |
| 207 | FORMAT (1H ,27X,6H(ATM.),6X,13H (DEG.KEL.) ,12H(SLUGS/FT3) ,2X, | OUTP1710 |
| 1 | 28H(LBM/FT-SEC) (LBF2-SEC3/FT6) ,16H (BTU/FT-SEC-R) ,//) | OUTP1720 |
| C | | OUTP1730 |
| | DO 101 I=1,NETA | OUTP1740 |
| C | | OUTP1750 |

C-3

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| TS = T(I)*TD | OUTP1760 |
| WRITE(6,8)ETA(I),YOND(I),PI(I),TS ,RHO(I),MU(I),RM (I),AK(I) | OUTP1770 |
| C | OUTP1780 |
| 8 FORMAT(1H F7.4,1P8E14.4) | OUTP1790 |
| 9 FORMAT(1H F7.4,1P7E14.4) | OUTP1800 |
| C | OUTP1810 |
| 101 CONTINUE | OUTP1820 |
| C | OUTP1830 |
| C ** WRITE SPECIES MASS FRACTIONS ** | OUTP1840 |
| C | OUTP1850 |
| WRITE(6,230) | OUTP1860 |
| 230 FORMAT (1H1,48X,26H-SPECIES MASS FRACTIONS-) | OUTP1870 |
| WRITE(6,231) | OUTP1880 |
| 231 FORMAT (1H ,14X,3H O2,11X,2HN2,11X,3H O ,11X,3H N ,11X,3H O+ , | OUTP1890 |
| 1 11X,3H N+ ,11X,3H E- ,//) | OUTP1900 |
| DO 102 I=1,NETA | OUTP1910 |
| WRITE(6,8) ETA(I),C(1,I),C(2,I),C(3,I),C(4,I),C(5,I), | OUTP1920 |
| 1 C(6,I),C(7,I) | OUTP1930 |
| 102 CONTINUE | OUTP1940 |
| WRITE(6,230) | OUTP1950 |
| WRITE(6,233) (SP(I),I=8,15) | OUTP1960 |
| 233 FORMAT(2X,4H ETA,1X,8(10X,A4)//) | OUTP1970 |
| WRITE(6,8) (ETA(I), (C(J,I),J= 8,15),I=1,NETA) | OUTP1980 |
| WRITE(6,230) | OUTP1990 |
| WRITE(6,234) (SP(I),I=16,20) | OUTP2000 |
| 234 FORMAT(2X,4H ETA,7X,3H CP,5(11X,A4),8X,4H AMW,//) | OUTP2010 |
| WRITE(6,9) (ETA(I),GP(I),(C(J,I),J=16,20),AMW(I),I=1,NETA) | OUTP2020 |
| C NONDIMENSIONALIZE | OUTP2030 |
| DO1001I=1,NETA | OUTP2040 |
| RHO(I) = RHO(I)/RDZ | OUTP2050 |
| MU(I) = MU(I)/MUDZ | OUTP2060 |
| RM(I) = RM(I)/RMDZ | OUTP2070 |
| E(I)=((E(I)*R)/(RINF*UINF**3))*20866.0*RZB | OUTP2080 |
| CP(I)=CP(I)*CPNF | OUTP2090 |
| 1001 AK(I)=AK(I)*AKNF | OUTP2100 |

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217  FORMAT(6E12.5)
1000 CONTINUE
C
      RETURN
      END
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OUTP2110
OUTP2120
OUTP2130
OUTP2140
OUTP2150
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| SUBROUTINE PUNCH | PONC 10 |
| COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDTIL | PONC 20 |
| COMMON/CONVI/HDAMP | PONC 30 |
| COMMON /DEL/ DELTA,DTIL,DTILS | PONC 40 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | PONC 50 |
| COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | PONC 60 |
| COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CPNF | PONC 70 |
| COMMON/NUMBER/NSP,NNS,NE,NC | PONC 80 |
| COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),EC(5,60) | PONC 90 |
| COMMON/PROP2/ MU(60),RM(60), AK(60) | PONC 100 |
| COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | PONC 110 |
| COMMON /RFLUX/ E(60),IRAD,ITYPE | PONC 120 |
| COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTCTAL | PONC 130 |
| COMMON /VEL/ F(60),FC(60),Z(60),V(60) | PONC 140 |
| COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | PONC 150 |
| COMMON /YL/ETA(60),YOND(60) | PONC 160 |
| COMMON/TIT/TITLE(18) | PONC 170 |
| COMMON/SP1/SS,TOL,NDEBUG | PONC 180 |
| REAL MU,MUDZ | PONC 190 |
| **** PUNCH-OUT CARDS FOR RESTART **** | PONC 200 |
| WRITE(7,566)TITLE,IEM | PONC 210 |
| 566 FORMAT(18A4,I8) | PONC 220 |
| WRITE(7,1801) FPRCT,TPRCT,DDAMP,TDAMP,PDTIL,HDAMP | PONC 230 |
| WRITE(7,1801)DELTA,DTIL,DTILS | PONC 240 |
| WRITE(7,1802) U INF, RINF, UINF2, R , RE, ITM, IEM, NETA | PONC 250 |
| WRITE(7,1803)KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | PONC 260 |
| PUNCH 1801, RDZ,MUDZ,RMDZ,AKNF,HNF,CPNF | PONC 270 |
| WRITE(7,1804) NSP,NNS,NE,NC | PONC 280 |
| WRITE(7,1801)(PI(J),RHO(J),T(J),J=1,NETA) | PONC 290 |
| WRITE(7,1801)(AMW(J),J=1,NETA) | PONC 300 |
| WRITE(7,1801)((C(I,J),J=1,NETA),I=1,NSP) | PONC 310 |
| WRITE(7,1801)(HM(J),J=1,NETA) | PONC 320 |
| WRITE(7,1801) (E(J),J=1,NETA) | PONC 330 |
| | PONC 340 |
| | PONC 350 |

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| WRITE(7,1804)IRAD,ITYPE | PONC 360 |
| PUNCH 1801, DUD,DPHI,TD,RZB,PD,HD,HTOTAL | PONC 370 |
| WRITE(7,1801)(F(J),FC(J),Z(J),V(J),J=1,NETA) | PONC 380 |
| PUNCH 1801, RVW,PRW,TWOLD | PONC 390 |
| WRITE(7,1801)(CWALL(I),I=1,NSP) | PONC 400 |
| WRITE(7,1801)(ECWALL(K),K=1,NE) | PONC 410 |
| WRITE(7,1801)(ETA(J),YOND(J),J=1,NETA) | PONC 420 |
| WRITE(7,715)NDEBUG,IAB,TOL | PONC 430 |
| WRITE(7,1801)((EC(K,J),J=1,NETA),K=1,NE) | PONC 440 |
| 1801 FORMAT(6E13.5) | PONC 450 |
| 1802 FORMAT(5E12.5,3I3) | PONC 460 |
| 1803 FORMAT(5I5,3L3,2I5) | PONC 470 |
| 1804 FORMAT(4I4) | PONC 480 |
| 715 FORMAT(2I5,E15.6) | PONC 490 |
| RETURN | PONC 500 |
| END | PONC 510 |

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| C | SUBROUTINE GAS (KODE) | GAS | 10 |
| C | | GAS | 20 |
| C | ** THERMODYNAMIC AND TRANSPORT PROPERTIES OF AIR ** | GAS | 30 |
| C | ** REFERENCE NASA TR R-50 ** | GAS | 40 |
| C | | GAS | 50 |
| C | THE FOLLOWING PROPERTIES ARE CALCULATED | GAS | 60 |
| C | TEMPERATURE AT WHICH PROPERTIES ARE WANTED (T) IN DEG R | GAS | 70 |
| C | PRESSURE AT WHICH PROPERTIES ARE WANTED (P) IN LB/IN**2 | GAS | 80 |
| C | RATIO OF SPECIFIC HEATS (GAMMA) IN DIMENSIONLESS | GAS | 90 |
| C | SPECIFIC HEAT AT CONSTANT PRESSURE (CP) IN BTU/LB-DEG R | GAS | 100 |
| C | ABSOLUTE VISCOSITY (V) IN LB/FT-SEC | GAS | 110 |
| C | PRANDTL NUMBER (PR) IN DIMENSIONLESS | GAS | 120 |
| C | THERMAL CONDUCTIVITY (XK) IN BTU/FT-SEC-DEG R | GAS | 130 |
| C | PRESSURE (P) IN ATMOSPHERES | GAS | 140 |
| C | DENSITY (DEN) IN LB/FT**3 | GAS | 150 |
| C | ENTHALPY (H) IN BTU/LB | GAS | 160 |
| C | ENTROPY (S) IN BTU/LB-DEG R | GAS | 170 |
| C | COMPRESSIBILITY (Z) IN DIMENSIONLESS | GAS | 180 |
| C | SPEED OF SOUND (SOS) IN FT/SEC | GAS | 190 |
| C | SPECIFIC HEAT AT CONSTANT VOLUME (CV) IN BTU/LB-DEG R | GAS | 200 |
| C | ENTROPY (S) IN FT**2/SEC**2 | GAS | 210 |
| C | VELOCITY (VEL) IN FT/SEC | GAS | 220 |
| C | PRESSURE (P) IN LBS/FT**2 | GAS | 230 |
| C | MACH NUMBER (M) IN DIMENSIONLESS | GAS | 240 |
| C | | GAS | 250 |
| C | | GAS | 260 |
| C | NOMENCLATURE 1=OXYGEN MOLECULES, 2=NITROGEN MOLECULES, 3=OXYGEN ATMOS | GAS | 270 |
| C | 4=NITROGEN ATMOS, 5=OXYGEN IONS, 6=NITROGEN IONS | GAS | 280 |
| C | 7=ELECTRONS | GAS | 290 |
| C | | GAS | 300 |
| C | | GAS | 310 |
| C | COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | GAS | 320 |
| C | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CNPF | GAS | 330 |
| C | COMMON/PROP1/PI(60),RHO(60),TI(60),AMW(60),C (20,60),CC(5,60) | GAS | 340 |
| C | COMMON/PROP2/ MU(60),RM(60), AK(60) | GAS | 350 |

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| COMMON/PROP3/CP5(20,60),HS(20,60),CPT(60),HM(60) | GAS | 360 |
| COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | GAS | 370 |
| COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | GAS | 380 |
| REAL MU,MUDZ | GAS | 390 |
| LOGICAL MCONV,GCONV,SCONV | GAS | 400 |
| DATA GASC /49721.7/ | GAS | 410 |
| C | GAS | 420 |
| C | GAS | 430 |
| DO 2000 I=KODE,NETA | GAS | 440 |
| T = TI(I) * TD | GAS | 450 |
| P = PI(I) | GAS | 460 |
| C | GAS | 470 |
| C THE FOLLOWING PART OF PROGRAM USES PRESSURE IN ATMOSPHERES | GAS | 480 |
| C AND TEMPERATURE IN DEG K | GAS | 490 |
| C | GAS | 500 |
| ITER=0 | GAS | 510 |
| C | GAS | 520 |
| C ** TEMPERATURE - ENTHALPY ITERATION ** | GAS | 530 |
| C | GAS | 540 |
| 900 CONTINUE | GAS | 550 |
| ITER=ITER+1 | GAS | 560 |
| IF(T.LT.100.) T=100. | GAS | 570 |
| A1=11390./T | GAS | 580 |
| A2=18990./T | GAS | 590 |
| A3=2270./T | GAS | 600 |
| A4=3390./T | GAS | 610 |
| A5=228./T | GAS | 620 |
| A6=326./T | GAS | 630 |
| A7=22800./T | GAS | 640 |
| A8=48600./T | GAS | 650 |
| A9=27700./T | GAS | 660 |
| A10=41500./T | GAS | 670 |
| A11=38600./T | GAS | 680 |
| A12=58200./T | GAS | 690 |
| A13=70.6/T | GAS | 700 |

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| A14=188.9/T | GAS 710 |
| A15=22000./T | GAS 720 |
| A16=47000./T | GAS 730 |
| A17=67900./T | GAS 740 |
| A18=2270./ (4.*T) | GAS 750 |
| A19=TANH(A18) | GAS 760 |
| A20=3390./ (4.*T) | GAS 770 |
| A21=TANH(A20) | GAS 780 |
| TT=1./T | GAS 790 |
| TSQ=T**2 | GAS 800 |
| TSQRT=T**0.5 | GAS 810 |
| A22=112.2222/T | GAS 820 |
| A23=T/59000. | GAS 830 |
| A24=T/113200. | GAS 840 |
| A25=T/75400. | GAS 850 |
| AA1=EXP(-A1) | GAS 860 |
| AA2=EXP(-A2) | GAS 870 |
| AA3=EXP(A3) | GAS 880 |
| AA4=EXP(A4) | GAS 890 |
| AA5=EXP(-A5) | GAS 900 |
| AA6=EXP(-A6) | GAS 910 |
| AA7=EXP(-A7) | GAS 920 |
| AA8=EXP(-A8) | GAS 930 |
| AA9=EXP(-A9) | GAS 940 |
| AA10=EXP(-A10) | GAS 950 |
| AA11=EXP(-A11) | GAS 960 |
| AA12=EXP(-A12) | GAS 970 |
| AA13=EXP(-A13) | GAS 980 |
| AA14=EXP(-A14) | GAS 990 |
| AA15=EXP(-A15) | GAS 1000 |
| AA16=EXP(-A16) | GAS 1010 |
| AA17=EXP(-A17) | GAS 1020 |
| C CALCULATING ENERGIES PER COMPONENT OF GAS MIXTURE ABOVE | GAS 1030 |
| C REFERENCE ENERGIES. | GAS 1040 |
| E1=2.5+((2.*AA1*A1+AA2*A2)/(3.+2.*AA1+AA2))+(A3/(AA3-1.)) | GAS 1050 |

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| ET=2.5+ (A4/(AA4-1.)) | GAS 1060 |
| E3=1.5+((3.*AA5*AA5+AA6*AA6+5.*AA7*AA7+AA8*AA8)/(5.+3.*AA5+AA6+5.*AA7+AA8)) | GAS 1070 |
| E4=1.5+((10.* AA9*AA9+6.*AA10*AA10)/(4.+10.*AA9+6.*AA10)) | GAS 1080 |
| E5=1.5+((10.*AA11*AA11+6.*AA12*AA12)/(4.+10.*AA11+6.*AA12)) | GAS 1090 |
| E6=1.5+((3.*AA13*AA13+5.*AA14*AA14+5.*AA15*AA15+AA16*AA16+5.*AA17*AA17)/1/(1.+3.*AA13+5.*AA14+5.*AA15+AA16+5.*AA17)) | GAS 1100 |
| E7=1.5 | GAS 1110 |
| C TOTAL ENERGY PER COMPONENT OF GAS MIXTURE | GAS 1120 |
| EN1=E1 | GAS 1130 |
| EN2=ET | GAS 1140 |
| EN3=E3+29500./T | GAS 1150 |
| EN4=E4+56600./T | GAS 1160 |
| EN5=E5+187500./T | GAS 1170 |
| EN6=E6+225400./T | GAS 1180 |
| EN7=E7 | GAS 1190 |
| C LOGS OF PARTITION FUNCTIONS | GAS 1200 |
| TL1=ALOG(T)*3.5 | GAS 1210 |
| TL2=ALOG(T)*2.5 | GAS 1220 |
| EQ1=TL1+.11+ALOG((3.+2.*AA1+AA2)/(1.-(1.0/AA3))) | GAS 1230 |
| EQ2=TL1-.42-ALOG((1.-(1.0/AA4))) | GAS 1240 |
| EQ3=TL2+.5+ALOG((5.+3.*AA5+AA6+5.*AA7+AA8)) | GAS 1250 |
| EQ4=TL2+.3+ALOG((4.+10.*AA9+6.*AA10)) | GAS 1260 |
| EQ5=TL2+.5+ALOG((4.+10.*AA11+6.*AA12)) | GAS 1270 |
| EQ6=TL2+.3+ALOG((1.+3.*AA13+5.*AA14+5.*AA15+AA16+5.*AA17)) | GAS 1280 |
| EQ7=TL2-14.24 | GAS 1290 |
| C EQUILIBRIUM CONSTANS FOR CHEMICAL REACTIONS | GAS 1300 |
| EK1=-59000./T+2.*EQ3-EQ1 | GAS 1310 |
| EK2=-113200./T+2.*EQ4-EQ2 | GAS 1320 |
| EK3=-158000./T+EQ5+EQ7-EQ3 | GAS 1330 |
| EK4=-168800./T+EQ6+EQ7-EQ4 | GAS 1340 |
| CCC=-79.9 | GAS 1350 |
| IF(EK1,LE,CCC) EK1=-79.9 | GAS 1360 |
| IF(EK2,LE,CCC) EK2=-79.9 | GAS 1370 |
| IF(EK3,LE,CCC) EK3=-79.9 | GAS 1380 |
| | GAS 1390 |
| | GAS 1400 |

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| IF(EK4.LE.CCC) EK4=-79.9 | GAS 1410 |
| XK1=EXP(EK1) | GAS 1420 |
| XK2=EXP(EK2) | GAS 1430 |
| XK3=EXP(EK3) | GAS 1440 |
| XK4=EXP(EK4) | GAS 1450 |
| XK34=.2*XK3+.8*XK4 | GAS 1460 |
| EE1=(-0.8+(.64+.8*(1.+(4.*P)/XK1))**.5)/(2.*(1.+4.*P/XK1)) | GAS 1470 |
| EE2=(-0.4+(.16+3.84*(1.+(4.*P)/(XK2))**.5)/(2.*(1.+4.*P/XK2)) | GAS 1480 |
| EE3= 1./((1.+P/XK34)**.5) | GAS 1490 |
| IF(EE1.GT..19999)EE1 = .19999 | GAS 1500 |
| IF(EE2.GT..79999)EE2 = .79999 | GAS 1510 |
| IF(EE3.GT..99999)EE3 = .99999 | GAS 1520 |
| C COMPRESSIBILITY (Z) DIMENSIONLESS | GAS 1530 |
| Z=1.+EE1+EE2+2.*EE3 | GAS 1540 |
| C COMPONENT MOL FRACTIONS IN AIR | GAS 1550 |
| X1=(.2-EE1)/Z | GAS 1560 |
| X2=(.8-EE2)/Z | GAS 1570 |
| X3=(2.*EE1-.4*EE3)/Z | GAS 1580 |
| X4=(2.*EE2-1.6*EE3)/Z | GAS 1590 |
| X5= .4*EE3/Z | GAS 1600 |
| X6= 1.6*EE3/Z | GAS 1610 |
| X7= 2.*EE3/Z | GAS 1620 |
| IF(X1.LE.0.) X1=1.E-20 | GAS 1630 |
| IF(X2.LE.0.) X2=1.E-20 | GAS 1640 |
| IF(X3.LE.0.) X3=1.E-20 | GAS 1650 |
| IF(X4.LE.0.) X4=1.E-20 | GAS 1660 |
| IF(X5.LE.0.) X5=1.E-20 | GAS 1670 |
| IF(X6.LE.0.) X6=1.E-20 | GAS 1680 |
| IF(X7.LE.0.) X7=1.E-20 | GAS 1690 |
| C ENERGY PER MOL OF INITIALLY UNDISSOCIATED AIR-DIMENSIONLESS | GAS 1700 |
| ER= Z*(X1*EN1+X2*EN2+X3*EN3+X4*EN4+X5*EN5+X6*EN6+X7*EN7) | GAS 1710 |
| C ENTHALPY PER INITIAL MOL OF AIR-DIMENSIONLESS | GAS 1720 |
| HR=ER+Z | GAS 1730 |
| C ENTHALPY PER INITIAL MOL OF AIR (H) IN BTU/LB | GAS 1740 |
| H=HR*T*.12348 | GAS 1750 |

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| IF(KODE.LT.NETA) GO TO 1000 | GAS 1760 |
| HRATO=.5*(H-HD)/H | GAS 1770 |
| AHR = ABS(HRATO) | GAS 1780 |
| IF(AHR .LE. 0.0010) GO TO 999 | GAS 1790 |
| IF(ITER .GT.1) GO TO 203 | GAS 1800 |
| TP=T | GAS 1810 |
| HP=HRATO | GAS 1820 |
| T = T *(1. - HRATO) | GAS 1830 |
| IF(ITER .LT. 15) GO TO 900 | GAS 1840 |
| 203 CONTINUE | GAS 1850 |
| TS=T*(1.0-HRATO) | GAS 1860 |
| IF(HRATO*HP .LT.0.0) TS=.5*(T+TP) | GAS 1870 |
| TP=T | GAS 1880 |
| T=TS | GAS 1890 |
| HP=HRATO | GAS 1900 |
| IF(ITER .LT. 15) GO TO 900 | GAS 1910 |
| WRITE(6,200) T,H,HT | GAS 1920 |
| 200 FORMAT(39H1TEMPERATURE-ENTHALPY DID NOT CONVERGE /3E15.6) | GAS 1930 |
| C CALL OUTPUT(4) | GAS 1940 |
| STOP | GAS 1950 |
| 999 CONTINUE | GAS 1960 |
| TD = T | GAS 1970 |
| C | GAS 1980 |
| 1000 CONTINUE | GAS 1990 |
| C ENTROPY PER INITIAL MOL OF AIR-DIMENSIONLESS | GAS 2000 |
| D1=E01+E1+1. | GAS 2010 |
| D2=E02+ET+1. | GAS 2020 |
| D3=E03+E3+1. | GAS 2030 |
| D4=E04+E4+1. | GAS 2040 |
| D5=E05+E5+1. | GAS 2050 |
| D6=E06+E6+1. | GAS 2060 |
| D7=E07+E7+1. | GAS 2070 |
| C TOTAL ENTROPY | GAS 2080 |
| SR=Z*(X1*D1+X2*D2+X3*D3+X4*D4+X5*D5+X6*D6+X7*D7)-Z*(X1*ALOG(X1) + | GAS 2090 |
| 1X2*ALOG(X2)+X3*ALOG(X3)+X4*ALOG(X4)+X5*ALOG(X5)+X6*ALOG(X6)+X7* | GAS 2100 |

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| 2A LOG(X7))-Z*A LOG(P) | GAS 2110 |
| C ENTROPY PER INITIAL MOL OF AIR (S) IN BTU/LB-DEG R | GAS 2120 |
| S =SR*0.0686 | GAS 2130 |
| C SPECIFIC HEAT AT CONSTANT VOLUME-CV | GAS 2140 |
| FF1=3.+2.*AA1+AA2 | GAS 2150 |
| CV1=2.5+(((2.*AA1*A1*A1+AA2*A2*A2)/FF1))-(((2.*AA1*A1+AA2*A2)/FF1)**GAS 2160 | |
| 12.))+((.25*A3*A3)/(((2.*A19)/(1.-A19*A19))**2)) | GAS 2170 |
| CV2=2.5+((.25*A4*A4)/(((2.*A21)/(1.-A21*A21))**2)) | GAS 2180 |
| CV3=1.5+((3.*AA5*A5*A5+AA6*A6*A6+5.*AA7*A7*A7+AA8*A8*A8)/(5.+3.*AAGAS 2190 | |
| 15+AA6+5.*AA7+AA8))-((E3-1.5)**2.) | GAS 2200 |
| CV4=1.5+((10.*AA9*A9*A9+6.*AA10*A10*A10)/(4.+10.*AA9+6.*AA10)) | GAS 2210 |
| 1-((E4-1.5)**2.) | GAS 2220 |
| CV5=1.5+((10.*AA11*A11*A11+6.*AA12*A12*A12)/(4.+10.*AA11+6.*AA12))GAS 2230 | |
| 1-((E5-1.5)**2.) | GAS 2240 |
| CV6=1.5+((3.*AA13*A13*A13+5.*AA14*A14*A14+5.*AA15*A15*A15+AA16*A16GAS 2250 | |
| 1*A16+5.*AA17*A17**2)/(1.+3.*AA13+5.*AA14+5.*AA15+AA16+5.*AA17))-((GAS 2260 | |
| 2E6-1.5)**2.) | GAS 2270 |
| CV7=1.5 | GAS 2280 |
| C LOGARITHMIC DERIVATIVES | GAS 2290 |
| CK1=TT*(59000./T+2.*E3-E1) | GAS 2300 |
| CK2=TT*(113200./T+2.*E4-E1) | GAS 2310 |
| CK3=TT*(158000./T+E5+E7-E3) | GAS 2320 |
| CK4=TT*(168800./T+E6+E7-E4) | GAS 2330 |
| CK34=.2*CK3+.8*CK4 | GAS 2340 |
| PK1= CK1+TT | GAS 2350 |
| PK2= CK2+TT | GAS 2360 |
| PK3= CK3+TT | GAS 2370 |
| PK4= CK4+TT | GAS 2380 |
| PK34=0.2*PK3+0.8*PK4 | GAS 2390 |
| C PARTIAL DERIVATIVES REQUIRED FOR CP | GAS 2400 |
| DE1P=(PK1*EE1*(1.+EE1)*(.2-EE1))/(.8*(.5-EE1)) | GAS 2410 |
| DE2P=(PK2*EE2*(1.2+EE2)*(.8-EE2))/(.4*(.8-EE2)) | GAS 2420 |
| DE3P=.5*PK34*EE3*(1.-EE3**2) | GAS 2430 |
| DZX1P=-DE1P | GAS 2440 |
| DZX2P=-DE2P | GAS 2450 |

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| DZX3P=2.*DE1P - .4*DE3P | GAS | 2460 |
| DZX4P=2.*DE2P-1.6*DE3P | GAS | 2470 |
| DZX5P=.4*DE3P | GAS | 2480 |
| DZX6P=1.6*DE3P | GAS | 2490 |
| DZX7P=2.* DE3P | GAS | 2500 |
| C EQUATION FOR SPECIFIC HEAT AT CONSTANT PRESSURE | GAS | 2510 |
| CPF=Z*(X1*(CV1+1.)+X2*(CV2+1.)+X3*(CV3+1.)+X4*(CV4+1.)+X5*(CV5+1. | GAS | 2520 |
| 1))+X6*(CV6+1.)+X7*(CV7+1.)) | GAS | 2530 |
| CPR = CPF + T*(DZX1P*(EN1+1.)+DZX2P*(EN2+1.)+DZX3P*(EGAS | 2540 | |
| 3N3+1.)+DZX4P*(EN4+1.)+DZX5P*(EN5+1.)+DZX6P*(EN6+1.)+DZX7P*(EN7+1.) | GAS | 2550 |
| 4) | GAS | 2560 |
| CPF = CPR | GAS | 2570 |
| C SPECIFIC HEAT AT CONSTANT PRESSURE (CP) IN BTU/LB-DEG R | GAS | 2580 |
| CP=CPR*.0686 | GAS | 2590 |
| CPF = CPF*.0686 | GAS | 2600 |
| C DENSITY (DEN) IN LB/FT**3 | GAS | 2610 |
| DEN=22.03703*P/(Z*T) | GAS | 2620 |
| C **TRANSPORT PROPERTIES** | GAS | 2630 |
| C COLLISION CROSS SECTIONS | GAS | 2640 |
| S2=31.4*1.E-16*(1.+(112./T)) | GAS | 2650 |
| SI2=(S2/3.1415927)**.5 | GAS | 2660 |
| SI4=(1.11676-(.01490* ALOG(1.-(1.-A23)**.5))-(.23654* ALOG | GAS | 2670 |
| 1(1.-(1.-A24)**.5))-(.11582* ALOG(1.-(1.-A25)**.5))) *1.0E-8 | GAS | 2680 |
| S4=3.1415927*(SI4)**2 | GAS | 2690 |
| SI24=(SI2+SI4)/2. | GAS | 2700 |
| S24=3.1415927*(SI24)**2 | GAS | 2710 |
| S47=9.40*1.0E-14/TSQRT | GAS | 2720 |
| FI=ALOG(1.042*1.0E-7*TSQ*(P*X7)**(-.5)) | GAS | 2730 |
| S7=8.55644*1.0E-6*(1./TSQ)*FI | GAS | 2740 |
| SIP4=(1.11676-(.0149*ALOG(1.-(1.-2.*A23)**.5))-(.23654*ALOG(1.-(1. | GAS | 2750 |
| 1-2.*A24)**.5))-(.11582* ALOG(1.-(1.-2.*A25)**.5))) *1.0E-8 | GAS | 2760 |
| SP4=3.145927*(SIP4)**2 | GAS | 2770 |
| SIP24=(SI2+SIP4)/2. | GAS | 2780 |
| SP24=3.145927*SIP24**2 | GAS | 2790 |
| C COMPONENT MOL FRACTIONS FOR INDEPENDENT REACTIONS | GAS | 2800 |

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| F1=1.+EE1 | GAS 2810 |
| F2=1.2+EE2 | GAS 2820 |
| F3=1.+EE3 | GAS 2830 |
| X10D=(.2-EE1)/F1 | GAS 2840 |
| X20D=.8/F1 | GAS 2850 |
| X30D=2.*EE1/F1 | GAS 2860 |
| X2ND=(.8-EE2)/F2 | GAS 2870 |
| X3ND=.4/F2 | GAS 2880 |
| X4ND=2.*EE2/F2 | GAS 2890 |
| X4I=(1.-EE3)/F3 | GAS 2900 |
| X6I=EE3/F3 | GAS 2910 |
| C MEAN FREE PATH RATIOS | GAS 2920 |
| SS1=S24/S2 | GAS 2930 |
| SS2=S4/S2 | GAS 2940 |
| SS3=S7/S2 | GAS 2950 |
| SS4=S47/S2 | GAS 2960 |
| FP10D=X10D+X20D*.9660918 +X30D*SS1*.8164966 | GAS 2970 |
| FP20D=X10D*1.032796+X20D+X30D*SS1*.8528029 | GAS 2980 |
| FP30D=X10D*1.154701*SS1+X2ND*SS1*1.128152+X30D*SS2 | GAS 2990 |
| FP2ND=X2ND+X4ND*SS1*.8164966+X3ND*SS1*.8528029 | GAS 3000 |
| FP3ND=X2ND*SS1*1.128152+X4ND*SS2*.9660918+X3ND*SS2 | GAS 3010 |
| FP4ND=X2ND*SS1*1.154701+X4ND*SS2+X3ND*SS2*1.032796 | GAS 3020 |
| FP4I=X4I*SS2+X6I*SS2 | GAS 3030 |
| FP6I=X4I*SS2+X6I*SS3 | GAS 3040 |
| FP7I=X4I*SS4*1.414186+X6I*SS3*1.414186+X6I*SS3 | GAS 3050 |
| C VISCOSTITES OF THE COMPONENTS FOR THE DIFFERENT REACTIONS | GAS 3060 |
| V10D=1.054093*X10D*1./FP10D | GAS 3070 |
| V20D=.9860133*X20D*1./FP20D | GAS 3080 |
| V30D=.745356*X30D*1./FP30D | GAS 3090 |
| V2ND=.9860133*X2ND*1./FP2ND | GAS 3100 |
| V3ND=.745356*X3ND*1./FP3ND | GAS 3110 |
| V4ND=.6972167*X4ND*1./FP4ND | GAS 3120 |
| V4I=.6972167*X4I*1./FP4I | GAS 3130 |
| V6I=.6972167*X6I*1./FP6I | GAS 3140 |
| V7I=.4367848*1.0E-2*X6I*1./FP7I | GAS 3150 |

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| VR0D=V10D+V20D+V30D | GAS 3160 |
| VRND=V2ND+V3ND+V4ND | GAS 3170 |
| VRI=V4I+V6I+V7I | GAS 3180 |
| F4=EE2/(.2-EE1+EE2) | GAS 3190 |
| F5=2.*EE3/(.8-EE2+2.*EE3) | GAS 3200 |
| VR=VR0D+(F4*(VRND-VR0D))+(F5*(VRI-VRND)) | GAS 3210 |
| C TOTAL VISCOSITY (V) IN LB/FT-SEC | GAS 3220 |
| V=VR*.9841838*1.0E-6*TSQRT/(1.+A22) | GAS 3230 |
| C CONDUCTIVITY DUE TO MOLECULAR COLLISIONS FOR DIFFERENT REACTIONS | GAS 3240 |
| G1=.2105263*CV1+.4736842 | GAS 3250 |
| G2=.2105263*CV2+.4736842 | GAS 3260 |
| G3=.2105263*CV3+.4736842 | GAS 3270 |
| G4=.2105363*CV4+.4736842 | GAS 3280 |
| G5=.2105363*CV6+.4736842 | GAS 3290 |
| G6=.2105363*CV7+.4736842 | GAS 3300 |
| XKN0D=(V10D*.9*G1)+(V20D*1.028571*G2)+(V30D*1.8*G3) | GAS 3310 |
| XKNND=(V2ND*1.028571*G2)+(V3ND*1.8*G3)+(V4ND*2.057143*G4) | GAS 3320 |
| XKNI=(V4I*2.057143*G4)+(V6I*2.057143*G5)+(V7I*52416.0*G6) | GAS 3330 |
| XKN=XKN0D+(F4*(XKNND-XKN0D))+(F5*(XKNI-XKNND)) | GAS 3340 |
| C CONDUCTIVITY DUE TO CHEMICAL REACTIONS FOR THE DIFFERENT REACTIONS | GAS 3350 |
| XKROD=(.178637*(T*PK1)**2)/((SP24/(1.732051*S2))*(((X30D+2.*X10D) | GAS 3360 |
| 1**2)/(X30D*X10D)+(4.*X20D/X30D))+(X20D/(1.414214*X10D))) | GAS 3370 |
| XKRND=(.178637*(T*PK2)**2)/((SP24/(1.732051*S2))*(((X4ND+2.*X2ND) | GAS 3380 |
| 1**2)/(X4ND*X2ND))+(X3ND/X2ND))+(SP4*2.*X3ND/(S2*X4ND))) | GAS 3390 |
| XKR1=(.178637*(T*PK34)**2)/(((.5*SP4/S2)+(4347826*1.0E-2*S47/S2))) | GAS 3400 |
| 1*(((X4I+X6I)**2)/(X4I*X6I))) | GAS 3410 |
| XKOD=XKN0D+XKROD | GAS 3420 |
| XKND=XKNND+XKRND | GAS 3430 |
| XKI=XKNI+XKR1 | GAS 3440 |
| XKR=XKOD+(F4*(XKND-XKOD))+(F5*(XKI-XKND)) | GAS 3450 |
| C TOTAL THERMAL CONDUCTIVITY (XK) IN BTU/FT-SEC-DEG R | GAS 3460 |
| XK=XKR*(.3206522*1.0E-6*TSQRT)/(1.+A22)) | GAS 3470 |
| C PRANDTL NUMBER (PR) DIMENSIONLESS | GAS 3480 |
| PRN = .2105263 * CPR * VR / XKR | GAS 3490 |
| IF(1.EQ. 1) PRW = PRN | GAS 3500 |

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| C FORM REQUIRED BY CALL STATEMENT | GAS 3510 |
| C | GAS 3520 |
| C ** RHO UNITS SLUGS/FT**3 | GAS 3530 |
| C ** MU UNITS LBM/FT-SEC | GAS 3540 |
| C ** RM UNITS LBF**2 SEC**3/FT**6 | GAS 3550 |
| C | GAS 3560 |
| MU (I) = V | GAS 3570 |
| RHO(I)=DEN/32.174 | GAS 3580 |
| RM(I)=RHO(I)*MU(I)/32.174 | GAS 3590 |
| AK(I) = XK | GAS 3600 |
| CPT(I) = CPF | GAS 3610 |
| C *** CALCULATE THE MEAN MOLECULAR WT. *** | GAS 3620 |
| REAL = 25050.*S *Z / SR | GAS 3630 |
| AMW(I)= GASC / REAL | GAS 3640 |
| C MASS FRACTIONS | GAS 3650 |
| C(1,I) = X1 *32.00/AMW(I) | GAS 3660 |
| C(2,I) = X2 *28.00/AMW(I) | GAS 3670 |
| C(3,I) = X3 *16.00/AMW(I) | GAS 3680 |
| C(4,I) = X4 *14.00/AMW(I) | GAS 3690 |
| C(6,I) = X6 *14.00/AMW(I) | GAS 3700 |
| C(5,I) = X5 *16.00/AMW(I) | GAS 3710 |
| C(7,I) = X7 /(1820.*AMW(I)) | GAS 3720 |
| C SPECIES ENTHALPY PER INITIAL MCLE OF AIR IN BTU/LB OF I | GAS 3730 |
| HS(1,I) = (Z*X1*EN1/C(1,I) +Z)*T*.12348 | GAS 3740 |
| HS(2,I) = (Z*X2*EN2/C(2,I) +Z)*T*.12348 | GAS 3750 |
| HS(3,I) = (Z*X3*EN3/C(3,I) +Z)*T*.12348 | GAS 3760 |
| HS(4,I) = (Z*X4*EN4/C(4,I) +Z)*T*.12348 | GAS 3770 |
| HS(5,I) = (Z*X5*EN5/C(5,I) +Z)*T*.12348 | GAS 3780 |
| HS(6,I) = (Z*X6*EN6/C(6,I) +Z)*T*.12348 | GAS 3790 |
| HS(7,I) = (Z*X7*EN7/C(7,I) +Z)*T*.12348 | GAS 3800 |
| 2000 CONTINUE | GAS 3810 |
| MUDZ= MU(NETA) | GAS 3820 |
| RDZ= RHO(NETA) | GAS 3830 |
| RMDZ= RM(NETA) | GAS 3840 |
| C | GAS 3850 |

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| C | | GAS 3860 |
| | DO 40 I=KODE,NETA | GAS 3870 |
| C | | GAS 3880 |
| C | ** NONDIMENSIONALIZE RHO AND MU ** | GAS 3890 |
| C | | GAS 3900 |
| | RHO(I) = RHO(I)/RDZ | GAS 3910 |
| | MU(I) = MU(I)/MUDZ | GAS 3920 |
| | RM(I) =RM(I)/RMDZ | GAS 3930 |
| | AK(I) = AK(I)*AKNF | GAS 3940 |
| | CPT(I) = CPT(I)*CPNF | GAS 3950 |
| C | NONDIMENSIONAL SPECIES ENTHALPY | GAS 3960 |
| | HS(1,I) = HS(1,I)*HNF | GAS 3970 |
| | HS(2,I) = HS(2,I)*HNF | GAS 3980 |
| | HS(3,I) = HS(3,I)*HNF | GAS 3990 |
| | HS(4,I) = HS(4,I)*HNF | GAS 4000 |
| | HS(5,I) = HS(5,I)*HNF | GAS 4010 |
| | HS(6,I) = HS(6,I)*HNF | GAS 4020 |
| | HS(7,I) = HS(7,I)*HNF | GAS 4030 |
| C | | GAS 4040 |
| C | | GAS 4050 |
| | 40 CONTINUE | GAS 4060 |
| | 100 FORMAT(1X,9E14.6) | GAS 4070 |
| | RETURN | GAS 4080 |
| | END | GAS 4090 |

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SUBROUTINE FG2 (Y,RHOA,TN,K)
C-----ORDER OF SPECIES ISO
C      CO      C      C2      C3      C+      C2H      CN
C      1      2      3      4      5      6      7
C      HCN      H2      H      N2      N      N+      O
C      8      9      10     11     12     13     14
C      O+      E-      C2H2     C3H     C4H
C      15     16     17     18     19
COMMON/FG2/ALL ,B ,C , T
DIMENSION Y(19),T1(5)
DIMENSION RHO(2),QK(16,2),PL(16,10), RATES(3,16),EQK(16)
DATA R/82.05/
DATA RATES/
1 .4352597E 02, .5000000E 00, .3572848E 05, .2683251E 02,
2 .5000000E 00, .1761263E 05, .4835429E 02, -.1500000E 01,
3 .1131738E 06, .4272747E 02, -.8200000E 00, .5193211E 05,
4 .2866064E 02, .5000000E 00, .1575073E 06, .2870602E 02,
5 .5000000E 00, .1675716E 06, .4684016E 02, -.1000000E 01,
6 .6632414E 05, .4357476E 02, .5000000E 00, .6038617E 05,
7 .4369782E 02, .5000000E 00, .7045053E 05, .4369782E 02,
8 .5000000E 00, .5887652E 05, .4588918E 02, -.1000000E 01,
9 .1297799E 06, .4374912E 02, .5000000E 00, .9561144E 05,
A .4367655E 02, .5000000E 00, .7799880E 05, .4368724E 02,
1 .5000000E 00, .1333528E 06, .4369782E 02, .5000000E 00,
2 .8303098E 05, .4369782E 02, .5000000E 00, .7296662E 05/
DIMENSION E1(95),E2(95),E3(34),E(16,7,2)
EQUIVALENCE (E(1,1,1),E1(1))
DATA E1/
1 0.3530E 00, 0.1816E 01, 0.1727E 01, 0.1642E 01, 0.2321E 01,
2 0.2753E 01, 0.1675E 01, 0.2257E 01, 0.3458E 01, 0.2094E 01,
3 0.2028E 01, 0.3053E 01, 0.7810E 00, 0.2497E 01, 0.2132E 01,
4 0.7030E 00,-0.7681E-03,-0.3571E-02,-0.8059E-03,-0.2810E-03,
5 0.2250E-03,-0.3726E-03,-0.6017E-03,-0.2955E-02,-0.3852E-02,
6-0.2155E-02,-0.1370E-02,-0.4032E-02,-0.1175E-03, 0.6404E-04,
7-0.2840E-02,-0.1406E-02, 0.5153E-06, 0.1634E-05, 0.1344E-06,

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| | | |
|---|-----|-----|
| 8-0.9244E-07,-0.9094E-07, 0.1884E-06,-0.6914E-07, 0.1359E-05, | FG2 | 360 |
| 9 0.1541E-05, 0.7206E-06, 0.4461E-06, 0.1731E-05,-0.8460E-07, | FG2 | 370 |
| A-0.5011E-07, 0.1138E-05, 0.5735E-06,-0.9618E-10,-0.2783E-09, | FG2 | 380 |
| 1 0.4980E-11, 0.2914E-10, 0.1275E-10,-0.3765E-10, 0.3996E-10, | FG2 | 390 |
| 2-0.2518E-09,-0.2492E-09,-0.1067E-09,-0.5622E-10,-0.2858E-09, | FG2 | 400 |
| 3 0.2854E-10, 0.6619E-11,-0.2020E-09,-0.8895E-10, 0.5427E-14, | FG2 | 410 |
| 4 0.1617E-13,-0.9780E-15,-0.2112E-14,-0.5056E-15, 0.2271E-14, | FG2 | 420 |
| 5-0.2991E-14, 0.1668E-13, 0.1406E-13, 0.5735E-14, 0.2436E-14, | FG2 | 430 |
| 6 0.1621E-13,-0.2197E-14,-0.2271E-15, 0.1335E-13, 0.4609E-14, | FG2 | 440 |
| 7 0.3485E 05, 0.1329E 05, 0.1132E 06, 0.5196E 05, 0.1580E 06, | FG2 | 450 |
| 8 0.1686E 06, 0.9406E 05, 0.5850E 05, 0.6525E 05, 0.5766E 05, | FG2 | 460 |
| 9 0.1289E 06, 0.8906E 05, 0.7297E 05, 0.1306E 06, 0.8068E 05/ | FG2 | 470 |
| EQUIVALENCE (E(16,6,1),E2(1)) | FG2 | 480 |
| DATA E2/ | FG2 | 490 |
| 1 0.7220E 05, 0.7500E-01,-0.8961E 01, 0.4274E 01, 0.2626E 01, | FG2 | 500 |
| 2-0.1181E 02,-0.1238E 02, 0.3698E 01, 0.1912E 01,-0.6335E 01, | FG2 | 510 |
| 3 0.3671E 01, 0.3773E 01, 0.1034E 01, 0.9378E 01,-0.1216E 02, | FG2 | 520 |
| 4 0.5461E 01, 0.1162E 02,-0.9100E-01,-0.1852E 01, 0.1765E 01, | FG2 | 530 |
| 5 0.4505E 01, 0.2904E 01, 0.2261E 01, 0.1414E 01, 0.3753E 01, | FG2 | 540 |
| 6 0.2653E 01, 0.2452E 01, 0.1323E 01,-0.1596E 02, 0.2560E 00, | FG2 | 550 |
| 7 0.2895E 01, 0.3483E 01, 0.2320E 00, 0.2624E-03, 0.1831E-02, | FG2 | 560 |
| 8-0.1257E-02,-0.4018E-02,-0.3576E-03, 0.3808E-03,-0.8027E-03, | FG2 | 570 |
| 9-0.4486E-02,-0.2187E-02,-0.2382E-02,-0.5403E-03, 0.1840E-01, | FG2 | 580 |
| A 0.1581E-03,-0.3234E-03,-0.4981E-02,-0.8811E-03,-0.8699E-09, | FG2 | 590 |
| 1-0.5850E-06, 0.3816E-06, 0.1254E-05, 0.6591E-07,-0.1210E-06, | FG2 | 600 |
| 2 0.1697E-06, 0.1768E-05, 0.6688E-06, 0.6930E-06, 0.1688E-06, | FG2 | 610 |
| 3-0.5690E-05,-0.3970E-07, 0.4932E-07, 0.2072E-05, 0.4090E-06, | FG2 | 620 |
| 4-0.5441E-11, 0.7641E-10,-0.3536E-10,-0.1592E-09,-0.3159E-11, | FG2 | 630 |
| 5 0.1070E-10,-0.1315E-10,-0.2902E-09,-0.8277E-10,-0.8428E-10, | FG2 | 640 |
| 6-0.1859E-10, 0.6841E-09, 0.2542E-11,-0.2579E-11,-0.3588E-09, | FG2 | 650 |
| 7-0.6840E-10, 0.2100E-15,-0.3417E-14, 0.1003E-14, 0.7013E-14, | FG2 | 660 |
| 8 0.2813E-16,-0.3032E-15, 0.3831E-15, 0.1768E-13, 0.3596E-14, | FG2 | 670 |
| 9 0.3630E-14, 0.5931E-15,-0.2842E-13, 0.2920E-17, 0.2381E-16/ | FG2 | 680 |
| EQUIVALENCE (E(15,5,2),E3(1)) | FG2 | 690 |
| DATA E3/ | FG2 | 700 |

| | | | |
|----|---|-----|------|
| 1 | 0.2234E-13, 0.3694E-14, 0.4160E 05, 0.1329E 05, 0.1132E 06, | FG2 | 710 |
| 2 | 0.5196E 05, 0.1580E 06, 0.1686E 06, 0.8731E 05, 0.6525E 05, | FG2 | 720 |
| 3 | 0.6525E 05, 0.5766E 05, 0.1289E 06, 0.8906E 05, 0.7297E 05, | FG2 | 730 |
| 4 | 0.1306E 06, 0.8068E 05, 0.7220E 05, 0.2066E 01, 0.1126E 02, | FG2 | 740 |
| 5 | 0.4450E 01, -0.1348E 02, -0.1538E 02, -0.1000E 02, 0.5594E 01, | FG2 | 750 |
| 6 | -0.6819E 01, -0.2220E 01, 0.1504E 01, 0.7660E 01, 0.1101E 03, | FG2 | 760 |
| 7 | 0.1266E 02, -0.1481E 02, -0.1881E 01, 0.1435E 02/ | FG2 | 770 |
| C | RHOA= DENSITY (LBM/FT3) | FG2 | 780 |
| | RHOB= .031085*RHOA | FG2 | 790 |
| | RHO(1)=.515362*RHOB | FG2 | 800 |
| | RHO(2)=RHO(1)*RHO(1) | FG2 | 810 |
| | X=0. | FG2 | 820 |
| | DO 6 I=1, 19 | FG2 | 830 |
| 6 | X=X+Y(I) | FG2 | 840 |
| | IF(ABS((TN-T)/TN)-.0001)20,20,10 | FG2 | 850 |
| 10 | T=TN | FG2 | 860 |
| | T1(1) = T | FG2 | 870 |
| | DO 12M=2,4 | FG2 | 880 |
| 12 | T1(M) = T1(1)*T1(M-1) | FG2 | 890 |
| | T1(5) = ALOG(T1(1)) | FG2 | 900 |
| | RT = R*T | FG2 | 910 |
| | L=1 | FG2 | 920 |
| | IF(T1(1).GT.6000.)L=2 | FG2 | 930 |
| | DO 18 I=1, 16 | FG2 | 940 |
| | QK(I,1)=T**RATES(2,I)* EXP(RATES(1,I)-RATES(3,I)/T) | FG2 | 950 |
| | QK(I,2) = E(I,1,L)*(1.-T1(5))-E(I,2,L)*T1(1)/2.-E(I,3,L)*T1(2)/6. | FG2 | 960 |
| | X-E(I,4,L)*T1(3)/12.-E(I,5,L)*T1(4)/20.+E(I,6,L)/T1(1)-E(I,7,L) | FG2 | 970 |
| | EQK(I) = EXP(-QK(I,2)) | FG2 | 980 |
| | QK(I,2) = QK(I,1)/ EQK(I) | FG2 | 990 |
| | IF(I.GT.2)QK(I,2) = RT*QK(I,2) | FG2 | 1000 |
| 18 | CONTINUE | FG2 | 1010 |
| 20 | PL(1, 1) = QK(1, 1)*RHO(1)*Y(12) | FG2 | 1020 |
| | IND = 0 | FG2 | 1030 |
| | PL(1, 2) = QK(1, 1)*RHO(1)*Y(1) | FG2 | 1040 |
| | PL(1, 6) = QK(1, 2)*RHO(1)*Y(14) | FG2 | 1050 |

| | |
|---|----------|
| PL(1, 7) = QK(1, 2)*RHO(1)*Y(7) | FG2 1060 |
| PL(2, 1) = QK(2, 1)*RHO(1)*Y(10) | FG2 1070 |
| PL(2, 2) = QK(2, 1)*RHO(1)*Y(6) | FG2 1080 |
| PL(2, 6) = QK(2, 2)*RHO(1)*Y(9) | FG2 1090 |
| PL(2, 7) = QK(2, 2)*RHO(1)*Y(3) | FG2 1100 |
| PL(3, 1) = QK(3, 1)*RHO(1)*X | FG2 1110 |
| PL(3, 6) = 2.*QK(3, 2)*RHO(2)*Y(12)*X | FG2 1120 |
| PL(3, 5) = QK(3, 1)*RHO(1)*Y(11) | FG2 1130 |
| PL(3, 10) = QK(3, 2)*RHO(2)*Y(12)*Y(12) | FG2 1140 |
| PL(4, 1) = QK(4, 1)*RHO(1)*X | FG2 1150 |
| PL(4, 6) = 2.*QK(4, 2)*RHO(2)*Y(10)*X | FG2 1160 |
| PL(4, 5) = QK(4, 1)*RHO(1)*Y(9) | FG2 1170 |
| PL(4, 10) = QK(4, 2)*RHO(2)*Y(10)*Y(10) | FG2 1180 |
| PL(5, 1) = QK(5, 1)*RHO(1)*X | FG2 1190 |
| PL(5, 6) = QK(5, 2)*RHO(2)*Y(16)*X | FG2 1200 |
| PL(5, 7) = QK(5, 2)*RHO(2)*Y(15)*X | FG2 1210 |
| PL(5, 5) = QK(5, 1)*RHO(1)*Y(14) | FG2 1220 |
| PL(5, 10) = QK(5, 2)*RHO(2)*Y(15)*Y(16) | FG2 1230 |
| PL(6, 1) = QK(6, 1)*RHO(1)*X | FG2 1240 |
| PL(6, 6) = QK(6, 2)*RHO(2)*Y(16)*X | FG2 1250 |
| PL(6, 7) = QK(6, 2)*RHO(2)*Y(13)*X | FG2 1260 |
| PL(6, 5) = QK(6, 1)*RHO(1)*Y(12) | FG2 1270 |
| PL(6, 10) = QK(6, 2)*RHO(2)*Y(13)*Y(16) | FG2 1280 |
| PL(7, 1) = QK(7, 1)*RHO(1)*X | FG2 1290 |
| PL(7, 6) = QK(7, 2)*RHO(2)*Y(12)*X | FG2 1300 |
| PL(7, 7) = QK(7, 2)*RHO(2)*Y(2)*X | FG2 1310 |
| PL(7, 5) = QK(7, 1)*RHO(1)*Y(7) | FG2 1320 |
| PL(7, 10) = QK(7, 2)*RHO(2)*Y(2)*Y(12) | FG2 1330 |
| PL(8, 1) = QK(8, 1)*RHO(1)*X | FG2 1340 |
| PL(8, 6) = QK(8, 2)*RHO(2)*Y(10)*X | FG2 1350 |
| PL(8, 7) = QK(8, 2)*RHO(2)*Y(7)*X | FG2 1360 |
| PL(8, 5) = QK(8, 1)*RHO(1)*Y(8) | FG2 1370 |
| PL(8, 10) = QK(8, 2)*RHO(2)*Y(7)*Y(10) | FG2 1380 |
| PL(9, 1) = QK(9, 1)*RHO(1)*X | FG2 1390 |
| PL(9, 6) = QK(9, 2)*RHO(2)*Y(10)*X | FG2 1400 |

| | |
|--|----------|
| PL(9, 7) = QK(9, 2)*RHO(2)*Y(3)*X | FG2 1410 |
| PL(9, 5) = QK(9, 1)*RHO(1)*Y(6) | FG2 1420 |
| PL(9, 10) = QK(9, 2)*RHO(2)*Y(3)*Y(10) | FG2 1430 |
| PL(10, 1) = QK(10, 1)*RHO(1)*X | FG2 1440 |
| PL(10, 6) = QK(10, 2)*RHO(2)*Y(10)*X | FG2 1450 |
| PL(10, 7) = QK(10, 2)*RHO(2)*Y(6)*X | FG2 1460 |
| PL(10, 5) = QK(10, 1)*RHO(1)*Y(17) | FG2 1470 |
| PL(10, 10) = QK(10, 2)*RHO(2)*Y(6)*Y(10) | FG2 1480 |
| PL(11, 1) = QK(11, 1)*RHO(1)*X | FG2 1490 |
| PL(11, 6) = QK(11, 2)*RHO(2)*Y(14)*X | FG2 1500 |
| PL(11, 7) = QK(11, 2)*RHO(2)*Y(2)*X | FG2 1510 |
| PL(11, 5) = QK(11, 1)*RHO(1)*Y(1) | FG2 1520 |
| PL(11, 10) = QK(11, 2)*RHO(2)*Y(2)*Y(14) | FG2 1530 |
| PL(12, 1) = QK(12, 1)*RHO(1)*X | FG2 1540 |
| PL(12, 6) = QK(12, 2)*RHO(2)*Y(2)*X | FG2 1550 |
| PL(12, 7) = QK(12, 2)*RHO(2)*Y(3)*X | FG2 1560 |
| PL(12, 5) = QK(12, 1)*RHO(1)*Y(4) | FG2 1570 |
| PL(12, 10) = QK(12, 2)*RHO(2)*Y(3)*Y(2) | FG2 1580 |
| PL(13, 1) = QK(13, 1)*RHO(1)*X | FG2 1590 |
| PL(13, 6) = 2.*QK(13, 2)*RHO(2)*Y(2)*X | FG2 1600 |
| PL(13, 5) = QK(13, 1)*RHO(1)*Y(3) | FG2 1610 |
| PL(13, 10) = QK(13, 2)*RHO(2)*Y(2)*Y(2) | FG2 1620 |
| PL(14, 1) = QK(14, 1)*RHO(1)*X | FG2 1630 |
| PL(14, 6) = QK(14, 2)*RHO(2)*Y(16)*X | FG2 1640 |
| PL(14, 7) = QK(14, 2)*RHO(2)*Y(5)*X | FG2 1650 |
| PL(14, 5) = QK(14, 1)*RHO(1)*Y(2) | FG2 1660 |
| PL(14, 10) = QK(14, 2)*RHO(2)*Y(5)*Y(16) | FG2 1670 |
| PL(15, 1) = QK(15, 1)*RHO(1)*X | FG2 1680 |
| PL(15, 6) = QK(15, 2)*RHO(2)*Y(2)*X | FG2 1690 |
| PL(15, 7) = QK(15, 2)*RHO(2)*Y(6)*X | FG2 1700 |
| PL(15, 5) = QK(15, 1)*RHO(1)*Y(18) | FG2 1710 |
| PL(15, 10) = QK(15, 2)*RHO(2)*Y(6)*Y(2) | FG2 1720 |
| PL(16, 1) = QK(16, 1)*RHO(1)*X | FG2 1730 |
| PL(16, 6) = QK(16, 2)*RHO(2)*Y(2)*X | FG2 1740 |
| PL(16, 7) = QK(16, 2)*RHO(2)*Y(18)*X | FG2 1750 |

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      PL( 16, 5) = QK( 16, 1)*RHO(1)*Y(19)                      FG2 1760
      PL( 16, 10) = QK( 16, 2)*RHO(2)*Y(18)*Y( 2)              FG2 1770
      GO TO (31,32,33,34,35,36,37,38,39,40,41,42,43,44,45,46,47,48,49),KFG2 1780
31 AA      = PL( 11, 10) - PL( 11, 5)                          FG2 1790
      AII      = AA      -PL(1,1)- PL(11,1)                      FG2 1800
      GO TO 90                                                  FG2 1810
32 BB= PL( 7, 6)+ PL( 7, 10)+ PL( 11, 6)+ PL( 11, 10)          FG2 1820
      BB=BB+ PL( 12, 7)+ PL( 12, 10)+2.*PL( 13, 6)+2.*PL( 13, 10) FG2 1830
      AA= PL( 7, 5)+ PL( 11, 5)+ PL( 12, 5)+2.*PL( 13, 5)      FG2 1840
      AA=AA+ PL( 14, 10) + PL(15, 5) + PL(16, 5)                FG2 1850
      BB=BB+ PL( 14, 1)+ PL( 14, 5) + PL(15,10) + PL(15, 7)     FG2 1860
      BB=BB+ PL( 16,10) + PL(16, 7)                             FG2 1870
      AII      = AA - BB                                         FG2 1880
      GO TO 90                                                  FG2 1890
33 AA= PL( 9, 5)+ PL( 12, 5)+ PL( 13, 10)                      FG2 1900
      BB= PL( 9, 10)+ PL( 12, 10)+ PL( 13, 5)                  FG2 1910
      AA      = AA - BB                                         FG2 1920
      AII      = AA      - PL( 2, 6) - PL( 9, 6) - PL( 12, 6) FG2 1930
      1 - PL( 13, 1)                                           FG2 1940
      GO TO 90                                                  FG2 1950
34 AA= PL( 12, 10)                                              FG2 1960
      BB= PL( 12, 5)                                           FG2 1970
      AA      = AA - BB                                         FG2 1980
      AII      = AA      - PL( 12, 1)                          FG2 1990
      GO TO 90                                                  FG2 2000
35 BB= PL( 14, 10)                                              FG2 2010
      AA= PL( 14, 5)                                           FG2 2020
      AA      = AA - BB                                         FG2 2030
      AII      = AA      - PL( 14, 6)                          FG2 2040
      GO TO 90                                                  FG2 2050
36 AA= PL( 9, 10)+ PL( 10, 5) + PL( 15, 5)                    FG2 2060
      BB= PL( 9, 5)+ PL( 10, 10) + PL( 15, 10) + PL( 15, 6)    FG2 2070
      AA      = AA - BB                                         FG2 2080
      AII      = AA      - PL( 2, 1) - PL( 9, 1) - PL( 10, 6) FG2 2090
      GO TO 90                                                  FG2 2100

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| | | | | |
|----|------------------|---------------------------------------|-------------------------------------|----------|
| 37 | AA= | PL(7, 10)+ | PL(8, 5) | FG2 2110 |
| | BB= | PL(7, 5)+ | PL(8, 10) | FG2 2120 |
| | AA | = AA - BB | | FG2 2130 |
| | AII | = AA | - PL(1, 6) - PL(7, 1) - PL(8, 6) | FG2 2140 |
| | GO TO | 90 | | FG2 2150 |
| 38 | AA= | PL(8, 10) | | FG2 2160 |
| | BB= | PL(8, 5) | | FG2 2170 |
| | AA | = AA - BB | | FG2 2180 |
| | AII | = AA | - PL(8, 1) | FG2 2190 |
| | GO TO | 90 | | FG2 2200 |
| 39 | AA= | PL(4, 10) | | FG2 2210 |
| | BB= | PL(4, 5) | | FG2 2220 |
| | AA | = AA - BB | | FG2 2230 |
| | AII | = AA | - PL(2, 7) - PL(4, 1) | FG2 2240 |
| | GO TO | 90 | | FG2 2250 |
| 40 | BB= | PL(2, 2)+2.*PL(4, 6)+2.*PL(4, 10)+ | PL(8, 7) | FG2 2260 |
| | BB=BB+ | PL(8, 10)+ | PL(9, 7)+ PL(9, 10)+ PL(10, 7) | FG2 2270 |
| | AA=2.*PL(4, 5)+ | PL(8, 5)+ | PL(9, 5)+ PL(10, 5) | FG2 2280 |
| | BB=BB+ | PL(10, 10) | | FG2 2290 |
| | AII | = AA - BB | | FG2 2300 |
| | GO TO | 90 | | FG2 2310 |
| 41 | AA= | PL(3, 10) | | FG2 2320 |
| | BB= | PL(3, 5) | | FG2 2330 |
| | AA | = AA - BB | | FG2 2340 |
| | AII | = AA | - PL(3, 1) | FG2 2350 |
| | GO TO | 90 | | FG2 2360 |
| 42 | BB= | PL(1, 2)+2.*PL(3, 6)+2.*PL(3, 10)+ | PL(6, 1) | FG2 2370 |
| | AA=2.*PL(3, 5)+ | PL(6, 10)+ | PL(7, 5) | FG2 2380 |
| | BB=BB+ | PL(6, 5)+ | PL(7, 7)+ PL(7, 10) | FG2 2390 |
| | AII | = AA - BB | | FG2 2400 |
| | GO TO | 90 | | FG2 2410 |
| 43 | AA= | PL(6, 5) | | FG2 2420 |
| | BB= | PL(6, 10) | | FG2 2430 |
| | AA | = AA - BB | | FG2 2440 |
| | AII | = AA | - PL(6, 6) | FG2 2450 |

| | | |
|----|--|----------|
| | GO TO 90 | FG2 2460 |
| 44 | AA= PL(5, 10)+ PL(11, 5) | FG2 2470 |
| | BB= PL(5, 5)+ PL(11, 10) | FG2 2480 |
| | AA = AA - BB | FG2 2490 |
| | AI1 = AA - PL(1, 7) - PL(5, 1) - PL(11, 7) | FG2 2500 |
| | GO TO 90 | FG2 2510 |
| 45 | AA= PL(5, 5) | FG2 2520 |
| | BB= PL(5, 10) | FG2 2530 |
| | AA = AA - BB | FG2 2540 |
| | AI1 = AA - PL(5, 6) | FG2 2550 |
| | GO TO 90 | FG2 2560 |
| 46 | AA= PL(5, 5)+ PL(6, 5)+ PL(14, 5) | FG2 2570 |
| | BB= PL(5, 10)+ PL(6, 10)+ PL(14, 10) | FG2 2580 |
| | AA = AA - BB | FG2 2590 |
| | AI1 = AA - PL(5, 7) - PL(6, 7) - PL(14, 7) | FG2 2600 |
| | GO TO 90 | FG2 2610 |
| 47 | AA = PL(10, 10) - PL(10, 5) | FG2 2620 |
| | AI1 = AA - PL(10, 1) | FG2 2630 |
| | GO TO 90 | FG2 2640 |
| 48 | AA = PL(15,10) - PL(15, 5) + PL(16, 5) - PL(16,10) | FG2 2650 |
| | AI1 =AA - PL(15, 1) - PL(16, 6) | FG2 2660 |
| | GO TO 90 | FG2 2670 |
| 49 | AA = PL(16,10) - PL(16, 5) | FG2 2680 |
| | AI1 = AA - PL(16, 1) | FG2 2690 |
| 90 | PL(1, 1)=Y(1)*PL(1, 1) | FG2 2700 |
| | PL(1, 6)=Y(7)*PL(1, 6) | FG2 2710 |
| | PL(2, 1)=Y(6)*PL(2, 1) | FG2 2720 |
| | PL(2, 6)=Y(3)*PL(2, 6) | FG2 2730 |
| | PL(3, 1)=Y(11)*PL(3, 1) | FG2 2740 |
| | PL(3, 6)=Y(12)*PL(3, 6)/2. | FG2 2750 |
| | PL(4, 1)=Y(9)*PL(4, 1) | FG2 2760 |
| | PL(4, 6)=Y(10)*PL(4, 6)/2. | FG2 2770 |
| | PL(5, 1)=Y(14)*PL(5, 1) | FG2 2780 |
| | PL(5, 6)=Y(15)*PL(5, 6) | FG2 2790 |
| | PL(6, 1)=Y(12)*PL(6, 1) | FG2 2800 |

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| PL(6, 6)=Y(13)*PL(6, 6) | FG2 2810 |
| PL(7, 1)=Y(7)*PL(7, 1) | FG2 2820 |
| PL(7, 6)=Y(2)*PL(7, 6) | FG2 2830 |
| PL(8, 1)=Y(8)*PL(8, 1) | FG2 2840 |
| PL(8, 6)=Y(7)*PL(8, 6) | FG2 2850 |
| PL(9, 1)=Y(6)*PL(9, 1) | FG2 2860 |
| PL(9, 6)=Y(3)*PL(9, 6) | FG2 2870 |
| PL(10, 1)=Y(17)*PL(10, 1) | FG2 2880 |
| PL(10, 6)=Y(6)*PL(10, 6) | FG2 2890 |
| PL(11, 1)=Y(1)*PL(11, 1) | FG2 2900 |
| PL(11, 6)=Y(2)*PL(11, 6) | FG2 2910 |
| PL(12, 1)=Y(4)*PL(12, 1) | FG2 2920 |
| PL(12, 6)=Y(3)*PL(12, 6) | FG2 2930 |
| PL(13, 1)=Y(3)*PL(13, 1) | FG2 2940 |
| PL(13, 6)=Y(2)*PL(13, 6)/2. | FG2 2950 |
| PL(14, 1)=Y(2)*PL(14, 1) | FG2 2960 |
| PL(14, 6)=Y(5)*PL(14, 6) | FG2 2970 |
| PL(15, 1)=Y(18)*PL(15, 1) | FG2 2980 |
| PL(15, 6)=Y(6)*PL(15, 6) | FG2 2990 |
| PL(16, 1)=Y(19)*PL(16, 1) | FG2 3000 |
| PL(16, 6)=Y(18)*PL(16, 6) | FG2 3010 |
| 191 GO TO (51,52,53,54,55,56,57,58,59,60,61,62,63,64,65,66,67,68,69),K | FG2 3020 |
| 51 AA= PL(1, 6)+ PL(11, 6) | FG2 3030 |
| BB= PL(1, 1)+ PL(11, 1) | FG2 3040 |
| B = AA - BB | FG2 3050 |
| GO TO 95 | FG2 3060 |
| 52 BB= PL(7, 6)+ PL(11, 6)+ PL(12, 6)+2.*PL(13, 6) | FG2 3070 |
| AA= PL(7, 1)+ PL(11, 1)+ PL(12, 1)+2.*PL(13, 1) | FG2 3080 |
| AA=AA+ PL(14, 6) + PL(15, 1) + PL(16, 1) | FG2 3090 |
| BB=BB+ PL(14, 1) + PL(15, 6) + PL(16,6) | FG2 3100 |
| B = AA - BB | FG2 3110 |
| GO TO 95 | FG2 3120 |
| 53 AA= PL(2, 1)+ PL(9, 1)+ PL(12, 1)+ PL(13, 6) | FG2 3130 |
| BB= PL(2, 6)+ PL(9, 6)+ PL(12, 6)+ PL(13, 1) | FG2 3140 |
| B = AA - BB | FG2 3150 |

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| | GO TO 95 | | FG2 3160 |
| 54 | AA= PL(12, 6) | | FG2 3170 |
| | BB= PL(12, 1) | | FG2 3180 |
| | B = AA - BB | | FG2 3190 |
| | GO TO 95 | | FG2 3200 |
| 55 | AA= PL(14, 1) | | FG2 3210 |
| | BB= PL(14, 6) | | FG2 3220 |
| | B = AA - BB | | FG2 3230 |
| | GO TO 95 | | FG2 3240 |
| 56 | AA= PL(2, 6)+ PL(9, 6)+ PL(10, 1) + PL(15, 1) | | FG2 3250 |
| | BB= PL(2, 1)+ PL(9, 1)+ PL(10, 6) + PL(15, 6) | | FG2 3260 |
| | B = AA - BB | | FG2 3270 |
| | GO TO 95 | | FG2 3280 |
| 57 | AA= PL(1, 1)+ PL(7, 6)+ PL(8, 1) | | FG2 3290 |
| | BB= PL(1, 6)+ PL(7, 1)+ PL(8, 6) | | FG2 3300 |
| | B = AA - BB | | FG2 3310 |
| | GO TO 95 | | FG2 3320 |
| 58 | AA= PL(8, 6) | | FG2 3330 |
| | BB= PL(8, 1) | | FG2 3340 |
| | B = AA - BB | | FG2 3350 |
| | GO TO 95 | | FG2 3360 |
| 59 | AA= PL(2, 1)+ PL(4, 6) | | FG2 3370 |
| | BB= PL(2, 6)+ PL(4, 1) | | FG2 3380 |
| | B = AA - BB | | FG2 3390 |
| | GO TO 95 | | FG2 3400 |
| 60 | AA= PL(2, 6)+2*PL(4, 1)+ PL(8, 1)+ PL(9, 1) | | FG2 3410 |
| | BB= PL(2, 1)+2*PL(4, 6)+ PL(8, 6)+ PL(9, 6) | | FG2 3420 |
| | AA=AA+ PL(10, 1) | | FG2 3430 |
| | BB=BB+ PL(10, 6) | | FG2 3440 |
| | B = AA - BB | | FG2 3450 |
| | GO TO 95 | | FG2 3460 |
| 61 | AA= PL(3, 6) | | FG2 3470 |
| | BB= PL(3, 1) | | FG2 3480 |
| | B = AA - BB | | FG2 3490 |
| | GO TO 95 | | FG2 3500 |

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|-----|---------|-------------------------|------------|------------|----------|
| 62 | AA= | PL(1, 6)+2.*PL(3, 1)+ | PL(6, 6)+ | PL(7, 1) | FG2 3510 |
| | BB= | PL(1, 1)+2.*PL(3, 6)+ | PL(6, 1)+ | PL(7, 6) | FG2 3520 |
| | B | = AA - BB | | | FG2 3530 |
| | GO TO | 95 | | | FG2 3540 |
| 63 | AA= | PL(6, 1) | | | FG2 3550 |
| | BB= | PL(6, 6) | | | FG2 3560 |
| | B | = AA - BB | | | FG2 3570 |
| | GO TO | 95 | | | FG2 3580 |
| 64 | AA= | PL(1, 1)+ | PL(5, 6)+ | PL(11, 1) | FG2 3590 |
| | BB= | PL(1, 6)+ | PL(5, 1)+ | PL(11, 6) | FG2 3600 |
| | B | = AA - BB | | | FG2 3610 |
| | GO TO | 95 | | | FG2 3620 |
| 65 | AA= | PL(5, 1) | | | FG2 3630 |
| | BB= | PL(5, 6) | | | FG2 3640 |
| | B | = AA - BB | | | FG2 3650 |
| | GO TO | 95 | | | FG2 3660 |
| 66 | AA= | PL(5, 1)+ | PL(6, 1)+ | PL(14, 1) | FG2 3670 |
| | BB= | PL(5, 6)+ | PL(6, 6)+ | PL(14, 6) | FG2 3680 |
| | B | = AA - BB | | | FG2 3690 |
| | GO TO | 95 | | | FG2 3700 |
| 67 | AA= | PL(10,6) | | | FG2 3710 |
| | BB= | PL(10,1) | | | FG2 3720 |
| | B | = AA - BB | | | FG2 3730 |
| | GO TO | 95 | | | FG2 3740 |
| 68 | AA= | PL(15, 6) + PL(16,1) | | | FG2 3750 |
| | BB= | PL(15, 1) + PL(16, 6) | | | FG2 3760 |
| | B | = AA - BB | | | FG2 3770 |
| | GO TO | 95 | | | FG2 3780 |
| 69 | AA= | PL(16, 6) | | | FG2 3790 |
| | BB= | PL(16, 1) | | | FG2 3800 |
| | B | = AA - BB | | | FG2 3810 |
| 95 | IF(IND) | 110,101,110 | | | FG2 3820 |
| 101 | IND = | 1 | | | FG2 3830 |
| | GAMMA = | 0. | | | FG2 3840 |
| | DO 1021 | =1,16 | | | FG2 3850 |

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| IF(I.GT.2)GAMMA = 1. | FG2 3860 |
| Q = (RATES(2,I) + RATES(3,I)/T)/T | FG2 3870 |
| PL(I,1) = Q*PL(I,1) | FG2 3880 |
| Q = GAMMA/RT + Q - E(I,1,L)/T1(1) - E(I,2,L)/2. - E(I,3,L)*T1(1)/3 | FG2 3890 |
| X. - E(I,4,L)*T1(2)/4. - E(I,5,L)*T1(3)/5. - E(I,6,L)/T1(2) | FG2 3900 |
| 102 PL(I,6) = Q*PL(I,6) | FG2 3910 |
| C = B | FG2 3920 |
| GO TO 191 | FG2 3930 |
| 110 RETURN | FG2 3940 |
| END | FG2 3950 |

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|---|---|----------|
| | SUBROUTINE OUPPUT (IOUT) | OUPP 10 |
| | COMMON/KKM/ALP(60),SVA(60) ,SDIFF(60),SAII(60),SB(60) | OUPP 20 |
| | COMMON/FOG/AII ,CWT, B, TOLD | OUPP 30 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | OUPP 40 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | OUPP 50 |
| | COMMON /YL/XXX(60),YOND(60) | OUPP 60 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | OUPP 70 |
| | COMMON/PROP1/PI(60),RHO(60), T(60),BMW(60),CM(20,60),CE(5,60) | OUPP 80 |
| | DIMENSION Y(20,60) | OUPP 90 |
| | EQUIVALENCE (CM(1),Y(1)) | OUPP 100 |
| | DIMENSION ETA(60) | OUPP 110 |
| | EQUIVALENCE (NETA,NP), (PD,PRE), (YCND(1),ETA(1)) | OUPP 120 |
| C | | OUPP 130 |
| C | **** PRINT RESULTS OF SPECIES ITERATIONS **** | OUPP 140 |
| C | | OUPP 150 |
| | NS = 19 | OUPP 160 |
| | GO TO (1, 2),IOUT | OUPP 170 |
| C | OUTPUT CONVERGED RESULTS | OUPP 180 |
| | 1 CONTINUE | OUPP 190 |
| | 11 WRITE(6,61) | OUPP 200 |
| | 61 FORMAT(/4X,'PT.',17X,'ETA',13X,'DENSITY',9X,'TEMPERATURE',12X,'VELOUPP 210 | |
| | 10CITY'/) | OUPP 220 |
| | WRITE(6,71)(I,ETA(I),RHO(I),T(I),V(I),I=1,NP) | OUPP 230 |
| | 71 FORMAT(I7,4E20,7) | OUPP 240 |
| C | OUTPUT INTERMEDIATE RESULTS | OUPP 250 |
| | 2 WRITE(6,20) | OUPP 260 |
| | 20 FORMAT(1H17X,'ETA',9X,'CO',10X,'C',9X,'C2',9X,'C3',9X,'C+',8X,'C2HOUPP 270 | |
| | 1',9X,'CN',8X,'HCN',9X,'H2',10X,'H') | OUPP 280 |
| | DO30J=1,NP | OUPP 290 |
| | 30 WRITE(6,40)ETA(J),(Y(I,J),I= 1,10) | OUPP 300 |
| | 40 FORMAT(11F11,7) | OUPP 310 |
| | WRITE(6,50) | OUPP 320 |
| | 50 FORMAT(8X,'ETA',9X,'N2',10X,'N',9X,'N+',10X,'O',9X,'O+',9X,'E-'OUPP 330 | |
| | X,7X,'C2H2',8X,'C3H',8X,'C4H') | OUPP 340 |
| | DO60J=1,NP | OUPP 350 |

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60 WRITE(6,40)ETA(J),(Y(I,J),I=11,19)
   RETURN
   END
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OUPP 360
OUPP 370
OUPP 380
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|-------------------|--|------|-----|
| SUBROUTINE COUPLE | | COUP | 10 |
| C | | COUP | 20 |
| C | -----THIS SUBROUTINE SOLVES THE SPECIES AND ENERGY EQUATIONS | COUP | 30 |
| C | | COUP | 40 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | COUP | 50 |
| | COMMON /CONV/ FPRCT,TPRCT,DDAMP,TDAMP,PDTIL | COUP | 60 |
| | COMMON/CONV1/HDAMP | COUP | 70 |
| | COMMON /DEL/ DELTA,DTIL,DTILS | COUP | 80 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | COUP | 90 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | COUP | 100 |
| | COMMON/NUMBER/NSP,NNS,NE,NC | COUP | 110 |
| | COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),CE(5,60) | COUP | 120 |
| | COMMON/PROP2/ MU(60),RM(60), AK(60) | COUP | 130 |
| | COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | COUP | 140 |
| | COMMON/VECTOR/SUB(60),DIAG(60),SUP(60),B(60) | COUP | 150 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | COUP | 160 |
| | COMMON/ID/SP(20),EL(5) | COUP | 170 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | COUP | 180 |
| | COMMON/SP2/BR,S(20),CSHOCK(5) | COUP | 190 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | COUP | 200 |
| | COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | COUP | 210 |
| | COMMON /YL/ETA(60),YOND(60) | COUP | 220 |
| | COMMON /OLD/ TOLD(60),EOLD(60),RHGS(60) | COUP | 230 |
| | COMMON/DD/D(60) | COUP | 240 |
| | DIMENSION RHOLD(60) | COUP | 250 |
| | REAL MU,MUDZ | COUP | 260 |
| | LOGICAL MCONV,ECONV,DCCNV | COUP | 270 |
| | COMMON/NETI/NETA1,NETA2 | COUP | 280 |
| | COMMON/KKB/RHOA,TA,VA,YA(19) | COUP | 290 |
| | COMMON/FOG/AII ,BI ,CI ,TOLD1 | COUP | 300 |
| | COMMON/IT/AC | COUP | 310 |
| | COMMON/CK/ISN(20),MWT(19) | COUP | 320 |
| | REAL MWT | COUP | 330 |
| | DIMENSION Y(20,60), SY(20,60) | COUP | 340 |
| | DIMENSION SACI(19,60) | COUP | 350 |

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| DIMENSION RED (60),RES(60),RET(60) | COUP 360 |
| DATA NS,SYNCH,SPER,TMCH/19.,.50.,.80.,.01/ | COUP 370 |
| C | COUP 380 |
| C-----INITIALIZE | COUP 390 |
| C | COUP 400 |
| NETA1 = NETA - 1 | COUP 410 |
| NETA2 = NETA - 2 | COUP 420 |
| ECONV = .FALSE. | COUP 430 |
| ITER = 0 | COUP 440 |
| DO 10 J=1,NETA | COUP 450 |
| 10 TOLD(J) = T(J) | COUP 460 |
| AC = 8.128E-08*(TD**1.659)/(PI(1)*R*UINF) | COUP 470 |
| AR = R/UINF | COUP 480 |
| C ** COMPUTE THE Y COORDINATE ** | COUP 490 |
| YOND(1) = 0.0 | COUP 500 |
| SUM = 0.0 | COUP 510 |
| DO 39 K=2,NETA | COUP 520 |
| DETA = ETA(K) -ETA(K-1) | COUP 530 |
| SUM = SUM +DETA*(1./RHC(K) +1./RHO(K-1))/2.0 | COUP 540 |
| YOND(K) = DTIL *SUM | COUP 550 |
| 39 CONTINUE | COUP 560 |
| DELTA = YOND(NETA) | COUP 570 |
| DO 49 K=1,NETA | COUP 580 |
| YOND(K) = YOND(K)/YOND(NETA) | COUP 590 |
| 49 CONTINUE | COUP 600 |
| IF(IAB.NE.0)GO TO 528 | COUP 610 |
| CALL SPECIE | COUP 620 |
| CALL PROPRT(NSP,1,NETA) | COUP 630 |
| C **** COMPUTE FLUX DIVERGENCE **** | COUP 640 |
| 528 IF(IRAD.EQ.3.AND.ITYPE.EQ.1)CALL EFLUX | COUP 650 |
| IF(IRAD.EQ.3.AND.ITYPE.EQ.0)CALL LRAD | COUP 660 |
| TOLD1 = 0. | COUP 670 |
| C RDZ = DENSITY AT SHOCK (SLUGS OF M / FT**3) | COUP 680 |
| RHOD = RDZ*32.174 | COUP 690 |
| C RHOD = DENSITY AT SHOCK (LBM OF M / FT**3) | COUP 700 |

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| C | | COUP 710 |
| C | ***** | COUP 720 |
| C | | COUP 730 |
| C | **** CALCULATE DIFFUSION COEFFICIENT **** | COUP 740 |
| | 110 DO120J=1,NETA | COUP 750 |
| | 120 D(J) = AC*(T(J)**1.659) | COUP 760 |
| | 130 CALL PROPRT (NSP,1,NETA) | COUP 770 |
| | DX1 = ETA(2) - ETA(1) | COUP 780 |
| | DO200J=2,NETA1 | COUP 790 |
| | DX = ETA(J+1) - ETA(J) | COUP 800 |
| | J1 = J - 1 | COUP 810 |
| | CO13 = 2.*RHO(J)*AK(J)/(DTIL*CP(J)) | COUP 820 |
| | CO14 = - RHO(J)*V(J) | COUP 830 |
| | CO15 = (C1(DX,DX1)*RHO(J+1)*AK(J+1)/CP(J+1) + C2(DX,DX1)*RHO(J)* | COUP 840 |
| X | AK(J)/CP(J) + C3(DX,DX1)*RHO(J-1)*AK(J-1)/CP(J-1))*2./DTIL | COUP 850 |
| | CON4 = 2.*RHO(J)*V(J)*V(J)*(C1(DX,DX1)*V(J+1) + C2(DX,DX1)*V(J) + | COUP 860 |
| X | C3(DX,DX1)*V(J-1)) | COUP 870 |
| | CON5 = 2.*DTIL*E(J)/RHO(J) | COUP 880 |
| | CON6 = 0. | COUP 890 |
| | CON7 = 0. | COUP 900 |
| | CO18 = 0. | COUP 910 |
| | DO2023I=1,NSP | COUP 920 |
| | CON6 = CON6 + HS(I,J)*(C4(DX,DX1)*C(I,J+1) + C5(DX,DX1)*C(I,J) + | COUP 930 |
| X | C6(DX,DX1)*C(I,J-1)) | COUP 940 |
| | CV = (C1(DX,DX1)*C(I,J+1) + C2(DX,DX1)*C(I,J) + | COUP 950 |
| X | C3(DX,DX1)*C(I,J-1)) | COUP 960 |
| | CON7 = CON7 + HS(I,J)*CV | COUP 970 |
| | CO18 = CO18 + CV*(C1(DX,DX1)*HS(I,J+1) + C2(DX,DX1)*HS(I,J) + | COUP 980 |
| X | C3(DX,DX1)*HS(I,J-1)) | COUP 990 |
| 2023 | CONTINUE | COUP1000 |
| | CV = (2.*RHO(J)*AK(J)/CP(J) - RHO(J)*RHO(J)*D(J))/DTIL | COUP1010 |
| | CO16 = CON6*CV | COUP1020 |
| | CD = (C1(DX,DX1)*RHO(J+1)*RHO(J+1)*D(J+1) + C2(DX,DX1)*RHO(J)* | COUP1030 |
| X | RHO(J)*D(J) + C3(DX,DX1)*RHO(J-1)*RHO(J-1)*D(J-1))/DTIL | COUP1040 |
| | CO17 = CON7*(CO15 -CD) | COUP1050 |

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| CO18 = CO18*CV | COUP1060 |
| SUB(J1) = CO13 | COUP1070 |
| DIAG(J1) = CO14 + CO15 | COUP1080 |
| SUP(J1) = 0. | COUP1090 |
| B(J1) = CON5 + CON4 + CO17 + CO18 + CO16 | COUP1100 |
| 200 DX1 = DX | COUP1110 |
| CALL SOLVE (HM(1),HM(NETA)) | COUP1120 |
| C-----CHECK FOR CONVERGENCE | COUP1130 |
| IF(IDERUG.E0.0)GO TO 249 | COUP1140 |
| DO1532K1=1,NETA2 | COUP1150 |
| KK = K1+1 | COUP1160 |
| 1532 WRITE(6,1533) K1,ETA(KK),HM(KK),B(K1) | COUP1170 |
| 1533 FORMAT(1X,I10,F15.4,2E20.6) | COUP1180 |
| 249 DO250J=2,NETA1 | COUP1190 |
| SRCT = (B(J-1) - HM(J))/HM(J) | COUP1200 |
| PRCT=ABS(SRCT) | COUP1210 |
| IF (PRCT.GT.TPRCT) GO TO 260 | COUP1220 |
| 250 CONTINUE | COUP1230 |
| GO TO 300 | COUP1240 |
| 260 ITER=ITER+1 | COUP1250 |
| DO270J=2,NETA1 | COUP1260 |
| SRCT = (B(J-1) - HM(J))/HM(J) | COUP1270 |
| SAP = HDAMP*SRCT | COUP1280 |
| IF(ABS(SAP).GT.TMCH)GO TO 1618 | COUP1290 |
| ZIP = SAP | COUP1300 |
| GO TO 1619 | COUP1310 |
| 1618 ZIP = TMCH*ABS(SAP)/SAP | COUP1320 |
| 1619 HM(J) = (1.+ZIP)*HM(J) | COUP1330 |
| IF(HM(J).LT.HM(1))HM(J) = 1.0001*HM(1) | COUP1340 |
| IF(HM(J).GT.1.0)HM(J) = .9999 | COUP1350 |
| 270 CONTINUE | COUP1360 |
| C | COUP1370 |
| C..... SOLVE FOR TEMPERATURE | COUP1380 |
| C | COUP1390 |
| DO818J=1,NETA | COUP1400 |

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| 818 RED(J) = T(J) | COUP1410 |
| DO756J=2,NETA1 | COUP1420 |
| B(J-1) = HM(J) | COUP1430 |
| ITEB = 0 | COUP1440 |
| 751 CALL PROPRT (NSP,J,J) | COUP1450 |
| DTT = (B(J-1) - HM(J))/CP(J) | COUP1460 |
| T(J) = T(J) + DTT | COUP1470 |
| DTT = DTT/T(J) | COUP1480 |
| ITEB = ITEB + 1 | COUP1490 |
| IF(ITEB.GT.20)GO TO 752 | COUP1500 |
| GO TO 754 | COUP1510 |
| 752 WRITE(6,753)J,DTT | COUP1520 |
| 753 FORMAT(1X,'TEMPERATURE AT THE ',I3,'TH POINT DOES NOT CONVERGE', | EXCOUP1530 |
| X , 'CHANGE IN TEMPERATURE=' ,E20.6) | COUP1540 |
| STOP | COUP1550 |
| C | COUP1560 |
| 754 IF(ABS(DTT).GT..010)GO TO 751 | COUP1570 |
| 756 CONTINUE | COUP1580 |
| DO9023J=2,NETA1 | COUP1590 |
| SRCT = (T(J) - RED(J))/RED(J) | COUP1600 |
| SAP = TDAMP*SRCT | COUP1610 |
| IF(ABS(SAP).GT.TMCH)GO TO 9021 | COUP1620 |
| ZIP = SAP | COUP1630 |
| GO TO 9022 | COUP1640 |
| 9021 ZIP = TMCH*ABS(SAP)/SAP | COUP1650 |
| 9022 T(J) = (1. + ZIP)*RED(J) | COUP1660 |
| IF(T(J).LT.T(1))T(J) = T(1) | COUP1670 |
| 9023 IF(T(J).GT.1.)T(J) = 1. | COUP1680 |
| IF(IDEBUG.EQ.0)GO TO 889 | COUP1690 |
| WRITE(6,817) (J,RED(J),T(J),J=1,NETA) | COUP1700 |
| 817 FORMAT(1X,I5,20X,2F20.6) | COUP1710 |
| C..... | COUP1720 |
| C | COUP1730 |
| 889 IF(ITER.GE.MAXE) GOTO 300 | COUP1740 |
| IF(IAB)891,891,892 | COUP1750 |

| | | |
|-----|---|----------|
| 891 | CALL SPECIE | COUP1760 |
| | GO TO 110 | COUP1770 |
| 892 | CALL ELRAT | COUP1780 |
| | CALL CHEMEQ (1,NETA) | COUP1790 |
| | GO TO 110 | COUP1800 |
| 300 | CONTINUE | COUP1810 |
| | IF(ITER.LT.MAXE) ECONV=.TRUE. | COUP1820 |
| | DO320J=2,NETA1 | COUP1830 |
| | T(J) = T(J) + .35*(TOLD(J) - T(J)) | COUP1840 |
| | SRCT = (T(J) - TOLD(J))/TOLD(J) | COUP1850 |
| C | IF(ABS(SRCT).GT.TMCH)T(J) = TOLD(J) + TMCH*SRCT*TOLD(J)/ABS(SRCT) | COUP1860 |
| 320 | CONTINUE | COUP1870 |
| | RETURN | COUP1880 |
| | END | COUP1890 |

| | |
|--|----------|
| SUBROUTINE ELRAT | ELRA 10 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | ELRA 20 |
| COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),CE(5,60) | ELRA 30 |
| COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,LT,IAB | ELRA 40 |
| COMMON/IT/AC | ELRA 50 |
| COMMON/NUMBER/NSP,NNS,NE,NC | ELRA 60 |
| COMMON /YL/ETA(60),YOND(60) | ELRA 70 |
| COMMON/NET1/NETA1,NETA2 | ELRA 80 |
| COMMON/VECTOR/SUB(60),DIAG(60),SUP(60),B(60) | ELRA 90 |
| COMMON /DEL/ DELTA,DTIL,DTILS | ELRA 100 |
| COMMON /VEL/ F(60),FC(60),Z(60),V(60) | ELRA 110 |
| COMMON/DD/D(60) | ELRA 120 |
| C----- SOLVE ELEMENT CONSERVATION EQUATIONS ----- | ELRA 130 |
| CE(1,NETA) = 1.E-03 | ELRA 140 |
| CE(2,NETA) = 1.E-03 | ELRA 150 |
| DO116K=1,NE | ELRA 160 |
| DX1 = ETA(2) - ETA(1) | ELRA 170 |
| DO106J=2,NETA1 | ELRA 180 |
| DX = ETA(J+1) - ETA(J) | ELRA 190 |
| J1 = J - 1 | ELRA 200 |
| SUB(J1) = RHO(J)*RHO(J)*D(J)/(DTIL*DTIL) | ELRA 210 |
| DIAG(J1) = ((C1(DX,DX1)*RHO(J+1)*RHO(J+1)*D(J+1) + C2(DX,DX1)* | ELRA 220 |
| XRHO(J)*RHO(J)*D(J) + C3(DX,DX1)*RHO(J-1)*RHO(J-1)*D(J-1)) - | ELRA 230 |
| XDTIL*RHO(J)*V(J))/(DTIL*DTIL) | ELRA 240 |
| SUP(J1) = 0. | ELRA 250 |
| B(J1) = 0. | ELRA 260 |
| 106 DX1 = DX | ELRA 270 |
| CALL SOLVE (CE(K, 1),CE(K,NETA)) | ELRA 280 |
| DO116J=2,NETA1 | ELRA 290 |
| 116 CE(K,J) = B(J-1) | ELRA 300 |
| DO1753K=1,5 | ELRA 310 |
| DO1753J=2,NETA1 | ELRA 320 |
| 1753 IF(CE(K,J).LT.0.)CE(K,J) = CE(K,J+1) | ELRA 330 |
| IF(IDEBUG.EQ.0)GO TO 11 | ELRA 340 |
| WRITE(6,117)(J,(CE(K,J),K=1,5),J=1,NETA) | ELRA 350 |

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117 FORMAT(1X,I5,5E20.7)
11 RETURN
END
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ELRA 360
FLRA 370
ELRA 380
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|--|--------------------|----|------|-----|-----|-----|----|------|-----|
| SUBROUTINE SPECIE | | | | | | | | SPEC | 10 |
| C-----STAGNATION LINE SOLUTION OF SPECIES CONTINUITY EQUATION (SPC4) | | | | | | | | SPEC | 20 |
| C-----ORDER OF SPECIES IS0 | | | | | | | | SPEC | 30 |
| C | CO | C | C2 | C3 | C+ | C2H | CN | SPEC | 40 |
| C | 1 | 2 | 3 | 4 | 5 | 6 | 7 | SPEC | 50 |
| C | HCN | H2 | H | N2 | N | N+ | O | SPEC | 60 |
| C | 8 | 9 | 10 | 11 | 12 | 13 | 14 | SPEC | 70 |
| C | O+ | E- | C2H2 | C3H | C4H | | | SPEC | 80 |
| C | 15 | 16 | 17 | 18 | 19 | | | SPEC | 90 |
| COMMON/COT/A1,CS | | | | | | | | SPEC | 100 |
| COMMON/N/NP1,NP2,NP3 | | | | | | | | SPEC | 110 |
| COMMON/KKB/RHOA,TA,VA,YA(19) | | | | | | | | SPEC | 120 |
| COMMON/KKM/ALP(60),SVA(60) ,SDIFF(60),SAII(60),SB(60) | | | | | | | | SPEC | 130 |
| COMMON/FOG/AII ,CWT ,B ,TOLD | | | | | | | | SPEC | 140 |
| COMMON/SAVE/SSUB(60),SDIAG(60),SSUP(60),SS(60),E(60) | | | | | | | | SPEC | 150 |
| DIMENSION IDEN(19) | | | | | | | | SPEC | 160 |
| C* | * | * | * | * | * | * | * | SPEC | 170 |
| COMMON /DEL/ DELTA,DTIL,DTILS | | | | | | | | SPEC | 180 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, F , RE, LXI, ITM, IEM, NETA | | | | | | | | SPEC | 190 |
| COMMON/PROPI/PI(60),RHO(60), T(60),BMW(60),CM(20,60),CE(5,60) | | | | | | | | SPEC | 200 |
| COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTCTAL | | | | | | | | SPEC | 210 |
| COMMON /YL/XXX(60),YOND(60) | | | | | | | | SPEC | 220 |
| C | XXX IS ETA IN MAIN | | | | | | | SPEC | 230 |
| COMMON /MAIM/KEEP,MAXE,MAXM,MAXD,IDEBUG,MCONV,ECONV,DCONV,L T, IAB | | | | | | | | SPEC | 240 |
| COMMON/VECTOR/ CA(60),CB(60),CC(60),ALT(60) | | | | | | | | SPEC | 250 |
| COMMON /VEL/ F(60),FC(60),Z(60),V(60) | | | | | | | | SPEC | 260 |
| COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | | | | | | | | SPEC | 270 |
| COMMON/NUMBER/NSP,NNS,NE,NC | | | | | | | | SPEC | 280 |
| COMMON/CK/ISN(20),MWT(19) | | | | | | | | SPEC | 290 |
| C* | * | * | * | * | * | * | * | SPEC | 300 |
| REAL MWT | | | | | | | | SPEC | 310 |
| DIMENSION Y(20,60) | | | | | | | | SPEC | 320 |
| COMMON/SIS/SY(20,60) | | | | | | | | SPEC | 330 |
| EQUIVALENCE (CM(1),Y(1)) | | | | | | | | SPEC | 340 |
| DIMENSION ETA(60) | | | | | | | | SPEC | 350 |

| | |
|---|----------|
| EQUIVALENCE (NETA,NP), (PD,PRE), (YCND(1),ETA(1)) | SPEC 360 |
| DIMENSION SUB(60),DIAG(60),SUP(60) | SPEC 370 |
| EQUIVALENCE (CA(1),SUB(1)), (CB(1),DIAG(1)), (CC(1),SUP(1)) | SPEC 380 |
| DATA NS,SYMCH,SPER/19,.50,.80/ | SPEC 390 |
| LOGICAL MCONV,ECONV,DCCNV | SPEC 400 |
| C RDZ = DENSITY AT SHOCK (SLUGS OF M/FT**3) | SPEC 410 |
| RHOD = RDZ*32.174 | SPEC 420 |
| C RHOD = DENSITY AT SHOCK (LBM OF M/FT**3) | SPEC 430 |
| ISIFI=0 | SPEC 440 |
| TOL2 = .02 | SPEC 450 |
| TOL1 = 0.250000*TOL2 | SPEC 460 |
| TOL3 = 2.*TOL2 | SPEC 470 |
| TOL4 = 4.*TOL2 | SPEC 480 |
| NID=2 | SPEC 490 |
| NIT=1 | SPEC 500 |
| IT=0 | SPEC 510 |
| TOLD=0. | SPEC 520 |
| C NP=TOTAL NUMBER OF PROFILE POINTS | SPEC 530 |
| C PRE =STATIC PRESSURE(LBF/FT2) | SPEC 540 |
| NP1=NP - 1 | SPEC 550 |
| NP2=NP - 2 | SPEC 560 |
| NP3=NP - 3 | SPEC 570 |
| C***** DIMENSIONALIZE | SPEC 580 |
| DO50 I=1,NP | SPEC 590 |
| RHO(I) = RHO(I)*RHOD | SPEC 600 |
| T(I) = T(I)*TD | SPEC 610 |
| 50 V(I) = V(I)*UINF | SPEC 620 |
| C ETA=Y(FT) | SPEC 630 |
| C RHO = DENSITY (LBM/FT3) | SPEC 640 |
| C T =TEMPERATURE PROFILE(OK) | SPEC 650 |
| C V =VELOCITY PROFILE(FT/SEC) | SPEC 660 |
| C NS=NO. OF SPECIES | SPEC 670 |
| C MWT=MOLECULAR WEIGHTS | SPEC 680 |
| C Y = MASS FRACTIONS | SPEC 690 |
| DO1111 I=1,NSP | SPEC 700 |

| | |
|---|----------|
| DO111J=1,NP | SPEC 710 |
| 111 SY(I,J) = Y(I,J) | SPEC 720 |
| DO112I=1,NSP | SPEC 730 |
| DO112J=1,NP | SPEC 740 |
| 112 Y(I,J) = SY(ISN(I),J) | SPEC 750 |
| C CALL OUPPUT (1) | SPEC 760 |
| DO100I=1,NS | SPEC 770 |
| DO100J=1,NP | SPEC 780 |
| 100 Y(I,J) = Y(I,J)/MWT(I) | SPEC 790 |
| C Y = MOLES OF I/MASS OF MIXTURE | SPEC 800 |
| DO1971J=1,NP | SPEC 810 |
| DO1971K=1,NS | SPEC 820 |
| 1971 SY(K,J)=Y(K,J) | SPEC 830 |
| C PATM=PRESSURE IN ATMOSPHERES | SPEC 840 |
| AC=8.128E-08 | SPEC 850 |
| AC=AC/PRE | SPEC 860 |
| C-----CALCULATE DIFFUSION COEFFICIENT(DIFF) | SPEC 870 |
| DO170J=1,NP | SPEC 880 |
| 170 SDIFF(J)= AC*(T(J)**1.659) | SPEC 890 |
| C DIFF=(FT2/SEC) | SPEC 900 |
| C*****COMPUTE PART OF THE ELEMENTS OF THE TRID MATRIX***** | SPEC 910 |
| DX1=ETA (2)-ETA (1) | SPEC 920 |
| DO1530J=2,NP1 | SPEC 930 |
| DX=ETA (J+1)-ETA (J) | SPEC 940 |
| SVA(J)= -V(J) + (C1(DX,DX1)*RHC(J+1)*SDIFF(J+1) + C2(DX,DX1)*RHO(SPEC 950 | |
| XJ)*SDIFF(J) + C3(DX,DX1)*RHC(J-1)*SDIFF(J-1))/RHO(J) | SPEC 960 |
| A =SDIFF(J)*C6(DX,DX1)+SVA(J)*C3(DX,DX1) | SPEC 970 |
| G =SDIFF(J)*C5(DX,DX1)+SVA(J)*C2(DX,DX1) | SPEC 980 |
| C =SDIFF(J)*C4(DX,DX1)+SVA(J)*C1(DX,DX1) | SPEC 990 |
| IF(J.EQ.2)A1=A | SPEC1000 |
| IF(J.EQ.NP1)CS=C | SPEC1010 |
| IF(J.EQ.NP1)GO TO 1520 | SPEC1020 |
| SSUP(J-1)=C | SPEC1030 |
| 1520 SDIAG(J-1)=G | SPEC1040 |
| IF(J.EQ.2)GO TO 1530 | SPEC1050 |

| | |
|---|----------|
| SSUB(J-2)=A | SPEC1060 |
| 1530 DX1=DX | SPEC1070 |
| C*** ** | SPEC1080 |
| DO321I=1,NS | SPEC1090 |
| 321 IDEN(I)=0 | SPEC1100 |
| C ***** OVERALL ITERATION LOOP ***** | SPEC1110 |
| 120 IT=IT+1 | SPEC1120 |
| YMCH=SYMCH | SPEC1130 |
| PER=SPER | SPEC1140 |
| IF(IT,LT,21)GO TO 1181 | SPEC1150 |
| 1175 YMCH=10. | SPEC1160 |
| 1181 CONTINUE | SPEC1170 |
| C ***** SPECIES ITERATION LOOP ***** | SPEC1180 |
| DO200I= 1,NS | SPEC1190 |
| IF(IDEN(I).GE.4)GO TO 200 | SPEC1200 |
| IDEN(I)=4 | SPEC1210 |
| DO127J=2,NP1 | SPEC1220 |
| AMW=0. | SPEC1230 |
| DO126K=1,NS | SPEC1240 |
| 126 AMW=AMW+Y(K,J) | SPEC1250 |
| AMW=1./AMW | SPEC1260 |
| 127 RHO(J)=.7608*PRE *AMW/T(J) | SPEC1270 |
| C ***** COMPUTE REACTION RATE ***** | SPEC1280 |
| DO550J=2,NP1 | SPEC1290 |
| JJ = J - 1 | SPEC1300 |
| DO530L=1,NS | SPEC1310 |
| 530 YA(L)=Y(L,J) | SPEC1320 |
| CALL FG2 (YA,RHO(J),T(J),I) | SPEC1330 |
| SAII(JJ) = AII*DELTA | SPEC1340 |
| 550 SB(JJ) = B*DELTA | SPEC1350 |
| 3923 CONTINUE | SPEC1360 |
| C*** ** | SPEC1370 |
| C***** COMPUTE ELEMENTS OF TRID. MATRIX AND SOLVE ***** | SPEC1380 |
| DO1010J=1,NP2 | SPEC1390 |
| JJ=J+1 | SPEC1400 |

| | |
|---|----------|
| DIAG(J)=SDIAG(J)+SAII(J) | SPEC1410 |
| IF(J.EQ.NP2)GO TO 1010 | SPEC1420 |
| SUB(J)=SSUB(J) | SPEC1430 |
| SUP(J)=SSUP(J) | SPEC1440 |
| 1010 ALT(J) = SAI(J)*Y(I,JJ) - SB(J) | SPEC1450 |
| ALT(1)=ALT(1) - A1*Y(I, 1) | SPEC1460 |
| ALT(NP2)=ALT(NP2) - CS*Y(I, NP) | SPEC1470 |
| CALL TRID (NP2) | SPEC1480 |
| DO1080J=2,NP1 | SPEC1490 |
| 1080 ALP(J)=ALT(J-1) | SPEC1500 |
| C***** | SPEC1510 |
| IF(IY.NE.1)GO TO 2694 | SPEC1520 |
| IF(I.NE.13)GO TO 2691 | SPEC1530 |
| DO2690JJ=2,NP1 | SPEC1540 |
| 2690 Y(13,JJ)=ALP(JJ) | SPEC1550 |
| 2691 IF(I.NE.15)GO TO 2694 | SPEC1560 |
| DO2692JJ=2,NP1 | SPEC1570 |
| 2692 Y(15,JJ)=ALP(JJ) | SPEC1580 |
| 2694 CONTINUE | SPEC1590 |
| 128 DO130J=2,NP1 | SPEC1600 |
| ABD=(ALP(J)-Y(I,J))/Y(I,J) | SPEC1610 |
| AR=ABS(ABD) | SPEC1620 |
| IF(ARS(Y(I,J)).LT.1.E-04.AND.ABS(ALP(J)).LT.1.E-04)GO TO 1344 | SPEC1630 |
| IF(IDEN(I).LT.4)GO TO 6010 | SPEC1640 |
| IF(AB.GT.TOL1)IDEN(I)=3 | SPEC1650 |
| 6010 IF(IDEN(I).LT.3)GO TO 6020 | SPEC1660 |
| IF(AB.GT.TOL2)IDEN(I)=2 | SPEC1670 |
| 6020 IF(IDEN(I).LT.2)GO TO 6030 | SPEC1680 |
| IF(AB.GT.TOL3)IDEN(I)=1 | SPEC1690 |
| 6030 IF(IDEN(I).LT.1)GO TO 6040 | SPEC1700 |
| IF(AR.GT.TOL4)IDEN(I)=0 | SPEC1710 |
| 6040 CONTINUE | SPEC1720 |
| 1344 CONTINUE | SPEC1730 |
| SAP=PER*ABD | SPEC1740 |
| IF(ABS(SAP).GT.YMCH)GO TO 1618 | SPEC1750 |

| | |
|--|----------|
| ZIP=SAP | SPEC1760 |
| GO TO 1619 | SPEC1770 |
| 1618 ZIP=YMCH*ABS(SAP)/SAP | SPEC1780 |
| 1619 CONTINUE | SPEC1790 |
| Y(I,J)=(1.+ZIP)*Y(I,J) | SPEC1800 |
| 130 CONTINUE | SPEC1810 |
| 200 CONTINUE | SPEC1820 |
| IF(IT.LT.NIT)GO TO 374 | SPEC1830 |
| IF(IDEBUG.EQ.0)GO TO 823 | SPEC1840 |
| WRITE(6,1463)(I,IDEN(I),I=1,NS) | SPEC1850 |
| 1463 FORMAT(6(4X,I3,2X,'=',I5,4X)) | SPEC1860 |
| 823 DO373I=1,NS | SPEC1870 |
| 373 IDEN(I)=0 | SPEC1880 |
| 374 CONTINUE | SPEC1890 |
| C IF(IT.NE.NID)GO TO 376 | SPEC1900 |
| DO375I=1,NS | SPEC1910 |
| IF(IDEN(I).LT.3)GO TO 376 | SPEC1920 |
| 375 CONTINUE | SPEC1930 |
| GO TO 210 | SPEC1940 |
| 376 IF(IT.LT.NIT)GO TO 3334 | SPEC1950 |
| NIT=NIT+5 | SPEC1960 |
| NID=NIT-4 | SPEC1970 |
| IF(IDEBUG.EQ.0)GO TO 3334 | SPEC1980 |
| WRITE(6,218)IT,YMCH | SPEC1990 |
| 218 FORMAT(//10X,'ITERATION NUMBER',I10,5X,'YMCH',F15,6//) | SPEC2000 |
| C CALL OUTPUT (2) | SPEC2010 |
| 3334 CONTINUE | SPEC2020 |
| GO TO 120 | SPEC2030 |
| 210 DO193J=1,NP | SPEC2040 |
| AMW = 0. | SPEC2050 |
| DO192I=1,NSP | SPEC2060 |
| 192 AMW = AMW + Y(I,J) | SPEC2070 |
| 193 BMW(J) = 1./AMW | SPEC2080 |
| DO191I=1,NS | SPEC2090 |
| DO191J=1,NP | SPEC2100 |

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191 Y(I,J) = Y(I,J)*MWT(I)
C    CALL OUPPUT (I)
C***** NON-DIMENSIONALIZE
      DO 51 I=1,NP
        RHO(I) = RHO(I)/RHOD
        T(I) = T(I)/TD
51  V(I) = V(I)/UINF
      DO 114 I=1,NSP
        DO 114 J=1,NP
114  SY(I,J) = Y(I,J)
      DO 115 I=1,NSP
        DO 115 J=1,NP
115  Y(ISN(I),J) = SY(I,J)
      RETURN
      END

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SPEC2110
SPEC2120
SPEC2130
SPEC2140
SPEC2150
SPEC2160
SPEC2170
SPEC2180
SPEC2190
SPEC2200
SPEC2210
SPEC2220
SPEC2230
SPEC2240
SPEC2250

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| | |
|--|----------|
| SUBROUTINE SOLVE (Z0,Z1) | SOLV 10 |
| C-----THIS SUBROUTINE SOLVES THE DIFFERENTIAL EQUATION | SOLV 20 |
| C A(X)Z'' + B(X)Z' + C(X)Z = D(X) | SOLV 30 |
| C WITH BOUNDARY CONDITIONS Z(0)=Z0, AND Z(1)=Z1. THE NUMERICAL | SOLV 40 |
| C INTEGRATION IS PERFORMED WITH NETA POINTS IN THE DOMAIN OF | SOLV 50 |
| C INTEREST. THE VARIABLE COEFFICIENTS A, B, C, AND D MUST BE | SOLV 60 |
| C EVALUATED AT NETA2=NETA-2 INTERIOR POINTS AND STORED IN SUB, | SOLV 70 |
| C DIAG, SUP, AND B, RESPECTIVELY. THE SOLUTION IS RETURNED IN B. | SOLV 80 |
| COMMON /NETI/NETA1,NETA2 | SOLV 90 |
| COMMON/VECTOR/SUB(60),DIAG(60),SUP(60),B(60) | SOLV 100 |
| COMMON /YL/ETA(60),YOND(60) | SOLV 110 |
| DX1 = ETA(2) - ETA(1) | SOLV 120 |
| DO 40 J=2,NETA1 | SOLV 130 |
| DX = ETA(J+1) - ETA(J) | SOLV 140 |
| J1 = J - 1 | SOLV 150 |
| J2 = J - 2 | SOLV 160 |
| C | SOLV 170 |
| CA = SUB(J1)*C6(DX,DX1) + DIAG(J1)*C3(DX,DX1) | SOLV 180 |
| CB = SUB(J1)*C5(DX,DX1) + DIAG(J1)*C2(DX,DX1) + SUP(J1) | SOLV 190 |
| CC = SUB(J1)*C4(DX,DX1) + DIAG(J1)*C1(DX,DX1) | SOLV 200 |
| DIAG(J1) = CB | SOLV 210 |
| SUP(J1) = CC | SOLV 220 |
| IF(J1.EQ.1)GO TO 20 | SOLV 230 |
| SUB(J2) = CA | SOLV 240 |
| GO TO 40 | SOLV 250 |
| 20 BC = CA | SOLV 260 |
| 40 DX1 = DX | SOLV 270 |
| B(1) = B(1) - BC*Z0 | SOLV 280 |
| B(NETA2) = B(NETA2) - SUP(NETA2)*Z1 | SOLV 290 |
| CALL TRID (NETA2) | SOLV 300 |
| RETURN | SOLV 310 |
| END | SOLV 320 |


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SUBROUTINE CHEMEQ( NI,NF)                                CHEM  10
C * * * * *                                           * CHEM  20
C * * * * *                                           * CHEM  30
C   THIS SUBPROGRAM IS A REVISION OF A PROGRAM ORIGINALLY REPORTED * CHEM  40
C   IN NASA-TN-D-5391 (AUGUST 1969).  THE PROGRAM COMPUTES FOR A * CHEM  50
C   PRESSURE ARRAY,PP(N),TEMPERATURE ARRAY,TT(N), AND AN ARRAY * CHEM  60
C   OF ELEMENTAL MASS FRACTIONS-CC(I,N), THE EQUILIBRIUM SPECIES * CHEM  70
C   MASS FRACTIONS AT EACH PCINT-N REPRESENTED BY THE GIVEN ARRAYS. * CHEM  80
C   THE EQUILIBRIUM COMPOSITIONS ARE STORED IN THE MATRIX CE(I,N). * CHEM  90
C * * * * *                                           * CHEM 100
C * * * * *                                           * CHEM 110
C * * * * *                                           * CHEM 120
C * * * * *                                           * CHEM 130
C * * * * *                                           * CHEM 140
C * * * * *                                           * CHEM 150
C * * * * *                                           * CHEM 160
C   IN = INITIAL POINT FOR EQUILIBRIUM CALCULATIONS. * CHEM 170
C   NF = FINAL PCINT. * CHEM 180
C * * * * *                                           * CHEM 190
COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) CHEM 200
COMMON/SP1/SS,TOL,NDBG CHEM 210
COMMON/EQ1/AI(20),BI(20),CI(20),DI(20),EI(20),FI(20),GI(20),CHEM 220
X   AII(20),BII(20),CII(20),DII(20),EII(20),FII(20),GII(20) CHEM 230
COMMON/EQ2/AA(20,5),ICODE(20) CHEM 240
COMMON/THERM1/C(20),FORT(20) CHEM 250
COMMON/ID/SP(20),EL(5) CHEM 260
COMMON/NUMBER/NS ,NNS,MM,NC CHEM 270
COMMON/WT/XMW(20),AWT(5) CHEM 280
COMMON /RH/ DUD,DPhi,TD,RZB,PD,HD,HTOTAL CHEM 290
COMMON /DY/ DYDT(20,60) CHEM 300
COMMON/PROP1/PP(60),RO(60),TT(60),AMW(60),CE(20,60),CC(5,60) CHEM 310
DIMENSION R( 7, 7),B( 7,1),YINT(20),FY(20),PI(7),FSUM( 20),YSUM( 20)CHEM 320
1 ,X(20),DELT(20),XLAN(20) CHEM 330
DIMENSION E(5),BB(5),Y(20) CHEM 340
DIMENSION EOLD(5) CHEM 350

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| | | |
|------|--|----------|
| 5 | CONTINUE | CHEM 360 |
| C | | CHEM 370 |
| | MA=1 | CHEM 380 |
| C | | CHEM 390 |
| C | ----- INITIAL GUESS FOR EQUILIBRIUM CALCULATIONS..... | CHEM 400 |
| C | | CHEM 410 |
| | DO 10 I=1,NS | CHEM 420 |
| 10 | Y(I) = CWall(I)*AMW(NI)/XMW(I) | CHEM 430 |
| C | | CHEM 440 |
| C | ----- COMPUTE THE SIZE OF THE MATRIX | CHEM 450 |
| C | | CHEM 460 |
| | NA=MM+1 | CHEM 470 |
| C | | CHEM 480 |
| C | NS=NUMBER OF SPECIES..... | CHEM 490 |
| C | | CHEM 500 |
| C | CRIT=CRITERIA FOR CONVERGENCE. | CHEM 510 |
| C | | CHEM 520 |
| | CRIT =TOL | CHEM 530 |
| | XBETA=CRIT | CHEM 540 |
| | BETA=0. | CHEM 550 |
| | LL=NS+1 | CHEM 560 |
| | TOLD=0.0 | CHEM 570 |
| C | | CHEM 580 |
| C | THE REMAINDER OF THE PROGRAM COMPUTES EQUILIBRIUM COMPOSITIONS | CHEM 590 |
| C | CORRESPONDING TO THE ELEMENTAL MASS FRACTIONS IN THE CC-ARRAY | CHEM 600 |
| C | FROM POINT N = NI TO POINT N = NF. | CHEM 610 |
| C | | CHEM 620 |
| | SUM = 0. | CHEM 630 |
| | DO 15 I=1,MM | CHEM 640 |
| 15 I | SUM = SUM + CC(I,1) | CHEM 650 |
| | DO 20 I=1,MM | CHEM 660 |
| 20 I | EOLD(I) = CC(I,1)/SUM | CHEM 670 |
| | DO 5000 N=NI,NF | CHEM 680 |
| | T = TT(N)*TD | CHEM 690 |
| | P=PP(N) | CHEM 700 |

| | |
|---|----------|
| BETOLD=0.0 | CHEM 710 |
| SUM=0.0 | CHEM 720 |
| DO 15 I=1,MM | CHEM 730 |
| 15 SUM=SUM + CC(I,N) | CHEM 740 |
| DO 20 I=1,MM | CHEM 750 |
| IF(CC(I,N).LT.1.0E-10)CC(I,N)=1.0E-10 | CHEM 760 |
| 20 E(I)=CC(I,N)/SUM | CHEM 770 |
| C | CHEM 780 |
| CALL ALTERY(E,EOLD,Y,TOLD) | CHEM 790 |
| C | CHEM 800 |
| TINCR=ABS(T-TOLD) | CHEM 810 |
| IF(TINCR.LE..01)GOTO1750 | CHEM 820 |
| 22 NT=1 | CHEM 830 |
| DO 25 J=1,MM | CHEM 840 |
| BB(J)=0.0 | CHEM 850 |
| DO 25 I=1,NS | CHEM 860 |
| 25 BB(J)=BB(J)+AA(I,J)*Y(I) | CHEM 870 |
| C | CHEM 880 |
| CALL THERMO(T,P) | CHEM 890 |
| C | CHEM 900 |
| C-----THERMO SUBPROGRAM CALCULATES F/RT FOR EACH COMPONENT..... | CHEM 910 |
| C | CHEM 920 |
| C | CHEM 930 |
| C-----SET-UP THE R-MATRIX AND THE B-VECTOR.... | CHEM 940 |
| C | CHEM 950 |
| 40 YBAR=0. | CHEM 960 |
| DO 50 I=1,NS | CHEM 970 |
| 50 YBAR=YBAR+Y(I) | CHEM 980 |
| C | CHEM 990 |
| C YBAR IS THE TOTAL NUMBER OF MOLES OF GAS SPECIES | CHEM1000 |
| C | CHEM1010 |
| C | CHEM1020 |
| C-----CALCULATE THE FREE ENERGY PARAMETER OF THE GAS SPECIES | CHEM1030 |
| C | CHEM1040 |
| DO 60 I=1,NS | CHEM1050 |

```

        FAC=Y(I)/YBAR
        IF(FAC.LT.1.E-73)FAC=1.E-73
60      FY(I)=Y(I)*(C(I)+ALOG(FAC))
C
C-----PROCEED TO CONSTRUCT THE R MATRIX
C
C-----INITIALIZE TO ZERO....
C
        DO75J=1,NA
        DO75K=1,NA
75      R(J,K)=0.0
C
        DO90J=1,MM
        DO90K=J,MM
        SUM=0.
        DO80I=1,NS
80      SUM=SUM+AA(I,J)*AA(I,K)*Y(I)
        R(J,K)=SUM
90      R(K,J)=SUM
C
        DO 103 K=1,MM
        SUM=0.
        DO 101 I=1,NS
101     SUM=SUM+AA(I,K)*Y(I)
        R(K,NA)=SUM
        R(NA,K)=SUM
103     CONTINUE
C
C ---PROCEED TO CALCULATE THE VECTOR B.
C
        DO140J=1,MM
        SUM=0.
        DO130I=1,NS
130     SUM=SUM+AA(I,J)*FY(I)
140     B(J,1)=SUM+BB(J)

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CHEM1060
CHEM1070
CHEM1080
CHEM1090
CHEM1100
CHEM1110
CHEM1120
CHEM1130
CHEM1140
CHEM1150
CHEM1160
CHEM1170
CHEM1180
CHEM1190
CHEM1200
CHEM1210
CHEM1220
CHEM1230
CHEM1240
CHEM1250
CHEM1260
CHEM1270
CHEM1280
CHEM1290
CHEM1300
CHEM1310
CHEM1320
CHEM1330
CHEM1340
CHEM1350
CHEM1360
CHEM1370
CHEM1380
CHEM1390
CHEM1400

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C
    SUM=0.
    DO 150 I=1,NS
150  SUM=SUM+FY(I)
    B(NA,1)=SUM
C
C-----MATRIX INVERSION IS CALLED TO PROVIDE THE SOLUTION FOR
C    THE LINEARIZED EQUATIONS. THE SOLUTION OF THE EQUATIONS
C    GIVES THE LAGRANGIAN MULTIPLIERS NEEDED TO COMPUTE THE MOLES
C    OF EACH GAS SPECIES.
C
    CALL MATINV(R,NA,B,MA,NMAX)
156  CONTINUE
C
    DO 160 I=1,NA
C
C    PI(I)=LAGRANGIAN MULTIPLIERS
C
160  PI(I)=B(I,1)
    U=PI(NA)
    XBAR=U*YBAR
C
C-----COMPUTE THE MOLFS OF EACH SPECIE....
C
    DO 170 I=1,NS
170  FSUM(I)=-FY(I)+(Y(I)/YBAR)*XBAR
    DO 200 I=1,NS
    PSUM=0.
    DO 180 J=1,MM
180  PSUM=PSUM+PI(J)*AA(I,J)
    YSUM(I)=PSUM*Y(I)
200  X(I)=FSUM(I)+YSUM(I)
C
C-----CHECK IF CONVERGENCE CRITERIA HAS BEEN MET. IF SO, GO TO 800
C

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CHEM1410
CHEM1420
CHEM1430
CHEM1440
CHEM1450
CHEM1460
CHEM1470
CHEM1480
CHEM1490
CHEM1500
CHEM1510
CHEM1520
CHEM1530
CHEM1540
CHEM1550
CHEM1560
CHEM1570
CHEM1580
CHEM1590
CHEM1600
CHEM1610
CHEM1620
CHEM1630
CHEM1640
CHEM1650
CHEM1660
CHEM1670
CHEM1680
CHEM1690
CHEM1700
CHEM1710
CHEM1720
CHEM1730
CHEM1740
CHEM1750

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| | |
|--|----------|
| BETA=0.0 | CHEM1760 |
| DO215 I=1,NS | CHEM1770 |
| DELT(I)=X(I)-Y(I) | CHEM1780 |
| 215 BETA=BETA+ABS(DELT(I)) | CHEM1790 |
| IF(BETA.GT.BETOLD)GOTO216 | CHEM1800 |
| IF(BETA.LT.XBETA)GOTO800 | CHEM1810 |
| 216 CONTINUE | CHEM1820 |
| C | CHEM1830 |
| C-----COMPLTE THE CONVERGENCE PARAMENTER XLAMBD | CHEM1840 |
| C | CHEM1850 |
| XLAMBD=1. | CHEM1860 |
| DO210 I=1,NS | CHEM1870 |
| IF(ABS(DELT(I)).LT.1.0E-20)DELT(I)=0.0 | CHEM1880 |
| IF(DELT(I).GE.0.)GOTO210 | CHEM1890 |
| IF(X(I).GT.0.)GOTO210 | CHEM1900 |
| XLAM(I)=-Y(I)/DELT(I) | CHEM1910 |
| XLAMBD=AMIN1(XLAMBD,XLAM(I)) | CHEM1920 |
| XLAMBD=.99*XLAMBD | CHEM1930 |
| 210 CONTINUE | CHEM1940 |
| XLAM1=XLAMBD | CHEM1950 |
| IF(XLAM1.EQ.0.)XLAM1=1.0E-5 | CHEM1960 |
| DEBAR=0. | CHEM1970 |
| DO220 I=1,NS | CHEM1980 |
| 220 DEBAR=DEBAR+DELT(I) | CHEM1990 |
| C | CHEM2000 |
| C-----DETERMINE THE SIZE OF THE UNIT VECTOR XLAMBD. | CHEM2010 |
| C | CHEM2020 |
| C APPLY THE CORRECTIONS TO OBTAIN A NEW SET OF ESTIMATES FOR THE | CHEM2030 |
| C NEXT ITERATION. WHEN THE VALUE OF XLAMBD IS VERY SMALL SET THE | CHEM2040 |
| C VALUES OF Y(I) EQUAL TO X(I) TO AVOID USING THE SAME VALUES OF | CHEM2050 |
| C Y(I) AS WAS USED IN THE PREVIOUS ITERATION | CHEM2060 |
| C | CHEM2070 |
| C | CHEM2080 |
| C-----DETERMINE THE FREE ENERGY GRADIENT. IF POSITIVE REDUCE XLAMBDA | CHEM2090 |
| C DFDL=FREE ENERGY GRADIENT | CHEM2100 |
| C | |

| | | |
|--------|--|----------|
| 230 | DFDL=0. | CHEM2110 |
| | DO280I=1,NS | CHEM2120 |
| | FAC=(Y(I)+XLAMB*DELT(I))/(YBAR+XLAMB*DEBAR) | CHEM2130 |
| | IF(FAC.LT.1.E-73)FAC=1.E-73 | CHEM2140 |
| C260 | DERP=(DELT(I)*YBAR-DEBAR*Y(I))/(YBAR+XLAMB*DEBAR) | CHEM2150 |
| C280 | DFDL=DFDL+DELT(I)*(C(I)+ALOG(FAC)) + DERP | CHEM2160 |
| 280 | DFDL=DFDL+DELT(I)*(C(I)+ALOG(FAC)) | CHEM2170 |
| | IF(DFDL.LT.0.000)GOTO300 | CHEM2180 |
| | XLAMB=.8*XLAMB | CHEM2190 |
| | IF(XLAMB.GT.1.E-08)GOTO230 | CHEM2200 |
| C | | CHEM2210 |
| C----- | THE VALUE OF XLAMB THAT ASSURES CONVERGENCE HAS BEEN FOUND | CHEM2220 |
| C | | CHEM2230 |
| 300 | DO350I=1,NS | CHEM2240 |
| | IF(XLAMB.GT.1.E-6)GOTO330 | CHEM2250 |
| | IF(DFDL.LT.0.)GOTO330 | CHEM2260 |
| | IF(XLAM1.LT.1.E-6)XLAM1=1.E-6 | CHEM2270 |
| C | | CHEM2280 |
| C----- | CALCULATE THE NEW COMPOSITION FOR THE NEXT ITERATION | CHEM2290 |
| C | | CHEM2300 |
| | Y(I)=Y(I)+XLAM1*DELT(I)*.1 | CHEM2310 |
| | GOTO340 | CHEM2320 |
| 330 | Y(I)=Y(I)+XLAMB*DELT(I) | CHEM2330 |
| 340 | IF(Y(I).LT.0.)Y(I)=1.E-73 | CHEM2340 |
| 350 | CONTINUE | CHEM2350 |
| C | | CHEM2360 |
| 532 | NT=NT+1 | CHEM2370 |
| | BETOLD=BETA | CHEM2380 |
| | TOLD=T | CHEM2390 |
| C | | CHEM2400 |
| C----- | IF THE NUMBER OF ITERATIONS EXCEED 900 STOP COMPUTATIONS | CHEM2410 |
| C | | CHEM2420 |
| | IF(NT.GT.900)GOTO6000 | CHEM2430 |
| | GOTO40 | CHEM2440 |
| 800 | CONTINUE | CHEM2450 |

| | |
|--|----------|
| C | CHEM2460 |
| C-----CONVERT Y(I) TO MOLE FRACTIONS..... | CHEM2470 |
| C | CHEM2480 |
| C CONVERT EQUILIBRIUM MOLE FRACTIONS TO MASS FRACTIONS AND STORE | CHEM2490 |
| C THESE VALUES IN THE CE-MATRIX. AMW(N) IS THE AVERAGE MOLECULAR | CHEM2500 |
| C WEIGHT AT THE POINT. N. | CHEM2510 |
| C | CHEM2520 |
| SUMY=0.0 | CHEM2530 |
| DO900 I=1,NS | CHEM2540 |
| 900 SUMY=SUMY+Y(I) | CHEM2550 |
| AMW(N) = 0.0 | CHEM2560 |
| DO1000 I=1,NS | CHEM2570 |
| Y(I)=Y(I)/SUMY | CHEM2580 |
| 1000 AMW(N) = AMW(N) + Y(I)*XMW(I) | CHEM2590 |
| DO1005 I=1,NS | CHEM2600 |
| 1005 CE(I,N) = Y(I)*XMW(I)/AMW(N) | CHEM2610 |
| GOTO1800 | CHEM2620 |
| C | CHEM2630 |
| C-----IF THE TEMPERATURE CHANGE IS LESS THAN 50 DEGREES FROM LAST | CHEM2640 |
| C F. E. M. CALCULATION ASSUME CURRENT VALUES AS EQUAL TO LAST | CHEM2650 |
| C VALUES... | CHEM2660 |
| C | CHEM2670 |
| 1750 DO1760 I=1,NS | CHEM2680 |
| 1760 CE(I,N)=CE(I,N-1) | CHEM2690 |
| AMW(N)=AMW(N-1) | CHEM2700 |
| 1800 CONTINUE | CHEM2710 |
| C | CHEM2720 |
| C OPTIONAL OUTPUT OF POSITION , TEMPERATURE AND EQUILIBRIUM | CHEM2730 |
| C COMPOSITIONS. | CHEM2740 |
| C | CHEM2750 |
| IF(NDEBUG.LT.1)GOTO3000 | CHEM2760 |
| PRINT 2000,P,T ,NT | CHEM2770 |
| 2000 FORMAT(/,,' P = ',F5.3,' T(OK) = ',F6.0,5X,'NUMBER OF ITERATIONS | CHEM2780 |
| X=',I3,/,11X,'Y(I)',12X,'C(I,N)',/) | CHEM2790 |
| PRINT 2005,(SP(I),Y(I),CE(I,N),I=1,NS) | CHEM2800 |

| | | |
|------|---|----------|
| 2005 | FORMAT(1X,A4,2E18.8) | CHEM2810 |
| C | | CHEM2820 |
| 3000 | CONTINUE | CHEM2830 |
| 3300 | XBETA=CRIT | CHEM2840 |
| 5000 | CONTINUE | CHEM2850 |
| 8000 | CONTINUE | CHEM2860 |
| | RETURN | CHEM2870 |
| 6000 | PRINT6001 | CHEM2880 |
| 6001 | FORMAT(' NUMBER OF ITERATIONS EXCEEDED 900, PROGRAM TERMINATING') | CHEM2890 |
| | RETURN | CHEM2900 |
| | END | CHEM2910 |

| | |
|--|----------|
| SUBROUTINE ALTERY(E,EOLD,Y,TOLD) | ALTE 10 |
| COMMON/WT/SMW(20),AWT(5) | ALTE 20 |
| COMMON/NUMBER/NSP,NNS,NE,NC | ALTE 30 |
| COMMON/EQ2/AA(20,5),ICODE(20) | ALTE 40 |
| COMMON/ELSP/LSP(5) | ALTE 50 |
| COMMON /SP1/SS,TOL,NDEBUG | ALTE 60 |
| DIMENSION E(5),Y(20),B(5),EOLD(5) | ALTE 70 |
| C | ALTE 80 |
| IF(NDEBUG.GT.1)PRINT101 | ALTE 90 |
| 101 FORMAT(' INITIAL GUESS ON MOLE FRACTIONS UPDATED') | ALTE 100 |
| C | ALTE 110 |
| C-----ASSUME ALL SPECIES HAVE THE SAME COMPOSITION..... | ALTE 120 |
| C | ALTE 130 |
| C | ALTE 140 |
| C-----COMPUTE GRAM-ATOMS OF EACH ELEMENT FROM KNOWN ELEMENTAL COMPOS | ALTE 150 |
| C DISTRIBUTION AND THE MAXIMUM POSSIBLE MOLECULAR WEIGHT..... | ALTE 160 |
| C | ALTE 170 |
| DO20J=1,NE | ALTE 180 |
| IF(E(J).GT.1.0E-8)GOTO20 | ALTE 190 |
| DO15I=1,NSP | ALTE 200 |
| IF(AA(I,J).LE.0.0)GOTO15 | ALTE 210 |
| Y(I)=1.0E-12 | ALTE 220 |
| 15 CONTINUE | ALTE 230 |
| 20 E(J)=E(J)*100./AWT(J) | ALTE 240 |
| C | ALTE 250 |
| C-----CALCULATE FOR EACH ELEMENT, THE NUMBER OF G-ATOMS BASED ON THE | ALTE 260 |
| C FIRST GUESS. ADJUST THE COMPOSITION OF EACH ELEMENT-SPECIE AS | ALTE 270 |
| C REQUIRED..... | ALTE 280 |
| C | ALTE 290 |
| DO30J=1,NE | ALTE 300 |
| R(J)=0.0 | ALTE 310 |
| DO30I=1,NSP | ALTE 320 |
| 30 B(J)=B(J)+AA(I,J)*Y(I) | ALTE 330 |
| DO40J=1,NE | ALTE 340 |
| EOLD(J) = E(J) | ALTE 350 |

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40  Y(LSP(J))=Y(LSP(J)) + (E(J)-B(J))  
    TOLD=0.0  
    RETURN  
    END
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ALTE 360  
ALTE 370  
ALTE 380  
ALTE 390
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C SUBROUTINE THERMO(T,P)
C

THER 10
THER 20
THER 30

| | | |
|---|------|-----|
| C-----SUBROUTINE THERMO CALCULATES THE FREE ENERGY FUNCTION FOR EACH | THER | 40 |
| C CHEMICAL SPECIE. | THER | 50 |
| C | THER | 60 |
| C | THER | 70 |
| COMMON/NUMBER/NQ ,NNS,NE,NC | THER | 80 |
| COMMON/EQ1/AI(20), BI(20), CI(20), DI(20), EI(20), FI(20), GI(20), | THER | 90 |
| X AII(20),BII(20),CII(20),DII(20),EII(20),FII(20),GII(20) | THER | 100 |
| COMMON/EQ2/AA(20,5),ICODE(20) | THER | 110 |
| COMMON/THERM1/C(20),FORT(20) | THER | 120 |
| C | THER | 130 |
| C T=TEMPERATURE IN OK | THER | 140 |
| C | THER | 150 |
| T1=T | THER | 160 |
| T2=T1*T | THER | 170 |
| T3=T2*T | THER | 180 |
| T4=T3*T | THER | 190 |
| T5=T4*T | THER | 200 |
| C | THER | 210 |
| C | THER | 220 |
| C-----CALCUALTE THE FREE ENERGY FUNCTION FCRT(I) | THER | 230 |
| C | THER | 240 |
| C FORT(I)=FREE ENERGY FUNCTION | THER | 250 |
| C | THER | 260 |
| DO 41 I=1,NQ | THER | 270 |
| IF(T.GT.6000.)GOTO6205 | THER | 280 |
| FORT(I)=AI(I)*(1.-ALOG(T))-BI(I)*T/2.-CI(I)*T2/6.-DI(I)*T3/12. | THER | 290 |
| 1 -FI(I)*T4/20.+FI(I)/T-GI(I) | THER | 300 |
| IF(ICODE(I).EQ.1)GOTO41 | THER | 310 |
| C(I)=FORT(I)+ALOG(P) | THER | 320 |
| GOTO41 | THER | 330 |
| 6205 FORT(I)=AII(I)*(1.-ALOG(T))-BII(I)*T/2.-CII(I)*T2/6.-DII(I)*T3/12. | THER | 340 |
| 1 -EII(I)*T4/20.+FII(I)/T-GII(I) | THER | 350 |
| IF(ICODE(I).EQ.1)GOTO41 | THER | 360 |
| C(I)=FORT(I)+ALOG(P) | THER | 370 |
| 41 CONTINUE | THER | 380 |

| | | |
|---|---|----------|
| | SUBROUTINE MATINV(A,N,B,M,NMAX) | MATI 10 |
| C | | MATI 20 |
| C | | MATI 30 |
| C | MATRIX INVERSION WITH ACCOMPANYING SOLUTION OF LINEAR EQUATIONS | MATI 40 |
| | DIMENSION A(7,7),B(7,1),IPIVOT(7),INDEX(7,2) | MATI 50 |
| | EQUIVALENCE (IROW,JROW), (ICOLUM,JCOLUM), (AMAX,T,SWAP) | MATI 60 |
| C | | MATI 70 |
| C | INITIALIZATION | MATI 80 |
| C | | MATI 90 |
| | 5 ISCALE=0 | MATI 100 |
| | 6 R1=10.0**18 | MATI 110 |
| | 7 R2=1.0/R1 | MATI 120 |
| | 10 DETERM=1.0 | MATI 130 |
| | 15 DO 20 J=1,N | MATI 140 |
| | 20 IPIVOT(J)=0 | MATI 150 |
| | 30 DO 550 I=1,N | MATI 160 |
| C | | MATI 170 |
| C | SEARCH FOR PIVOT ELEMENT | MATI 180 |
| C | | MATI 190 |
| | 40 AMAX=0.0 | MATI 200 |
| | 45 DO 105 J=1,N | MATI 210 |
| | 50 IF (IPIVOT(J)-1)60,105,60 | MATI 220 |
| | 60 DO 100 K=1,N | MATI 230 |
| | 70 IF (IPIVOT(K)-1)80,100,740 | MATI 240 |
| | 80 IF (ABS(AMAX)-ABS(A(J,K)))85,100,100 | MATI 250 |
| | 85 IROW=J | MATI 260 |
| | 90 ICOLUM=K | MATI 270 |
| | 95 AMAX=A(J,K) | MATI 280 |
| | 100 CONTINUE | MATI 290 |
| | 105 CONTINUE | MATI 300 |
| | IF (AMAX)110,106,110 | MATI 310 |
| | 106 DETERM=0.0 | MATI 320 |
| | ISCALE=0 | MATI 330 |
| | GO TO 740 | MATI 340 |
| | 110 IPIVOT(ICOLUM)=IPIVOT(ICOLUM)+1 | MATI 350 |

| | | |
|---|---|----------|
| C | | MATI 360 |
| C | INTERCHANGE ROWS TO PUT PIVOT ELEMENT ON DIAGONAL | MATI 370 |
| C | | MATI 380 |
| | 130 IF (IROW-ICOLUM)140,260,140 | MATI 390 |
| | 140 DETERM=-DETERM | MATI 400 |
| | 150 DO 200 L=1,N | MATI 410 |
| | 160 SWAP=A(IROW,L) | MATI 420 |
| | 170 A(IROW,L)=A(ICOLUM,L) | MATI 430 |
| | 200 A(ICOLUM,L)=SWAP | MATI 440 |
| | 205 IF(M)260,260,210 | MATI 450 |
| | 210 DO 250 L=1,M | MATI 460 |
| | 220 SWAP=B(IROW,L) | MATI 470 |
| | 230 B(IROW,L)=B(ICOLUM,L) | MATI 480 |
| | 250 B(ICOLUM,L)=SWAP | MATI 490 |
| | 260 INDEX(1,1)=IROW | MATI 500 |
| | 270 INDEX(1,2)=ICOLUM | MATI 510 |
| | 310 PIVOT=A(ICOLUM,ICOLUM) | MATI 520 |
| C | | MATI 530 |
| C | SCALE THE DETERMINANT | MATI 540 |
| C | | MATI 550 |
| | 1000 PIVOTI=PIVOT | MATI 560 |
| | 1005 IF(ABS(DETERM)-R1)1030,1010,1010 | MATI 570 |
| | 1010 DETERM=DETERM/R1 | MATI 580 |
| | ISCALE=ISCALE+1 | MATI 590 |
| | IF(ABS(DETERM)-R1)1060,1020,1020 | MATI 600 |
| | 1020 DETERM=DETERM/R1 | MATI 610 |
| | ISCALE=ISCALE+1 | MATI 620 |
| | GO TO 1060 | MATI 630 |
| | 1030 IF(ABS(DETERM)-R2)1040,1040,1060 | MATI 640 |
| | 1040 DETERM=DETERM*R1 | MATI 650 |
| | ISCALE=ISCALE-1 | MATI 660 |
| | IF(ABS(DETERM)-R2)1050,1050,1060 | MATI 670 |
| | 1050 DETERM=DETERM*R1 | MATI 680 |
| | ISCALE=ISCALE-1 | MATI 690 |
| | 1060 IF(ABS(PIVOTI)-R1)1090,1070,1070 | MATI 700 |

| | |
|--|-----------|
| 1070 PIVOTI=PIVOTI/R1 | MATI 710 |
| ISCALE=ISCALE+1 | MATI 720 |
| IF (ABS(PIVOTI)-R1) 320,1080,1080 | MATI 730 |
| 1080 PIVOTI=PIVOTI/R1 | MATI 740 |
| ISCALE=ISCALE+1 | MATI 750 |
| GO TO 320 | MATI 760 |
| 1090 IF (ABS(PIVOTI)-R2) 2000,2000,320 | MATI 770 |
| 2000 PIVOTI=PIVOTI*R1 | MATI 780 |
| ISCALE=ISCALE-1 | MATI 790 |
| IF (ABS(PIVOTI)-R2) 2010,2010,320 | MATI 800 |
| 2010 PIVOTI=PIVOTI*R1 | MATI 810 |
| ISCALE=ISCALE-1 | MATI 820 |
| 320 DETERM=DETERM*PIVOTI | MATI 830 |
| C | MATI 840 |
| C DIVIDE PIVOT ROW BY PIVOT ELEMENT | MATI 850 |
| C | MATI 860 |
| 330 A(ICOLUM,ICOLUM)=1.0 | MATI 870 |
| 340 DO 350 L=1,N | MATI 880 |
| 350 A(ICOLUM,L)=A(ICOLUM,L)/PIVOT | MATI 890 |
| 355 IF (M) 380,380,360 | MATI 900 |
| 360 DO 370 L=1,M | MATI 910 |
| 370 B(ICOLUM,L)=B(ICOLUM,L)/PIVOT | MATI 920 |
| C | MATI 930 |
| C REDUCE NON-PIVOT ROWS | MATI 940 |
| C | MATI 950 |
| 380 DO 550 L1=1,N | MATI 960 |
| 390 IF (L1-ICOLUM) 400,550,400 | MATI 970 |
| 400 T=A(L1,ICOLUM) | MATI 980 |
| 420 A(L1,ICOLUM)=0.0 | MATI 990 |
| 430 DO 450 L=1,N | MATI 1000 |
| 450 A(L1,L)=A(L1,L)-A(ICOLUM,L)*T | MATI 1010 |
| 455 IF (M) 550,550,460 | MATI 1020 |
| 460 DO 500 L=1,M | MATI 1030 |
| 500 B(L1,L)=B(L1,L)-B(ICOLUM,L)*T | MATI 1040 |
| 550 CONTINUE | MATI 1050 |


```

C
C      INTERCHANGE COLUMNS
600 DO 710 I=1,N
C
610 L=N+1-I
620 IF( INDEX(L,1)-INDEX(L,2))630,710,630
630 JROW=INDEX(L,1)
640 JCOLUM=INDEX(L,2)
650 DO 705 K=1,N
660 SWAP=A(K,JROW)
670 A(K,JROW)=A(K,JCOLUM)
700 A(K,JCOLUM)=SWAP
705 CONTINUE
710 CONTINUE
740 RETURN
      END

```

```

MATI1060
MATI1070
MATI1080
MATI1090
MATI1100
MATI1110
MATI1120
MATI1130
MATI1140
MATI1150
MATI1160
MATI1170
MATI1180
MATI1190
MATI1200
MATI1210

```

| | | | |
|---|--|------|-----|
| | SUBROUTINE PROPRT(NSP,NI,NF) | PROP | 10 |
| C | * * * * * | PROP | 20 |
| C | | PROP | 30 |
| C | SUBROUTINE FOR THE CALCULATION OF THERMODYNAMIC AND TRANSPORT | PROP | 40 |
| C | PROPERTIES | PROP | 50 |
| C | | PROP | 60 |
| C | NOMENCLATURE..... | PROP | 70 |
| C | Y(I)....MOLE FRACTION OF COMPONENT I | PROP | 80 |
| C | C(I,N)....MASS FRACTION OF COMPONENT I AT POINT N | PROP | 90 |
| C | T1.....TEMPERATURE, DEG. K | PROP | 100 |
| C | CP(I).....SPECIES SPECIFIC HEAT, CAL/GMOLE OF I-K | PROP | 110 |
| C | CPM(N)....MIXTURE SPECIFIC HEAT, CAL/GMOLE OF M-K | PROP | 120 |
| C | H(I).....SPECIES ENTHALPY, CAL/GMOLE OF I | PROP | 130 |
| C | HM(N)....MIXTURE ENTHALPY, CAL/GMOLE OF M | PROP | 140 |
| C | VIS(I)....SPECIES VISCOSITY, LBM/FT-SEC | PROP | 150 |
| C | VISM(N)....MIXTURE VISCOSITY, LBM/FT-SEC | PROP | 160 |
| C | TC(I).....SPECIES THERMAL CONDUCTIVITY, BTU/FT-SEC-DEG. R | PROP | 170 |
| C | TCM(N)....MIXTURE THERMAL CONDUCTIVITY, BTU/FT-SEC-DEG. R | PROP | 180 |
| | COMMON /BLOCK1/V1(20),V2(20),V3(20) | PROP | 190 |
| | COMMON/BLOCK3/K1(20),K2(20) | PROP | 200 |
| | COMMON/F01/AI(20), BI(20), CI(20), DI(20), EI(20), FI(20), GI(20), | PROP | 210 |
| X | AII(20),BII(20),CII(20),DII(20),EII(20),FII(20),GII(20) | PROP | 220 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2,RAD, RE, LXI, ITM, IEM,NETA | PROP | 230 |
| | COMMON/PROP1/PI(60),RHO(60),T(60),AMW(60),C (20,60),CC(5,60) | PROP | 240 |
| | COMMON/PROP2/VISM(60),RM(60),TCM(60) | PROP | 250 |
| | COMMON/PROP3/CP(20,60),H(20,60),CPM(60),HM(60) | PROP | 260 |
| | COMMON/WT/SMW(20),AWT(5) | PROP | 270 |
| | COMMON /RH/ DUD,DPHI,T0,RZB,PD,HD,HTOTAL | PROP | 280 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CPNF | PROP | 290 |
| | DIMENSION PHI(20,20),YPHI(20),Y(20),TC(20),VIS(20) | PROP | 300 |
| | DIMENSION SAMW(2) | PROP | 310 |
| | REAL MU,MUDZ,K1,K2 | PROP | 320 |
| | DATA R /1.98716/ | PROP | 330 |
| C | | PROP | 340 |
| | DO500N=NI,NF | PROP | 350 |

| | |
|--|----------|
| T1= T(N) *TD | PROP 360 |
| T2=T1*T1 | PROP 370 |
| T3=T2*T1 | PROP 380 |
| T4=T3*T1 | PROP 390 |
| T5=T4*T1 | PROP 400 |
| DO150I=1,NSP | PROP 410 |
| IF(T1.GT.6000.)GOTO 50 | PROP 420 |
| CP(I,N)=(AI(I)+ BI(I)*T1+ CI(I)*T2+ DI(I)*T3+ EI(I)*T4)*R | PROP 430 |
| H(I,N)=(AI(I)*T1+ BI(I)*T2/2.+ CI(I)*T3/3.+ DI(I)*T4/4. | PROP 440 |
| X + EI(I)*T5/5.+ FI(I))*R | PROP 450 |
| GOTO 100 | PROP 460 |
| 50 CP(I,N)=(AII(I)+BII(I)*T1+CII(I)*T2+DII(I)*T3+EII(I)*T4)*R | PROP 470 |
| H(I,N)=(AII(I)*T1+BII(I)*T2/2.+CII(I)*T3/3.+DII(I)*T4/4. | PROP 480 |
| X +EII(I)*T5/5.+FII(I))*R | PROP 490 |
| 100 Y(I)=C(I,N)*AMW(N)/SMW(I) | PROP 500 |
| VIS(I)=(V1(I) + V2(I)*T1 + V3(I)*T2)*1.0E-05 | PROP 510 |
| TC(I)=(K1(I)+K2(I)*T1)*1.0E-5 | PROP 520 |
| 150 CONTINUE | PROP 530 |
| C-----CALCULATE PHI(I,J) PARAMETERS FOR MIXTURE PROPERTIES.... | PROP 540 |
| C | PROP 550 |
| DO200I=1,NSP | PROP 560 |
| DO200J=1,NSP | PROP 570 |
| VIS12=SQRT(VIS(I)/VIS(J)) | PROP 580 |
| SMW14=(SMW(J)/SMW(I))*0.25 | PROP 590 |
| PHI(I,J)=.354*((1.+VIS12*SMW14)**2)/SQRT(1.+SMW(I)/SMW(J)) | PROP 600 |
| 200 CONTINUE | PROP 610 |
| C | PROP 620 |
| C-----CALCULATION OF MIXTURE PROPERTIES... | PROP 630 |
| C | PROP 640 |
| DO250I=1,NSP | PROP 650 |
| YPHI(I)=0.0 | PROP 660 |
| DO250J=1,NSP | PROP 670 |
| 250 YPHI(I)=YPHI(I) + Y(J)*PHI(I,J) | PROP 680 |
| VISM(N)=0.0 | PROP 690 |
| TCM(N) = 0. | PROP 700 |

| | |
|---|----------|
| DO 300 I=1, NSP | PROP 710 |
| IF (Y(I)).LT.1.E-5) GO TO 300 | PROP 720 |
| VISM(N)=VISM(N)+Y(I)*VIS(I)/YPHI(I) | PROP 730 |
| 300 TCM(N) = TCM(N) + Y(I)*TC(I)/YPHI(I) | PROP 740 |
| 500 CONTINUE | PROP 750 |
| C | PROP 760 |
| C----- NONDIMENSIONALIZE QUANTITIES ----- | PROP 770 |
| C | PROP 780 |
| DO 550 N=NI,NF | PROP 790 |
| VISM(N) = VISM(N)/MUDZ | PROP 800 |
| TCM(N) = TCM(N)*AKNF | PROP 810 |
| RM(I)=RHO(N)*VISM(N) | PROP 820 |
| CPM(N) = 0. | PROP 830 |
| HM(N) = .0. | PROP 840 |
| DO 550 I=1, NSP | PROP 850 |
| CP(I,N) = CP(I,N)*CPNF/SMW(I) | PROP 860 |
| H(I,N) = H(I,N)*HNF*1.80/SMW(I) | PROP 870 |
| CPM(N) = CPM(N) + CP(I,N)*C(I,N) | PROP 880 |
| HM(N) = HM(N) + H(I,N)*C(I,N) | PROP 890 |
| 550 CONTINUE | PROP 900 |
| RETURN | PROP 910 |
| END | PROP 920 |

| | | |
|---|--|----------|
| | SUBROUTINE STPSIZE | STPS 10 |
| C | | STPS 20 |
| C | ** ROUTINE TO ADJUST STEP SIZE AS NEEDED | STPS 30 |
| C | TO MAINTAIN ACCURACY ** | STPS 40 |
| C | | STPS 50 |
| C | | STPS 60 |
| | COMMON /DEL/ DELTA,DTIL,DTILS | STPS 70 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | STPS 80 |
| | COMMON/PROP1/PI(60),RHO(60),G (60),AMW(60),C (20,60),EC(5,60) | STPS 90 |
| | COMMON/PROP2/ MU(60),RM(60), AK(60) | STPS 100 |
| | COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | STPS 110 |
| | COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | STPS 120 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | STPS 130 |
| | COMMON /VEL/ F(60),FC(60),Z(60),V(60) | STPS 140 |
| | COMMON/WALL/RVW,PRW,TWOLD,FLUX(20),CWALL(20),ECWALL(5) | STPS 150 |
| | COMMON /YL/ETA(60),YOND(60) | STPS 160 |
| | COMMON /OLD/ TOLD(60),EOLD(60),RHOS(60) | STPS 170 |
| C | | STPS 180 |
| | N=2 | STPS 190 |
| 1 | CONTINUE | STPS 200 |
| | I2=2 | STPS 210 |
| | IF(NETA.GE. 59) GO TO 5 | STPS 220 |
| C | | STPS 230 |
| | DO 2 I=N,NETA | STPS 240 |
| | L=I | STPS 250 |
| | CHECK = ABS(G(I)-G(I-1)) | STPS 260 |
| | IF(CHECK .GT. .05) GO TO 3 | STPS 270 |
| 2 | CONTINUE | STPS 280 |
| C | | STPS 290 |
| | GO TO 5 | STPS 300 |
| C | | STPS 310 |
| 3 | CONTINUE | STPS 320 |
| | M = NETA - L + 1 | STPS 330 |
| C | | STPS 340 |
| | DO 4 I=1,M | STPS 350 |

| | | |
|-----|---|----------|
| | K = NETA - I + 1 | STPS 360 |
| | G(K+1) = G(K) | STPS 370 |
| | F(K+1) = F(K) | STPS 380 |
| | DO100JJ=1,5 | STPS 390 |
| 100 | EC(JJ,K+1)=EC(JJ,K) | STPS 400 |
| | TOLD(K+1)=TOLD(K) | STPS 410 |
| | RHOS(K+1)=RHOS(K) | STPS 420 |
| | RHO(K+1) = RHO(K) | STPS 430 |
| | RM (K+1) = RM (K) | STPS 440 |
| | IF(IRAD.EQ. 3) E(K+1) = E(K) | STPS 450 |
| | ETA(K+1) = ETA(K) | STPS 460 |
| 4 | CONTINUE | STPS 470 |
| C | | STPS 480 |
| | G(L) = (G(L-1) + G(L+1)) / 2.0 | STPS 490 |
| | F(L) = (F(L-1)+F(L+1))/2. | STPS 500 |
| | DO101JJ=1,5 | STPS 510 |
| 101 | EC(JJ,L)=(EC(JJ,L-1)+EC(JJ,L+1))/2. | STPS 520 |
| | TOLD(L)=(TOLD(L-1)+TOLD(L+1))/2.0 | STPS 530 |
| | RHOS(L)=(RHOS(L-1)+RHOS(L+1))/2.0 | STPS 540 |
| | RHO(L) = (RHO(L-1) +RHO(L+1))/2. | STPS 550 |
| | RM (L) = (RM (L-1) +RM (L+1))/2. | STPS 560 |
| | IF(IRAD.EQ. 3) E(L) =(E(L-1)+ E(L+1))/2.0 | STPS 570 |
| | ETA(L) = (ETA(L-1) + ETA(L+1))/2.0 | STPS 580 |
| | NETA = NETA + 1 | STPS 590 |
| | N=L | STPS 600 |
| C | | STPS 610 |
| | IF(NETA .LT. 59) GO TO 1 | STPS 620 |
| C | | STPS 630 |
| 5 | CONTINUE | STPS 640 |
| C | | STPS 650 |
| | IF(I2 .GE. NETA) GC TO 10 | STPS 660 |
| | DO 6 I=I2,NETA,2 | STPS 670 |
| | L=I | STPS 680 |
| | IF (L.EQ.NETA) GO TO 6 | STPS 690 |
| | IF(ETA(I).EQ. 0.98) GO TC 6 | STPS 700 |

| | | |
|-----|---------------------------------------|----------|
| | CHECK =ABS(G(I+1) - G(I-1)) | STPS 710 |
| | IF(CHECK .LT. 0.005) GO TO 7 | STPS 720 |
| 6 | CONTINUE | STPS 730 |
| C | | STPS 740 |
| | GO TO 10 | STPS 750 |
| 7 | CONTINUE | STPS 760 |
| | I2=L+1 | STPS 770 |
| | IF(ETA(L+1)-ETA(L-1).GT. .04) GO TO 5 | STPS 780 |
| C | | STPS 790 |
| | DO 8 I=L,NETA | STPS 800 |
| | G(I) = G(I+1) | STPS 810 |
| | F(I) = F(I+1) | STPS 820 |
| | DO102JJ=1,5 | STPS 830 |
| 102 | EC(JJ,I)=EC(JJ,I+1) | STPS 840 |
| | TOLD(I)=TOLD(I+1) | STPS 850 |
| | RHOS(I)=RHOS(I+1) | STPS 860 |
| | RHO(I) = RHO(I+1) | STPS 870 |
| | RM (I) = RM (I+1) | STPS 880 |
| | IF(IRAD.EQ. 3) E(I) =E(I+1) | STPS 890 |
| | ETA(I)=ETA(I+1) | STPS 900 |
| 8 | CONTINUE | STPS 910 |
| C | | STPS 920 |
| | NETA=NETA-1 | STPS 930 |
| | IF (NETA .GT. 1) GO TO 5 | STPS 940 |
| C | | STPS 950 |
| 10 | CONTINUE | STPS 960 |
| | NN = NETA-2 | STPS 970 |
| | DO 20 I=1,NN | STPS 980 |
| | Z(I) = ETA(I+1)/DTIL | STPS 990 |
| 20 | CONTINUE | STPS1000 |
| | RETURN | STPS1010 |
| | END | STPS1020 |

| | |
|--|----------|
| SUBROUTINE TRID (M) | TRID 10 |
| C**** TRID --TRIDIAGONAL EQUATION SOLVER OBTAINED FROM CONTE P-184 *** | TRID 20 |
| C SUBROUTINE SOLVES AX = B FOR THE VECTOR X (WHERE A IS TRIDIAGONAL) | TRID 30 |
| C M = ORDER OF SYSTEM | TRID 40 |
| C SUP = SUPER DIAGONAL OF A | TRID 50 |
| C SUB = SUB DIAGONAL OF A | TRID 60 |
| C DIAG = MAIN DIAGONAL OF A | TRID 70 |
| C B = CONSTANT VECTOR | TRID 80 |
| C SUP AND DIAG ARE DESTROYED | TRID 90 |
| C SOLUTION VECTOR IS RETURNED IN B | TRID 100 |
| C | TRID 110 |
| COMMON/VECTOR/ SUB(60),DIAG(60),SUP(60),B(60) | TRID 120 |
| C | TRID 130 |
| N = M | TRID 140 |
| NN = N -1 | TRID 150 |
| SUP(1) = SUP(1)/DIAG(1) | TRID 160 |
| B(1) = B(1)/DIAG(1) | TRID 170 |
| DO 10 I=2,N | TRID 180 |
| II = I -1 | TRID 190 |
| C-----DECOMPOSE A TO FORM A = LU WHERE L IS LOWER TRIANGULAR, | TRID 200 |
| C AND U IS UPPER TRIANGULAR ----- | TRID 210 |
| DIAG(I) = DIAG(I) - SUP(II)*SUB(II) | TRID 220 |
| IF(I .EQ. N) GO TO 10 | TRID 230 |
| SUP(I) = SUP(I) / DIAG(I) | TRID 240 |
| C-----COMPUTE Z WHERE LZ = B | TRID 250 |
| 10 B(I) = (B(I) - SUB(II) *B(II))/ DIAG(I) | TRID 260 |
| C-----COMPUTE X BY BACK SUBSTITUTION WHERE UX = Z | TRID 270 |
| DO 20 K =1,NN | TRID 280 |
| I = N - K | TRID 290 |
| 20 B(I) = B(I) -SUP(I) *B(I+1) | TRID 300 |
| RETURN | TRID 310 |
| END | TRID 320 |


```

C      FUNCTION QUAD (X,FX,I)
C
C      **  TRAPEZOIDAL QUADRATURE FUNCTION  **
C
      DIMENSION X(60),FX(60)
      DX=X(I)-X(I-1)
      QUAD = (DX/2.0) * (FX(I) + FX(I-1) )
      RETURN
      END

```

```

QU      10
QU      20
QU      30
QU      40
QU      50
QU      60
QU      70
QU      80
QU      90

```

| | |
|--|----------|
| SUBROUTINE EFLUX | EFLU 10 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NETA | EFLU 20 |
| COMMON /RH/ DUD,DPHI,TD,RZB,PD,HD,HTOTAL | EFLU 30 |
| COMMON /RFLUX/ E(60),IRAD,ITYPE | EFLU 40 |
| COMMON/PROPI/PI(60),RHO(60), T(60),AMW(60),C (20,60),CC(5,60) | EFLU 50 |
| DIMENSION EOLD(60) | EFLU 60 |
| DO 100 I=1,NETA | EFLU 70 |
| IF(IEM.GT.1) EOLD(I) = E(I) | EFLU 80 |
| PL = ALOG10(PI(I)) | EFLU 90 |
| TSI= T(I)*TD | EFLU 100 |
| TS = 1100. * PL +13800. | EFLU 110 |
| IF (TS -TSI) 300,200,200 | EFLU 120 |
| 200 EP = 10.**(.0005 *TSI +1.15*PL -3.15) | EFLU 130 |
| GO TO 350 | EFLU 140 |
| 300 EP = 10.** (1.875 *PL + 3.903) | EFLU 150 |
| C **** EP HAS UNITS OF WATTS/CM**3 *** | EFLU 160 |
| C | EFLU 170 |
| 350 E(I) =(EP*R/(RINF *UINF2 *U INF))* 20866.0 | EFLU 180 |
| 400 E(I) =E(I) *RZB | EFLU 190 |
| C IF(IEM.GT.1) E(I) = .6 *E(I) +.4 *EOLD(I) | EFLU 200 |
| C | EFLU 210 |
| C **** E IS NONDIMENSIONAL *** | EFLU 220 |
| C | EFLU 230 |
| 100 CONTINUE | EFLU 240 |
| RETURN | EFLU 250 |
| END | EFLU 260 |

```
FUNCTION C1 (DX,DX1)  
C1 = DX1/DX/(DX+DX1)  
RETURN  
END
```

| | |
|----|----|
| C1 | 10 |
| C1 | 20 |
| C1 | 30 |
| C1 | 40 |

```
FUNCTION C2 (DX,DX1)
C2= (DX-DX1)/DX/DX1
RETURN
END
```

```
C2    10
C2    20
C2    30
C2    40
```

```
FUNCTION C3 (DX,DX1)
C3=-DX/DX1/(DX+DX1)
RETURN
END
```

| | |
|----|----|
| C3 | 10 |
| C3 | 20 |
| C3 | 30 |
| C3 | 40 |

```
FUNCTION C4 (DX,DX1)
C4=2.00/DX/(DX+DX1)
RETURN
END
```

| | |
|----|----|
| C4 | 10 |
| C4 | 20 |
| C4 | 30 |
| C4 | 40 |

```
FUNCTION C5 (DX,DX1)
C5=-2.00/DX/DX1
RETURN
END
```

```
C5      10
C5      20
C5      30
C5      40
```

```
FUNCTION C6 (DX,DX1)
C6=2*D0/DX1/(DX+DX1)
RETURN
END
```

```
C6      10
C6      20
C6      30
C6      40
```


| | |
|---|----------|
| BLOCK DATA | DATA 10 |
| COMMON/FINV/ NHVL,NIHVC,FHVC(12),DJ(9),HVJ(9),ZKZ | DATA 20 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NETA | DATA 30 |
| COMMON/GUESS/TG1(60),TG2(60) | DATA 40 |
| COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),EC(5,60) | DATA 50 |
| COMMON/PROP2/ MU(60),RM(60), AK(60) | DATA 60 |
| COMMON/PROP3/CPS(20,60),HS(20,60),CP (60),HM(60) | DATA 70 |
| COMMON/NUMBER/NSP,NNS,NE,NC | DATA 80 |
| COMMON/ELSP/LSP(5) | DATA 90 |
| COMMON/ID/SP(20),EL(5) | DATA 100 |
| COMMON/WT/SMW(20),AWT(5) | DATA 110 |
| COMMON /BLOCK1/V1(20),V2(20),V3(20) | DATA 120 |
| COMMON/BLOCK3/K1(20),K2(20) | DATA 130 |
| COMMON/EQ1/AI(20), BI(20), CI(20), DI(20), EI(20), FI(20), GI(20), | DATA 140 |
| X AII(20),RII(20),CII(20),DII(20),EII(20),FII(20),GII(20) | DATA 150 |
| COMMON/EQ2/AA(20,5),ICODE(20) | DATA 160 |
| COMMON/EQ3/IA(20,5) | DATA 170 |
| REAL K1,K2 | DATA 180 |
| COMMON/CK/ISN(20),MWT(19) | DATA 190 |
| REAL MWT | DATA 200 |
| DATA ISN/ | DATA 210 |
| 1 11, 8, 19, 12, 20, 14, 13, 18, 10, 9, 2, 4, 6, | DATA 220 |
| 2 3, 5, 7, 15, 16, 17, 1/ | DATA 230 |
| DATA MWT/ | DATA 240 |
| 1 28.011, 12.011, 24.022, 36.033, 12.011, 25.030, 26.019, 27.027, | DATA 250 |
| 2 02.016, 01.008, 28.016, 14.008, 14.008, 16.000, 16.000, .0005486, | DATA 260 |
| 3 26.038, 37.041, 49.052/ | DATA 270 |
| DATA NETA/Q/ | DATA 280 |
| DATA RHO /25.1,14.3,8.85,6.50,4.37,3.01,2.49,2.17,1.90,1.67,1.46, | DATA 290 |
| 1 1.29,1.16,1.08,1.03,1.00,44*1.0/ | DATA 300 |
| DATA RM /10.0,7.71,5.89,5.10,4.18,3.54,3.31,3.10,2.83,2.48,2.09, | DATA 310 |
| 1 1.72,1.42,1.22,1.09,1.02,44*1.0/ | DATA 320 |
| DATA TG1 / .1033,.2294,.3531,.4719,.5777,.6531,.6867,.7034,.7145, | DATA 330 |
| 1 .7236,.7321,.7401,.7479,.7554,.7628,.7699,.7769,.7836,.7902, | DATA 340 |
| 2 .7967,.8030,.8092,.8153,.8213,.8272,.8331,.8389,.8447,.8504, | DATA 350 |

| | | | |
|---|---|--|----------|
| | 3 | .8562..8619..8676..8734..8791..8850..8908..8968..9028..9089, | DATA 360 |
| | 4 | .9151..9215..9280..9347..9417..9488..9563..9641..9723..9809, | DATA 370 |
| | 5 | .9901.10*1.0/ | DATA 380 |
| | | DATA TG2 / .3325..3325..3325..3325..3325..3325..3325..3326..3328, | DATA 390 |
| | 1 | .3331..3336..3344..3357..3378..3408..3452..3515..3601..3718, | DATA 400 |
| | 2 | .3873..4076..4335..4665..5075..5560..6054..6487..6857..7161, | DATA 410 |
| | 3 | .7404..7595..7749..7878..7993..8100..8203..8302..8399..8496, | DATA 420 |
| | 4 | .8594..8693..8797..8904..9019..9142..9278..9476..9609..9757, | DATA 430 |
| | 5 | .9877.10*1.0/ | DATA 440 |
| C | | | DATA 450 |
| | | DATA NSP,NNS,NE,NC/20.0,5,20/ | DATA 460 |
| | | DATA SP/ 'O2 ' , 'N2 ' , 'O ' , 'N ' , 'O+ ' , | DATA 470 |
| | 1 | 'N+ ' , 'E- ' , 'C ' , 'H ' , 'H2 ' , | DATA 480 |
| | 2 | 'CO ' , 'C3-G' , 'CN ' , 'C2H ' , 'C2H2' , | DATA 490 |
| | 3 | 'C3H ' , 'C4H ' , 'HCN ' , 'C2 ' , 'C+ ' / | DATA 500 |
| C | | | DATA 510 |
| | | DATA SMW/32.000 , 28.016 , 16.000 , 14.008 , 16.000 , | DATA 520 |
| | 1 | 14.008 , 5.486E-4 , 12.011 , 1.008 , 2.016 , | DATA 530 |
| | 2 | 28.011 , 36.033 , 26.019 , 25.030 , 26.038 , | DATA 540 |
| | 3 | 37.041 , 49.052 , 27.027 , 24.022 , 12.011/ | DATA 550 |
| C | | | DATA 560 |
| | | DATA V1/0.1693E 01,0.9704E 00,0.1519E 01,0.2534E 00,0.0 , | DATA 570 |
| | 1 | 0.0 ,0.0 ,0.1997E 01,0.2941E 00,-.7944E-01, | DATA 580 |
| | 2 | 0.2404E 01,0.2019E 01,0.2404E 01,0.2404E 01,0.1396E 01, | DATA 590 |
| | 3 | 0.2019E 01,0.2019E 01,0.1378E 01,0.1931E 01,0.0 / | DATA 600 |
| C | | | DATA 610 |
| | | DATA V2/0.1496E-02,0.1613E-02,0.1875E-02,0.2206E-02,0.5000E-03, | DATA 620 |
| | 1 | 0.5000E-03,0.5000E-03,0.1772E-02,0.8893E-03,0.7907E-03, | DATA 630 |
| | 2 | 0.1363E-02,0.1179E-02,0.1363E-02,0.1363E-02,0.8423E-03, | DATA 640 |
| | 3 | 0.1179E-02,0.1179E-02,0.9651E-03,0.1393E-02,0.5000E-03/ | DATA 650 |
| C | | | DATA 660 |
| | | DATA V3/-0.2276E-07,-0.1916E-07,-0.2228E-07,-0.3737E-07,-0.1000E-07, | DATA 670 |
| | 1 | -0.1000E-07,-0.1000E-07,-0.3378E-07,-0.8111E-08,-0.8864E-08, | DATA 680 |
| | 2 | -0.2184E-07,-0.1655E-07,-0.2184E-07,-0.2184E-07,-0.6939E-08, | DATA 690 |
| | 3 | -0.1655E-07,-0.1655E-07,-0.9481E-08,-0.2575E-07,-0.1000E-07/ | DATA 700 |

| | | | | | | | | |
|---|------|---------------|------------|------------|------------|-------------|-----|----------|
| C | | | | | | | | DATA 710 |
| | DATA | AI/0.3316E | 01.0.3221E | 01.0.2670E | 01.0.2474E | 01.0.2491E | 01. | DATA 720 |
| | 1 | 0.2727E | 01.0.2500E | 01.0.2612E | 01.0.2500E | 01.0.3358E | 01. | DATA 730 |
| | 2 | 0.3254E | 01.0.4002E | 01.0.3411E | 01.0.3485E | 01.0.3891E | 01. | DATA 740 |
| | 3 | 0.3965E | 01.0.5874E | 01.0.3654E | 01.0.4443E | 01.0.2609E | 01/ | DATA 750 |
| C | | | | | | | | DATA 760 |
| | DATA | BI/0.1151E-02 | 0.5878E-03 | -.1970E-03 | 0.9097E-04 | 0.2762E-04 | | DATA 770 |
| | 1 | -.2820E-03 | 0.3440E-06 | -.2030E-03 | -.8243E-06 | 0.2794E-03 | | DATA 780 |
| | 2 | 0.9698E-03 | 0.3541E-02 | 0.4897E-03 | 0.3563E-02 | 0.5717E-02 | | DATA 790 |
| | 3 | 0.6200E-02 | 0.7403E-02 | 0.3444E-02 | -.2885E-03 | -.1393E-03/ | | DATA 800 |
| C | | | | | | | | DATA 810 |
| | DATA | CI/-.3726E-06 | -.2907E-06 | 0.7193E-07 | -.7814E-07 | -.1881E-07 | | DATA 820 |
| | 1 | 0.1105E-06 | -.1954E-09 | 0.1095E-06 | 0.6421E-09 | 0.9372E-07 | | DATA 830 |
| | 2 | -.2647E-06 | -.1318E-05 | 0.1005E-06 | -.1237E-05 | -.1957E-05 | | DATA 840 |
| | 3 | -.2265E-05 | -.2729E-05 | -.1258E-05 | 0.3036E-06 | 0.5959E-07/ | | DATA 850 |
| C | | | | | | | | DATA 860 |
| | DATA | DI/0.6186E-10 | 0.3938E-10 | -.8901E-11 | 0.2218E-10 | 0.3807E-11 | | DATA 870 |
| | 1 | -.1551E-10 | 0.3937E-13 | -.1695E-10 | -.1720E-12 | -.2948E-10 | | DATA 880 |
| | 2 | 0.3037E-10 | 0.2064E-09 | -.3473E-10 | 0.1866E-09 | 0.2931E-09 | | DATA 890 |
| | 3 | 0.3717E-09 | 0.4437E-09 | 0.2169E-09 | -.6244E-10 | -.1037E-10/ | | DATA 900 |
| C | | | | | | | | DATA 910 |
| | DATA | EI/-.3666E-14 | -.2000E-14 | 0.4002E-15 | -.1489E-14 | -.1028E-15 | | DATA 920 |
| | 1 | 0.7847E-15 | -.2573E-17 | 0.8590E-15 | 0.1457E-16 | 0.2141E-14 | | DATA 930 |
| | 2 | -.1177E-14 | -.1144E-13 | 0.2361E-14 | -.1013E-13 | -.1585E-13 | | DATA 940 |
| | 3 | -.2262E-13 | -.2637E-13 | -.1430E-13 | 0.3915E-14 | 0.6345E-15/ | | DATA 950 |
| C | | | | | | | | DATA 960 |
| | DATA | FI/-.1044E | 04.0.1043E | 04.0.2915E | 05.0.5609E | 05.0.1879E | 06. | DATA 970 |
| | 1 | 0.2254E | 06.0.7450E | 03.0.8542E | 05.0.2547E | 05.0.1018E | 04. | DATA 980 |
| | 2 | -.1434E | 05.0.9423E | 05.0.4745E | 05.0.5809E | 05.0.2590E | 05. | DATA 990 |
| | 3 | 0.6283E | 05.0.7605E | 05.0.1442E | 05.0.9787E | 05.0.2168E | 06/ | DATA1000 |
| C | | | | | | | | DATA1010 |
| | DATA | GI/0.5393E | 01.0.4326E | 01.0.4504E | 01.0.4300E | 01.0.4424E | 01. | DATA1020 |
| | 1 | 0.3645E | 01.0.1173E | 02.0.4144E | 01.0.4612E | 00.0.3548E | 01. | DATA1030 |
| | 2 | 0.4875E | 01.0.2020E | 01.0.4746E | 01.0.4784E | 01.0.6520E | 00. | DATA1040 |
| | 3 | 0.3467E | 01.0.4010E | 01.0.2373E | 01.0.1090E | 01.0.3709E | 01/ | DATA1050 |

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| C | DATA AII/0.3721E 01,0.3727E 01,0.2548E 01,0.2746E 01,0.2944E 01, | DATA1060 |
| | 1 0.2499E 01,0.2508E 01,0.2141E 01,0.3934E 01,0.3363E 01, | DATA1070 |
| | 2 0.3366E 01,0.2213E 02,0.3473E 01,0.5307E 01,0.6789E 01, | DATA1080 |
| | 3 0.3965E 01,0.5874E 01,0.3654E 01,0.4026E 01,0.2528E 01/ | DATA1090 |
| C | DATA BII/0.4254E-03,0.4684E-03,-.5952E-04,-.3909E-03,-.4108E-03, | DATA1100 |
| | 1 -.3725E-05,-.6332E-05,0.3219E-03,-.1776E-02,0.4656E-03, | DATA1110 |
| | 2 0.8027E-03,-.1759E-01,0.7337E-03,0.8966E-03,0.1503E-02, | DATA1120 |
| | 3 0.6200E-02,0.7403E-02,0.3444E-02,0.4857E-03,0.4869E-05/ | DATA1130 |
| C | DATA CII/-,2835E-07,-.1140E-06,0.2701E-07,0.1338E-06,0.9156E-07, | DATA1140 |
| | 1 0.1147E-07,0.1364E-08,-.5498E-07,0.6013E-06,-.5127E-07, | DATA1150 |
| | 2 -.1968E-06,0.5565E-05,-.9088E-07,-.1378E-06,-.2295E-06, | DATA1160 |
| | 3 -.2265E-05,-.2729E-05,-.1258E-05,-.7026E-07,-.7026E-08/ | DATA1170 |
| C | DATA DII/0.6050E-12,0.1154E-10,-.2798E-11,-.1191E-10,-.5848E-11, | DATA1180 |
| | 1 -.1102E-11,-.1094E-12,0.3604E-11,-.7819E-10,0.2802E-11, | DATA1190 |
| | 2 0.1940E-10,-.6758E-09,0.4847E-11,0.9251E-11,0.1534E-10, | DATA1200 |
| | 3 0.3717E-09,0.4437E-09,0.2169E-09,0.4666E-11,0.1134E-11/ | DATA1210 |
| C | DATA FII/-,5186E-17,-.3293E-15,0.9380E-16,0.3369E-15,0.1190E-15, | DATA1220 |
| | 1 0.3078E-16,0.2934E-17,-.5564E-16,0.3482E-14,-.4905E-16, | DATA1230 |
| | 2 -.5549E-15,0.2825E-13,-.1018E-15,-.2278E-15,-.3763E-15, | DATA1240 |
| | 3 -.2262E-13,-.2637E-13,-.1430E-13,-.1142E-15,-.3476E-16/ | DATA1250 |
| C | DATA FII/-,1044E 04,-.1043E 04,0.2915E 05,0.5609E 05,0.1879E 06, | DATA1260 |
| | 1 0.2254E 06,-.7450E 03,0.8542E 05,0.2547E 05,-.1018E 04, | DATA1270 |
| | 2 -.1434E 05,0.9423E 05,0.5420E 05,0.5809E 05,0.2590E 05, | DATA1280 |
| | 3 0.6283E 05,0.7605E 05,0.1442E 05,0.9787E 05,0.2168E 06/ | DATA1290 |
| C | DATA GII/0.3254E 01,0.1294E 01,0.5049E 01,0.2872E 01,0.1750E 01, | DATA1300 |
| | 1 0.4950E 01,-.1208E 02,0.6874E 01,-.8598E 01,-.3716E 01, | DATA1310 |
| | 2 0.4263E 01,-.1021E 03,0.4152E 01,-.5288E 01,-.1539E 02, | DATA1320 |
| | 3 0.3467E 01,-.4010E 01,0.2373E 01,0.1090E 01,0.4139E 01/ | DATA1330 |
| | | DATA1340 |
| | | DATA1350 |
| | | DATA1360 |
| | | DATA1370 |
| | | DATA1380 |
| | | DATA1390 |
| | | DATA1400 |

| | | |
|---|---|----------|
| C | DATA K1/0.1019E 01.0.6541E 00.0.1250E 01.0.1281E 01.2.6000E 01. | DATA1410 |
| | 1 2.6000E 01.2.6000E 01.0.2506E 01.0.2496E 01.0.3211E 01. | DATA1420 |
| | 2 0.8589E 00.0.6304E 00.0.8589E 00.0.1126E 01.0.1126E 01. | DATA1430 |
| | 3 0.6304E 00.0.6304E 00.0.4855E 00.0.8589E 00.0.1000E-04/ | DATA1440 |
| C | DATA K2/0.4901E-03.0.6457E-03.0.7092E-03.0.8593E-03.0.0000E-03. | DATA1450 |
| | 1 0.0000E-03.0.0000E-03.0.7479E-03.0.5129E-02.0.5344E-02. | DATA1460 |
| | 2 0.6233E-03.0.5804E-03.0.6233E-03.0.7439E-03.0.7439E-03. | DATA1470 |
| | 3 0.5804E-03.0.5804E-03.0.8714E-03.0.6233E-03.0.7350E-03/ | DATA1480 |
| C | DATA ICODE/20*0/ | DATA1490 |
| C | DATA EL/ 'C', 'H', 'N', 'O', 'E' / | DATA1500 |
| | DATA AWT/ 12.011, 1.008, 14.008, 16.000, 5.486E-4 / | DATA1510 |
| C | DATA IA/ 0.0.0.0.0.0.0.0.1.0.0.1.3.1.2.2.3.4.1.2.1. | DATA1520 |
| H | 0.0.0.0.0.0.0.0.1.2.0.0.0.1.2.1.1.1.0.0. | DATA1530 |
| N | 0.2.0.1.0.1.0.0.0.0.0.1.0.0.0.0.1.0.0. | DATA1540 |
| O | 2.0.1.0.1.0.0.0.0.0.1.0.0.0.0.0.0.0.0.0. | DATA1550 |
| E | 2.2.1.1.0.0.1.1.0.0.2.3.2.2.2.3.4.2.2.0/ | DATA1560 |
| C | DATA LSP/20.9.6.5.7/ | DATA1570 |
| | DATA NHVL /9/, NIHVC /12/ | DATA1580 |
| | DATA FHVC /5.0, 6.0, 7.0, 8.0, 9.0, 10.0, 10.8, 11.1, | DATA1590 |
| | 1 12.0, 13.4, 14.3, 20.0/ | DATA1600 |
| | DATA DJ /0.6, 2.2, 1.5, 1.65, 1.4, 1.0, 1.2, 1.4, 1.0/ | DATA1610 |
| | DATA HVJ /1.3, 2.7, 5.75, 7.57, 9.1, 10.4, 11.4, 12.7, 13.9/ | DATA1620 |
| | DATA ZKZ /7.26E-16/ | DATA1630 |
| | END | DATA1640 |
| | | DATA1650 |
| | | DATA1660 |
| | | DATA1670 |
| | | DATA1680 |
| | | DATA1690 |
| | | DATA1700 |

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|-----|---|-----------|
| | SUBROUTINE LRAD | LRAD 10 |
| C | | LRAD 20 |
| C | | LRAD 30 |
| C | ** THIS IS A DRIVER PROGRAM FOR SUBROUTINE TRANS WHICH CALCULATES | **LRAD 40 |
| C | THE RADIATIVE FLUX DIVERGENCE THROUGH A ONE-DIMENSIONAL SLAB | LRAD 50 |
| C | FOR A GIVEN TEMPERATURE AND SPECIES DISTRIBUTION | LRAD 60 |
| | COMMON /SFLUX/ QRI(3) | LRAD 70 |
| | COMMON /TRN/ NUT(60), FMC(12,60), FPC(12,60), | LRAD 80 |
| 1 | FM(9,60), FP(9,60), LINES | LRAD 90 |
| | COMMON /TEST/ETZ(60),IEZ | LRAD 100 |
| | COMMON /YL/ETA(60),YOND(60) | LRAD 110 |
| | COMMON/PROP1/PI(60),RHC(60), T(60),AMW(60),C (20,60),EC(5,60) | LRAD 120 |
| | COMMON/SIS/SY(20,60) | LRAD 130 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, F , RE, LXI, ITM, IEM, NETA | LRAD 140 |
| | COMMON /DEL/ DELTA,DTIL,DTILS | LRAD 150 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | LRAD 160 |
| | COMMON /MAIM/KEEP,MAXE,MAXM,MAXD, IDG,MCONV,ECONV,DCONV,LT,IAB | LRAD 170 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | LRAD 180 |
| | COMMON /NUMDEN/ SNDD2(60), SNDN2(60), SNDD(60), SNDN(60), | LRAD 190 |
| 1 | SNDE(60), SNDC(60), | LRAD 200 |
| 2 | SNDH(60), SNDC2(60), SNDH2(60), SNDCO(60), | LRAD 210 |
| 3 | SNDC3(60),SNDC2H(60) | LRAD 220 |
| | COMMON /SPEC/ MF, XMDL | LRAD 230 |
| C | | LRAD 240 |
| | DO100I=1,20 | LRAD 250 |
| | DO100J=1,NETA | LRAD 260 |
| 100 | SY(I,J) = C(I,J) | LRAD 270 |
| | DO200I=1,20 | LRAD 280 |
| | DO200J=1,NETA | LRAD 290 |
| 200 | IF(C(I,J).LT.0.)C(I,J) = 1.E-20 | LRAD 300 |
| C | | LRAD 310 |
| C | ** NETA = NUMBER OF ETA POINTS | LRAD 320 |
| C | MF = 1 IF SPECIE MOLE FRACTIONS ARE INPUT AND NUMBER DENSITY | LRAD 330 |
| C | TO BE COMPUTED | LRAD 340 |
| C | 0 IF SPECIE NUMBER DENSITIES ARE INPUT | LRAD 350 |

| | | |
|----|---|----------|
| C | NS = NUMBER OF SPECIES TO BE INPUT | LRAD 360 |
| C | LINES= 1 IF LINE CALCULATION IS TO BE DONE | LRAD 370 |
| C | 0 IF ONLY CONTINUUM CALCULATION IS TO BE DONE | LRAD 380 |
| C | IDG = 0 ONLY FINAL PRINT IS GIVEN | LRAD 390 |
| C | 1 PRINT IS GIVEN FOR EACH ETA | LRAD 400 |
| C | 99 COMPLETE PRINT | LRAD 410 |
| C | | LRAD 420 |
| C | | LRAD 430 |
| C | ** R = BODY RADIUS (FT) | LRAD 440 |
| C | DELTA = NONDIMENSIONAL STAND-OFF DISTANCE | LRAD 450 |
| C | DTIL = TRANSFORMED STAND-OFF DISTANCE | LRAD 460 |
| C | XMOL = 1.0 FOR RUN WITH MOLECULES | LRAD 470 |
| C | 0.0 FOR RUN WITHOUT MOLECULES | LRAD 480 |
| C | | LRAD 490 |
| | XMOL = 0.0 | LRAD 500 |
| | XMOL = 1.0 | LRAD 510 |
| C | | LRAD 520 |
| C | ** DETERMINE ETZ PCINTS | LRAD 530 |
| | N2 = NETA-2 | LRAD 540 |
| | K = 0 | LRAD 550 |
| | IEZ = 0 | LRAD 560 |
| | DO 20 I=1,N2,2 | LRAD 570 |
| | K=K+1 | LRAD 580 |
| 20 | ETZ(K) = ETA(1) | LRAD 590 |
| | ETZ(K+1) = ETA(NETA-1) | LRAD 600 |
| | ETZ(K+2) = ETA(NETA) | LRAD 610 |
| | IEZ = K + 1 | LRAD 620 |
| C | | LRAD 630 |
| C | ** COMPUTE THE Y COORDINATE ** | LRAD 640 |
| | YOND(1) = 0.0 | LRAD 650 |
| | SUM = 0.0 | LRAD 660 |
| | DO 30 K=2,NETA | LRAD 670 |
| | DETA = ETA(K) -ETA(K-1) | LRAD 680 |
| | SUM = SUM +DETA*(1./RHO(K) +1./RHO(K-1))/2.0 | LRAD 690 |
| | YOND(K) = DTIL *SUM | LRAD 700 |

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|-----|--|----------|
| 30 | CONTINUE | LRAD 710 |
| | DELTA = YOND(NETA) | LRAD 720 |
| | DO 40 K=1,NETA | LRAD 730 |
| | YOND(K) = YOND(K)/YOND(NETA) | LRAD 740 |
| 40 | CONTINUE | LRAD 750 |
| C | | LRAD 760 |
| | LINES= 1 | LRAD 770 |
| | IDGS = IDG | LRAD 780 |
| | IDG = 0 | LRAD 790 |
| | CALL TRANS(1) | LRAD 800 |
| | IDG= IDGS | LRAD 810 |
| C | | LRAD 820 |
| C | ** INDEX IS NUMBER GIVEN SPECIE FOR USE IN STORING ARRAYS ** | LRAD 830 |
| C | 1 = O2 | LRAD 840 |
| C | 2 = N2 | LRAD 850 |
| C | 3 = O | LRAD 860 |
| C | 4 = N | LRAD 870 |
| C | 5 = E- | LRAD 880 |
| C | 6 = C | LRAD 890 |
| C | 7 = H | LRAD 900 |
| C | 8 = C2 | LRAD 910 |
| C | 9 = H2 | LRAD 920 |
| C | 10 = CO | LRAD 930 |
| C | 11 = C3 | LRAD 940 |
| C | 12 = C2H | LRAD 950 |
| C | | LRAD 960 |
| C | | LRAD 970 |
| | DO300I=1,20 | LRAD 980 |
| | DO300J=1,NETA | LRAD 990 |
| 300 | C(I,J) = SY(I,J) | LRAD1000 |
| | RETURN | LRAD1010 |
| | END | LRAD1020 |

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|---|---------------|---------------|-----------------|------|-----|
| SUBROUTINE RADIN | | | | RADI | 10 |
| COMMON /DEBUG/ QLC(60), QCL(60), QLL(60), DQN(60), QCC(60), | | | | RADI | 20 |
| 1 | BEEC(12,60), | FMUC(12,60), | EM(12,60), | RADI | 30 |
| 2 | EP(12,60), | TAUC(12,60), | BEEL(9,60), | RADI | 40 |
| 3 | QCCP(12), | WMM(9,60), | GMM(9,60), | RADI | 50 |
| 4 | EEM(9,60), | XLMM(9,60), | QLCF(9), | RADI | 60 |
| 5 | QCLP(9), | QLLP(9), | DELTA, IY, IYY, | RADI | 70 |
| 6 | WPP(9,60), | GPP(9,60), | EEP(9,60), | RADI | 80 |
| 7 | XLPP(9,60), | FG(9,4), | GP(9,4), | RADI | 90 |
| 8 | WN(9,4), | FMUL(9,60), | SSM(9,4,60), | RADI | 100 |
| 9 | GGM(9,4,60), | ETAM(9,4,60), | SBM(9,4,60), | RADI | 110 |
| A | TAUL(9,60) | | | RADI | 120 |
| C | ** GROUP 1 ** | | | RADI | 130 |
| | WN(1,1)=0. | | | RADI | 140 |
| | FG(1,1)=0. | | | RADI | 150 |
| | GP(1,1)=0. | | | RADI | 160 |
| | WN(1,2)=18. | | | RADI | 170 |
| | WN(1,3)=15. | | | RADI | 180 |
| | WN(1,4)=5. | | | RADI | 190 |
| C | ** GROUP 2 ** | | | RADI | 200 |
| | WN(2,1)=3.0 | | | RADI | 210 |
| | WN(2,2)=5.0 | | | RADI | 220 |
| | WN(2,3)=11.0 | | | RADI | 230 |
| | WN(2,4)=10. | | | RADI | 240 |
| C | ** GROUP 3 ** | | | RADI | 250 |
| | WN(3,1)=0. | | | RADI | 260 |
| | FG(3,1)=0. | | | RADI | 270 |
| | GP(3,1)=0. | | | RADI | 280 |
| | WN(3,2)=2.0 | | | RADI | 290 |
| | WN(3,3)=0. | | | RADI | 300 |
| | FG(3,3)=0. | | | RADI | 310 |
| | GP(3,3)=0. | | | RADI | 320 |
| | WN(3,4)=0. | | | RADI | 330 |
| | FG(3,4)=0. | | | RADI | 340 |
| | GP(3,4)=0. | | | RADI | 350 |

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C ** GROUP 4 **
  WN(4,1)=0.
  FG(4,1)=0.
  GP(4,1)=0.
  WN(4,2)=8.0
  WN(4,3)=2.0
  WN(4,4)=0.
  FG(4,4)=0.
  GP(4,4)=0.
C ** GROUP 5 **
  WN(5,1)=0.
  FG(5,1)=0.
  GP(5,1)=0.
  WN(5,2)=14.
  WN(5,3)=4.0
  WN(5,4)=1.0
C ** GROUP 6 **
  WN(6,1)=1.0
  WN(6,2)=4.0
  WN(6,3)=13.0
  WN(6,4)=2.0
C ** GROUP 7 **
  WN(7,1)=0.
  FG(7,1)=0.
  GP(7,1)=0.
  WN(7,2)=6.0
  WN(7,3)=14.0
  WN(7,4)=3.0
C ** GROUP 8 **
  WN(8,1)=2.0
  WN(8,2)=2.0
  WN(8,3)=11.
  WN(8,4)=15.
C ** GROUP 9 **
  WN(9,1)=0.

```

```

RADI 360
RADI 370
RADI 380
RADI 390
RADI 400
RADI 410
RADI 420
RADI 430
RADI 440
RADI 450
RADI 460
RADI 470
RADI 480
RADI 490
RADI 500
RADI 510
RADI 520
RADI 530
RADI 540
RADI 550
RADI 560
RADI 570
RADI 580
RADI 590
RADI 600
RADI 610
RADI 620
RADI 630
RADI 640
RADI 650
RADI 660
RADI 670
RADI 680
RADI 690
RADI 700

```

```
FG(9,1)=0.
GP(9,1)=0.
WN(9,2)=1.0
WN(9,3)=11.
WN(9,4)=10.
RETURN
END
```

```
RADI 710
RADI 720
RADI 730
RADI 740
RADI 750
RADI 760
RADI 770
```

| | | |
|------------------------|---|----------|
| SUBROUTINE TRANS (ISW) | | TRAN 10 |
| C | | TRAN 20 |
| C | -----THIS IS A MODIFIED VERSION OF SUBROUTINE TRANS FROM K WILSON | TRAN 30 |
| C | TRANS IS DOCUMENTED IN LMSC-687209 APRIL 69 ----- | TRAN 40 |
| C | | TRAN 50 |
| | COMMON /ZPI/ ZPO(6), ZPN(6), ZPH(2), ZPC(7) | TRAN 60 |
| | COMMON /FINV/ NHVL, NIHVC, FHVC(12), DJ(9), HVJ(9), ZKZ | TRAN 70 |
| | COMMON /SFLUX/ QRI(3) | TRAN 80 |
| | COMMON /TRN/ NUT(60), FMC(12,60), FPC(12,60), | TRAN 90 |
| 1 | FM(9,60), FP(9,60), LINES | TRAN 100 |
| | COMMON /YL/ETA(60), YD(60) | TRAN 110 |
| | COMMON /PROPI/ P(60), R(60), T(60), AMW(60), C(20,60), EC(5,60) | TRAN 120 |
| | COMMON /FRSTRM/ U INF, RINF, UINF2, XL, RE, LXI, ITM, IEM, NES | TRAN 130 |
| | COMMON /DEL/ W(1), DTIL, DTILS | TRAN 140 |
| | COMMON /NON/ RDZ, MUDZ, RMDZ, AKNF, HNF, CFNF | TRAN 150 |
| | COMMON /MAIM/ KEEP, MAXE, MAXM, MAXD, IDG, MCONV, ECONV, DCONV, LT, IAB | TRAN 160 |
| | COMMON /RFLUX/ E(60), IRAD, ITYPE | TRAN 170 |
| | COMMON /RH/ DUD, DPHI, TD, RZB, PD, HD, HTOTAL | TRAN 180 |
| | COMMON /WT/ SMW(20), AWT(5) | TRAN 190 |
| | COMMON /TEST/ ETZ(60), IEZ | TRAN 200 |
| | COMMON /NUMDEN/ SNDO2(60), SNON2(60), SNDO(60), SNDN(60), | TRAN 210 |
| 1 | SNDE(60), SNDC(60), | TRAN 220 |
| 2 | SNDH(60), SNDC2(60), SNDH2(60), SNDCO(60), | TRAN 230 |
| 3 | SNDC3(60), SNDC2H(60) | TRAN 240 |
| | COMMON /DEBUG/ QLC(60), QCL(60), QLL(60), DQN(60), QCC(60), | TRAN 250 |
| 1 | BEEC(12,60), FMUC(12,60), EM(12,60), | TRAN 260 |
| 2 | EP(12,60), TAUC(12,60), BEEL(9,60), | TRAN 270 |
| 3 | QCCP(12), WMM(9,60), GMM(9,60), | TRAN 280 |
| 4 | EEM(9,60), XLMM(9,60), QLCP(9), | TRAN 290 |
| 5 | QCLP(9), QLLP(9), DELTA, IY, IYY, | TRAN 300 |
| 6 | WPP(9,60), GPP(9,60), EEP(9,60), | TRAN 310 |
| 7 | XLPP(9,60), FG(9,4), GP(9,4), | TRAN 320 |
| 8 | WN(9,4), FMUL(9,60), SSM(9,4,60), | TRAN 330 |
| 9 | GGM(9,4,60), ETAM(9,4,60), SBM(9,4,60), | TRAN 340 |
| A | TAUL(9,60) | TRAN 350 |

```

COMMON /SPEC/ MF, XMOL
DIMENSION XKT(60), DQ(60)
C
C ** BAND AVERAGE ABSORPTION CROSS SECTION (EQ. A2) **
C
SIGMA(ZH,ZA,ZB,ZG)= ((5.0E+03*T1*ZG*ZKZ)/BE) * (EXP(ZDL/T1)
1 *ZH*(ZA+ZB*(ZH**2)/3.0) +
2 T1 * (ZA+2.0*ZB*T12) -T1*EXP((ZH-ZHVP)/T1)
3 *(ZA+ZB*(ZHVP-ZH)**2) -T1*EXP((ZH-ZHVP)/T1)
4 *2.0*ZB*T1*(ZHVP-ZH+T1))
SIGMA2(ZH,ZG,ZE,ZY)=7.26E-16*T1*ZG*EXP((-ZE+ZY+ZDL)/T1)/ZH**3
GAMMA(ZX)=(1.0+(1.5707963*ZX)**1.25)**(-0.4)
XLAMB(ZX)=(1.0+ZX*EXP(-ZX))/SQRT(1.0+6.283185 *ZX)
C
C ** W(GROUP)/D CORRELATION (EQ. 88) **
C
PHI1(ZX)=(ATAN(1.570796 *ZX)/1.570796 )
C
C ** FLUX DIVERGENCE OVERLAPPING FUNCTION (EQ. 92) **
C
PHI2(ZX)=EXP(-ZX)
C
DO 400 I=1,NES
400 T(I)= T(I)*TD
ZHVP=5.0
YI=0.0
CONVER = 3.10375E+23 *R (I) *RDZ
SNDE(NES) = CONVER * C( 7,NES)/SMW(7)
XNE=SNDE(NES)
FNE=(4.71E-6 * XNE**(2.0/7.0))/((T(NES)/11606.)*(1.0/7.0))
ZDL=AMIN1(0.20,FNE)
C
C ** DEBUG PRINT **
C
IF (IDG.NE.0) CALL BUGPR (1)

```

```

TRAN 360
TRAN 370
TRAN 380
TRAN 390
TRAN 400
TRAN 410
TRAN 420
TRAN 430
TRAN 440
TRAN 450
TRAN 460
TRAN 470
TRAN 480
TRAN 490
TRAN 500
TRAN 510
TRAN 520
TRAN 530
TRAN 540
TRAN 550
TRAN 560
TRAN 570
TRAN 580
TRAN 590
TRAN 600
TRAN 610
TRAN 620
TRAN 630
TRAN 640
TRAN 650
TRAN 660
TRAN 670
TRAN 680
TRAN 690
TRAN 700

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| | |
|--|----------|
| DELTA=W(1) * XL * 30.48006 | TRAN 710 |
| CALL BUGPR (2) | TRAN 720 |
| 6001 CONTINUE | TRAN 730 |
| DO 91 L=1,NES | TRAN 740 |
| XKT(L)=T(L)/11606. | TRAN 750 |
| T1=XKT(L) | TRAN 760 |
| CALL SND(L) | TRAN 770 |
| C | TRAN 780 |
| C ** PARTITION FUNCTIONS FOR H, C, N, O ** | TRAN 790 |
| C | TRAN 800 |
| 94 IF(T(L).GT.15000.) GO TO 6 | TRAN 810 |
| C | TRAN 820 |
| C ** LOW TEMPERATURE ** | TRAN 830 |
| C | TRAN 840 |
| SUMH=2.0 | TRAN 850 |
| SUMC=9.0 + 5.0 * EXP(-1.264/T1) + EXP(-2.684/T1) + | TRAN 860 |
| 1 5.0 * EXP(-4.183/T1) | TRAN 870 |
| SUMN=4.0 + 10.0 * EXP(-2.384/T1) + 6.0 * EXP(-3.576/T1) | TRAN 880 |
| SUMO= 9.0 + 5.0 * EXP(-1.975/T1) | TRAN 890 |
| GO TO 7 | TRAN 900 |
| C | TRAN 910 |
| C ** HIGH TEMPERATURE ** | TRAN 920 |
| C | TRAN 930 |
| 6 SUMH=2.0 | TRAN 940 |
| SUMC=2.71818 + 6.40677 * T(L)/1.0E4 -0.45466 * (T(L)/1.0E4)**2 | TRAN 950 |
| SUMN=5.938216 - 0.225593 * T(L)/1.0E3 + 0.015408 * (T(L)/1.0E3)**2 | TRAN 960 |
| SUMO=11.79563 -0.317964 * T(L)/1.0E3 + 0.013765 * (T(L)/1.0E3)**2 | TRAN 970 |
| 7 CONTINUE | TRAN 980 |
| T12=T1**2 | TRAN 990 |
| GH = 6.4994 | TRAN1000 |
| DO 5 K=1,12 | TRAN1010 |
| GF=FHVC(K)/T1 | TRAN1020 |
| GHM=GH | TRAN1030 |
| GH=EXP(-GF) *GF * (GF**2 + 3.0 *GF +6.0 + 6.0/GF) | TRAN1040 |
| C | TRAN1050 |

```

C **   PLANK MEAN ABSORPTION COEFFICIENT FOR BAND INTERVALS (EQ.A3) **   TRAN1060
C                                           TRAN1070
      BEEC(K,L)=5.04E3 * (T12**2) * (GHM-GH)   TRAN1080
      BE=BEEC(K,L)   TRAN1090
C                                           TRAN1100
C **   ABSORPTION CROSS SECTIONS **   TRAN1110
C     SPECIES --   TRAN1120
C           N       N2       CO   TRAN1130
C           O       O2           TRAN1140
C           C       C2       C2H  TRAN1150
C           H       H2       C3   TRAN1160
C                                           TRAN1170
      SGH=0.   TRAN1180
      SGN=0.   TRAN1190
      SGC=0.   TRAN1200
      SGO=0.   TRAN1210
      SGC0=0.  TRAN1220
      SGC2=0.  TRAN1230
      SGO2=0.  TRAN1240
      SGN2=0.  TRAN1250
      SGH2=0.  TRAN1260
      SGC3=0.  TRAN1270
      SGC2H = 0.0   TRAN1280
      GO TO (581,582,583,584,585,586,587,588,589,590,591,592),K   TRAN1290
581 SGH=SIGMA(2.4,1.0,0.0,1.0) * EXP(-13.56/T1)   TRAN1300
      SGC=SIGMA(3.78, 0.3, 0.0488, 1.33) * EXP(-11.26/T1)   TRAN1310
      SGN=SIGMA(4.22, 0.24, 0.0426, 4.5) * EXP(-14.54/T1)   TRAN1320
      SGO=SIGMA(4.22, 0.24, 0.0426, .8888889) * EXP(-13.61/T1)   TRAN1330
      GO TO 38   TRAN1340
582 ZZHV=5.5   TRAN1350
      SGC2=8.0E-18 * EXP(-0.5/T1) + 3.0E-18   TRAN1360
      SGC3=4.0E-18   TRAN1370
593 CALL ZHV(ZZHV,ZZO,ZZN,ZZI,ZZC)   TRAN1380
      SGC=SIGMA2(ZZHV, 1.33, 11.26, 3.78) * ZZC + SGC   TRAN1390
      SGN=SIGMA2(ZZHV,4.50, 14.54, 4.22) * ZZN   TRAN1400

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| | | |
|-----|--|----------|
| 594 | SGO=SIGMA2 (ZZHV, .889, 13.61, 4.22) * ZZ0 | TRAN1410 |
| 595 | SGH=SIGMA2 (ZZHV, 1.00, 13.56, 2.40) | TRAN1420 |
| | GO TO 38 | TRAN1430 |
| 583 | ZZHV=6.5 | TRAN1440 |
| | SGC2=1.0E-18 | TRAN1450 |
| | SGC0=3.0E-18 * EXP(-0.7/T1) | TRAN1460 |
| | GO TO 593 | TRAN1470 |
| 584 | ZZHV=7.5 | TRAN1480 |
| | SGC=5.0E-17 * EXP(-4.18/T1)/SUMC | TRAN1490 |
| | SGC0=1.9E-17 * EXP(-0.5/T1) | TRAN1500 |
| | SGO2=6.0E-19 | TRAN1510 |
| | SGC2H = 1.3E-18 | TRAN1520 |
| | GO TO 593 | TRAN1530 |
| 585 | ZZHV=8.5 | TRAN1540 |
| | SGC=5.0E-17 * EXP(-4.18/T1)/SUMC + | TRAN1550 |
| 1 | 2.2E-17 * EXP(-2.68/T1)/SUMC | TRAN1560 |
| | SGC0=2.5E-17 | TRAN1570 |
| | SGO2=2.0E-19 | TRAN1580 |
| | SGC2H = 8.5E-19 | TRAN1590 |
| | GO TO 593 | TRAN1600 |
| 586 | ZZHV=9.5 | TRAN1610 |
| | SGC=5.0E-17 * EXP(-4.18/T1)/SUMC + | TRAN1620 |
| 1 | 2.2E-17 * EXP(-2.68/T1)/SUMC | TRAN1630 |
| | SGC0=5.0E-18 | TRAN1640 |
| | SGO2=1.0E-18 | TRAN1650 |
| | GO TO 593 | TRAN1660 |
| 587 | SGN=3.2E-18 * T1 * EXP(-10.2/T1)/SUMN | TRAN1670 |
| | SGO2=6.0E-19 | TRAN1680 |
| | ZZHV=10.4 | TRAN1690 |
| | CALL ZHV(ZZHV,ZZ0,ZZN,ZZI,ZZC) | TRAN1700 |
| 596 | SGC=(8.5E-17 * EXP(-1.26/T1) + 2.2E-17 * EXP(-2.75/T1) | TRAN1710 |
| 1 | + 5.0E-17 * EXP(-4.18/T1))/SUMC | TRAN1720 |
| | GO TO 594 | TRAN1730 |
| 588 | ZZHV=10.9 | TRAN1740 |
| | CALL ZHV(ZZHV,ZZ0,ZZN,ZZI,ZZC) | TRAN1750 |

| | |
|---|----------|
| SGN=(5.16E-17 *EXP(-3.50/T1))/SUMN | TRAN1760 |
| GO TO 596 | TRAN1770 |
| 589 ZZHV=11.6 | TRAN1780 |
| CALL ZHV(ZZHV,ZZO,ZZN,ZZI,ZZC) | TRAN1790 |
| SGN2=1.0E-18 | TRAN1800 |
| SGN=(5.16E-17 * EXP(-3.50))/SUMN | TRAN1810 |
| 598 SGC=(9.9E-17 + 8.5E-17 * EXP(-1.26/T1) +2.2E-17 * EXP(-2.75/T1) | TRAN1820 |
| 1 + 5.0E-17 * EXP(-4.18/T1))/SUMC | TRAN1830 |
| IF (K.LT.11) GO TO 594 | TRAN1840 |
| GO TO 38 | TRAN1850 |
| 590 ZZHV=12.7 | TRAN1860 |
| CALL ZHV (ZZHV,ZZO,ZZN,ZZI,ZZC) | TRAN1870 |
| SGN2=2.0E-18 | TRAN1880 |
| SGH2 = 2.7E-17 | TRAN1890 |
| 599 SGN=(6.4E-17 * EXP(-2.30/T1) + 5.16E-17 * EXP(-3.50/T1))/SUMN | TRAN1900 |
| 1 + SGN | TRAN1910 |
| GO TO 598 | TRAN1920 |
| 591 SGH=1.18E-17/SUMH | TRAN1930 |
| SGO=3.6E-17/SUMO | TRAN1940 |
| SGN2=1.0E-17 | TRAN1950 |
| SGH2 = 2.7E-17 | TRAN1960 |
| GO TO 599 | TRAN1970 |
| 592 SGN=3.6E-17/SUMN | TRAN1980 |
| SGN2=1.0E-18 | TRAN1990 |
| GO TO 599 | TRAN2000 |
| 38 CONTINUE | TRAN2010 |
| FMUC(K,L)= SNDH(L)*SGH + SNDC(L)*SGC + SNDN(L)*SGN + SNDO(L)*SGO | TRAN2020 |
| 1 + XMOL * (SNDN2(L)*SGN2 + SNDO2(L)*SGO2 + | TRAN2030 |
| 2 SNDC2(L)*SGC2 + SNDH2(L)*SGH2 + SNDCO(L)*SGCO + | TRAN2040 |
| 3 SNDC3(L)*SGC3 +SNDC2H(L)*SGC2H) | TRAN2050 |
| IF (L.GT.1) GO TO 8 | TRAN2060 |
| TAUC(K,L)=0. | TRAN2070 |
| GO TO 5 | TRAN2080 |
| 8 TAUC(K,L)=TAUC(K,L-1)+(YD(L)-YD(L-1))* | TRAN2090 |
| 1 (FMUC(K,L-1)+FMUC(K,L)) * DELTA | TRAN2100 |

5 CONTINUE

IF (LINES.EQ.0) GO TO 91

C

C ** FRACTIONAL POPULATION STATES FOR H, N, O, C **

C

CALL ZP (T1,SUMN,SUMO,SUMH,SUMC)

C ** CALCULATION OF PARAMETERS FOR 9 LINE GROUPEs **

C

WN -- NUMBER OF LINES

C

FG -- EFFECTIVE F-NUMBER

C

GP -- EFFECTIVE HALF-WIDTH

C

C GROUP 1

FG(1,2)=(1.02 * ZPC(5) + .795 * ZPC(6) + 0.114 * ZPC(7))

1 /WN(1,2)

GP(1,2)=(8.16E-11 * SQRT(ZPC(5)) + 1.25E-10 * SQRT(ZPC(6))

1 +2.55E-10 * SQRT(ZPC(7)))**2 /((FG(1,2) * WN(1,2)**2)

FG(1,3)=(1.040 * ZPN(4) + 1.29 * ZPN(5) + 0.00 * ZPN(6))

1 /WN(1,3)

GP(1,3)=(6.65E-11 * SQRT(ZPN(4)) + 1.71E-10 * SQRT(ZPN(5))

1 + 0.00E-10 * SQRT(ZPN(6)))**2 /((FG(1,3) * WN(1,3)**2)

FG(1,4)=(1.00 * ZPO(5) + .978 * ZPO(6))/WN(1,4)

GP(1,4)=(3.90E-11 * SQRT(ZPO(5)) + 9.68E-11 * SQRT(ZPO(6)))**2

1 /((FG(1,4) * WN(1,4)**2)

FMUL(1,L)=FMUC(1,L)

C GROUP 2

FG(2,1)=0.805 * ZPH(2)/WN(2,1)

GP(2,1)=2.37E-10 * 2.37E-10 * ZPH(2)/((FG(2,1) * WN(2,1)**2)

FG(2,2)=(0.00E-2 * ZPC(5) + 6.71E-2 * ZPC(6))/WN(2,2)

GP(2,2)=(0.00E-12 * SQRT(ZPC(5)) + 7.15E-11 * SQRT(ZPC(6)))**2

1 /((FG(2,2) * WN(2,2)**2)

FG(2,3)=(0.047 * ZPN(4) + 2.85E-2 * ZPN(5))/WN(2,3)

GP(2,3)=(1.11E-10 * SQRT(ZPN(4)) + 6.07E-11 * SQRT(ZPN(5)))**2

1 /((FG(2,3) * WN(2,3)**2)

FG(2,4)=(.0217 * ZPO(4) + 8.25E-2 * ZPO(5))/WN(2,4)

GP(2,4)=(2.61E-11 * SQRT(ZPO(4)) + 7.19E-11 * SQRT(ZPO(5)))**2

TRAN2110

TRAN2120

TRAN2130

TRAN2140

TRAN2150

TRAN2160

TRAN2170

TRAN2180

TRAN2190

TRAN2200

TRAN2210

TRAN2220

TRAN2230

TRAN2240

TRAN2250

TRAN2260

TRAN2270

TRAN2280

TRAN2290

TRAN2300

TRAN2310

TRAN2320

TRAN2330

TRAN2340

TRAN2350

TRAN2360

TRAN2370

TRAN2380

TRAN2390

TRAN2400

TRAN2410

TRAN2420

TRAN2430

TRAN2440

TRAN2450

| | | |
|---|---|----------|
| 1 | /(FG(2,4) * WN(2,4)**2) | TRAN2460 |
| | FMUL(2,L)=FMUC(1,L) | TRAN2470 |
| C | GROUP 3 | TRAN2480 |
| | FG(3,2)=(7.29E-2 * ZPC(2) + 6.76E-2 * ZPC(3))/WN(3,2) | TRAN2490 |
| | GP(3,2)=(9.08E-12 * SQRT(ZPC(2)) + 8.75E-12 * SQRT(ZPC(3)))*2 | TRAN2500 |
| 1 | /(FG(3,2) * WN(3,2)**2) | TRAN2510 |
| | FMUL(3,L)=FMUC(2,L) | TRAN2520 |
| C | GROUP 4 | TRAN2530 |
| | FG(4,2)=(1.05 * ZPC(1) + 1.10E-2 * ZPC(2) + 0.150 * ZPC(3)) | TRAN2540 |
| 1 | /WN(4,2) | TRAN2550 |
| | GP(4,2)=(9.57E-12 * SQRT(ZPC(1)) + 4.86E-12 * SQRT(ZPC(2)) | TRAN2560 |
| 1 | + 5.93E-10 * SQRT(ZPC(3)))*2/(FG(4,2) * WN(4,2)**2) | TRAN2570 |
| | FG(4,3)=(7.40E-2 * ZPN(2) + 6.34E-2 * ZPN(3))/WN(4,3) | TRAN2580 |
| | GP(4,3)=(8.22E-12 * SQRT(ZPN(2)) + 7.60E-12 * SQRT(ZPN(3)))*2 | TRAN2590 |
| 1 | /(FG(4,3) * WN(4,3)**2) | TRAN2600 |
| | FMUL(4,L)=FMUC(4,L) | TRAN2610 |
| C | GROUP 5 | TRAN2620 |
| | FG(5,2)=(0.329 * ZPC(1) + 0.118 * ZPC(2) + 0.226 * ZPC(4)) | TRAN2630 |
| 1 | /WN(5,2) | TRAN2640 |
| | GP(5,2)=(3.65E-11 * SQRT(ZPC(1)) + 5.77E-10 * SQRT(ZPC(2)) | TRAN2650 |
| 1 | + 6.56E-11 * SQRT(ZPC(4)))*2/(FG(5,2) * WN(5,2)**2) | TRAN2660 |
| | FG(5,3)=0.108 * ZPN(3)/WN(5,3) | TRAN2670 |
| | GP(5,3)=3.09E-11 * 3.09E-11 * ZPN(3)/(FG(5,3) * WN(5,3)**2) | TRAN2680 |
| | FG(5,4)=4.71E-2 * ZPD(1)/WN(5,4) | TRAN2690 |
| | GP(5,4)=5.08E-12 * 5.08E-12 * ZPD(1)/(FG(5,4) * WN(5,4)**2) | TRAN2700 |
| | FMUL(5,L)=FMUC(6,L) | TRAN2710 |
| C | GROUP 6 | TRAN2720 |
| | FG(6,1)=0.416 * ZPH(1)/WN(6,1) | TRAN2730 |
| | GP(6,1)=3.02E-11 * 3.02E-11 * ZPH(1)/(FG(6,1) * WN(6,1)**2) | TRAN2740 |
| | FG(6,2)=8.65E-2 * ZPC(1)/WN(6,2) | TRAN2750 |
| | GP(6,2)=2.35E-10 * 2.35E-10 * ZPC(1)/(FG(6,2) * WN(6,2)**2) | TRAN2760 |
| | FG(6,3)=(0.184 * ZPN(1) + 0.290 * ZPN(2) + 8.52E-2 * ZPN(3)) | TRAN2770 |
| 1 | /WN(6,3) | TRAN2780 |
| | GP(6,3)=(1.07E-11 * SQRT(ZPN(1)) + 4.28E-11 * SQRT(ZPN(2)) | TRAN2790 |
| 1 | + 2.09E-10 * SQRT(ZPN(3)))*2/(FG(6,3) * WN(6,3)**2) | TRAN2800 |

| | | |
|---|--|----------|
| | FG(6,4)=(0.120 * ZPO(2) + 0.151 * ZFO(3))/WN(6,4) | TRAN2810 |
| | GP(6,4)=(8.85E-12 * SQRT(ZPO(2)) + 9.93E-12 * SQRT(ZPO(3)))**2 | TRAN2820 |
| | 1 / (FG(6,4) * WN(6,4)**2) | TRAN2830 |
| | FMUL(6,L)=FMUC(7,L) | TRAN2840 |
| C | GROUP 7 | TRAN2850 |
| | FG(7,2)=(4.51E-2 * ZPC(1) + 0.705 * ZPC(2))/WN(7,2) | TRAN2860 |
| | GP(7,2)=(6.07E-10 * SQRT(ZPC(1)) + 2.10E-10 * SQRT(ZPC(2)))**2 | TRAN2870 |
| | 1 / (FG(7,2) * WN(7,2)**2) | TRAN2880 |
| | FG(7,3)=(0.454 * ZPN(1) + 9.66E-2 * ZPN(2) | TRAN2890 |
| | 1 + 0.178 * ZPN(3))/WN(7,3) | TRAN2900 |
| | GP(7,3)=(2.71E-12 * SQRT(ZPN(1)) + 2.34E-10 * SQRT(ZPN(2)) | TRAN2910 |
| | 1 + 2.46E-11 * SQRT(ZPN(3)))**2 / (FG(7,3) * WN(7,3)**2) | TRAN2920 |
| | FG(7,4)=4.23E-2 * ZPO(3)/WN(7,4) | TRAN2930 |
| | GP(7,4)=2.52E-11 * 2.52E-11 * ZPO(3)/(FG(7,4) * WN(7,4)**2) | TRAN2940 |
| | FMUL(7,L)=FMUC(9,L) | TRAN2950 |
| C | GROUP 8 | TRAN2960 |
| | FG(8,1)=0.108 * ZPH(1)/WN(8,1) | TRAN2970 |
| | GP(8,1)=1.32E-10 * 1.32E-10 * ZPH(1) | TRAN2980 |
| | 1 / (FG(8,1) * WN(8,1)**2) | TRAN2990 |
| | FG(8,2)=(0.379 * ZPC(1) + 1.05 * ZPC(3))/WN(8,2) | TRAN3000 |
| | GP(8,2)=(1.95E-11 * SQRT(ZPC(1)) + 1.27E-10 * SQRT(ZPC(3)))**2 | TRAN3010 |
| | 1 / (FG(8,2) * WN(8,2)**2) | TRAN3020 |
| | FG(8,3)=(0.155 * ZFN(1) + 0.142 * ZPN(2) + 3.75E-2 * ZPN(3)) | TRAN3030 |
| | 1 / WN(8,3) | TRAN3040 |
| | GP(8,3)=(2.98E-11 * SQRT(ZPN(1)) + 7.08E-11 * SQRT(ZPN(2)) | TRAN3050 |
| | 1 + 1.33E-10 * SQRT(ZPN(3)))**2 / (FG(8,3) * WN(8,3)**2) | TRAN3060 |
| | FG(8,4)=(0.146 * ZPO(1) + 8.61E-2 * ZPO(2) | TRAN3070 |
| | 1 + 9.33E-2 * ZPO(3))/WN(8,4) | TRAN3080 |
| | GP(8,4)=(1.97E-10 * SQRT(ZPO(1)) + 1.80E-11 * SQRT(ZPO(2)) | TRAN3090 |
| | 1 + 8.13E-11 * SQRT(ZPO(3)))**2 / (FG(8,4) * WN(8,4)**2) | TRAN3100 |
| | FMUL(8,L)=FMUC(10,L) | TRAN3110 |
| C | GROUP 9 | TRAN3120 |
| | FG(9,2)=2.95 * ZPC(2)/WN(9,2) | TRAN3130 |
| | GP(9,2)=5.85E-12 * 5.85E-12 * ZPC(2)/(FG(9,2) * WN(9,2)**2) | TRAN3140 |
| | FG(9,3)=(0.224 * ZPN(1) + 2.92E-2 * ZPN(2))/WN(9,3) | TRAN3150 |

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      GP(9,3)=(3.41E-10 * SQRT(ZPN(1)) + 1.48E-10 * SQRT(ZPN(2)))**2  TRAN3160
1      /(FG(9,3) * WN(9,3)**2)  TRAN3170
      FG(9,4)=(5.24E-2 * ZPO(1) + 7.22E-2 * ZPO(2)  TRAN3180
1      + 6.04E-2 * ZPO(3))/WN(9,4)  TRAN3190
      GP(9,4)=(5.76E-12 * SQRT(ZPO(1)) + 7.20E-11 * SQRT(ZPO(2))  TRAN3200
1      + 8.05E-11 * SQRT(ZPO(3)))**2/(FG(9,4) * WN(9,4)**2)  TRAN3210
      FMUL(9,L)=FMUC(11,L)  TRAN3220
C  TRAN3230
C **  PLANCK FUNCTION  **  TRAN3240
C  TRAN3250
      DO 9 J=1,NHVL  TRAN3260
      BEEL(J,L)=5.04E3 * HVJ(J)**3 / (EXP(HVJ(J)/T1) - 1.0)  TRAN3270
C  TRAN3280
C **  INDUCED EMISSION FACTOR (EQ 81)  **  TRAN3290
C  TRAN3300
      SSM(J,1,L)=1.10E-16*SNDH (L)*(1.0-EXP(-HVJ(J)/T1)) * FG(J,1)  TRAN3310
      SSM(J,2,L)=1.10E-16*SNDC (L)*(1.0-EXP(-HVJ(J)/T1)) * FG(J,2)  TRAN3320
      SSM(J,3,L)=1.10E-16*SNDN (L)*(1.0-EXP(-HVJ(J)/T1)) * FG(J,3)  TRAN3330
      SSM(J,4,L)=1.10E-16*SNDD (L)*(1.0-EXP(-HVJ(J)/T1)) * FG(J,4)  TRAN3340
      DO 10 M=1,4  TRAN3350
      GGM(J,M,L)=GP(J,M) * SNDE(L) * (T(L)/1.0E4)**0.25  TRAN3360
1      + 1.0E-6  TRAN3370
      IF(L.GT.1) GO TO 11  TRAN3380
      ETAM(J,M,1)=0.  TRAN3390
      SBM (J,M,1)=0.  TRAN3400
      GO TO 10  TRAN3410
11 ETAM(J,M,L)=ETAM(J,M,L-1)+ (YD(L)-YD(L-1))  TRAN3420
1      *(SSM(J,M,L-1) * GGM(J,M,L-1) + SSM(J,M,L) * GGM(J,M,L))  TRAN3430
2      * DELTA/3.14159265  TRAN3440
      SBM(J,M,L)=SBM(J,M,L-1) + (YD(L)-YD(L-1))  TRAN3450
1      * (SSM(J,M,L-1)+SSM(J,M,L)) * DELTA  TRAN3460
10 CONTINUE  TRAN3470
      IF (L.GT.1) GO TO 12  TRAN3480
      TAUL(J,1)=0.  TRAN3490
      GO TO 9  TRAN3500

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| 12 | TAUL(J,L)=TAUL(J,L-1) + (YD(L)-YD(L-1)) | TRAN3510 |
| 1 | * (FMUL(J,L-1)+FMUL(J,L)) * DELTA | TRAN3520 |
| 9 | CONTINUE | TRAN3530 |
| | IF (IDG.NE.99) GO TO 91 | TRAN3540 |
| | CALL BUGPR (7) | TRAN3550 |
| C | | TRAN3560 |
| 91 | CONTINUE | TRAN3570 |
| | IEZ=IEZ+1 | TRAN3580 |
| | ETZ(IEZ)=1.0 | TRAN3590 |
| C | | TRAN3600 |
| C ** | CONTINUUM - CONTINUUM FLUX DIVERGENCE CALCULATION ** | TRAN3610 |
| C | | TRAN3620 |
| | DO 300 K=1,IEZ | TRAN3630 |
| | DO 31 LK=1,NES | TRAN3640 |
| | I=LK | TRAN3650 |
| | NUT(K)=I | TRAN3660 |
| | IF (ABS(ETZ(K)-ETA(LK)) - 1.0E-5) 300,300,31 | TRAN3670 |
| 31 | CONTINUE | TRAN3680 |
| 300 | CONTINUE | TRAN3690 |
| | DO 1612 J=1,9 | TRAN3700 |
| | QCLP(J)=0. | TRAN3710 |
| | QLCP(J)=0. | TRAN3720 |
| | QLLP(J)=0. | TRAN3730 |
| | DO 1612 L=1,NES | TRAN3740 |
| | FM(J,L)=0. | TRAN3750 |
| 1612 | FP(J,L)=0. | TRAN3760 |
| | DO 1613 L=1,IEZ | TRAN3770 |
| | QCL(L)=0. | TRAN3780 |
| | QLC(L)=0. | TRAN3790 |
| 1613 | QLL(L)=0. | TRAN3800 |
| | DO 49 IYY=1,IEZ | TRAN3810 |
| | IY=NUT(IYY) | TRAN3820 |
| | DO 20 K=1,12 | TRAN3830 |
| | FMC(K,IY)=0. | TRAN3840 |
| | FPC(K,IY)=0. | TRAN3850 |

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| IF (IY.EQ.1) GO TO 44 | TRAN3860 |
| DO 40 L=1,IY | TRAN3870 |
| C | TRAN3880 |
| C ** MINUS EMISSIVITY FUNCTION (EQ 47) * | TRAN3890 |
| C | TRAN3900 |
| EM(K,L)=1.0 - EXP(TAUC(K,L)-TAUC(K,IY)) | TRAN3910 |
| IF (L.EQ.1) GO TO 40 | TRAN3920 |
| C | TRAN3930 |
| C ** MINUS CONTINUUM FLUX (EQ 46) ** | TRAN3940 |
| C | TRAN3950 |
| FMC(K,IY)=FMC(K,IY) - (EM(K,L)-EM(K,L-1)) | TRAN3960 |
| 1 * (BEEC(K,L-1)+BEEC(K,L))/2. | TRAN3970 |
| 40 CONTINUE | TRAN3980 |
| 44 IF (IY.EQ.NES) GO TO 41 | TRAN3990 |
| DO 42 L=IY,NES | TRAN4000 |
| C | TRAN4010 |
| C ** POSITIVE EMISSIVITY FUNCTION (EQ 47) ** | TRAN4020 |
| C | TRAN4030 |
| EP(K,L)=1.0 - EXP(TAUC(K,IY)-TAUC(K,L)) | TRAN4040 |
| IF (L.EQ.IY) GO TO 42 | TRAN4050 |
| C | TRAN4060 |
| C ** POSITIVE EMISSIVITY CONTINUUM FLUX (EQ 46) ** | TRAN4070 |
| C | TRAN4080 |
| FPC(K,IY)=FPC(K,IY) + (EP(K,L)-EP(K,L-1)) | TRAN4090 |
| 1 * (BEEC(K,L-1)+BEEC(K,L))/2. | TRAN4100 |
| 42 CONTINUE | TRAN4110 |
| C | TRAN4120 |
| C ** POSITIVE EMISSIVITY CONTINUUM FLUX DIVERGENCE (EQ 51) ** | TRAN4130 |
| C | TRAN4140 |
| 41 QCCP(K)=6.2831853 * FMUC(K,IY) * | TRAN4150 |
| 1 (FMC(K,IY) + FPC(K,IY) - 2.0 * BEEC(K,IY)) | TRAN4160 |
| FMC(K,IY)=FMC(K,IY) * 3.14159265 | TRAN4170 |
| FPC(K,IY)=FPC(K,IY) * 3.14159265 | TRAN4180 |
| 20 CONTINUE | TRAN4190 |
| C | TRAN4200 |

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| C ** | DEBUG PRINT ** | TRAN4210 |
| C | | TRAN4220 |
| | IF (IDG.NE.99) GO TO 21 | TRAN4230 |
| | CALL BUGPR (3) | TRAN4240 |
| 21 | QCC(IYY)=0. | TRAN4250 |
| | DO 24 K=1,12 | TRAN4260 |
| C | | TRAN4270 |
| C ** | LINE AND CROSS TERM FLUX DIVERGENCE CALCULATION ** | TRAN4280 |
| C | | TRAN4290 |
| | 24 QCC(IYY)=QCC(IYY) + QCCP(K) | TRAN4300 |
| | IF (LINES.EQ.0) GO TO 1614 | TRAN4310 |
| C | | TRAN4320 |
| C ** | INTEGRATION FROM 1 TO IY ** | TRAN4330 |
| C | | TRAN4340 |
| | IF (IY.EQ.1) GO TO 68 | TRAN4350 |
| | DO 65 J=1,9 | TRAN4360 |
| | DO 66 L=1,IY | TRAN4370 |
| | WIM=0. | TRAN4380 |
| | SUM1=0. | TRAN4390 |
| | SUM2=0. | TRAN4400 |
| | DO 67 M=1,4 | TRAN4410 |
| | DIF=ETAM(J,M,IY) - ETAM(J,M,L) | TRAN4420 |
| | DIFSBM = SBM(J,M,IY)-SBM(J,M,L) | TRAN4430 |
| | IF (ABS(DIFSBM).LT.1.E-10) DIFSBM = 1.E-10 | TRAN4440 |
| | BETAM=DIF / (DIFSBM) * 3.14159265 | TRAN4450 |
| | IF (L.EQ.IY) BETAM=GGM(J,M,L) | TRAN4460 |
| | IF (ABS(DIF).GT.1.E-10) GO TO 9001 | TRAN4470 |
| | TM = 1.E-10 | TRAN4480 |
| | GO TO 9002 | TRAN4490 |
| 9001 | CONTINUE | TRAN4500 |
| | TM=DIF/2.0/BETAM**2 | TRAN4510 |
| 9002 | RRM=DIF/2.0/GGM(J,M,IY)**2 | TRAN4520 |
| | WWM=6.2831853 * WN(J,M) * BETAM * GAMMA(TM) * TM | TRAN4530 |
| | SUM1=SUM1 + GAMMA(TM) * WN(J,M) * SSM(J,M,IY) | TRAN4540 |
| | SUM2=SUM2 + XLAMB(RFM) * WN(J,M) * SSM(J,M,IY) | TRAN4550 |

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| 67 WIM=WIM + WWM | TRAN4560 |
| ALPHAM=WIM/DJ(J) | TRAN4570 |
| C | TRAN4580 |
| C ** OVERLAPPING LINE CALCULATIONS ** | TRAN4590 |
| C | TRAN4600 |
| C | TRAN4610 |
| C ** GROUP EQUIVALENT WIDTHS (EQ.88) ** | TRAN4620 |
| C | TRAN4630 |
| WMM(J,L)=DJ(J) * PHI1(ALPHAM) * EXP(TAUL(J,L)-TAUL(J,IY)) | TRAN4640 |
| C | TRAN4650 |
| C ** GROUP GAMMA -- LINE TRANSPORT FUNCTION (EQ.92) ** | TRAN4660 |
| C | TRAN4670 |
| GMM(J,L)=PHI2(ALPHAM) * SUM1 | TRAN4680 |
| C | TRAN4690 |
| C ** MINUS EMISSIVITY FUNCTION FOR LINES (EQ.47) ** | TRAN4700 |
| C | TRAN4710 |
| EFM(J,L)=1.0 - EXP(TAUL(J,L)-TAUL(J,IY)) | TRAN4720 |
| 66 XLMM(J,L)=PHI2(ALPHAM) * SUM2 | TRAN4730 |
| 65 CONTINUE | TRAN4740 |
| IF (IDG.EQ.99) CALL BUGPR(1) | TRAN4750 |
| IF (IDG.EQ.99) CALL BUGPR(4) | TRAN4760 |
| 68 IF (IY.EQ.NES) GO TO 72 | TRAN4770 |
| C | TRAN4780 |
| C ** INTEGRATION FROM IY TO NES ** | TRAN4790 |
| C | TRAN4800 |
| DO 69 J=1,9 | TRAN4810 |
| DO 70 L=IY,NES | TRAN4820 |
| WIP=0. | TRAN4830 |
| SUM1=0. | TRAN4840 |
| SUM2=0. | TRAN4850 |
| DO 71 M=1,4 | TRAN4860 |
| DIF=ETAM(J,M,L) - ETAM(J,M,IY) | TRAN4870 |
| DIFSBM = SBM(J,M,L)-SBM(J,M,IY) | TRAN4880 |
| IF(ABS(DIFSBM).LT.1.E-10) DIFSBM = 1.E-10 | TRAN4890 |
| BETAP=DIF / (DIFSBM) * 3.14159265 | TRAN4900 |

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| IF (L.EQ.IY) BETAP=GGM(J,M,L) | TRAN4910 |
| IF(ABS(DIF).GT.1.E-10) GO TO 9003 | TRAN4920 |
| TP = 1.E-10 | TRAN4930 |
| GO TO 9004 | TRAN4940 |
| 9003 CONTINUE | TRAN4950 |
| TP=DIF/2.0/BETAP**2 | TRAN4960 |
| 9004 RRP=DIF/2.0/GGM(J,M,IY)**2 | TRAN4970 |
| WWP=6.2831853 * WN(J,M) * BETAP * GAMMA(TP) * TP | TRAN4980 |
| SUM1=SUM1 + GAMMA(TP) * WN(J,M) * SSM(J,M,IY) | TRAN4990 |
| SUM2=SUM2 + XLAMB(RRP) * WN(J,M) * SSM(J,M,IY) | TRAN5000 |
| 71 WIP=WIP+WWP | TRAN5010 |
| ALPHAP=WIP/DJ(J) | TRAN5020 |
| WPP(J,L)=DJ(J) * PHI1(ALPHAP) * EXP(TAUL(J,IY)-TAUL(J,L)) | TRAN5030 |
| GPP(J,L)=PHI2(ALPHAP) * SUM1 | TRAN5040 |
| C | TRAN5050 |
| C ** POSITIVE EMISSIVITY FUNCTION FOR LINES (EQ.47) ** | TRAN5060 |
| C | TRAN5070 |
| EEP(J,L)=1.0 - EXP(TAUL(J,IY)-TAUL(J,L)) | TRAN5080 |
| 70 XLPP(J,L)=PHI2(ALPHAP) * SUM2 | TRAN5090 |
| 69 CONTINUE | TRAN5100 |
| C | TRAN5110 |
| C ** DEBUG PRINT ** | TRAN5120 |
| IF (IDG.EQ.99) CALL BUGPR (5) | TRAN5130 |
| C | TRAN5140 |
| 72 DO 80 J=1,9 | TRAN5150 |
| ASM1=0. | TRAN5160 |
| ASM2=0. | TRAN5170 |
| FM(J,IY)=0. | TRAN5180 |
| IF (IY.EQ.1) GO TO 81 | TRAN5190 |
| DO 82 L=2,IY | TRAN5200 |
| FM(J,IY)=FM(J,IY) - (WMM(J,L)-WMM(J,L-1)) | TRAN5210 |
| 1 *(BEEL(J,L-1)+BEEL(J,L)) * 1.5707963 | TRAN5220 |
| IF (L.EQ.IY) GO TO 82 | TRAN5230 |
| ASM1=ASM1 - (EEM(J,L)-EEM(J,L-1)) | TRAN5240 |
| 1 *(BEEL(J,L-1) * XLMP(J,L-1) + BEEL(J,L) * XLMM(J,L))/2. | TRAN5250 |

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| ASM2=ASM2 - (XLMM(J,L)-XLMM(J,L-1)) | TRAN5260 |
| 1 * (BEEL(J,L-1) * EXP(TAUL(J,L-1)-TAUL(J,IY)) + BEEL(J,L) | TRAN5270 |
| 2 * EXP(TAUL(J,L)-TAUL(J,IY)))/2.0 | TRAN5280 |
| 82 CONTINUE | TRAN5290 |
| 81 ASP1=0. | TRAN5300 |
| ASP2=0. | TRAN5310 |
| IYP=IY+1 | TRAN5320 |
| IF (IY.EQ.NES) GO TO 83 | TRAN5330 |
| DO 84 L=IYP,NES | TRAN5340 |
| FP(J,IY)=FP(J,IY) + (WPP(J,L)-WPP(J,L-1)) | TRAN5350 |
| 1 * (BEEL(J,L-1)+BEEL(J,L)) * 1.5707963 | TRAN5360 |
| IF (L.EQ.IYP) GO TO 84 | TRAN5370 |
| ASP1=ASP1 + (EEP(J,L)-EEP(J,L-1)) | TRAN5380 |
| 1 * (BEEL(J,L-1) * XLPP(J,L-1) + BEEL(J,L) * XLPP(J,L))/2.0 | TRAN5390 |
| ASP2=ASP2 + (XLPP(J,L)-XLPP(J,L-1)) * | TRAN5400 |
| 1 (BEEL(J,L-1) * EXP(TAUL(J,IY)-TAUL(J,L-1)) + BEEL(J,L) | TRAN5410 |
| 2 * EXP(TAUL(J,IY)-TAUL(J,L)))/2.0 | TRAN5420 |
| 84 CONTINUE | TRAN5430 |
| 83 QLCP(J)=2.0 * FMUL(J,IY) * (FM(J,IY)+FP(J,IY)) | TRAN5440 |
| SUMS=1.0 | TRAN5450 |
| SUMT=0. | TRAN5460 |
| DO 86 M=1,4 | TRAN5470 |
| 86 SUMT=SUMT + SSM(J,M,IY) * WN(J,M) | TRAN5480 |
| ATM1=0. | TRAN5490 |
| IF (IY.NE.1) ATM1=(BEEL(J,IY-1)+BEEL(J,IY)) /2.0 * EEM(J,IY-1) | TRAN5500 |
| 1 * XLMM(J,IY-1) | TRAN5510 |
| ATP1=0. | TRAN5520 |
| IF (IY.NE.NES) ATP1=(BEEL(J,IY+1)+BEEL(J,IY))/2.0 * EEP(J,IY+1) | TRAN5530 |
| 1 * XLPP(J,IY+1) | TRAN5540 |
| QCLP(J)=6.2831853 * SUMS * (ASM1+ASP1+ATM1+ATP1) | TRAN5550 |
| IF (IY.EQ.1) ATM2=-BEEL(J,IY) * SUMT | TRAN5560 |
| IF (IY.NE.1) ATM2=(BEEL(J,IY-1)-BEEL(J,IY)) * GMM(J,IY-1) | TRAN5570 |
| 1 - BEEL(J,IY-1) * XLMM(J,IY-1) | TRAN5580 |
| IF (IY.EQ.NES) ATP2=-BEEL(J,IY) * SUMT | TRAN5590 |
| IF (IY.NE.NES) ATP2=(BEEL(J,IY+1)-BEEL(J,IY)) * GPP(J,IY+1) | TRAN5600 |

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| 1 | - BEEL(J,IY+1) * XLPP(J,IY+1) | TRANS5610 |
| | QLLP(J)=6.2831853 * SUMS*(-ASM2-ASP2+ATM2+ATP2) | TRANS5620 |
| 80 | CONTINUE | TRANS5630 |
| | QCL(IYY)=0. | TRANS5640 |
| | QLC(IYY)=0. | TRANS5650 |
| | QLL(IYY)=0. | TRANS5660 |
| | DO 85 J=1,9 | TRANS5670 |
| | QCL(IYY)=QCL(IYY) + QCLP(J) | TRANS5680 |
| | QLC(IYY)=QLC(IYY) + QLCP(J) | TRANS5690 |
| 85 | QLL(IYY)=QLL(IYY) + QLLP(J) | TRANS5700 |
| 1614 | CONTINUE | TRANS5710 |
| | DQN(IYY)=- (QCC(IYY)+QCL(IYY)+QLC(IYY)+QLL(IYY)) | TRANS5720 |
| C | | TRANS5730 |
| C ** | DEBUG PRINT ** | TRANS5740 |
| C | | TRANS5750 |
| | IF (IDG.EQ.0) GO TO 49 | TRANS5760 |
| | CALL BUGPR(6) | TRANS5770 |
| 49 | CONTINUE | TRANS5780 |
| | IEZ=IEZ-1 | TRANS5790 |
| | DQ(1)=DQN(1) | TRANS5800 |
| | L=2 | TRANS5810 |
| | DO 1 N=2,NES | TRANS5820 |
| | DO 2 I=2,IEZ | TRANS5830 |
| | NP=I | TRANS5840 |
| | IF (ETZ(I).GT.ETA(N)) GO TO 3 | TRANS5850 |
| 2 | CONTINUE | TRANS5860 |
| 3 | NN=NP-1 | TRANS5870 |
| | AA=0.0 | TRANS5880 |
| | ZB=(DQN(NN)-DQN(NP)) / (ETZ(NN)-ETZ(NP)) | TRANS5890 |
| | CC=DQN(NN) - ZB * ETZ(NN) | TRANS5900 |
| | DQ(N)=AA * ETA(N)**2 + ZB * ETA(N) + CC | TRANS5910 |
| | GO TO 1 | TRANS5920 |
| 4 | DQ(N)=DQN(NN) | TRANS5930 |
| 1 | CONTINUE | TRANS5940 |
| C | | TRANS5950 |

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| C ** | NON-DIMENSIONALIZE E(I) | ** | TRAN5960 |
| | DO 250 I=1,NES | | TRAN5970 |
| | T(I) = T(I)/TD | | TRAN5980 |
| 250 | E(I) = ((DQ(I)*XL)/(RINF*UINF**3))*20866.0 | *RZB | TRAN5990 |
| | RETURN | | TRAN6000 |
| | END | | TRAN6010 |

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| SUBROUTINE TRANS2 | TRAN | 10 |
| COMMON /SFLUX/ QRI(3) | TRAN | 20 |
| COMMON /YL/ETA(60),YOND(60) | TRAN | 30 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R , RE, LXI, ITM, IEM, NES | TRAN | 40 |
| COMMON /TRN/ NUT(60), FMC(12,60), FPC(12,60), | TRAN | 50 |
| 1 FM(9,60), FP(9,60), LINES | TRAN | 60 |
| COMMON /FINV/ NHVL,NIHVC,FHVC(12),DJ(9),HVJ(9),ZKZ | TRAN | 70 |
| COMMON /TEST/ETZ(60),IEZ | TRAN | 80 |
| COMMON /NUMDEN/ SNDO2(60), SOND2(60), SNDO(60), SOND(60), | TRAN | 90 |
| 1 SNDE(60), SNDC(60), | TRAN | 100 |
| 2 SNDH(60), SNDC2(60), SNDH2(60), SNDCO(60), | TRAN | 110 |
| 3 SNDC3(60),SNDC2H(60) | TRAN | 120 |
| COMMON /SPEC/ MF, XMOL | TRAN | 130 |
| DIMENSION ETOUT(3) | TRAN | 140 |
| NETA=NES | TRAN | 150 |
| ETOUT(1)=0.0 | TRAN | 160 |
| ETOUT(2)=0.5 | TRAN | 170 |
| ETOUT(3)=1.0 | TRAN | 180 |
| NOUT=3 | TRAN | 190 |
| C | TRAN | 200 |
| C OUTPUT FLUX | TRAN | 210 |
| C | TRAN | 220 |
| WRITE (6,600) | TRAN | 230 |
| WRITE (6,603) (ETA(I),SOND2(I),SNDO2(I),SOND(I),SNDO(I), | TRAN | 240 |
| 1 SNDE(I), SONDH(I), | TRAN | 250 |
| 2 SNDC(I),SNDC2(I),SONDH2(I),SNDCO(I),SNDC3(I), | TRAN | 260 |
| 3 SNDC2H(I), I=1,NETA) | TRAN | 270 |
| C ** CONTINUUM CONTRIBUTION TO THE SPECTRAL FLUX ** | TRAN | 280 |
| C | TRAN | 290 |
| WRITE (6,4103) | TRAN | 300 |
| DO 8040 K=1,NOUT | TRAN | 310 |
| DO 8041 LK=1,NES | TRAN | 320 |
| NUT(K)=LK | TRAN | 330 |
| IF (ABS(ETOUT(K)-ETA(LK)) - 1.0E-05) 8040,8040,8041 | TRAN | 340 |
| 8041 CONTINUE | TRAN | 350 |

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| 8040 | CONTINUE | TRAN 360 |
| | L1=NUT(1) | TRAN 370 |
| | L2=NUT(2) | TRAN 380 |
| | L3=NUT(3) | TRAN 390 |
| | WRITE (6,8037) (ETOUT(IL),IL=1,3) | TRAN 400 |
| | FM1=0.0 | TRAN 410 |
| | FP1=0.0 | TRAN 420 |
| | FM2=0.0 | TRAN 430 |
| | FP2=0.0 | TRAN 440 |
| | FM3=0.0 | TRAN 450 |
| | FP3=0.0 | TRAN 460 |
| | DO 4104 KL=1,NIHVC | TRAN 470 |
| | WRITE (6,8042) KL, FHVC(KL), FMC(KL,L1), FPC(KL,L1), | TRAN 480 |
| | 1 FMC(KL,L2), FPC(KL,L2), FMC(KL,L3), FPC(KL,L3) | TRAN 490 |
| | FM1=FM1 + FMC(KL,L1) | TRAN 500 |
| | FP1=FP1 + FPC(KL,L1) | TRAN 510 |
| | FM2=FM2 + FMC(KL,L2) | TRAN 520 |
| | FP2=FP2 + FPC(KL,L2) | TRAN 530 |
| | FM3=FM3 + FMC(KL,L3) | TRAN 540 |
| | FP3=FP3 + FPC(KL,L3) | TRAN 550 |
| 4104 | CONTINUE | TRAN 560 |
| | WRITE (6,8045) FM1, FP1, FM2, FP2, FM3, FP3 | TRAN 570 |
| | QRI(1)=FM1+FP1 | TRAN 580 |
| | QRI(2)=FM2+FP2 | TRAN 590 |
| | QRI(3)=FM3+FP3 | TRAN 600 |
| C | | TRAN 610 |
| C ** | LINE CONTRIBUTION TO THE SPECTRAL FLUX ** | TRAN 620 |
| C | | TRAN 630 |
| | IF (LINES.EQ.0) RETURN | TRAN 640 |
| | WRITE (6,8035) | TRAN 650 |
| | WRITE (6,8037) (ETOUT(IL),IL=1,3) | TRAN 660 |
| | FM1=0.0 | TRAN 670 |
| | FP1=0.0 | TRAN 680 |
| | FM2=0.0 | TRAN 690 |
| | FP2=0.0 | TRAN 700 |

| | |
|--|----------|
| FM3=0.0 | TRAN 710 |
| FP3=0.0 | TRAN 720 |
| C | TRAN 730 |
| C ** TOTAL FLUX CALCULATION ** | TRAN 740 |
| C | TRAN 750 |
| DO 8043 KL=1,NHVL | TRAN 760 |
| WRITE (6,8042) KL, HVJ(KL), FM(KL,L1), FP(KL,L1), | TRAN 770 |
| 1 FM(KL,L2), FP(KL,L2), FM(KL,L3), FP(KL,L3) | TRAN 780 |
| FM1=FM1 + FM(KL,L1) | TRAN 790 |
| FP1=FP1 + FP(KL,L1) | TRAN 800 |
| FM2=FM2 + FM(KL,L2) | TRAN 810 |
| FP2=FP2 + FP(KL,L2) | TRAN 820 |
| FM3=FM3 + FM(KL,L3) | TRAN 830 |
| FP3=FP3 + FP(KL,L3) | TRAN 840 |
| 8043 CONTINUE | TRAN 850 |
| WRITE (6,8045) FM1, FP1, FM2, FP2, FM3, FP3 | TRAN 860 |
| QRI(1)=QRI(1) + FM1 + FP1 | TRAN 870 |
| QRI(2)=QRI(2) + FM2 + FP2 | TRAN 880 |
| QRI(3)=QRI(3) + FM3 + FP3 | TRAN 890 |
| C | TRAN 900 |
| 600 FORMAT (1H1,33HNUMBER DENSITIES (PARTICLES/CM3) ///5X,3HETA, 8X, | TRAN 910 |
| 1 2HN2, 8X,2HO2, 8X,1HN, 8X,1HC, 8X, 2HE-,8X, | TRAN 920 |
| 2 1HH,8X,1HC, 8X,2HC2, 8X,2HH2, 8X,2HCQ, 8X,2HC3,8X,3HC2H///) | TRAN 930 |
| 603 FORMAT (1P13E10.2) | TRAN 940 |
| 4103 FORMAT (44H1CONTINUUM CONTRIBUTION TO THE SPECTRAL FLUX) | TRAN 950 |
| 8035 FORMAT (39H0LINE CONTRIBUTION TO THE SPECTRAL FLUX) | TRAN 960 |
| 8037 FORMAT (/22X,5HETA =F7.3,13X,5HETA =F7.3,13X,5HETA =F7.3//3X,1HI, | TRAN 970 |
| 1 3X,3HHNU,8X,6HQMINUS,7X,5HQPLUS,8X,6HQMINUS,7X,5HQPLUS,8X, | TRAN 980 |
| 2 6HQMINUS,7X,5HQPLUS/) | TRAN 990 |
| 8042 FORMAT (14,F8.3,1PEE13.3) | TRAN1000 |
| 8045 FORMAT (12H0TOTAL FLUX ,1PEE13.3) | TRAN1010 |
| RETURN | TRAN1020 |
| END | TRAN1030 |

| | | | |
|---|---|----------|---------|
| | SUBROUTINE SND(I) | | SND(10 |
| | COMMON/PROP1/PI(60),RHO(60), T(60),AMW(60),C (20,60),EC(5,60) | | SND(20 |
| | COMMON /RFLUX/ E(60),IRAD,ITYPE | | SND(30 |
| | COMMON /NON/RDZ,MUDZ,RMDZ,AKNF,HNF,CFNF | | SND(40 |
| | COMMON/WT/SMW(20),AWT(5) | | SND(50 |
| | COMMON /NUMDEN/ SNDO2(60), SNDN2(60), SNDO(60), SNDN(60), | | SND(60 |
| | 1 SNDE(60), SNDC(60), | | SND(70 |
| | 2 SNDH(60), SNDC2(60), SNDH2(60), SNDCO(60), | | SND(80 |
| | 3 SNDC3(60),SNDCH(60) | | SND(90 |
| C | ** CALCULATE SPECIE NUMBER DENSITIES BASED ON MOLE FRACTIONS ** | SND(100 | |
| C | | SND(110 | |
| | CONVER = 3.10375E+23 *RHO(I) *RDZ | SND(120 | |
| C | | SND(130 | |
| | SNDO2(I) = CCNVER * C(1,I)/SMW(1) | SND(140 | |
| | SNDN2(I) = CONVER * C(2,I)/SMW(2) | SND(150 | |
| | SNDO (I) = CCNVER * C(3,I)/SMW(3) | SND(160 | |
| | SNDN (I) = CCNVER * C(4,I)/SMW(4) | SND(170 | |
| | SNDE (I) = CCNVER * C(7,I)/SMW(7) | SND(180 | |
| | SNDC (I) = CONVER * C(8,I)/SMW(8) | SND(190 | |
| | SNDH (I) = CONVER * C(9,I)/SMW(9) | SND(200 | |
| | SNDH2(I) = CCNVER * C(10,I)/SMW(10) | SND(210 | |
| | SNDCO(I) = CCNVER * C(11,I)/SMW(11) | SND(220 | |
| | SNDC3(I) = CCNVER * C(12,I)/SMW(12) | SND(230 | |
| | SNDC2(I) = CONVER * C(19,I)/SMW(19) | SND(240 | |
| | SNDCH(I) = CCNVER * C(14,I)/SMW(14) | SND(250 | |
| | RETURN | SND(260 | |
| | END | SND(270 | |

| | | |
|---|---|----------|
| C | SUBROUTINE ZP(T1,SUMN,SUM0,SUMH,SUMC) | ZP(T 10 |
| C | ** FRACTIONAL POPULATION STATES FOR N, O, H, C ** | ZP(T 20 |
| C | | ZP(T 30 |
| C | COMMON /ZP1/ ZPO(6), ZPN(6), ZPH(2), ZPC(7) | ZP(T 40 |
| | ZPH(1)=2.0/SUMH | ZP(T 50 |
| | ZPH(2)=8.0 * EXP(-10.20/T1)/SUMH | ZP(T 60 |
| | ZPC(1)=9.0/SUMC | ZP(T 70 |
| | ZPC(2)=5.0 * EXP(-1.264/T1)/SUMC | ZP(T 80 |
| | ZPC(3)=EXP(-2.684/T1)/SUMC | ZP(T 90 |
| | ZPC(4)=5.0 * EXP(-4.183/T1)/SUMC | ZP(T 100 |
| | ZPC(5)=12.0 * EXP(-7.532/T1)/SUMC | ZP(T 110 |
| | ZPC(6)=36.0*EXP(-8.722/T1)/SUMC | ZP(T 120 |
| | ZPC(7)=60.0 * EXP(-9.724/T1)/SUMC | ZP(T 130 |
| | ZPN(1)=4.0/SUMN | ZP(T 140 |
| | ZPN(2)=10.0* EXP(-2.384/T1)/SUMN | ZP(T 150 |
| | ZPN(3)=6.0 * EXP(-3.576/T1)/SUMN | ZP(T 160 |
| | ZPN(4)=18.0 * EXP(-10.452/T1)/SUMN | ZP(T 170 |
| | ZPN(5)=54.0 * EXP(-11.877/T1)/SUMN | ZP(T 180 |
| | ZPN(6)=90.0 * EXP(-13.002/T1)/SUMN | ZP(T 190 |
| | ZPO(1)=9.0/SUM0 | ZP(T 200 |
| | ZPO(2)=5.0 * EXP(-1.967/T1)/SUM0 | ZP(T 210 |
| | ZPO(3)=EXP(-4.188/T1)/SUM0 | ZP(T 220 |
| | ZPO(4)=8.0 * EXP(-9.283/T1)/SUM0 | ZP(T 230 |
| | ZPO(5)=24.0 * EXP(-10.830/T1)/SUM0 | ZP(T 240 |
| | ZPO(6)=40.0 * EXP(-12.077/T1)/SUM0 | ZP(T 250 |
| C | RETURN | ZP(T 260 |
| | END | ZP(T 270 |
| | | ZP(T 280 |
| | | ZP(T 290 |

| | | |
|--|------|-----|
| SUBROUTINE BUGPR (IDGSW) | BUGP | 10 |
| COMMON /FRSTRM/ U INF, RINF, UINF2, R, RE, LXI, ITM, IEM, NES | BUGP | 20 |
| COMMON /YL/ETA(60), YD(60) | BUGP | 30 |
| COMMON /TRN/ NUT(60), FMC(12,60), FPC(12,60), | BUGP | 40 |
| 1 FM(9,60), FP(9,60), LINES | BUGP | 50 |
| COMMON /DEBUG/ QLC(60), QCL(60), QLL(60), DQN(60), QCC(60), | BUGP | 60 |
| 1 BEEC(12,60), FMUC(12,60), EM(12,60), | BUGP | 70 |
| 2 EP(12,60), TAUC(12,60), BEEL(9,60), | BUGP | 80 |
| 3 QCCP(12), WMM(9,60), GMM(9,60), | BUGP | 90 |
| 4 EEM(9,60), XLMM(9,60), QLCP(9), | BUGP | 100 |
| 5 QCLP(9), QLLP(9), DELTA, IY, IYY, | BUGP | 110 |
| 6 WPP(9,60), GPP(9,60), EEP(9,60), | BUGP | 120 |
| 7 XLPP(9,60), FG(9,4), GP(9,4), | BUGP | 130 |
| 8 WN(9,4), FMUL(9,60), SSM(9,4,60), | BUGP | 140 |
| 9 GGM(9,4,60), ETAM(9,4,60), SBM(9,4,60), | BUGP | 150 |
| A TAUL(9,60) | BUGP | 160 |
| GO TO (10,20,30,40,50,60,70), IDGSW | BUGP | 170 |
| 10 WRITE (6,194) | BUGP | 180 |
| 194 FORMAT (1H1) | BUGP | 190 |
| RETURN | BUGP | 200 |
| 20 WRITE (6,7182) DELTA | BUGP | 210 |
| 7182 FORMAT (7H0DELTA=1PE14.7,3H CM) | BUGP | 220 |
| RETURN | BUGP | 230 |
| 30 WRITE (6,190) IY, YD(IY) | BUGP | 240 |
| 190 FORMAT (4H1IY=I3,2X,3HYD=1PE12.5//2X,1HK,2X,1HL,7X,3HETA,13X,2HYD, | BUGP | 250 |
| 1 13X,2HMU,11X,3HTAU,14X,1HE,11X,3HBEE//) | BUGP | 260 |
| DO 22 K=1,12 | BUGP | 270 |
| IF (IY.EQ.1) GO TO 23 | BUGP | 280 |
| WRITE(6,191) (K, L, ETA(L), YD(L), FMUC(K,L), TAUC(K,L), | BUGP | 290 |
| 1 EM(K,L), BEEC(K,L), L=1,IY) | BUGP | 300 |
| 191 FORMAT (2I3,1P6E15.5) | BUGP | 310 |
| WRITE (6,192) | BUGP | 320 |
| 192 FORMAT (//) | BUGP | 330 |
| 23 IF (IY.EQ.NES) GO TO 22 | BUGP | 340 |
| WRITE (6,191) (K, L, ETA(L), YD(L), FMUC(K,L), | BUGP | 350 |

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1          TAUC(K,L), EP(K,L), BEEC(K,L), L=IY,NES)          BUGP 360
22 WRITE (6,193) FMC(K,IY), FPC(K,IY), QCCP(K)              BUGP 370
193 FORMAT (SH0FIM=1PE12.5 ,2X,4HFIP=E12.5,2X,5HQCCP=E12.5)  BUGP 380
      RETURN                                                  BUGP 390
40          WRITE (6,195) IY, YD(IY), ((J, L, YD(L),          BUGP 400
1          WMM(J,L), GMM(J,L), XLMM(J,L), EEM(J,L),          BUGP 410
2          BEEL(J,L), L=1,IY), J=1,9)                        BUGP 420
195 FORMAT (4H0IY=I3,2X,3HYY=1PE12.5//2X,1HJ,2X,1HL,7X,2HYD,12X,3HWMM,BUGP 430
1 12X,3HGMM,11X,4HXLMM,13X,3HEEM,13X,3HBEE// (2I3,6E16.5))  BUGP 440
      RETURN                                                  BUGP 450
50          WRITE (6,196) IY, YD(IY), ((J, L, YD(L),          BUGP 460
1          WPP(J,L), GPP(J,L), XLPP(J,L), EEP(J,L),          BUGP 470
2          BEEL(J,L), L=IY,NES), J=1,9)                      BUGP 480
196 FORMAT (4H0IY=I3,2X,3HYY=1PE12.5//2X,1HJ,2X,1HL,7X,2HYD,13X,3HWPP,BUGP 490
1 2X,3HGPP,11X,4HXLPP,13X,3HEEP,13X,3HBEE// (2I3,6E16.5))  BUGP 500
      RETURN                                                  BUGP 510
60 WRITE (6,198) IY, ETA(IY), YD(IY)                          BUGP 520
198 FORMAT (4H0IY=I3,2X,4HETA=1PE12.5,2X,3HYY=E12.5//2X,1HJ,5X,3HQCC,BUGP 530
1 11X,3HFMC,11X,3HFPC,11X,3HQCL,11X,3HQLC,11X,3HQLL,12X,2HFM,12X,BUGP 540
2 2HFP,11X,3HDQN//)                                          BUGP 550
      WRITE (6,199) (J, QCCP(J), FMC(J,IY), FPC(J,IY),          BUGP 560
1          QCLP(J), QLCP(J), QLLP(J), FM(J,IY),FP(J,IY),      BUGP 570
2          J=1,9)                                             BUGP 580
199 FORMAT (I3,1P8E14.5)                                     BUGP 590
      WRITE (6,8069) (J, QCCP(J), FMC(J,IY), FPC(J,IY), J=10,12)BUGP 600
8069 FORMAT (I3,1P3E14.5)                                     BUGP 610
      WRITE (6,200) QCC(IYY), QCL(IYY), QLC(IYY), QLL(IYY),    BUGP 620
1          DQN(IYY)                                           BUGP 630
200 FORMAT (1H0,2X,1PE14.5,28X,3E14.5,28X,E14.5)           BUGP 640
      RETURN                                                  BUGP 650
70 WRITE (6,197) L, ETA(L), YD(L), ((J, M, WN(J,M),          BUGP 660
1          FG(J,M), GP(J,M), FMUL(J,L), TAUL(J,L),          BUGP 670
2          SSM(J,M,L), GGM(J,M,L), ETAM(J,M,L), SBM(J,M,L),BUGP 680
3          M=1,4),J=1,9)                                     BUGP 690
197 FORMAT (3H0L=I3,2X,4HETA=1PE12.5,2X,3HYD=E12.5//2X,1HJ,2X,1HM,7X,BUGP 700

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1 1HN.13X.1HF.13X.1HG.11X.3HFNU.11X.3HTAU.11X.3HSSM.11X.3HGGM.10X. BUGP 710
2 4HETAM.11X.3HSBM//((213,9E14.5)) BUGP 720
  RETURN BUGP 730
  END BUGP 740
```

| | | | | |
|--------------------------------|---|-------------------|-------------------|----------|
| SUBROUTINE ZHV(HV,ZO,ZN,ZI,ZC) | | | | ZHV(10 |
| C | | | | ZHV(20 |
| C | | | | ZHV(30 |
| C | ** THIS SUBROUTINE CALCULATES THE QUANTUM MECHANICAL CORRECTION | | | ZHV(40 |
| C | FACTORS GIVEN A FREQUENCY (HV) ** | | | ZHV(50 |
| C | | | | ZHV(60 |
| | X= HV | | | ZHV(70 |
| | X2 =X*X | | | ZHV(80 |
| | X3 =X2*X | | | ZHV(90 |
| | X4 =X3*X | | | ZHV(100 |
| | X5 =X4*X | | | ZHV(110 |
| | X6 =X5*X | | | ZHV(120 |
| | X7 =X6*X | | | ZHV(130 |
| | IF (X -9.82) 1,1,2 | | | ZHV(140 |
| 1 | ZO = .9999795 | - .3155480*X | +2.824548 E-02*X2 | ZHV(150 |
| 1 | +6.677328 E-03*X3 | -3.644585 E-03*X4 | +8.058070 E-04*X5 | ZHV(160 |
| 2 | -7.708637 E-05*X6 | +2.668133 E-06*X7 | | ZHV(170 |
| | GO TO 3 | | | ZHV(180 |
| 2 | ZO = (X/9.82)**3 | | | ZHV(190 |
| 3 | IF (X -8.35) 4,4,5 | | | ZHV(200 |
| 4 | ZN = 1.000148 | - .4183535 *X | + .1680359 *X2 | ZHV(210 |
| 1 | -9.779458 E-02*X3 | +3.354635 E-02*X4 | -5.609353 E-03*X5 | ZHV(220 |
| 2 | +4.515535E-04*X6 | -1.403585 E-05*X7 | | ZHV(230 |
| | GO TO 6 | | | ZHV(240 |
| 5 | ZN = (X/8.35)**3 | | | ZHV(250 |
| 6 | Y = X/4.0 | | | ZHV(260 |
| | IF (Y-6.6) 9,9,10 | | | ZHV(270 |
| 9 | Y2 =Y*Y | | | ZHV(280 |
| | Y3 =Y2*Y | | | ZHV(290 |
| | Y4 =Y3*Y | | | ZHV(300 |
| | Y5 =Y4*Y | | | ZHV(310 |
| | Y6 =Y5*Y | | | ZHV(320 |
| | Y7 =Y6*Y | | | ZHV(330 |
| | ZI = 1.000379 | - .2964767 *Y | +7.505242 E-02*Y2 | ZHV(340 |
| 1 | -1.702948E-02*Y3 | +3.279554 E-03*Y4 | -2.128469 E-04*Y5 | ZHV(350 |

| | | | | |
|----|----------------------|-------------------|-------------------|----------|
| | GO TO 11 | | | ZHV(360 |
| 10 | ZI = (Y/6.6)**3 | | | ZHV(370 |
| 11 | IF (X-7.37) 12,12,13 | | | ZHV(380 |
| 12 | ZC = .9974367 | - .4341812 *X | +8.531314 E-02*X2 | ZHV(390 |
| | 1 -1.393917 E-02*X3 | +4.038545 E-03*X4 | -5.426425 E-04*X5 | ZHV(400 |
| | 2 +2.812126 E-05*X6 | -3.883530 E-07*X7 | | ZHV(410 |
| | GO TO 14 | | | ZHV(420 |
| 13 | ZC = (X/7.37)**3 | | | ZHV(430 |
| 14 | RETURN | | | ZHV(440 |
| | END | | | ZHV(450 |

APPENDIX C
DETAILED RESULTS

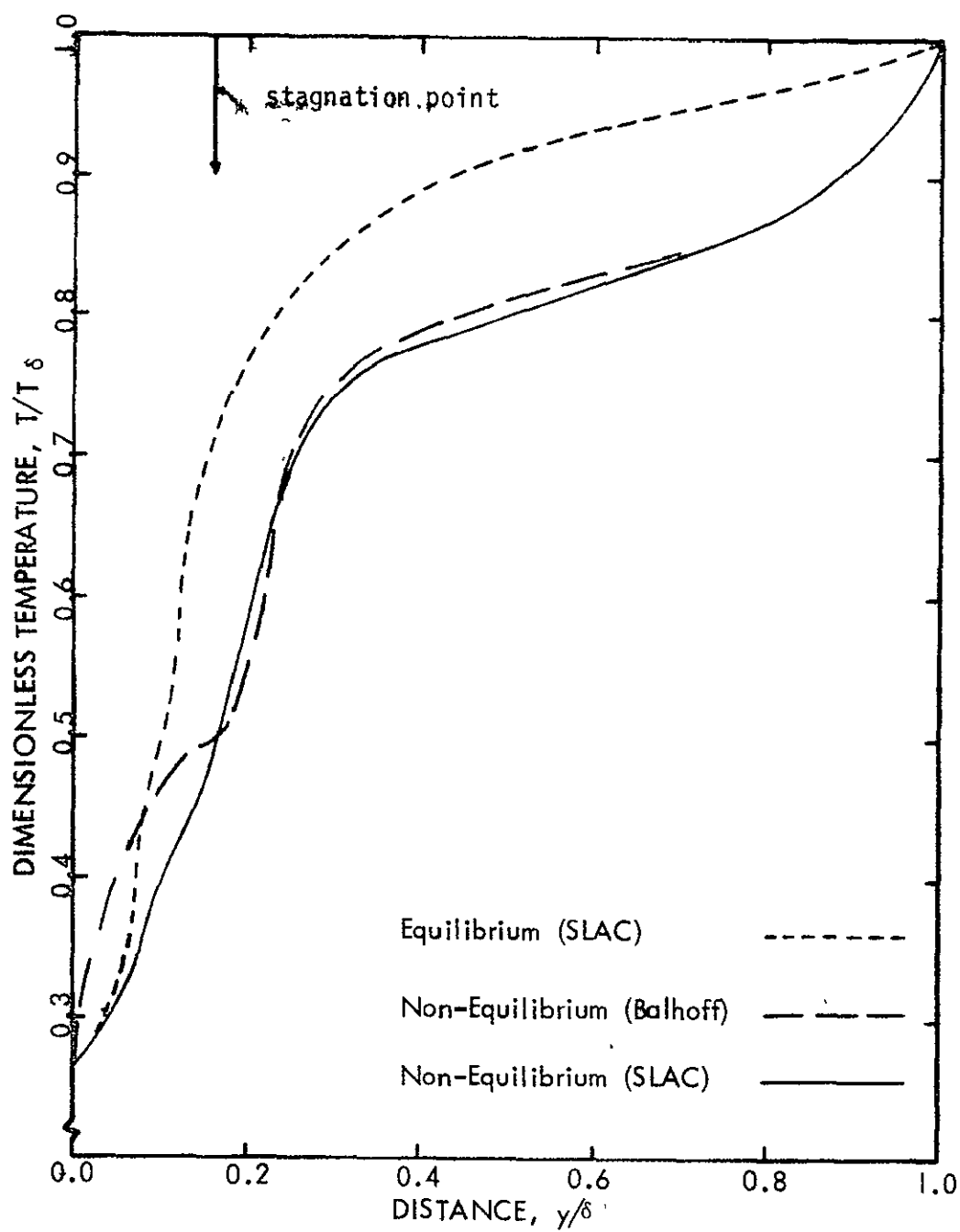


Figure C.1 Temperature Profiles

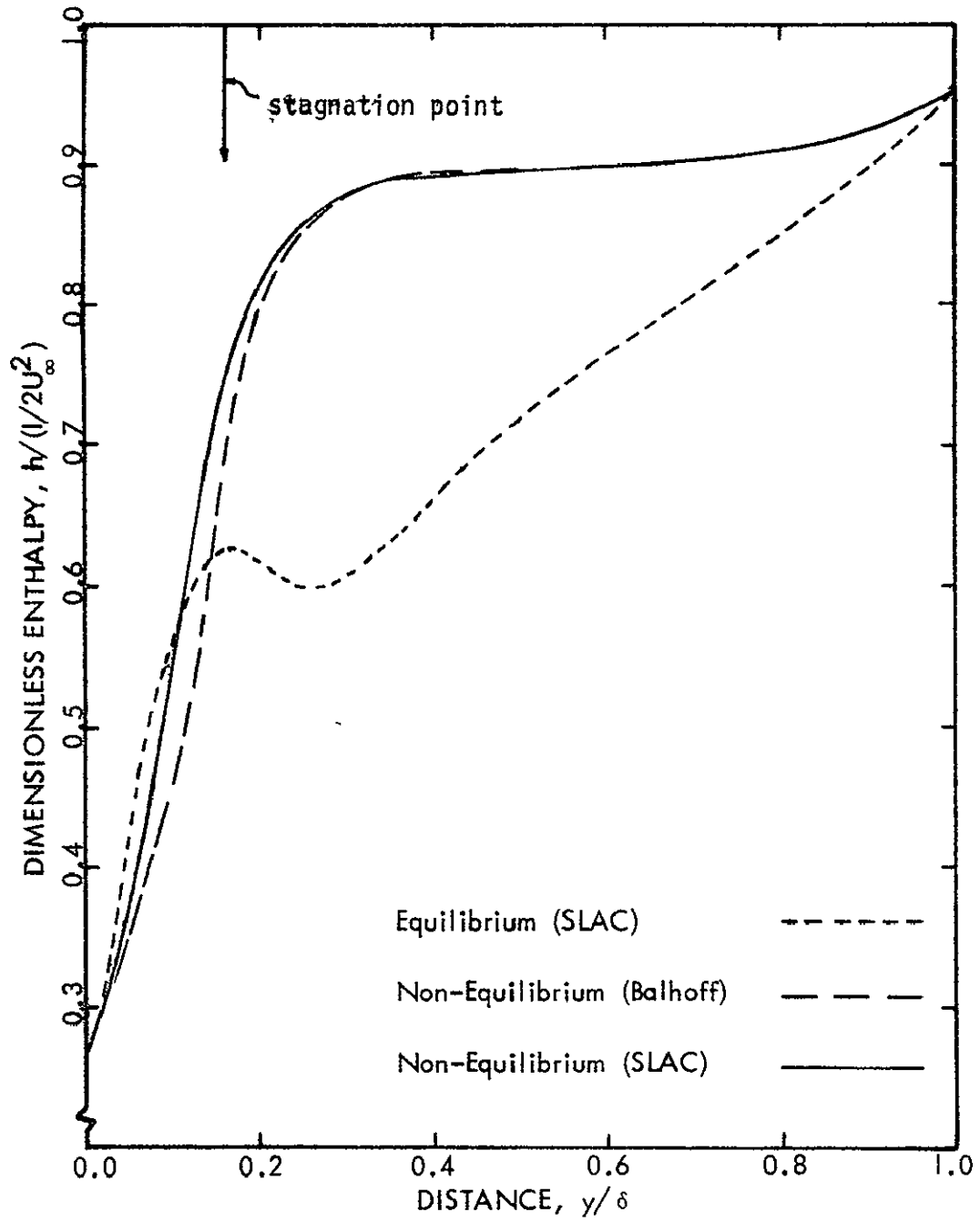


Figure 6.2 Enthalpy Profiles

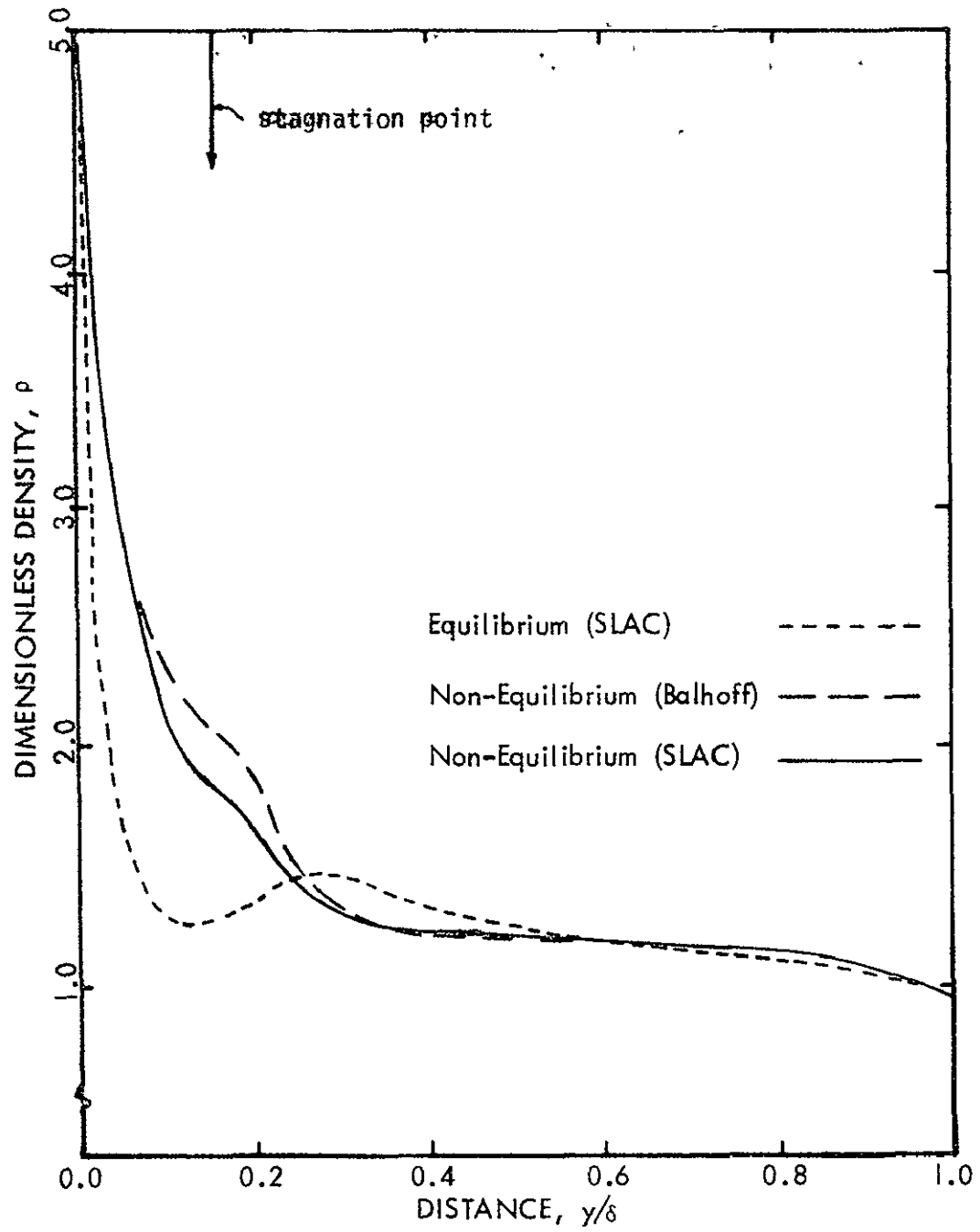


Figure C.3 Density Profiles

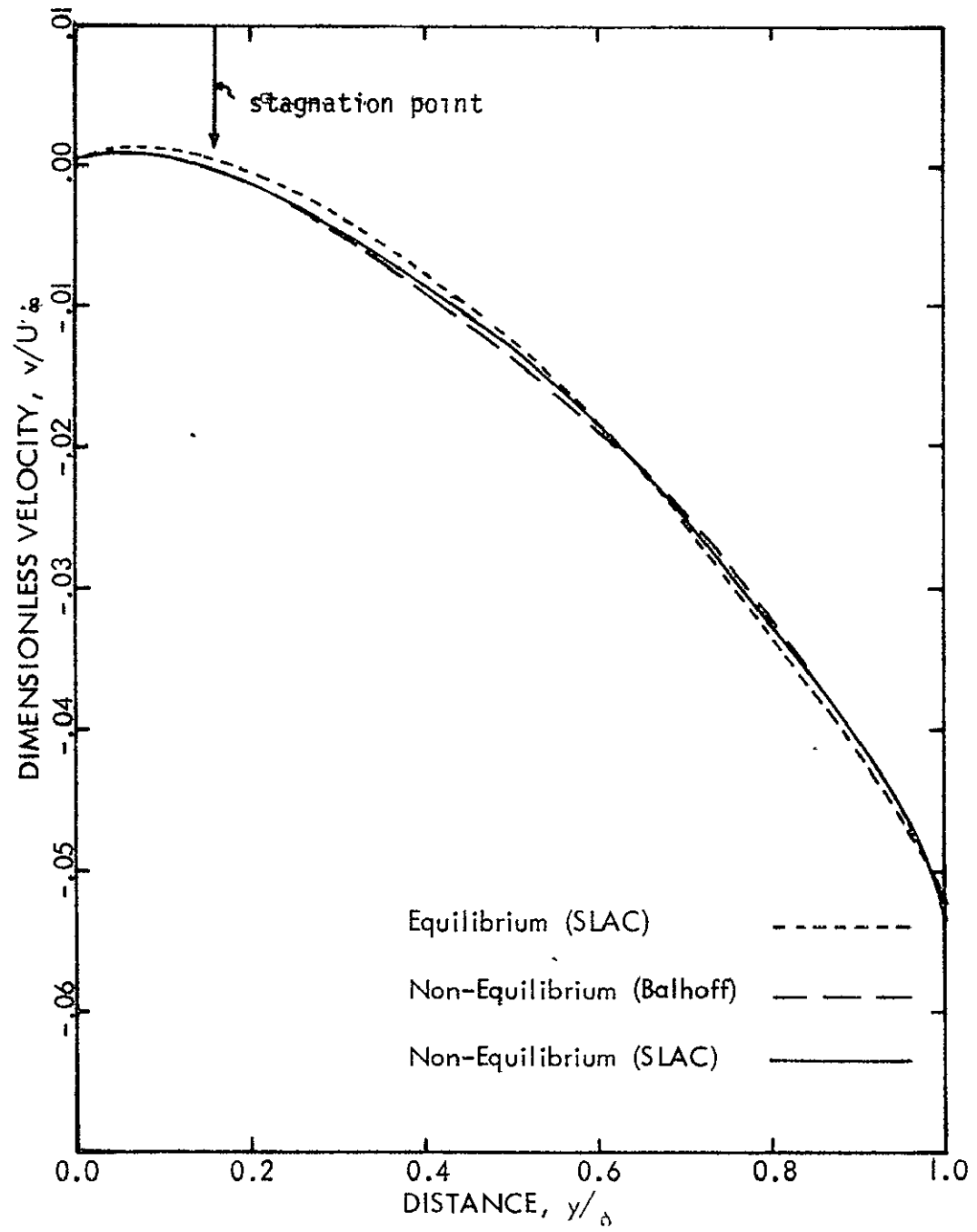


Figure C.4 Velocity Profiles

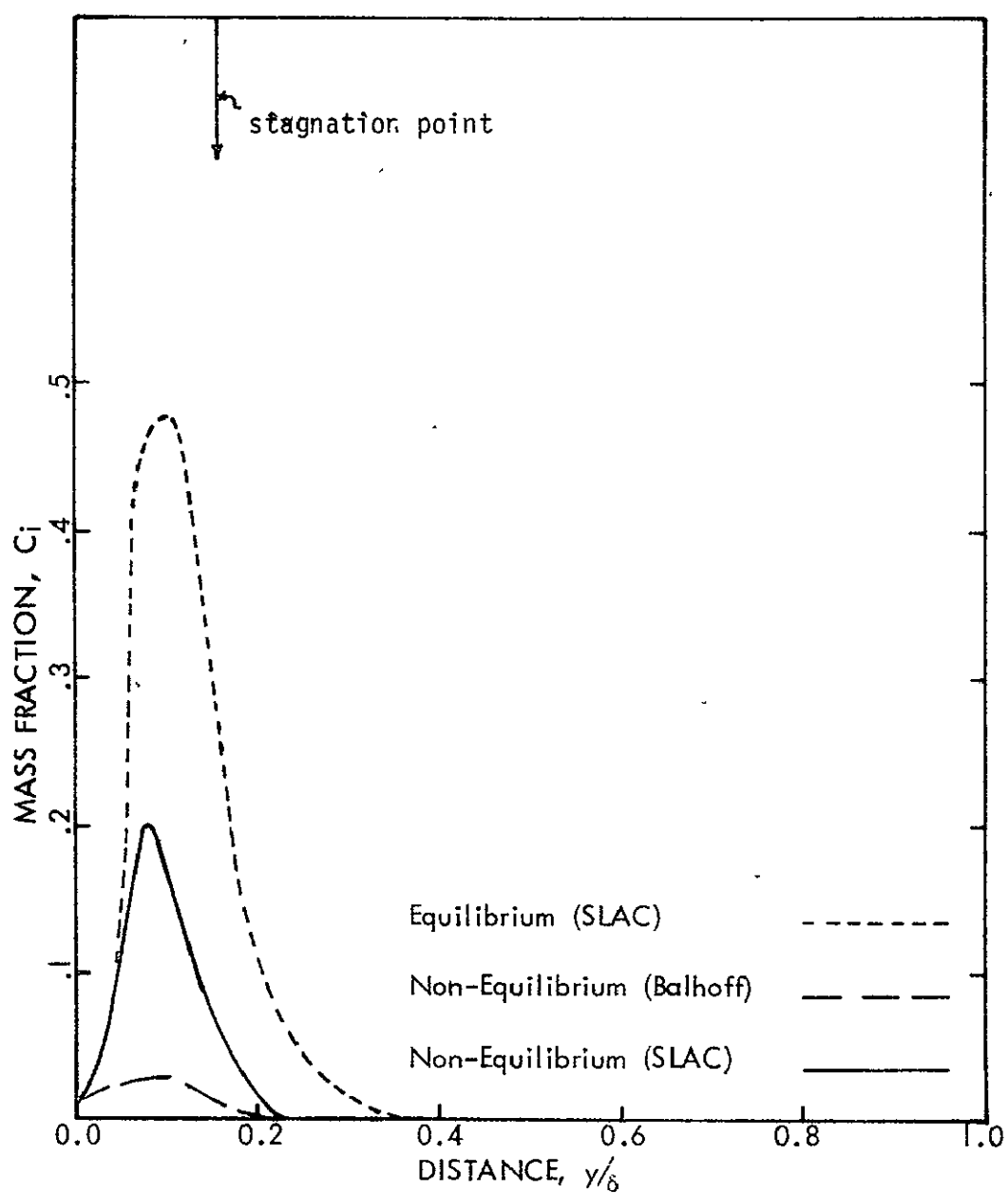


Figure 6.5 Mass Fraction Profiles for C

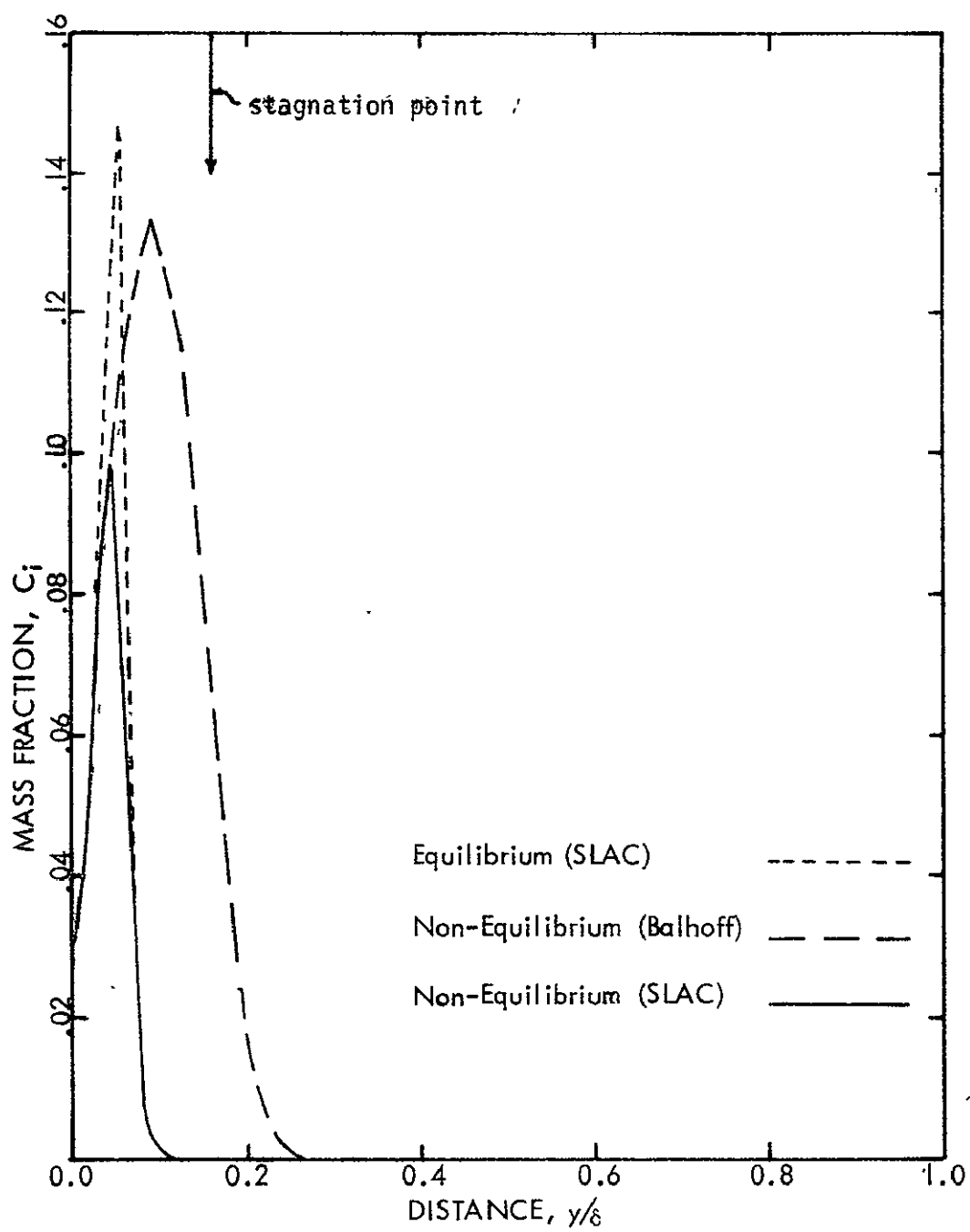


Figure C.6 Mass Fraction Profiles for C_2

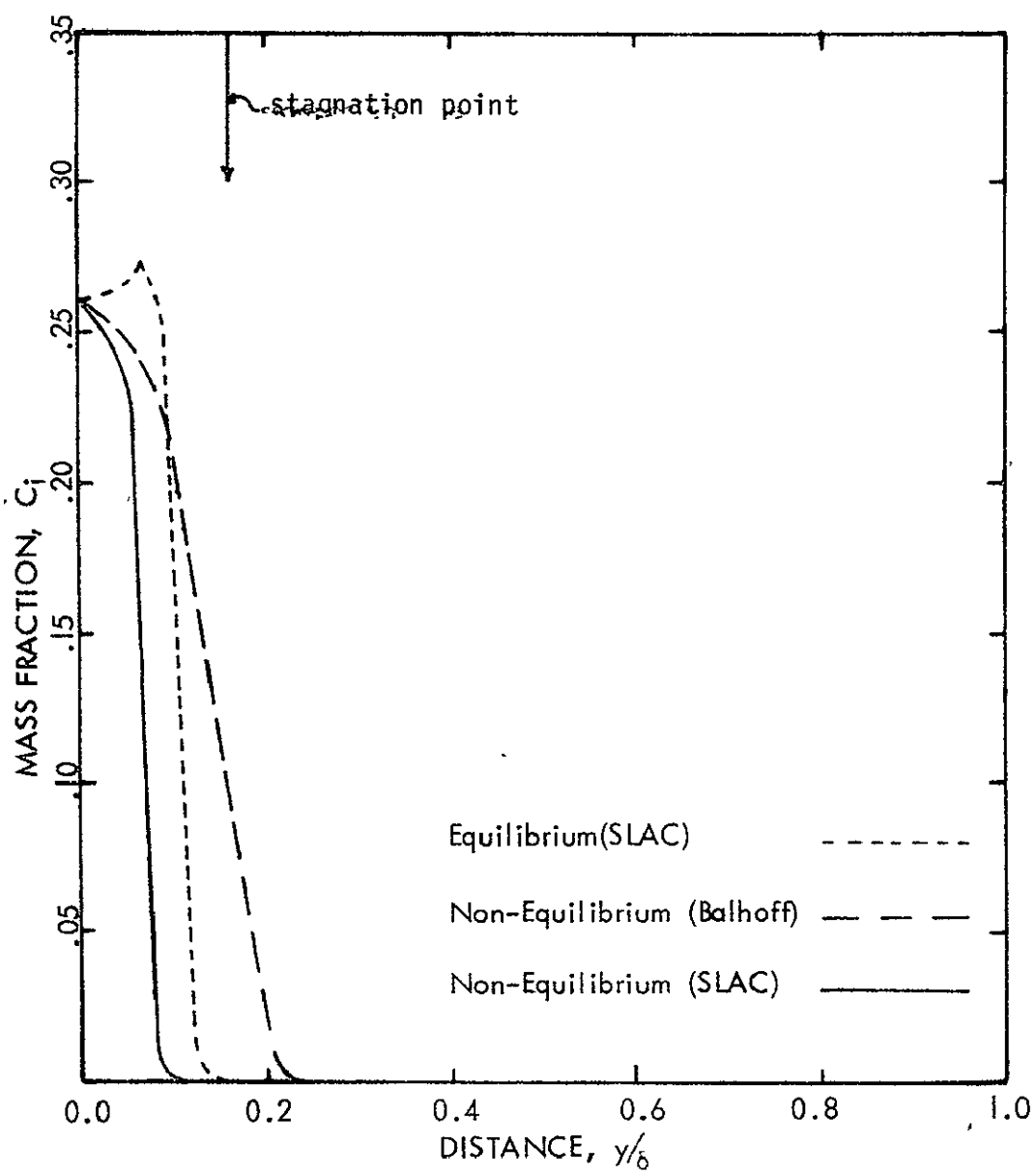


Figure C.7 Mass Fraction Profiles for CO

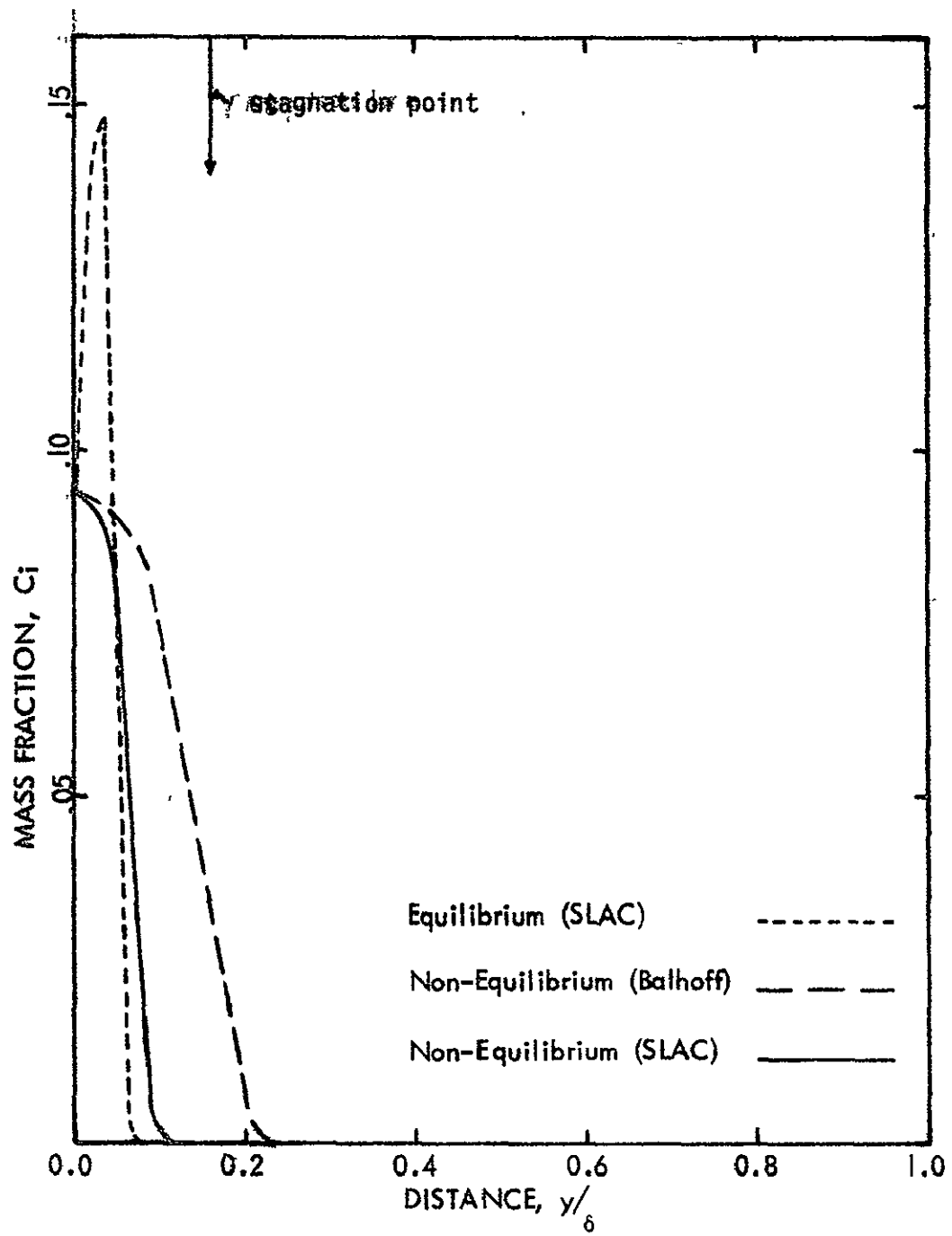


Figure 3.8 Mass Fraction Profiles for C_3

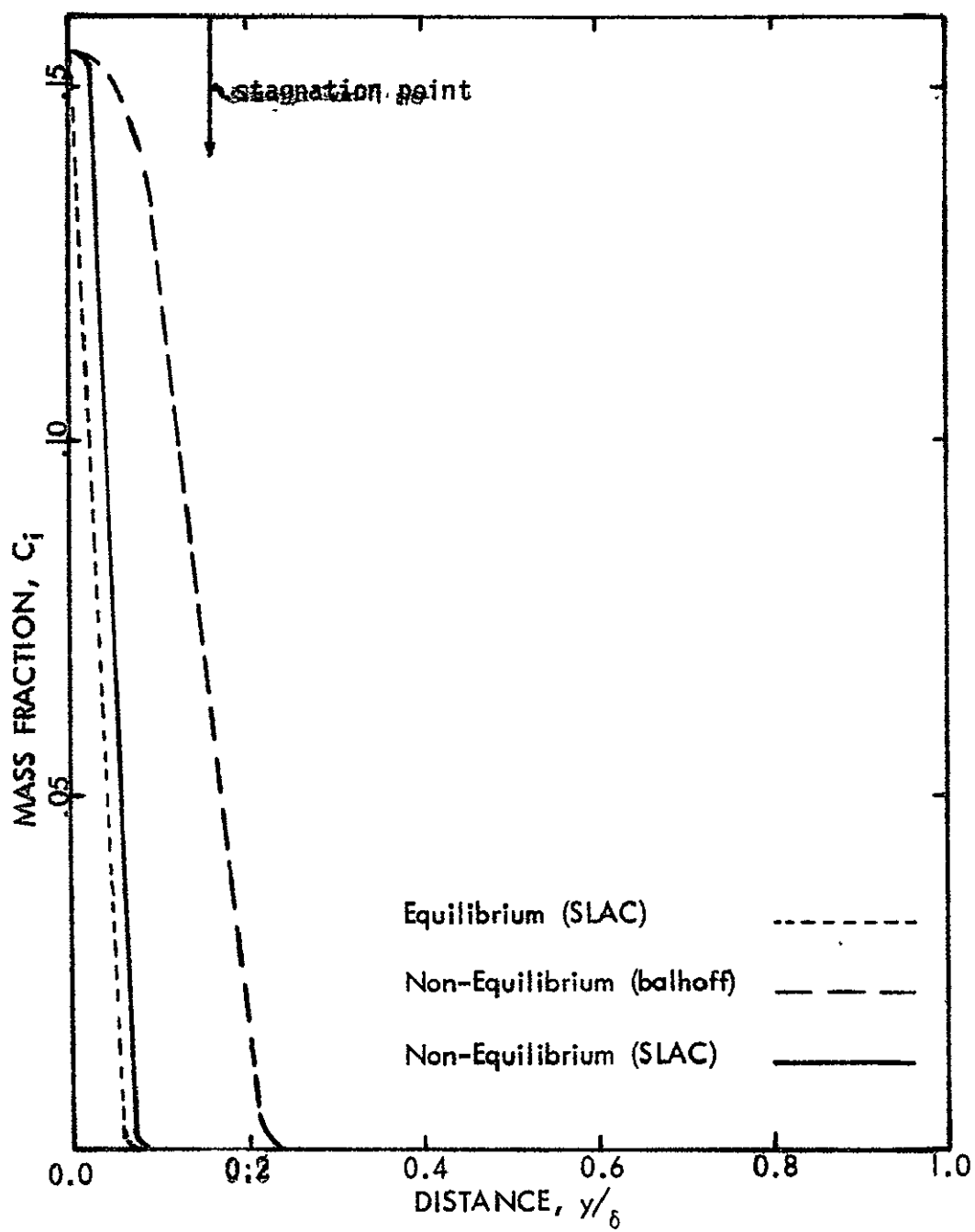


Figure 6.9 Mass Fraction Profiles for C_3H

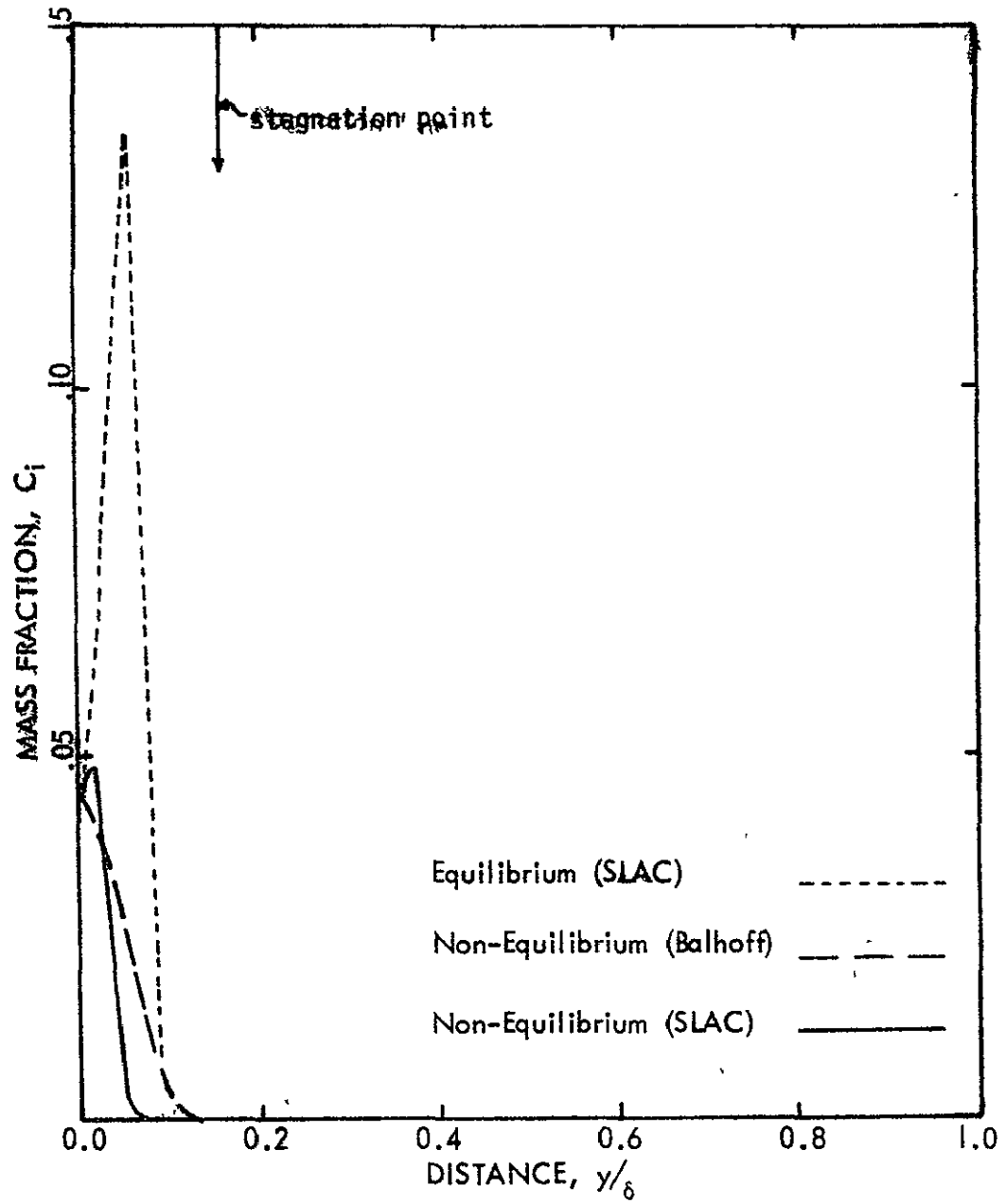


Figure 8.10 Mass Fraction Profiles for CN

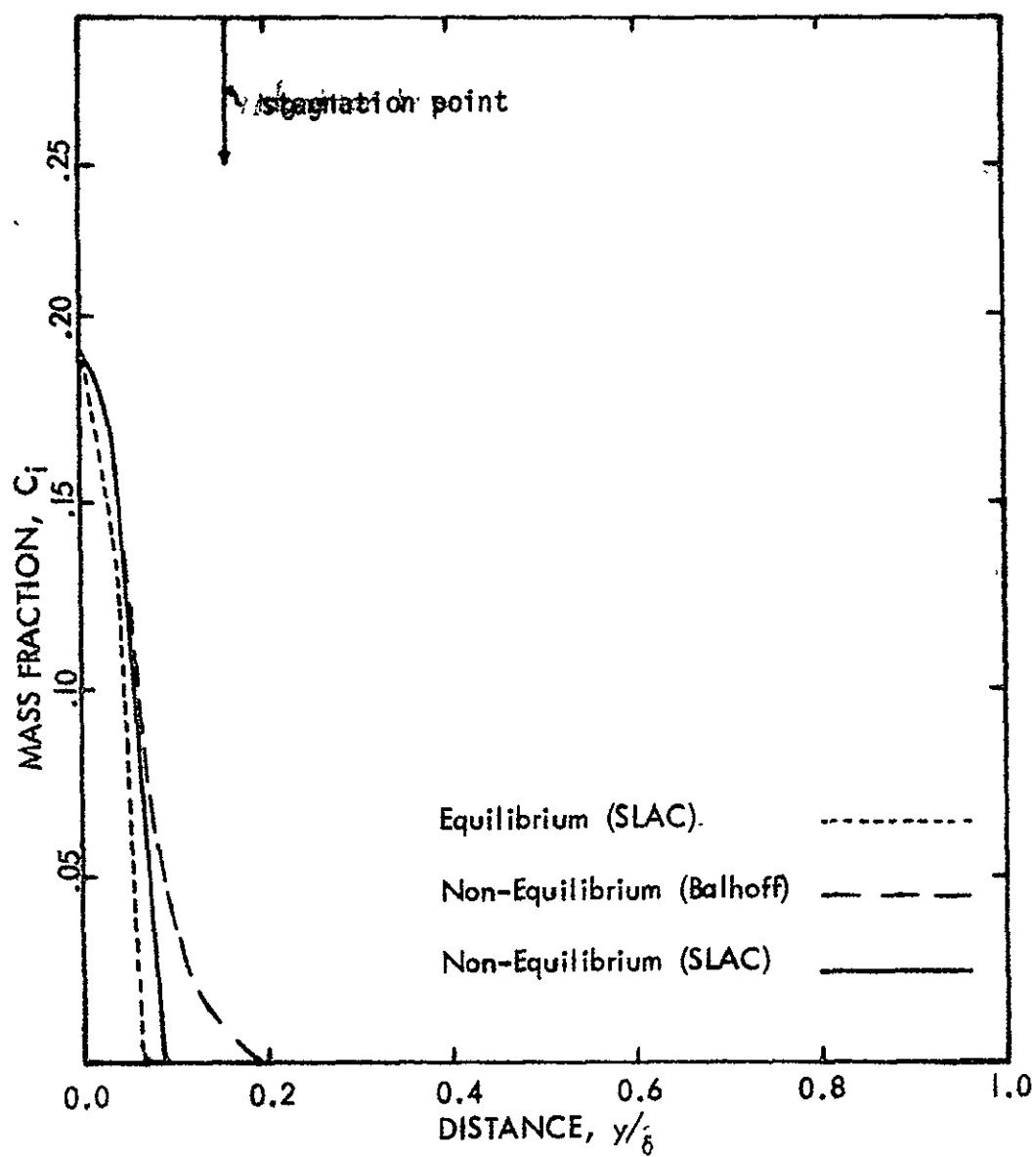


Figure 6.11 Mass Fraction Profiles for C_2H

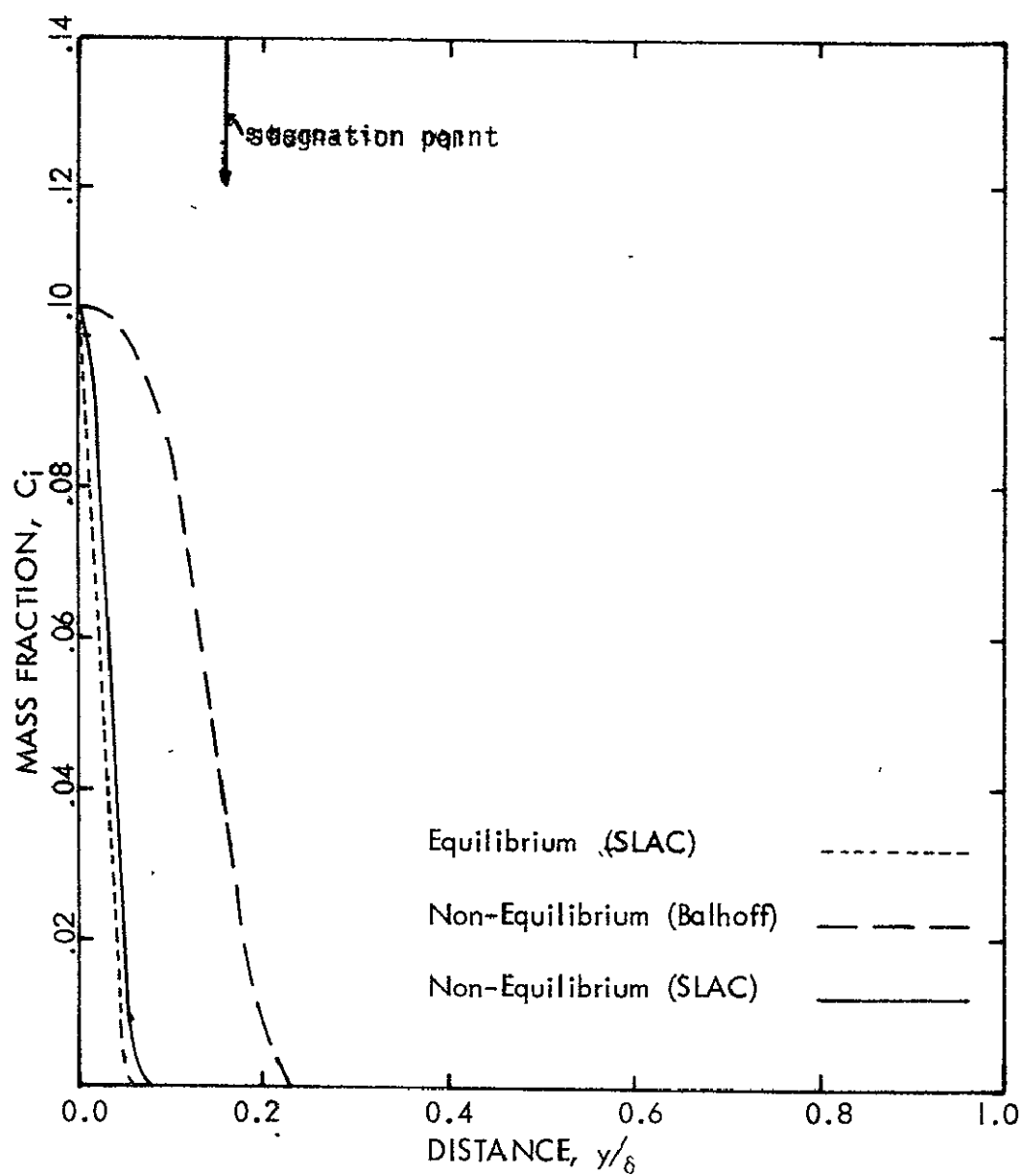


Figure C.12 Mass Fraction Profiles for C_4H_6

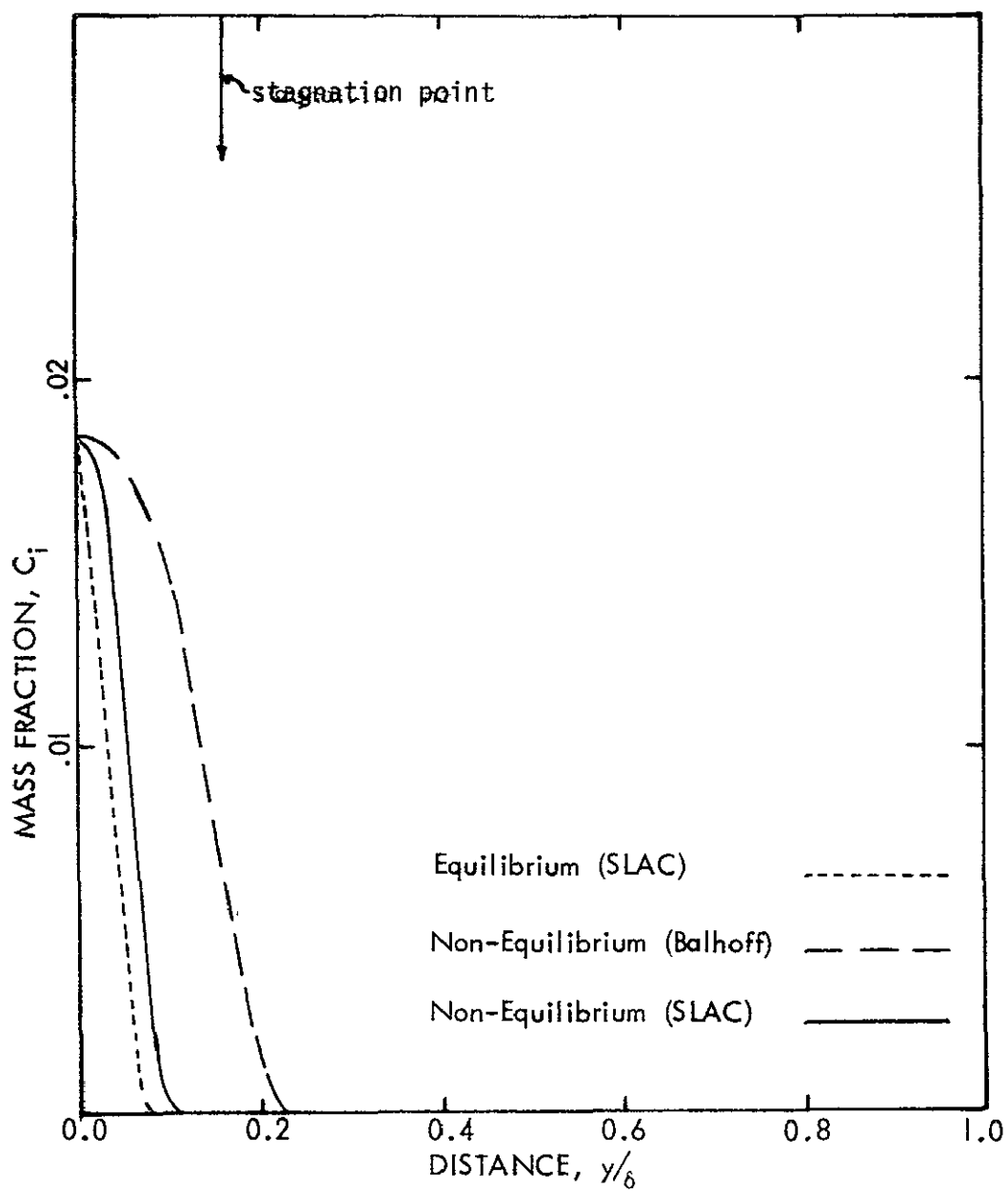


Figure E.13 Mass Fraction Profiles for HCN

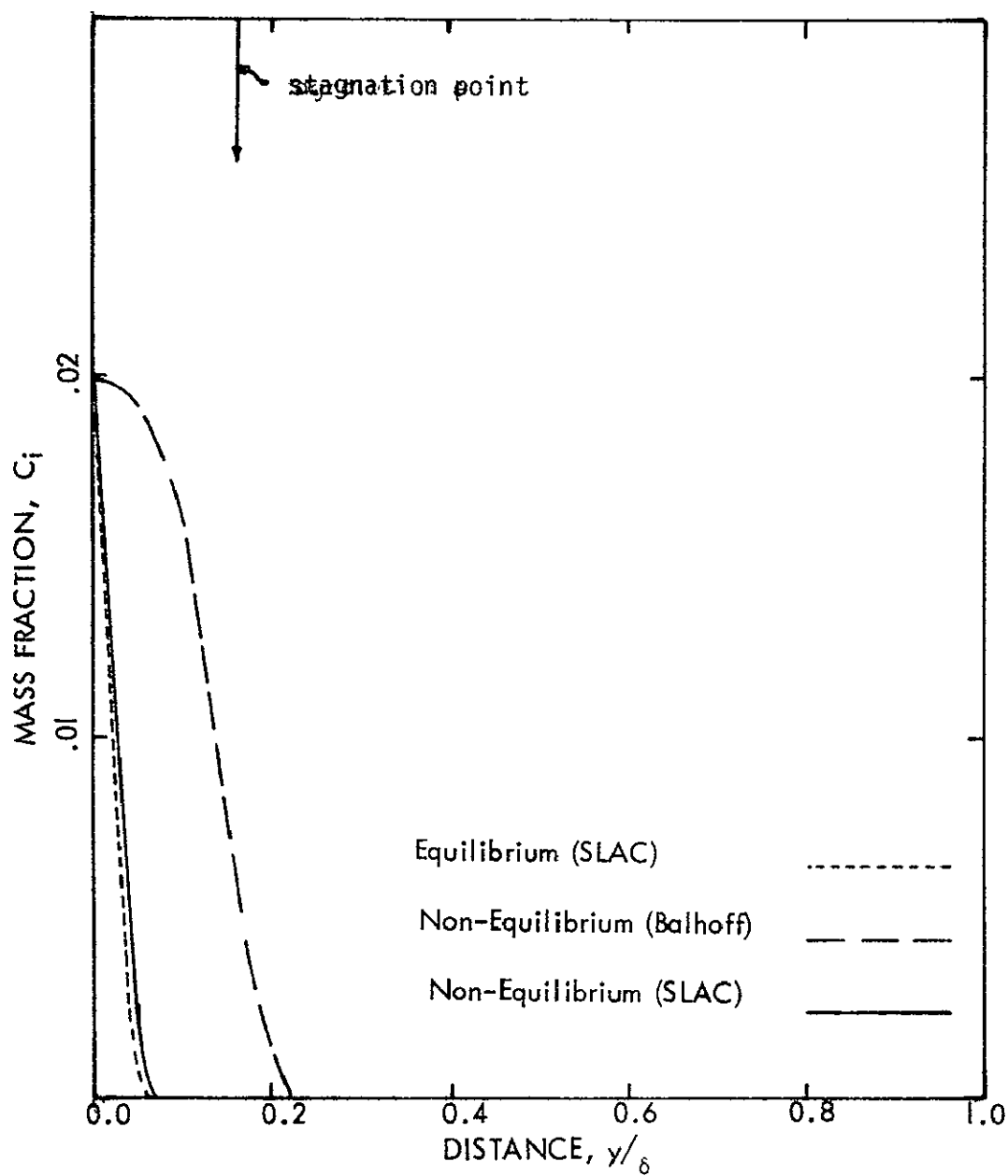


Figure C.14 Mass Fraction Profiles for C_2H_2

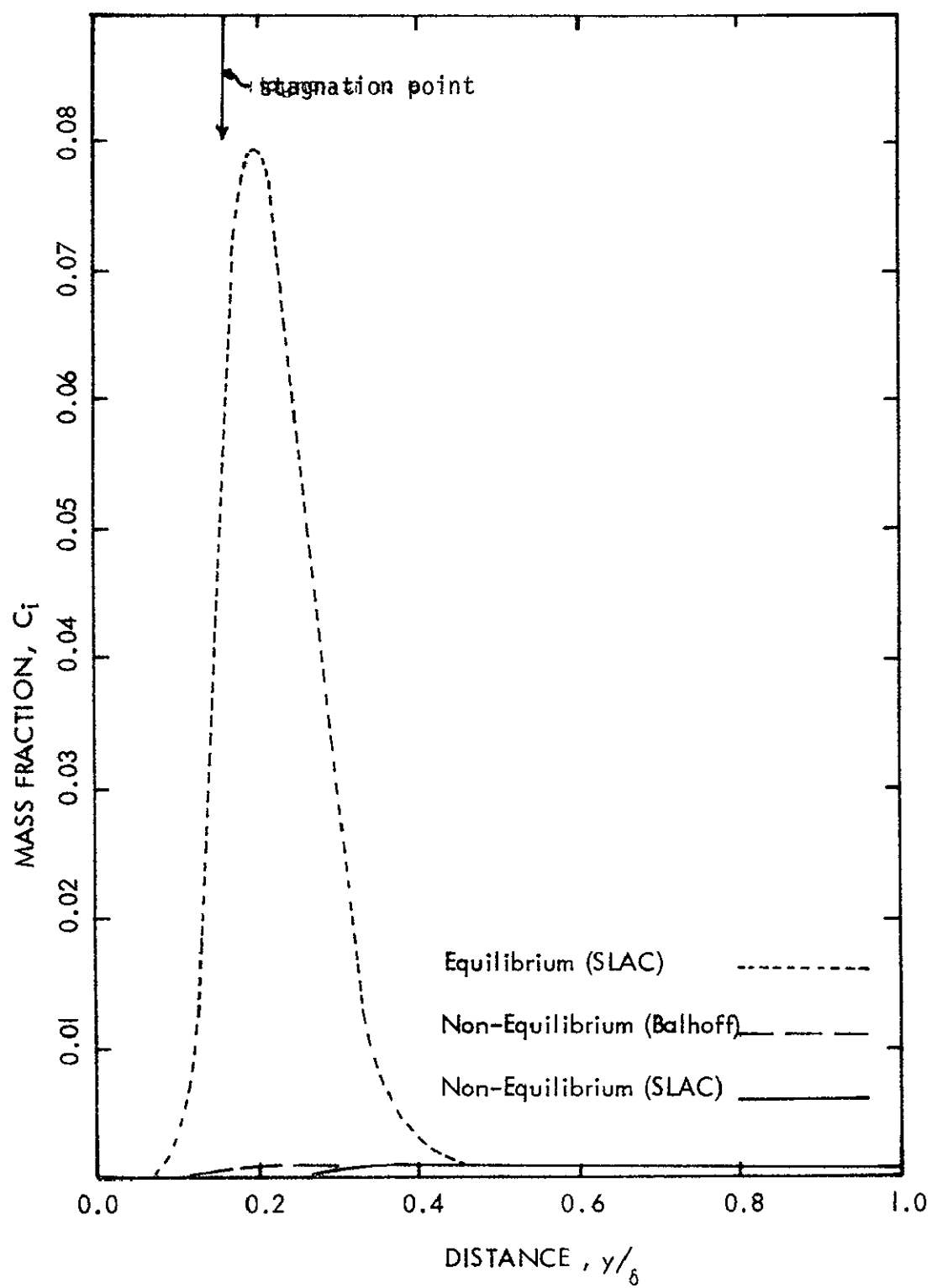


Figure C.15 Mass Fraction Profiles for C^+

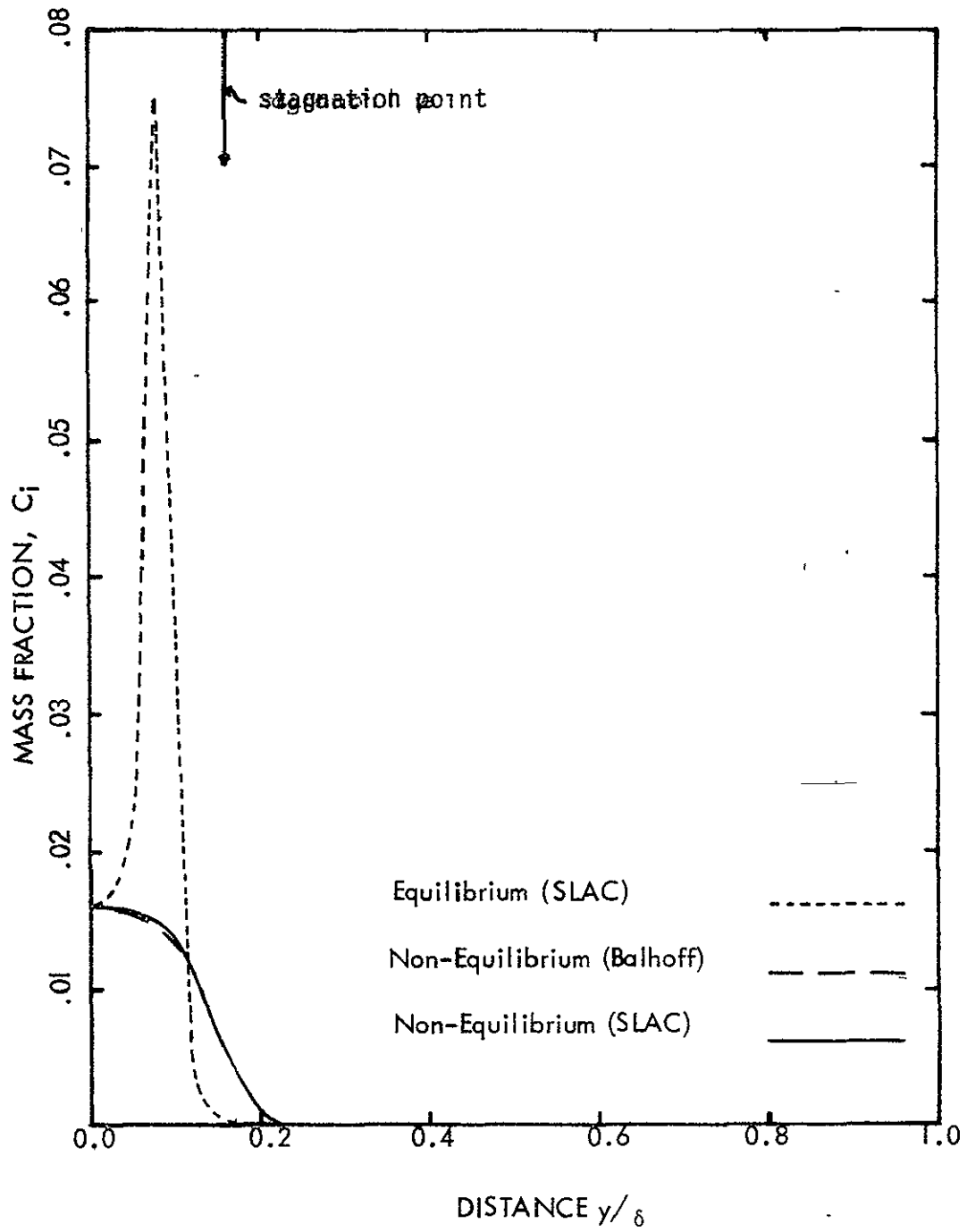


Figure C.16 Mass Fraction Profiles for N_2

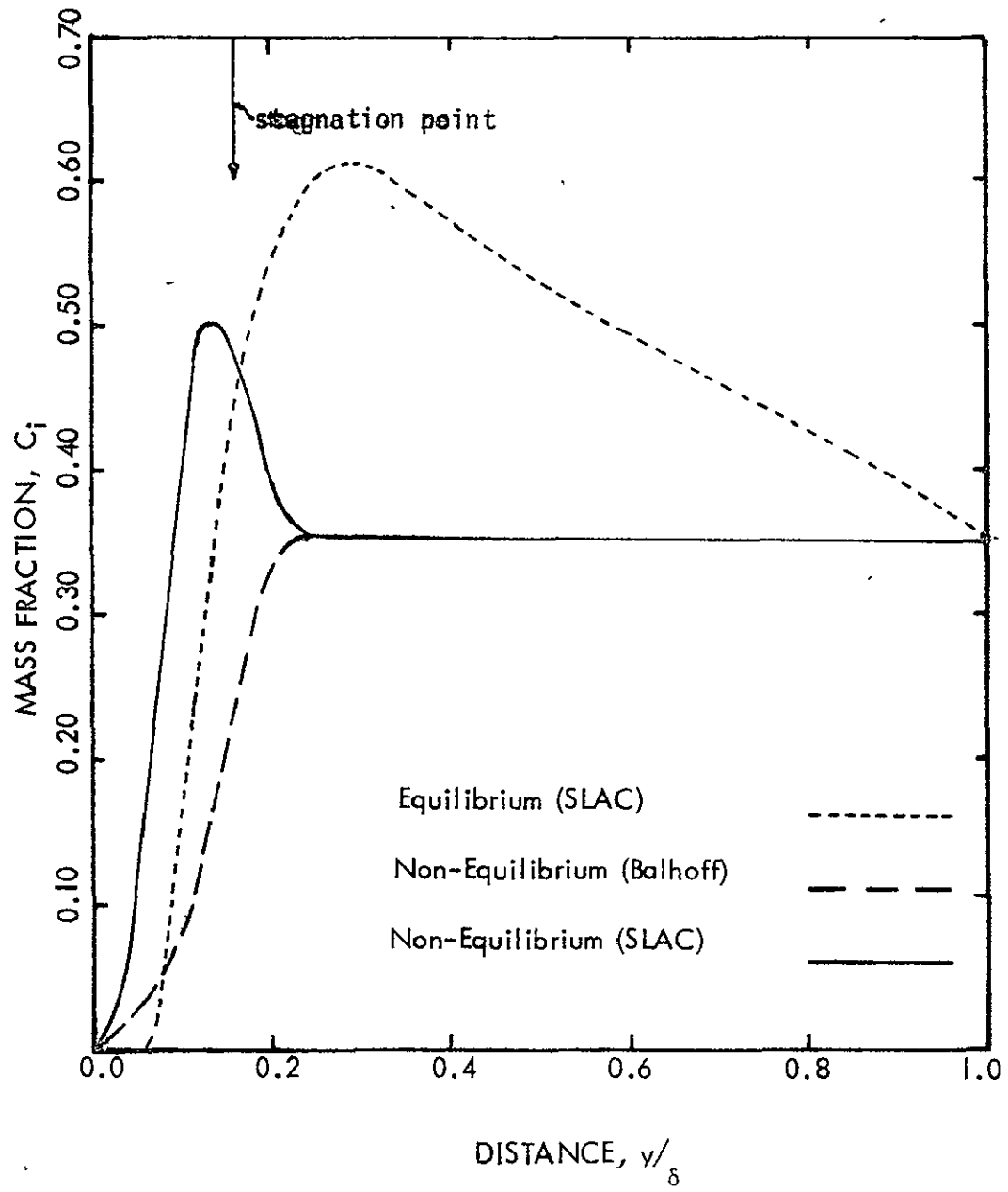


Figure 6.17 Mass Fraction Profiles for N

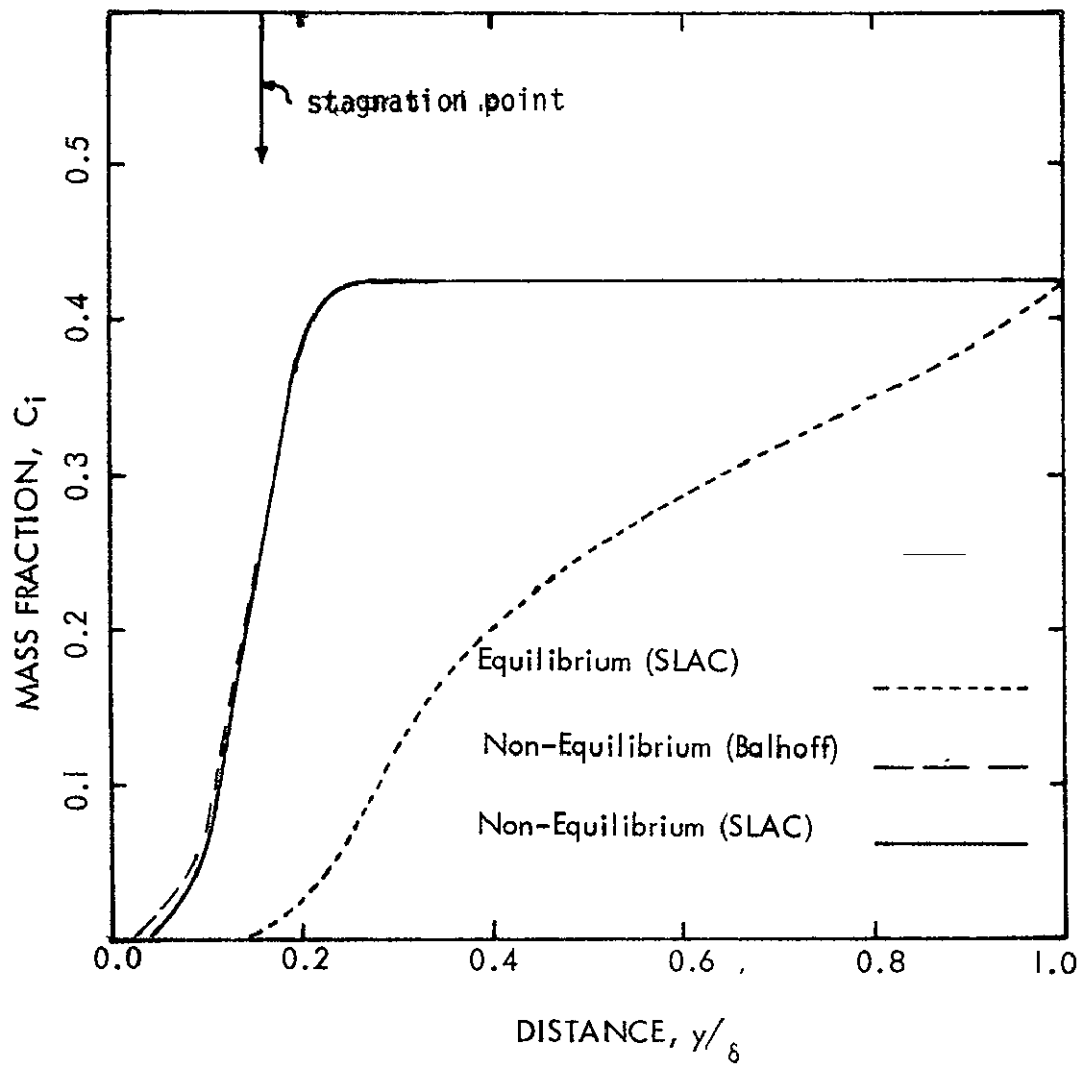


Figure C.18 Mass Fraction Profiles for N^+

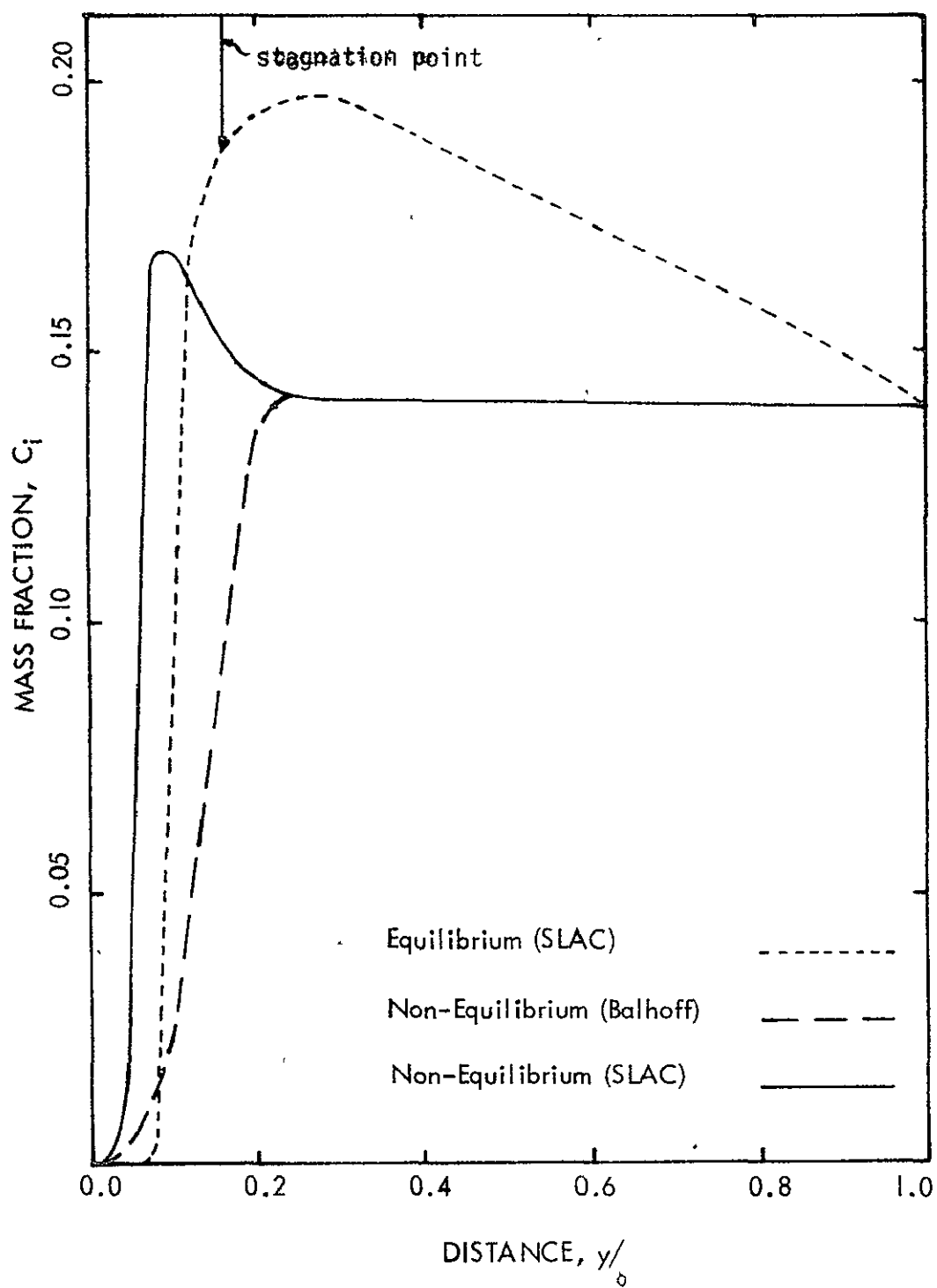


Figure C. 19 Mass Fraction Profiles for O

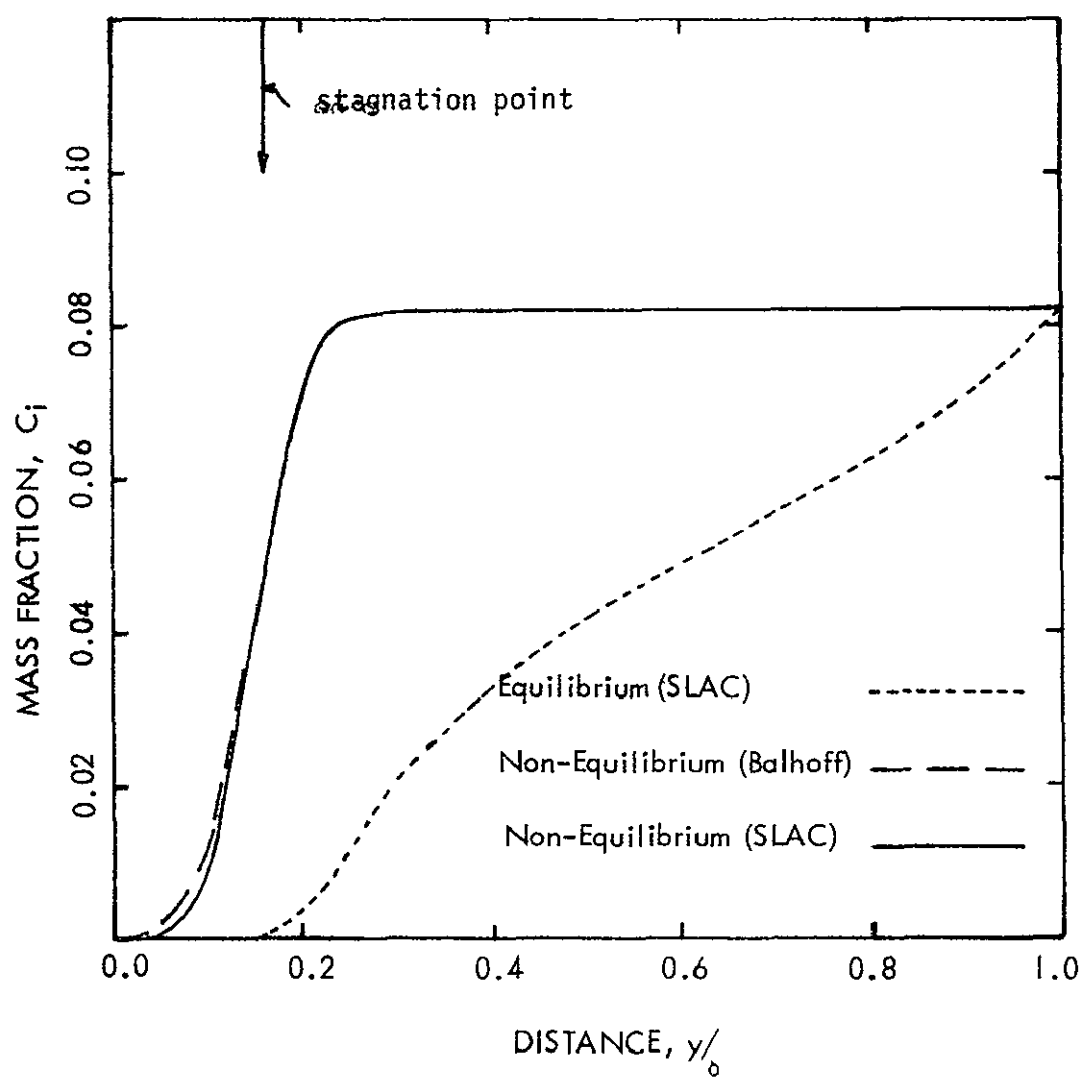


Figure C.20 Mass Fraction Profiles for O^+

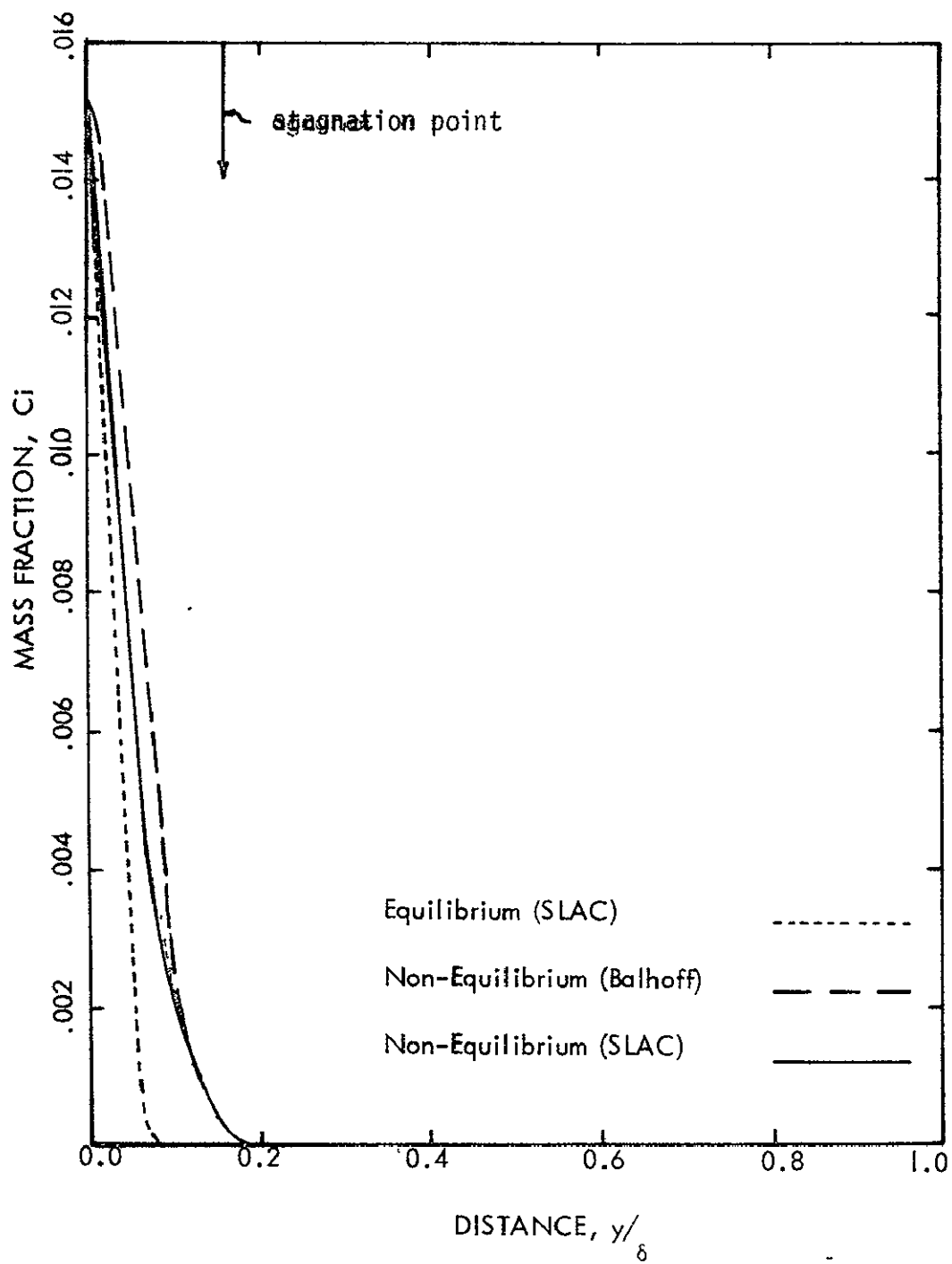


Figure C.21 Mass Fraction Profiles for H_2

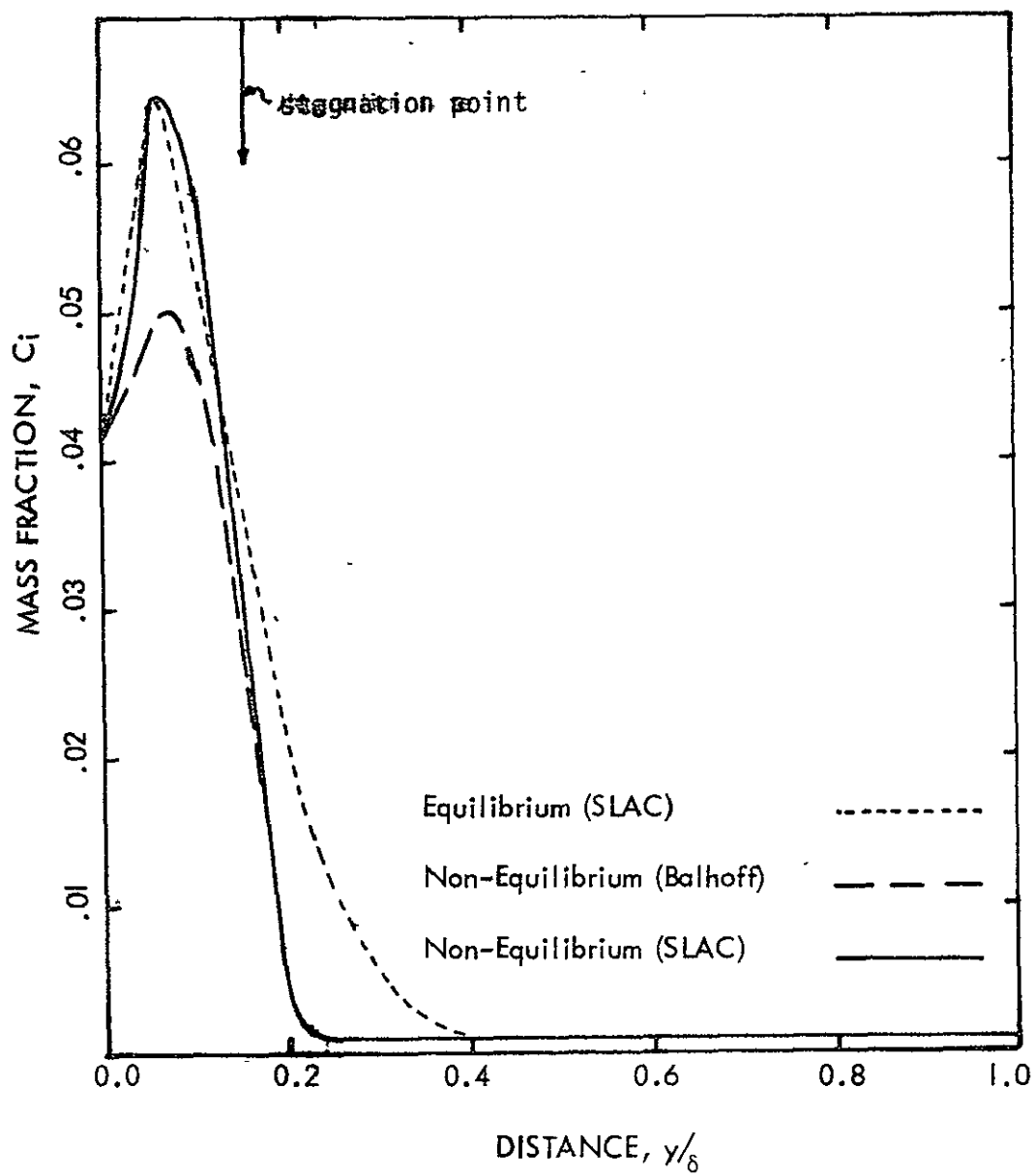


Figure C.22 Mass Fraction Profiles for H

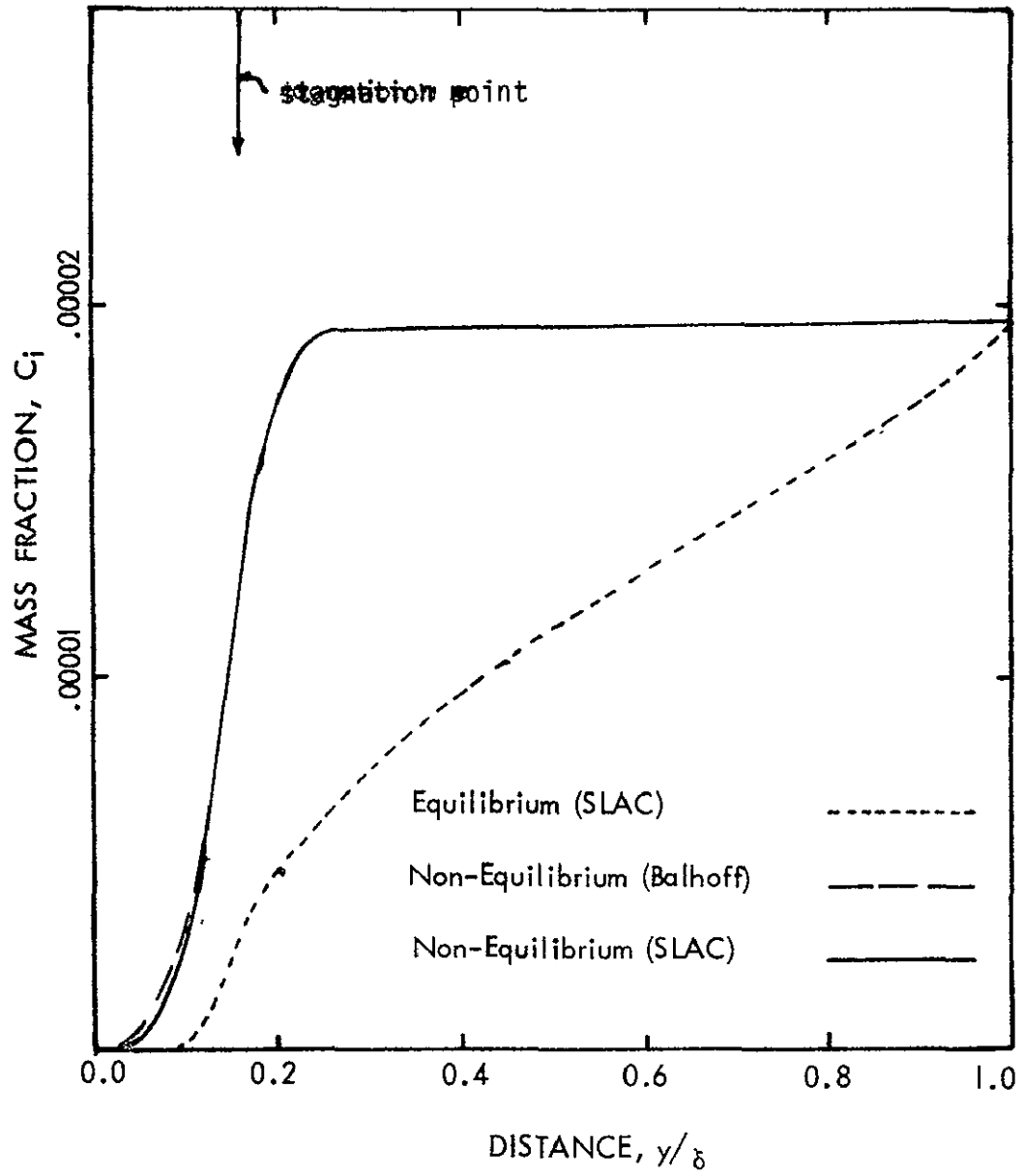


Figure C. 23 Mass Fraction Profiles for e^-

VITA

Guillermo Perez was born on [REDACTED]

[REDACTED] He completed his secondary education at University High School, Rio Piedras, Puerto Rico in May, 1962. From August, 1962 until May, 1967 he attended the University of Puerto Rico at Mayaguez receiving the Bachelor of Science degree in Mechanical Engineering. In September 1967, following a brief period of employment at the Puerto Rico Water Resources Authority, he enrolled in the Graduate School of Louisiana State University, and in January 1970, he received the Master of Science degree in Mechanical Engineering. From February, 1970 until June 1972 Mr. Perez continued his studies towards the Doctor of Philosophy degree in Chemical Engineering at L.S.U. In July he went to work as Director of the Resource Management Division of Puerto Rico's Environment Quality Board (EQB). While working at the EQB Mr. Perez was subsequently named Director of the Water Quality Bureau (November 1972) and Technical Assistant to the Executive Director (March 1973). Since March 1974, he has been a partner in the consulting firm of Cummings, Perez & Co., a firm specializing in environmental planning. He intends to continue working towards the development of Cummings, Perez & Co.